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PRE-TEST PREDICTION OF FIX II INTERMEDIATE BREAK
EXPERIMENT WITH THYDE-PI
(CSNI INTERNATIONAL STANDARD PROBLEM NO. 15)

August 1985

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Pre-Test Prediction of FIX II Intermediate Break
Experiment with THYDE-P1
(CSNI International Standard Problem No. 15)

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This report contains the results from the pre-test prediction calculation of FIX II Intermediate Break Experiment with the THYDE-P1 computer code. The experiment simulated the conditions corresponding to an intermediate size break loss-of-coolant accident (LOCA) in one of the recirculation lines of a boiling water reactor (BWR) with external pumps. The test was the first LOCA experiment in the FIX II facility and was chosen as the OECD CSNI International Standard Problem No 15.

Although an extensive verification calculations have been performed with THYDE-P1 for pressurized water reactors (PWRs), the present calculation is the first application to BWR. The major purposes of the present calculation are to verify the applicability of THYDE-P1 to BWR and to identify the model deficiencies for further code modification.

After the experimental data being released, the predicted results with THYDE-P1 have been compared with the experimental data. The comparison shows that the present calculation has adequately predicted the important phenomena observed in the experiment.

Key Words: THYDE-P1, LOCA, Pre-Test Prediction, FIX II, Intermediate Break, CSNI International Standard Problem No. 15, BWR External pump, Verification

THYDE-P1によるFIX I I中破断実験の実験前予測計算
(CSNI国際標準問題No.11)

日本原子力研究所東海研究所原子炉安全工学部

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(1985年7月15日受理)

THYDE-P1コードによるFIX I I中破断実験の実験前予測計算結果を報告する。本実験は、外付き再循環ポンプ沸騰水型軽水炉(BWR)に於ける再循環ライン中破断冷却水喪失事故(LOCA)を模擬した実験である。本実験はFIX I I実験装置での初めてのLOCA実験であり、OECD-CSNI国際標準問題No.15に採用された。

これまでTHYDE-P1の検証計算を、加圧水型軽水炉に対して広範に行って来たが、本計算がBWRへの初めての適用である。本計算の目的は、THYDE-P1がBWRに適用可能であることを検証すると同時に、今後のコードの改良のためにモデルの不備を検討することにある。

実験結果が公開された後、予測結果と実験結果の比較を行った。その結果、本予測計算は、実験で観測された重要な事象を適切に予測したことが示された。

Contents

1.	Introduction	1
2.	Description of FIX II Intermediate Break Experiment	1
2.1	System Description	1
2.2	Test Procedure	2
2.3	Initial Conditions and Boundary Conditions	3
3.	Analytical Models	3
3.1	Nodalization	3
3.2	CHF and Heat Transfer Correlations	4
3.3	Break Flow Model	5
3.4	Relief Valve Model	5
3.5	Pump Model	5
3.6	Pressure Loss Coefficients	6
3.7	Ambient Heat Loss	6
3.8	Material Properties	7
3.9	Phase Separator Model	7
4.	Calculated Results and Discussions	8
4.1	Pressure Transient	8
4.2	Cladding Surface Temperature Transient	8
4.3	Core Hydraulics	9
4.4	Downcomer Behavior	10
4.5	Break Flow Transient	11
4.6	Bypass Hydraulics	11
5.	Concluding Remarks	13
	References	14
Appendix A	List of Input Data	57
Appendix B	Description and Geometrical Data of components in FIX II Loop	66
Appendix C	Brief Description of Code Characteristics and Information on Computer Resources Utilized in Present Calculation	71
Appendix D	Pump Characteristic Curves Used in Present Calculation	75

目 次

1. 序	1
2. F I X I I 中破断実験の記述	1
2.1 実験装置	1
2.2 実験手順	2
2.3 初期条件及び境界条件	3
3. 解析モデル	3
3.1 ノード分割	3
3.2 CHF及び熱伝達モデル	4
3.3 破断流モデル	5
3.4 逃し弁モデル	5
3.5 ポンプモデル	5
3.6 圧損係数	6
3.7 系外への熱損失	6
3.8 物性値	7
3.9 気水分離器モデル	7
4. 計算結果及び議論	8
4.1 圧力の過渡変化	8
4.2 被覆管表面温度の過渡変化	8
4.3 炉心流動	9
4.4 ダウンカム挙動	10
4.5 破断流の過渡変化	11
4.6 バイパス流動	11
5. 結論	13
参考文献	14
付録 A 入力データリスト	57
B F I X I I ループの記述及び形状データ	66
C コードの簡単な記述及び本計算に使用した計算機資源	71
D 本計算で使用したポンプ特性曲線	75

List of Figures

Figure No.	Title	Page
Fig. 2.1.1	Lay-out of FIX II loop	23
Fig. 2.1.2	Schematic diagram of central loop section	24
Fig. 2.1.3	Split break simulation	25
Fig. 2.1.4	Horizontal cut view of test section and radial power distribution	26
Fig. 2.1.5	Structure of fuel rod simulator	27
Fig. 2.1.6	Axial power distribution and instrumentation location	28
Fig. 2.1.7	Structure of bypass and control rod guide tube volume	29
Fig. 2.1.8	Axial power distribution of bypass channel	30
Fig. 2.1.9	Structure above test section	31
Fig. 2.3.1	Planned core and bypass power histories	32
Fig. 2.3.2	Measured core and bypass power histories	32
Fig. 2.3.3	Planned pump speed histories	33
Fig. 2.3.4	Measured pump speed histories	33
Fig. 3.1.1	Nodalization chart	34
Fig. 3.5.1	Pump characteristic curve for pump P1	35
Fig. 3.5.2	Pump characteristic curve for pump P2	35
Fig. 3.6.1	Flow diagram of FIX II loop	36
Fig. 3.6.2	Pressure losses at core and bypass	37
Fig. 3.8.1	Specific heat of fuel rod simulator filler material	38
Fig. 3.8.2	Thermal conductivity of fuel rod simulator filler material	38
Fig. 3.9.1	Schematic diagram of phase separation	39
Fig. 4.1.1	Pressure at steam dome	39
Fig. 4.1.2	Mass flow rate through steam relief valve	40
Fig. 4.1.3	Schematic diagram of valve actions	40
Fig. 4.1.4	Measured fluid temperatures at lower plenum and upper plenum	41
Fig. 4.2.1	Vertical distribution of calculated cladding surface temperature	42
Fig. 4.2.2	Three-Dimensional view of calculated cladding surface temperature	43

Fig. 4.2.3	Experimental cladding surface temperatures in upper part part of core	43
Fig. 4.2.4	Experimental cladding surface temperatures in lower part part of core	44
Fig. 4.2.5	Effect of lower plenum flashing on cladding surface temperature	45
Fig. 4.2.6	Cladding surface temperature at level 1	45
Fig. 4.2.7	Cladding surface temperature at level 3	46
Fig. 4.2.8	Cladding surface temperature at level 5	46
Fig. 4.2.9	Cladding surface temperature at level 7	47
Fig. 4.2.10	Cladding surface temperature at level 9	47
Fig. 4.2.11	Cladding surface temperature at level 10	48
Fig. 4.2.12	Cladding surface temperature at level 12	48
Fig. 4.2.13	Cladding surface temperature at level 15	49
Fig. 4.3.1	Bundle inlet mass flow rate	49
Fig. 4.3.2	Differential pressure through core	50
Fig. 4.3.3	Mass flow rate through pump P1	50
Fig. 4.3.4	Mass flow rate through pump P2	51
Fig. 4.3.5	Calculated mass flow rates at core inlet and both inlet and outlet of lower plenum	51
Fig. 4.3.6	Calculated void fractions at core	52
(a)	Lower part of core	52
(b)	Upper part of core	52
Fig. 4.4.1	Differential pressure through downcomer	53
Fig. 4.4.2	Fluid temperature at downcomer inlet	53
Fig. 4.5.1	Break flow	54
Fig. 4.5.2	Integrated break mass flow	54
Fig. 4.5.3	Calculated mass flow rates at break, vessel side and pump side of broken recirculation line	55
Fig. 4.5.4	Calculated mass flow rates at lower plenum inlet, vessel side of broken recirculation line and intact recirculation line	55
Fig. 4.6.1	Differential pressure through bypass	56
(a)	Calculated	56
(b)	Measured	56

List of Tables

Table No.	Title	
Table 2.2.1	Planned performance chronology	16
Table 2.2.2	Measured performance chronology	16
Table 2.3.1	Planned initial conditions	17
Table 2.3.2	Planned initial conditions and rated values for recirculation pumps	17
Table 2.3.3	Measured initial conditoinis	18
Table 3.1.1	Description of nodes and heat slabs	19
Table 3.6.1	Loss coefficients in loop	20
Table 3.7.1	Ambient heat losses evaluated before experiment	21
Table 3.7.2	Measured ambient heat losses	21
Table 4.1	Chronology of key events	22

1. Introduction

The THYDE-P1 (1-10) code is a computer code which is applicable to various types of transient including a loss-of-coolant accident (LOCA) in both pressurized water reactors (PWRs) and boiling water reactors (BWRs). Although an extensive verification calculations have been performed for PWRs using the experimental data such as Loss-Of-Fluid Test (LOFT)(11) data, the present calculation is the first application to BWR.

The experiment(12-14) has been performed in May 1983 as the very first LOCA experiment in the FIX II facility, which has simulated the Swedish Oskarshamn 2 reactor, a 570 MWe BWR with external recirculation pumps. The volume scaling is 1:777. Components are included to steam dome, downcomer, two recirculation lines with one pump each, lower plenum, core with a 36-rod full-length bundle, control-rod guide tube volume, core bypass, upper plenum and steam separators.

The transient performed has simulated the conditions corresponding to an intermediate size split break in one of the main recirculation lines. The general objective with the test program is to obtain data for heat transfer between rod cladding and coolant during specified LOCAs and transients included in the studies of Swedish BWRs.

The test has been chosen as the OECD CSNI International Standard Problem No. 15 (ISP No.15). A blind calculation has been performed with THYDE-P1 based on the information provided about the test loop geometry, data on flow resistance, initial conditions and the test procedure(12).

In the present pre-test prediction calculation, the system characteristics is unknown because the experiment is the very first LOCA experiment in the facility. In addition, the planned initial conditions and chronology of performance can be different from those actually realized in the experiment. In these points of view, the present calculation can be said what is called a double-blind calculation. In spite of the difficulties expected to this type of calculation, the calculated results are reasonably in agreement with the experimental data.

2. Description of FIX II Intermediate Break Experiment

The intermediate break experiment, denoted Experiment 3025, has been conducted as the first LOCA experiment in the FIX II facility. The test facility is utilized for loss-of-coolant heat transfer experiment during the blowdown phase in an external recirculation pump BWR.

2.1 System Description

Figure 2.1.1 shows the lay-out of the FIX II loop. The description and the geometrical data of each component shown in this figure are summarized in App. B. The principal outline of the central loop section is schematically shown in Fig. 2.1.2. The test apparatus comprises the following main parts:

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- A pressure vessel with a 36-rod test section and a steam dome,
- A piping system with recirculation pumps and arrangements for break simulations,
- An external piping and auxiliary systems for steady-state operation of the loop, and
- Power supply and power control systems.

The downcomer is simulated by the external piping. Two recirculation lines simulate 4 recirculation lines in BWR. The one simulates the three intact lines and the other simulates one broken line. Thus, 1/4 of the total flow is circulated through the broken line. The valves v103, v117, v120 and v123 are used to simulate the break. In Experiment 3025, a simulated split break is accomplished by opening the valve 120 to the break flow receiver as shown in Fig. 2.1.3. The valve 103 is closed when simulating a double-ended break.

For the removal of the heat generation during the steady state operation, a portion of the downcomer flow is circulated through the cooler. The flow is fed back to the system as feed water and as spray to the top of the steam dome. During the steady-state operation, the valve v112 is closed and all the steam generated at the core is condensated by the spray flow. The flow path through the cooler is closed immediately after the break.

Figure 2.1.4 shows the horizontal cut view of the test section and the radial power distribution. The test section consists of a container with a 6x6 bundle of electorically heated fuel rod simulators. The measurment lines from the test section are cooled by circulating water in the gap between the teflon filling and the pressure vessel. This causes a heat loss, which will be discussed later.

Figure 2.1.4 shows the horizontal cut view of the test section and the radial power distribution. The test section consists of a container with a 6 × 6 bundle of electorically heated fuel rod simulators. The measurement lines from the test section are cooled by circulating water in the gap between the teflon filling and the pressure vessel. This causes a heat loss, which will be discussed later.

The core bypass and guide tube volume are simulated by separate pipes outside the pressure vessel as shown in Fig. 2.1.7. An electrically heated rod is installed within the core bypass. Its axial power distribution is shown in Fig. 2.1.8.

Figure 2.1.9 shows the structures above the test section. A spray condensor is located on top of the test section. A simplified steam separator is located in the lower part of the condensor, and its upper part is used as a steam dome, to which the steam relief line is connected.

2.2 Test Procedure

Table 2.2.1 shows the planned chronology of performance, which has been provided for the blind calculation for ISP No. 15. Table 2.2.2 shows the chronology of performance actually measured in the

experiment. When these tables are compared, it is clear that the planned chronology of performance is a little different from that actually measured in the experiment. The difference in these two tables will be discussed in detail later. Here, the test procedure is stated based on the measured chronology of performance shown in Table 2.2.2. The test has been initiated by opening the break valve at 0.0 s. The steam relief valve has started to open at 0.4 s and been fully open at 1.0 s. The steam relief valve has started to close again at 1.6 s. The flow through the cooler has been closed by 2 s into the transient. The steam relief valve has started to open for the second time at 12.05 s. The experiment has been terminated by closing the steam relief valve and the break valve at 76 s.

2.3 Initial Conditions and Boundary Conditions

Table 2.3.1 shows the planned initial conditions provided(12) for the blind calculation. Also the initial conditions for the pumps and the rated pump values are summarized in Table 2.3.2. Table 2.3.3 shows, on the other hand, the initial conditions actually measured in the experiment. The planned initial conditions are also slightly different from those measured in the experiment. For example, the planned initial core power is about 3 % larger than the actual power and the planned bypass power is also 8 % larger than the actual value.

Figure 2.3.1 shows the planned core and bypass power histories. Corresponding to this figure, the measured power histories are shown in Fig. 2.3.2. The planned normalized core power, for example, at 10 s is, in turn, about 10% smaller than the measured value. Then, considering the difference in the initial power, it is concluded that the power level used in the calculation at 10 s is slightly smaller than the actual power applied in the experiment, and the difference is small.

Figures 2.3.3 and 2.3.4 show planned and measured pump rotational speeds, respectively. Apparent difference appears after 50 s for pump P2. Except this point, the planned speeds are in good agreement with those measured in the experiment.

3. Analytical Models

As to the detailed description of the methods and the models of THYDE-P1, Ref. (1) should be referred. The brief summary of the code characteristics and the options applied are shown in Table C.1 in App. C.

3.1 Nodalization

Figure 3.1.1 shows the nodalization diagram. The description of the nodes and the heat slabs are shown in Table 3.1.1. The present calculation has been performed with 54 nodes and 50 junctions. The test section is represented by a single channel. The focal point in the present calculation is the core heat trans-

experiment. When these tables are compared, it is clear that the planned chronology of performance is a little different from that actually measured in the experiment. The difference in these two tables will be discussed in detail later. Here, the test procedure is stated based on the measured chronology of performance shown in Table 2.2.2. The test has been initiated by opening the break valve at 0.0 s. The steam relief valve has started to open at 0.4 s and been fully open at 1.0 s. The steam relief valve has started to close again at 1.6 s. The flow through the cooler has been closed by 2 s into the transient. The steam relief valve has started to open for the second time at 12.05 s. The experiment has been terminated by closing the steam relief valve and the break valve at 76 s.

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fer during the blowdown and therefore the phase separation phenomenon is considered to be very important. In order to simulate the phase separation phenomenon adequately, the test section should be discretized vertically by many nodes because a drift flux model is applied in THYDE-P1 and a water level tracking model is not implemented. In the present calculation, the test section and the downcomer are nodalized 12 and 11 nodes, respectively. The average rod and the highest power rod are simulated separately by the heat slabs. The power ratio of the highest power rod to the average rod is 1.034 based on the radial power distribution shown in Fig. 2.1.4.

The core bypass and the control rod guide tube volume are simulated by the flow path parallel to the test section. The heater in the core bypass is also modeled by the heat slabs.

The flow path through the cooler with the circulation pump P3 and several controlling valves are also modeled to simulate the steady-state operation in the experiment. The spray line is, however, assumed to be directly connected to the lower part of the steam dome. It is because that, in the present version of THYDE-P1, the velocity difference between the phases are not yet considered at the steady state calculation. This means the node average enthalpy is set equal to the average enthalpy upstream of the node, if the heat source does not exist. Then, the spray line from the evaporation cooler, which is filled with subcooled water, cannot be connected at the top of steam dome, filled with saturated steam. This simulation method may not affect adversely the overall trend of the calculation, because the spray valve starts to close just after the test initiation, i.e. 1.78 s after break in the planned sequence.

The list of the input data used in the present calculation is shown in App. A.

3.2 CHF and Heat Transfer Correlations

In the present calculation, the correlations for the heat transfer coefficients and the critical heat flux (CHF) are selected by options as follows:

- The Dittus-Boeltler correlation(18) for the forced convection condition,
- The Thom correlation(19) for the nucleate boiling,
- The Bromley-Pomeranz correlation(20, 21) for the low flow film boiling,
- The Groenevelt(22) and Dougal-Rohsenow(23) correlations for high flow film boiling, and
- The McEligot(24) correlation for the forced convection due to superheated steam.

The CHF correlations applied in the present calculation are:

- The Biasi correlation(25) for the high flow condition, and
- The Zuber correlation(26) for the low flow condition.

3.3 Break Flow Model

The modified Zaloudek equation(16) and the Moody correlation(15) are applied under the subcooled condition and the saturated condition, respectively. The discharge coefficient for the Moody correlation is assumed to be 0.6. The modified Zaloudek equation is as follows:

$$G_c = cd \sqrt{2 \rho (P - c_2 P_{sat})} \quad \text{Eq. 3.3.1}$$

where

- G_c : Critical mass flow (kg/s.m²)
- P : Pressure upstream of break (pa)
- P_{sat} : Saturation pressure (pa)
- ρ : Fluid density (kg/m³)
- Cd : Discharge coefficient (=0.6)
- C_2 : Constant (=0.8)

The transition quality is assumed to be 0.02. When the coolant is subcooled, the modified Zaloudek equation is used and when it becomes larger than 0.02, the Moody correlation is used. When the quality is between 0.0 and 0.02, the critical flow is linearly interpolated between them.

3.4 Relief Valve Model

The boundary condition at the steam outflow is very important to determine the pressure transient. In the experiment, the steam flow is calibrated to give 1 kg/s at a steam dome pressure of 70 bars. This is done by the adjustment of the valve flow area.

In the present calculation, the following model(13) is tentatively implemented in THYDE-P1 and used:

$$W_s = A \sqrt{\kappa \left(\frac{2}{\kappa+1} \right)^{\frac{\kappa+1}{\kappa-1}} \cdot \rho_{fs} \cdot P} \quad \text{Eq. 3.4.1}$$

where

- W_s : Steam mass flow (kg/s),
- A : Valve flow area (m²),
- P : Pressure upstream of valve (pa),
- ρ_{fs} : Fluid saturation density (kg/m³),
- κ : Ratio of isobaric specific heat to specific heat at constant volume (-).

The flow area A is adjusted to give 1 kg/s at a steam dome pressure of 70 bar corresponding to the flow calibration in the experiment. It is because the relief valve model in THYDE-P1 is yet to be developed.

3.5 Pump Model

Figures 3.5.1 and 3.5.2 show so called Q-H curves for recirculation pumps p1 and p2, respectively. These are the only data available from the experimental basis. In the transient analysis, however, the full pump characteristic curves are necessary including those for two-phase flows.

Although the two-phase pump characteristics are very important to predict the core hydraulics, no experimental data are available for the recirculation pumps. The pump characteristic curves for both single phase flow and two-phase flow used in the present calculation are shown in App. D. These data are recommended(13) to be used for blind calculations because of lack of data, in spite of the fact that the two-phase data are to be used for the different pump from that installed in FIX II test facility.

3.6 Pressure Loss Coefficients

Table 3.6.1 shows the data table provided(12) for the pressure loss coefficients for the blind calculation. It is found that there is a inconsistency in this table. Figure 3.6.2 schematically shows the pressure drops calculated based on the items 1 to 3 and 9 in Table 3.6.1. Since the wall frictional pressure drops through the test section and the bypass are small in comparison with the pressure losses at the flow restrictions, they are not necessary to be considered here. As shown in this figure, the pressure loss through the bypass is much larger than that through the core. Thus, we can conclude that some data are missing for the pressure losses at the test section or the loss coefficient at the bypass inlet is erroneous, i.e. too large. In the present blind calculation, the loss coefficient at the bypass inlet is reduced and adjustment has been performed at the steam separator and the pumps to obtain a consistent pressure distribution at the steady state.

3.7 Ambient Heat Loss

The detailed information on the structures in the test loop has not been provided for the blind calculations. In the present calculation, therefore, the structural heat and ambient heat loss are not taken into consideration.

The major heat losses from the loop are caused by the cooling of the measurement lines and heat losses from exposed surfaces. As stated in Subsec. 2.1, the measurement lines from the test section are cooled by circulating water in the gap between the teflon filling and the pressure vessel. The heat loss from this section estimated before the experiment is 250 kw. The other heat losses estimated are shown in Table 3.7.1. The total heat loss is evaluated more than 10% of the rated core power. Table 3.7.2 shows the measured heat loss from the measurement lines. The evaluated heat loss from this section is turned out to be the same order of magnitude as that measured.

3.8 Material Properties

Figures 3.8.1 and 3.8.2 show the specific heat capacity and the thermal conductivity of the compacted magnesium oxide as a function of temperature provided(12) for the blind calculation. These data are used in the present calculation.

3.9 Phase Separator Model

In the present version of THYDE-Pl, a phase separator model yet to be developed. Therefore, in the present calculation, a very simple model is tentatively implemented into THYDE-Pl and used. The modeling is schematically explained in Fig. 3.9.1. In this model, the integration scheme of the enthalpy transportation term in the energy conservation equation is changed only at the phase separator as follows:

- (1) If mixing junction 42 (mj42) is saturated and the node 14 (nl4) inlet flow is positive, only saturated fluid is transported from mj42 to nl4.
- (2) If mj42 is saturated and the n50 inlet flow is positive, only saturated steam is transported from mj42 to n 50.
- (3) If nl4 is saturated and the nl4 inlet flow is negative, only the saturated steam is transported from nl4 to mj42.
- (4) If nl4 is saturated and the nl4 inlet flow is positive, only saturated fluid is transported from nl4 to mj44.
- (5) In the other cases than (1) to (4), the integration method is the same as that for the other nodes and junctions.

4. Calculated Results and Discussions

The calculated chronology of key events are compared with the measured data in Table 4.1. In the following subsections, the predicted results with THYDE-P1 are compared with the experimental data and discussed with special attention to the key events shown in the table.

4.1 Pressure Transient

The calculated pressure transient at the steam dome is shown in Fig.4.1.1 along with the experimental data. In the early stage, the pressure is overestimated but is underestimated in the late stage. In order to clarify the reasons for the difference, the valve actions in the calculation should be compared with those actually performed in the experiment because the steam flow rate through the relief valve is very important to determine the pressure history especially in the early stage of the blowdown. Figure 4.1.2 shows the comparison between the calculated mass flow rate through the relief valve and the corresponding experimental data. Overall trend of the calculated steam mass flow rate is in good agreement with the experimental data. This means the model stated in Subsec. 3.4 gives a good approximation to the steam mass flow rate. Although it is not clear in this figure, the relief valve is closed earlier in the calculation than in the experiment. Figure 4.1.3 shows the valve actions both in the calculation and experiment. The valve actions in the calculation are based on the planned sequence of events, which is different from the measured one. The steam relief valve (v 112) is closed about 0.5 s earlier in the calculation than in the experiment. In addition, both spray flow valve (v 106) and feed water valve (v 104) are also calculated to be closed earlier than in the experiment. These differences in the boundary condition are considered to be one of the reasons for the overestimated pressure in the early stage of the blowdown.

The other reason for the difference may come from the ambient heat loss. As stated in Subsec. 3.7, the ambient heat loss during the steady state operation is evaluated more than 10 % of the rated power. This value is comparable to the decay power. When the ambient heat losses are not taken into account by modeling the structures, the pressure may be overestimated in the early stage and be underestimated in the later stage. It is because that they behave as heat sinks at the beginning and as the heat sources later on. In fact, in the experiment, superheated steam was observed after about 50 s at the upper plenum due to the structural heat as shown in Fig. 4.1.4.

4.2 Cladding Surface Temperature Transient

The focal point of the experiment is the core heat transfer during the blowdown. Figure 4.2.1 shows the vertical distribution of the cladding surface temperatures. The level numbers denoted right sides of the curves are corresponding to the levels of the

thermo-couples in the test section shown in Fig. 2.1.6. The dotted lines discriminate the area where the cladding temperature stays at the saturation temperature. The effect of the lower plenum flashing is clearly shown in this figure. Figure 4.2.2 shows the three-dimensional view of also the calculated cladding surface temperatures. The first DNB is calculated to occur at 16 s at the top of core and the core wide dries out at about 38 s. At the lower part of the test section, temperature excursion is not calculated to occur. The distributions of the measured cladding temperatures in the upper part and lower part of the core are in Figs 4.2.3 and 4.2.4, respectively. The DNB occurred just after the break at the top of core and the cladding temperature soon returns to the saturation temperature. The core uncover occurs first at about 50 s at the top of core. The experimental data imply a clear water level has been formed and it slowly moved down to the bottom within about 10 s. It should be noted that in Fig. 4.2.4, a small peak has been observed at level 7. This is caused by the lower plenum flashing just after the dryout. Figure 4.2.5 shows the cladding surface temperature calculated at level 7 along with the experimental data. With respect to the small peak, a very good agreement was obtained. This means the core hydraulics related to the lower plenum flashing is properly predicted by the calculation.

Figures 4.2.6 to 4.1.13 show the calculated cladding surface temperatures at the different levels with the corresponding experimental data. The tendencies shown in these figures are summarized as follows:

- (1) The observed small peak of the cladding temperature at the beginning of the blowdown is not predicted to occur,
- (2) The observed small peak just before the lower plenum flashing is well predicted by the calculation,
- (3) At the upper part of the test section, the temperature excursion is calculated to start earlier than in the experiment and at the lower part, on the other hand, it is delayed in the calculation.
- (4) In the experiment, the whole test section is uncovered within 10 s starting at about 50 s. In the calculation, however, the cladding temperature at the lower part of the core stays almost at the saturation temperature.

One of the reasons for (1), (3) and (4) seems to be that the model for the phase separation, i.e. the drift flux model, does not correctly predict the phase separation in the core. These figures imply the present drift flux model overpredicts the relative velocity between the phases, resulting in higher void fraction in the upper part and lower void fraction in the lower part in comparison with the experiment, which will be shown in the next subsection.

4.3 Core Hydraulics

The calculated bundle inlet mass flow rate and the differential pressure through the core are shown in Figs. 4.3.1 and 4.3.2, respectively, along with the experimental data. A good agreement is obtained in both core inlet flow and the differential pressure through the core. In the experiment, the core inlet flow suddenly decreases just after the break and reaches very close to zero, i.e. flow stagnation. This instantaneous flow stagnation brings about the initial dryout. In the calculation, on the other hand, the core flow also suddenly decreases but does not reach close to zero. This is because the core flow recovery is calculated to occur earlier than in the experiment. As mentioned in Subsec 4.1, the valve is calculated to be closed earlier than in the experiment. Then, the core flow recovery comes on before the core flow stagnation in the calculation. This is one of the reasons why the initial dryout is not calculated to occur.

After the relief valve, spray valve and feed water valve being closed, the core inlet flow slowly decreases according to the flow coastdown in pump P2, which is shown in Fig. 4.3.3. The calculated mass flow rate through pump P2 is in good agreement with the experimental data.

The core inlet flow suddenly decreases and the flow reversal occurs at about 12 s in the calculation and 15 s in the experiment. These timings are corresponding to when the cavitation starts in pump P1, as shown in Fig. 4.3.4. These timings are also corresponding to the timings of downcomer dryout as will be shown in the next section. The core flow reversal will also be discussed in conjunction with the break flow transient. Relatively high enthalpy coolant flows into the lower plenum due to the core flow reversal and flashing starts. The core inlet flow again returns to be positive due to the lower plenum flashing.

The calculated mass flow rates at the inlet and outlet of the lower plenum and at the core inlet are shown in Fig. 4.3.5. Although the lower plenum inlet flow is downward, the core inlet flow is forced to be upward due to the flashing. The difference in the mass flow rates between the lower plenum outlet and the core inlet shows the mass flow rate from the core bypass to the bottom of the test section. It can be concluded from this figure that the effects of the core bypass and the guide tube volume to the core hydraulics are small in the calculation.

Figure 4.3.6(a) and (b) show the calculated void fractions in the lower part and upper part of the core, respectively. These figures are helpful to understand the discussions stated above. The void fractions start to increase just after the break but soon start to decrease before reaching 1.0 s due to the closure of the relief valve. Sudden increase in the void fraction at about 15 s is brought about by the core flow reversal. The effect of the lower plenum flashing is again clearly shown after about 20 s. After about 36 s, the upper part of the core is filled with only steam but at the lower part two-phase mixture still remains.

4.4 Downcomer behavior

The calculated differential pressure through the downcomer is shown in Fig. 4.4.1 along with the experimental data. A good agreement is obtained. This figure shows that the downcomer becomes almost empty at about 12 s in the calculation and at about 15 s in the experiment. This downcomer dryout causes the pump degradation of pump P2. The fluid temperature at the downcomer inlet is shown in Fig. 4.4.2 along with the experimental data. After the feedwater valve and the spray valve being closed, the saturated fluid at the bottom of the steam dome is drained through the downcomer. Therefore the fluid temperature increases. The flashing of the downcomer starts earlier in the calculation than in the experiment. After the flashing, the fluid temperature stays at the saturation temperature. The discrepancy of the temperature between the calculation and the experiment after about 30 s is corresponding to the underprediction of the pressure.

4.5 Break Flow Transient

The calculated break flow is shown in Fig.4.5.1 along with the experimental data. The experimental data just after the break are not available. The calculation underpredicts the break flow until about 60 s. Figure 4.5.2 shows the integrated break mass flow rates. This figure also shows that the break flow is underestimated especially until 30 s.

The calculated mass flow rates at the break, and both the pump side and the vessel side of the broken recirculation line are shown in Fig.4.5.3. A local minimum of the break flow is calculated to occur at about 14 s. This behavior is caused by the change in the ratio of the contribution to the break flow from the pump P2 side to that from the vessel side. At about 14 s, the pump cavitation is calculated to occur at pump p2 caused by the downcomer dryout and the mass flow rate from pump P2 side suddenly decreases. Even when it suddenly decreases, the break flow stays unchanged since the other side flow, i.e. the flow from the vessel side, compensates it. But the change in the mixing ratio changes the enthalpy upstream of the break and therefore it changes the break flow behavior. This is one of the characteristic behaviors in the split type break.

Figure 4.5.4 shows the calculated mass flow rates through the flow paths connected to the lower plenum. Although the pump P2 flow suddenly decreases, the break flow slowly decrease and therefore the mass flow rate from the lower plenum to the break suddenly increases as stated above. As the result of this sudden increase in the flow rate, the core flow reversal is calculated to occur as clearly shown in this figure. The flow from pump P1 to the lower plenum also once increases accompanied by the sudden increase in the flow from the lower plenum to the break.

4.6 Bypass Hydraulics

Figures 4.6.1(a) and (b) show the calculated and measured differential pressures through the bypass, respectively. The sum of

the experimental curves for DPT20, DPT21 and DPT22 in Fig. 4.6.1(a) is corresponding to the predicted curve in Fig. 4.6.1(b). Both in the experiment and calculation, flashing starts at the bypass from 10 to 20 s and at the guide tube volume from 20 to 30 s. The two-phase mixture is forced into both the top of core and the bottom of core due to the flashing of the guide tube volume. As already shown in Fig. 4.3.5, the core inlet flow rate is not very different from the lower plenum outlet flow. Therefore the effect of the bypass to the core hydraulics is considered to be small.

5. Concluding Remarks

The present calculation has been performed based on the data provided for the blind calculation for ISP No. 15(12), which contain the system geometrical data and the planned initial conditions and boundary conditions, being different from those measured in the experiment. In addition, the system characteristics is unknown before the calculation starts, because the experiment is the first LOCA experiment in the FIX II test facility. In these sense, the present pre-test prediction calculation can be categorized as what is called a double-blind calculation. In spite of the fact, the physical phenomena observed in the experiment are adequately predicted in the present calculation. The major concluding remarks derived from the present calculation are:

- (1) The pressure transient is well predicted. In detail, it is overestimated in the early stage and is underestimated estimated in the late portion. One of the reasons for the overestimation is that the steam relief valve is closed earlier in the calculation than in the experiment. The planned valve actions are meaningfully different from the those actually performed in the experiment.
- (2) Ambient heat loss is considered to be one of the reasons for both overestimation and underestimation of pressure in the early stage and late stage, respectively.
- (3) The early dryout observed just after the break is not predicted to occur because the relief valve is closed earlier in the calculation. The core wide dryout is predicted to occur early in the upper part of the core and late in the upper part of the bundle in comparison with the experimental data. This means the phase separation in the test section is not correctly predicted.
- (4) Most of the key phenomena observed in the experiment such as the downcomer dryout, core flow reversal and lower plenum flashing are adequately predicted to occur in the present pre-test prediction calculation. Especially the effect of the lower plenum flashing is in good agreement with the experimental data, and
- (5) THYDE-P1 is applicable to BWR. Improvement should be performed with special attention to the drift flux model.

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Table 2.2.1 Planned performance chronology

Start opening of the break valve (V120)	0.	s
Break valve fully open (Area 1.13 cm ²)	0.2	s
Steam relief valve starts to open (V112)	0.2	s
Steam relief valve fully open	0.5	s
Steam relief valve starts to close	1.35	s
Spray flow, feed water flow and cooler flow valves start to close (V104, V106, V107)	1.4	s
Steam relief valve closed	1.65	s
Spray flow, feed water flow, cooler flow valves closed	1.8	s
Steam relief valve starts to open	11.8	s
Steam relief valve fully open	12.1	s

The test is terminated on the first occurrence of one of the following conditions:

- a) 120 s
- b) Clad temperature exceeds 700 °C
- c) Pressure drops below 2 MPa

Table 2.2.2 Measured performance chronology

Break valve starts to open	0 s
Break valve fully open	0.08 s
Steam relief valve (SRV) starts to open	0.4 s
SRV open *	1.0 s
SRV starts to close	1.6 s
Spray flow valve starts to close	1.65 s
Valve V104 (to cooler) starts to close,	1.78 s
Spray flow closed	1.9 s
Feed water valve starts to close	1.9 s
Valve V104 closed	2.0 s
Feed water closed	2.0 s
SRV closed	2.3 s
SRV starts to open	12.05 s
SRV open *	12.65 s
Test termination	76 s

Table 2.3.1 Planned initial conditions

Steam dome pressure	7.0	MPa
Core inlet flow	5.34	kg/s
Core power	3.5	MW
Core inlet subcooling relative to steam dome temperature	19	°C
Bypass flow	0.66	kg/s
Bypass power	62.5	kW
Flow through pump P1	4.5	kg/s*
Flow through pump P2	1.5	kg/s*
Water spray to the steam dome	5.905	kg/s
Water spray subcooling relative to steam dome conditions	106	°C
Feed water flow	2.95	kg/s
Feed water subcooling relative to steam dome conditions	106	°C
Mass flow rate through the cooler	8.855	kg/s

Table 2.3.2 Planned initial conditions and rated values for recirculation pumps

Pump P1:		
Rated pump speed	303.69	rad/s
Initial pump speed ratio	0.5399*	
Rated pump flow	0.0383	m ³ /s
Rated pump head	50	m
Rated pump torque	81.7	Nm
Pump rated density	1000	kg/m ³
Pump P2:		
Rated pump speed	303.69	rad/s
Initial pump speed ratio	0.7018*	
Rated pump flow	0.01056	m ³ /s
Rated pump head	25	m
Rated pump torque	13.2	Nm
Pump rated density	1000	kg/m ³

Table 2.3.3 Measured initial conditions

Steam dome pressure	6.997 MP
Bundle inlet flow	5.51 kg/s
Bundle power	3.385 MW
Bundle inlet temperature	542.0 K
Bypass flow	0.6 kg/s
Bypass power	57.4 KW
Flow through pump P1	4.56 kg/s
Flow through pump P2	1.55 kg/s
Spray flow	5.36 kg/s
Feedwater flow	2.49 kg/s
Feedwater and spray temperature	454.0 K
Mass flow through cooler	7.85 kg/s
Water level in steam dome above loop reference plane	6.34 m
Pump speed, P1	160 rad/s
Pump speed, P2	210 rad/s
Steam dome TE21	Saturated
Steam dome bottom TE25	542.5 K
Downcomer inlet TE26	541.8
Downcomer bottom TE31	541.6
Intact recirculation line TE32	541.0 K
Lower plenum TE2	542.0 K
Upstream break TE34	539.1 K
Bypass bottom TE17	541.5 K
Guide tube volume TE16	540.6 K
Bypass outlet TE20	556.4 K

Table 3.1.1 Description of nodes and heat slabs

Node No.	Description
1	Lower plenum
2	Lower plenum baffle inlet
3-12	Test section
13	Upper plenum lower part and upper part
14	Steam separator
15	Lower part of spray condensor
16-20	Downcomer, upper horizontal part
21-26	Downcomer, upper vertical part
27	Downcomer, vertical part to pump P1
28	Pump P1
29	Intact recirculation line pressure vessel side
30	Pump 2 suction line to pump P2
31	Pump P2
32	Broken recirculation line, pump P2 to split break T-connection
33	Broken recirculation line, split break T-connection to break from lower plenum
34	Broken recirculation line back to lower plenum
35	Bypass channel lower horizontal part
36-40	Bypass channel vertical part
41	Bypass channel upper horizontal part
42	Coolant flow line from downcomer to pump P3
43	Pump P3
44	From pump P3 to evaporation cooler
45	Evaporation cooler
46	From outlet of evaporation cooler to feed water line
47-48	Spray line
49	Feed water line
50	Spray condensor (Steam dome)
51	Relief valve line
52	Guide tube simulator volume
53-54	Split break line, from T-connection to break flow restrictor KI2

Slab No.	Description
1	Non-heated part of average heater rod
2-11	Average heater rod
12	Non-heated part of average heater rod
13	Non-heated part of hottest heater rod
14-23	Hottest heater rod
24	Non-heated part of hottest heater rod
25-29	Bypass heater rod
30	Evaporation cooler

Table 3.6.1 Loss coefficients in loop

Item no	Description	Minimum flow area (cm ²)	Loss coefficient*	Reference flow area *** (cm ²)
1	Bundle inlet orifice **	4.73	72	60.34
2	Spacer		0.65	60.34
3	Upper tie plate	76	0.2	60.34
4	Steam separator flow restrictor	27.3	21	78.5
5	Pump P1 exit flow restriction	9.08	400	112.5
6	Measuring orifice, K1 (see figure 2:1)	13.9	14	42.7
7	Pump P2 exit flow restriction	3.80	230	34.3
8	Measuring orifice, K2	7.02	9.4	19.0
9	Bypass inlet, incl measuring orifice, K7 and valve V114	1.54	102	5.7
10	Split break line flow restriction, K12**	1.131	-	42.7
11	Guillotine break line from lower plenum flow restriction, K13**	3.664	-	34.3
12	Guillotine break line from downcomer flow restriction K11**	2.011-3.664	-	34.3
13	Steam line measuring orifice K6	14.3	8.4	42.6

*) Loss coefficient is defined as : $\xi = \frac{2 \cdot \Delta p}{\rho \cdot u^2}$

referred to the reference flow area. From preliminary, measured pressure drop; variation with Re has to be taken into consideration.

**) Variable parameter that will be subject to change between different experiments

***) Area to be used for sudden area change

Table 3.7.1 Ambient heat losses evaluated before experiment conducted

Steam condenser	13 KW
Upper plenum	12 KW
Test section	4 KW
Lower plenum	14 KW
Bypass channel	7 KW
Downcomer and recirculation lines	20 KW
Pump P1	5 KW
Pump P2	6 KW
Pump P3	5 KW
Spray and feedwater lines	<u>22 KW</u>
Total	<u>108 KW</u>

Table 3.7.2 Measured ambient heat losses

Cooling flow between teflon filler and vessel:	
Inlet temperature	403 K
Outlet temperature	493 K
Mass flow	0.565 kg/s
Total loss	204 kW

Table 4.1 Chronology of key events

<u>Key Event</u>	<u>Time (s)</u>	
	<u>Experiment</u>	<u>Prediction</u>
Spray condensor emptied	6.5	5.5
Maximum steamdome pressure	7.6	7.0
Bypass inlet flow reversal	10	8.8
Start of flashing in bypass	12	11
Start of flashing in broken loop recirculation line (downcomer side)	16	12
Start of flashing in intact loop recirculation line	21	14
Start of flashing in broken loop recirculation line (lower plenum side)	22	20
Start of flashing in broken loop recirculation line (lower plenum side)	22	20
Start of flashing in lower plenum	22	19
Start of flashing in control rod guide tube volume	25	26
Core uncover	50-64	38
Transient terminated	76	120

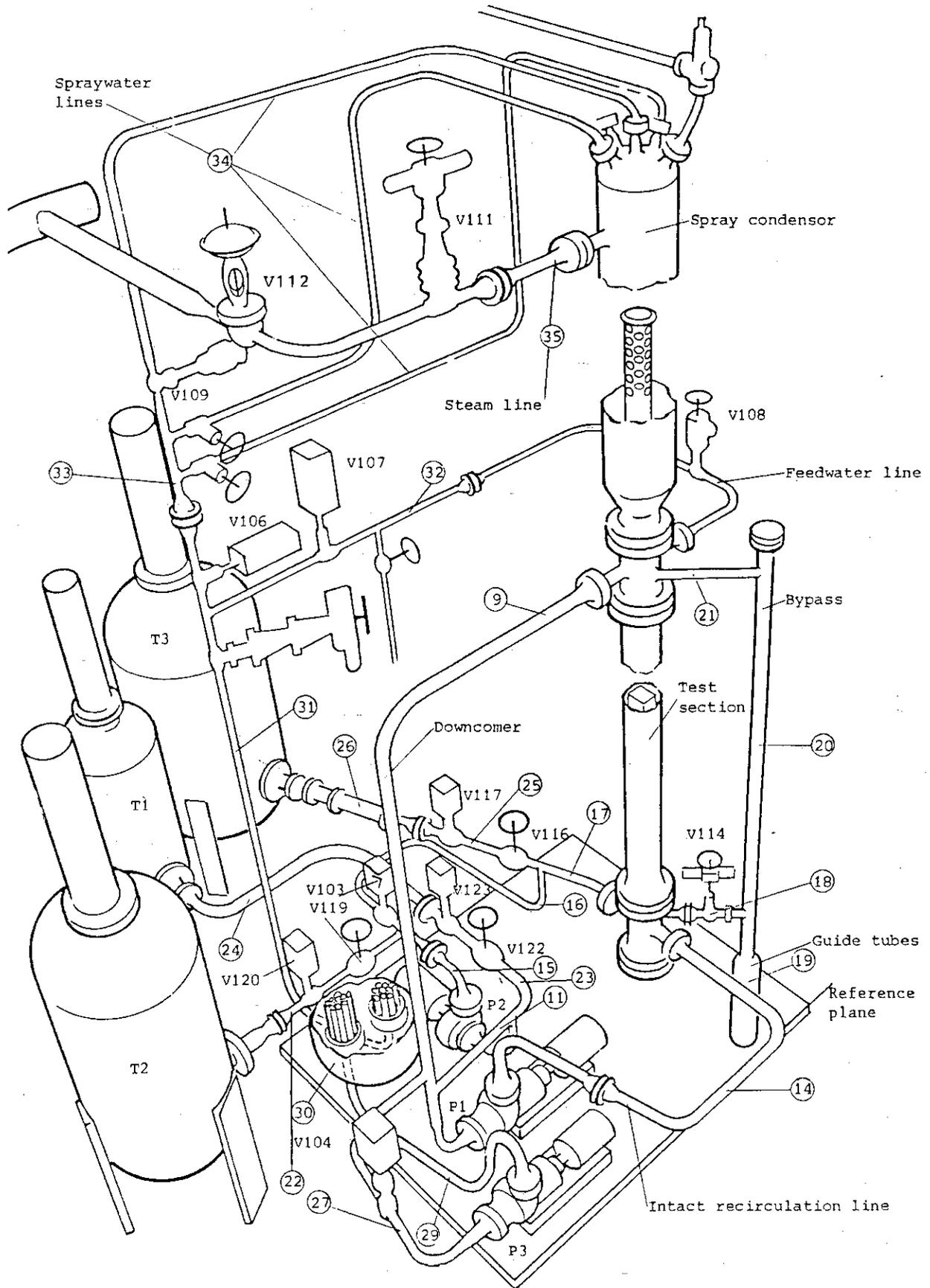


Fig. 2.1.1 Lay-out of FIX II loop

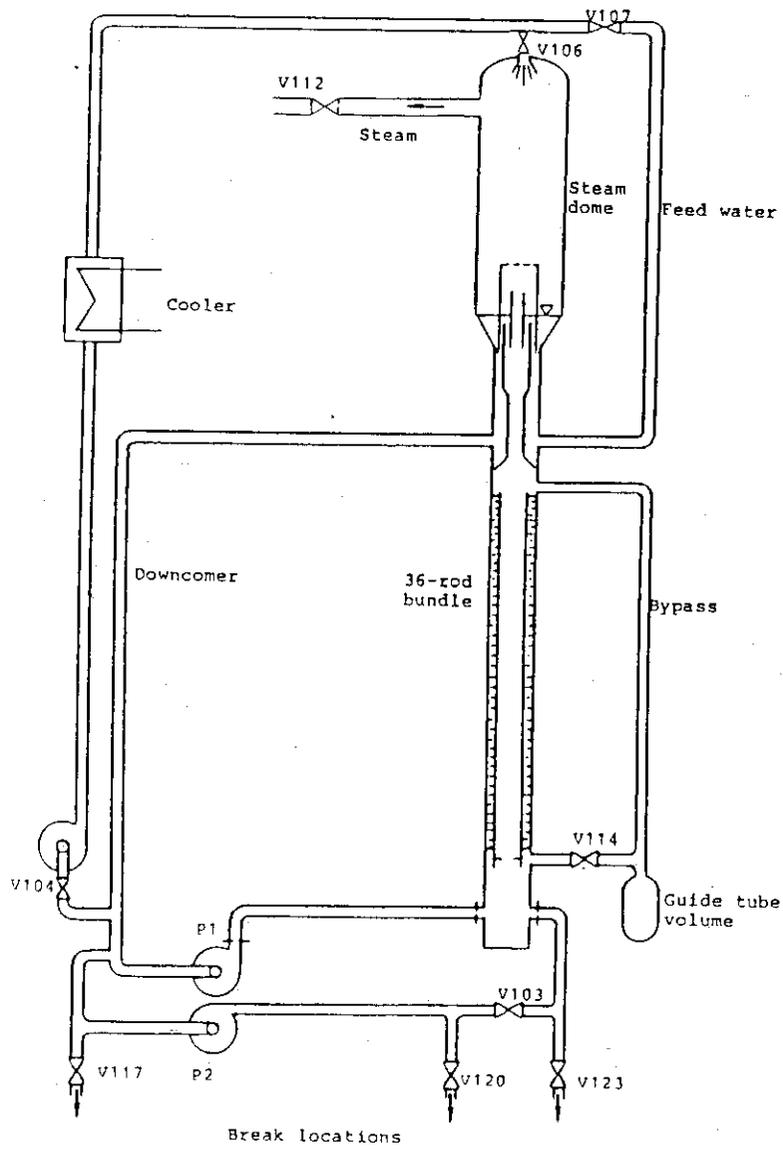


Fig. 2.1.2 Schematic diagram of central loop section

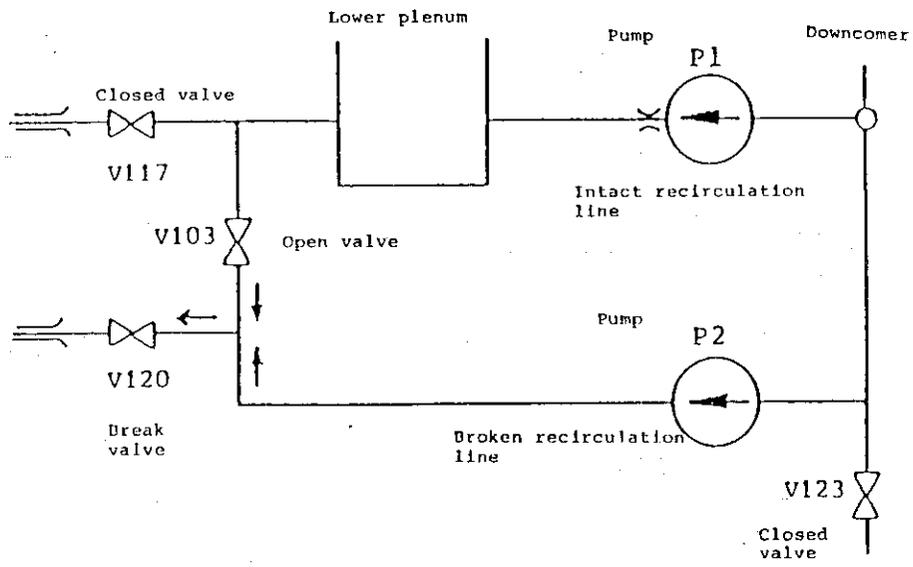
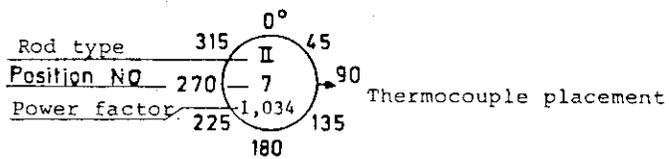
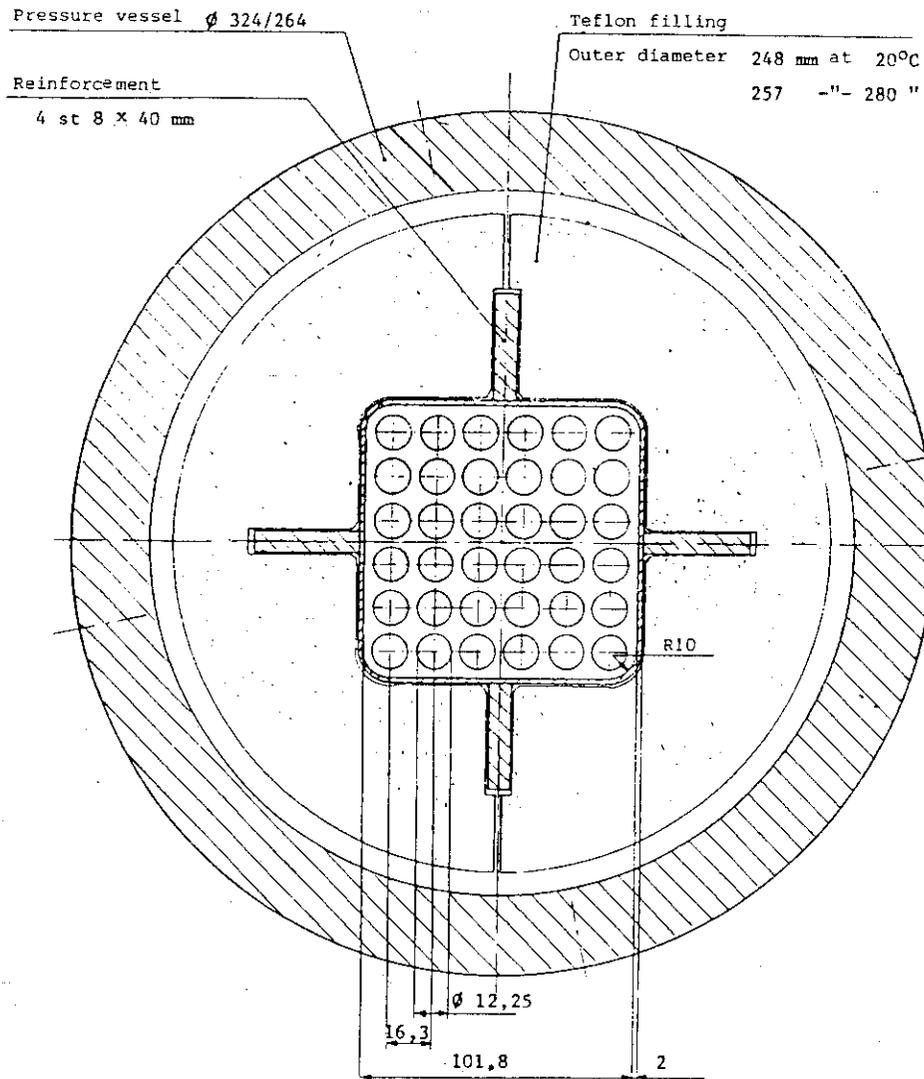


Fig. 2.1.3 Split break simulation



II 1 1,029	II 2 1,007	III 3 ,974	I 4 ,992	I 5 1,009	I 6 ,978
II 7 1,034	II 8 ,979	I 9 ,979	I 10 ,993	I 11 1,000	III 12 1,019
II 13 ,990	II 14 1,032	II 15 ,984	I 16 1,034	IV 17 1,005	II 18 ,992
IV 19 1,014	II 20 ,972	III 21 1,000	IV 22 1,016	IV 23 1,016	IV 24 1,011
I 25 ,989	III 26 ,957	III 27 ,996	III 28 1,032	IV 29 1,008	IV 30 ,989
III 31 ,999	III 32 1,009	III 33 ,979	I 34 1,005	II 35 1,007	IV 36 ,972

Fig. 2.1.4 Horizontal cut view of test section and radial power distribution

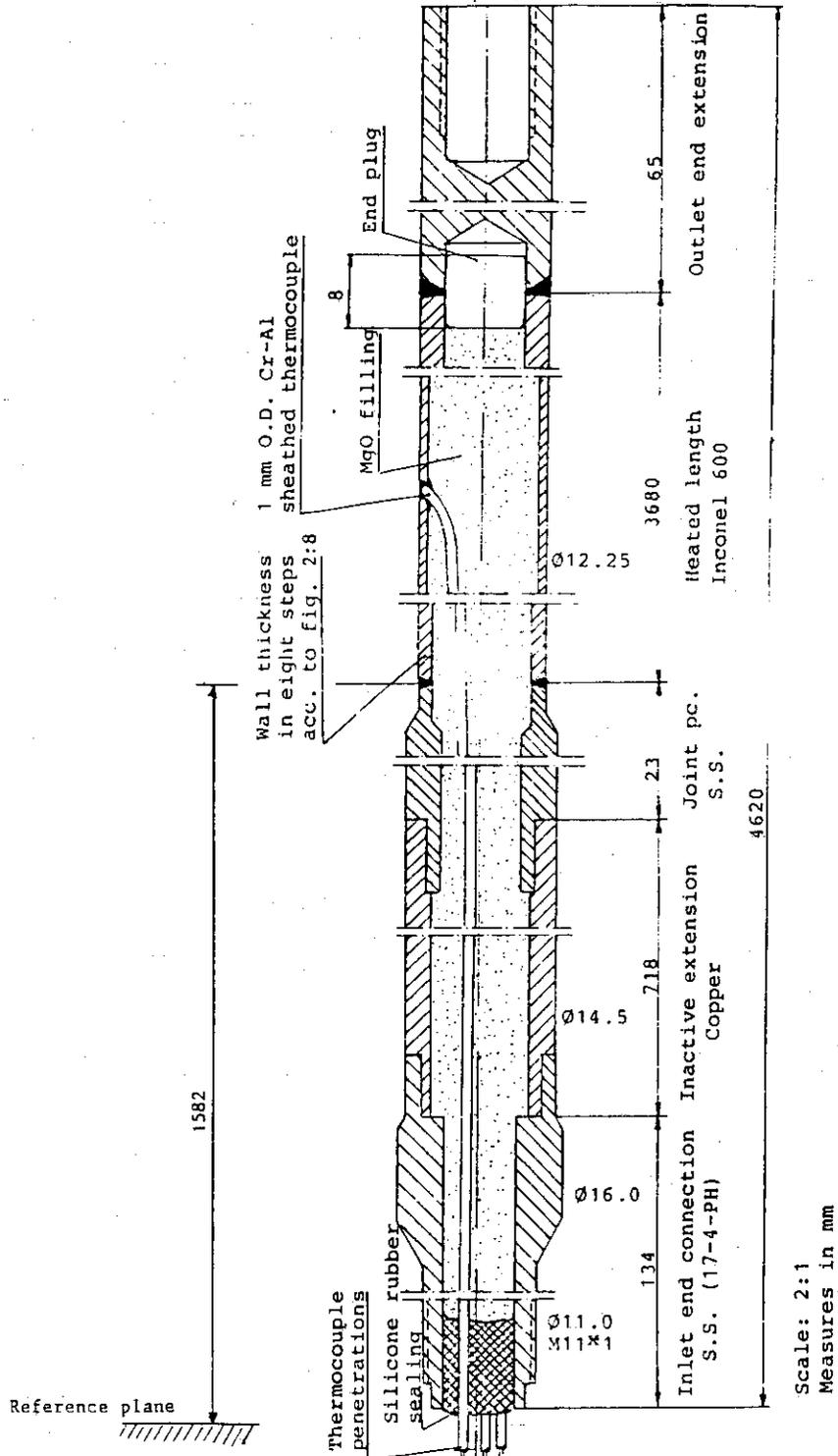


Fig. 2.1.5 Structure of fuel rod simulator

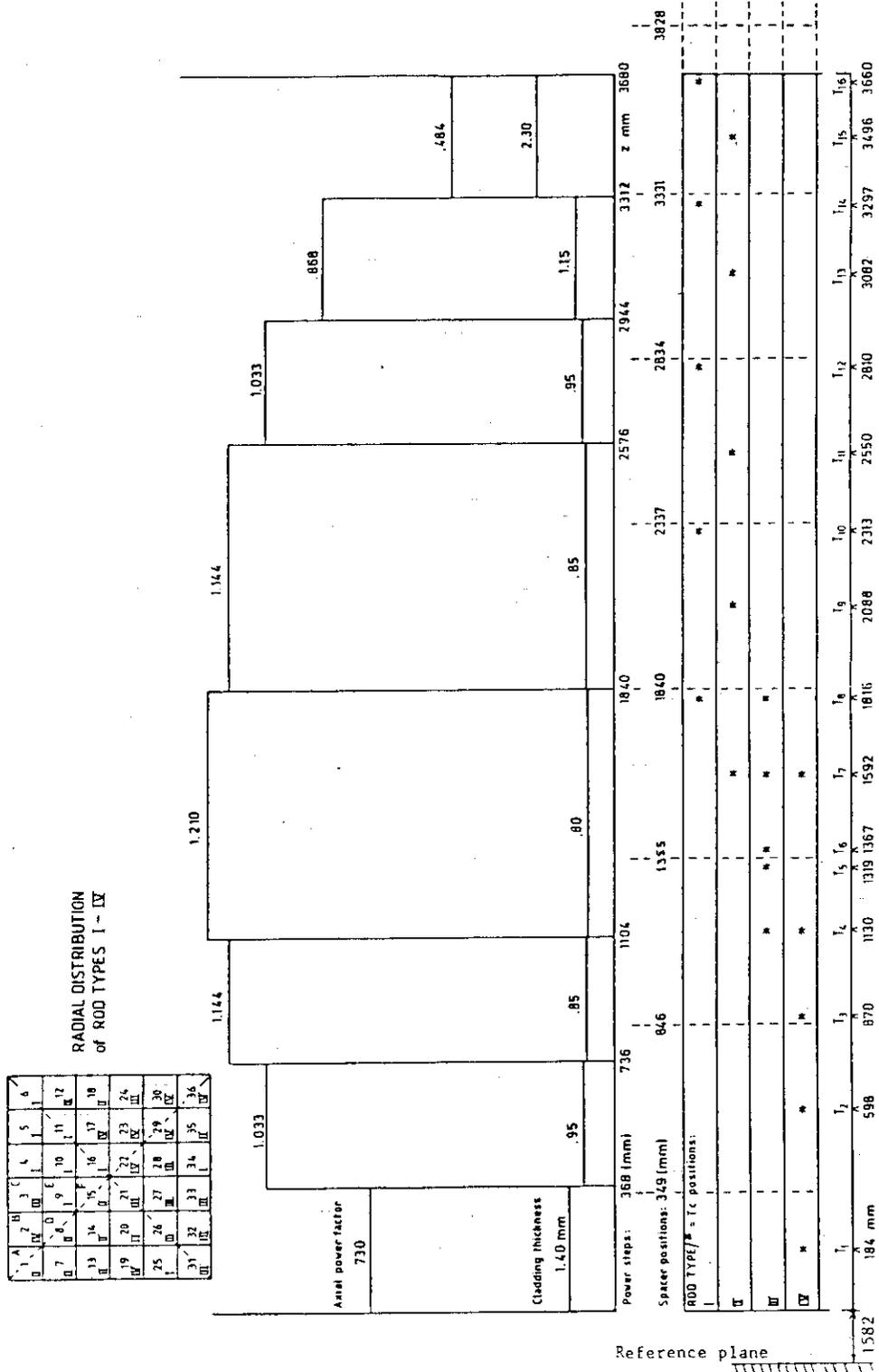


Fig. 2.1.6 Axial power distribution and instrumentation location

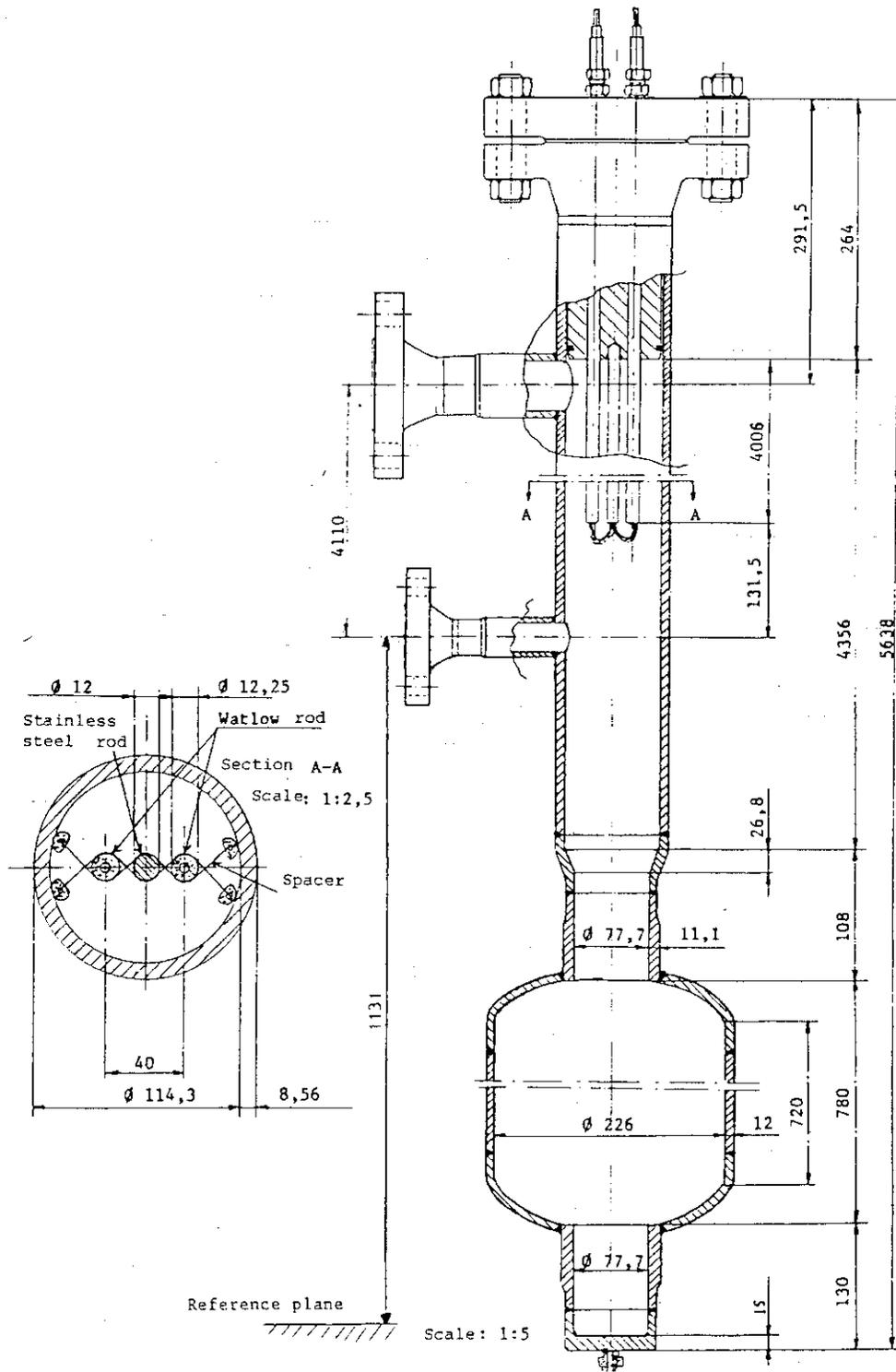


Fig. 2.1.7 Structure of bypass and control rod guide tube volume

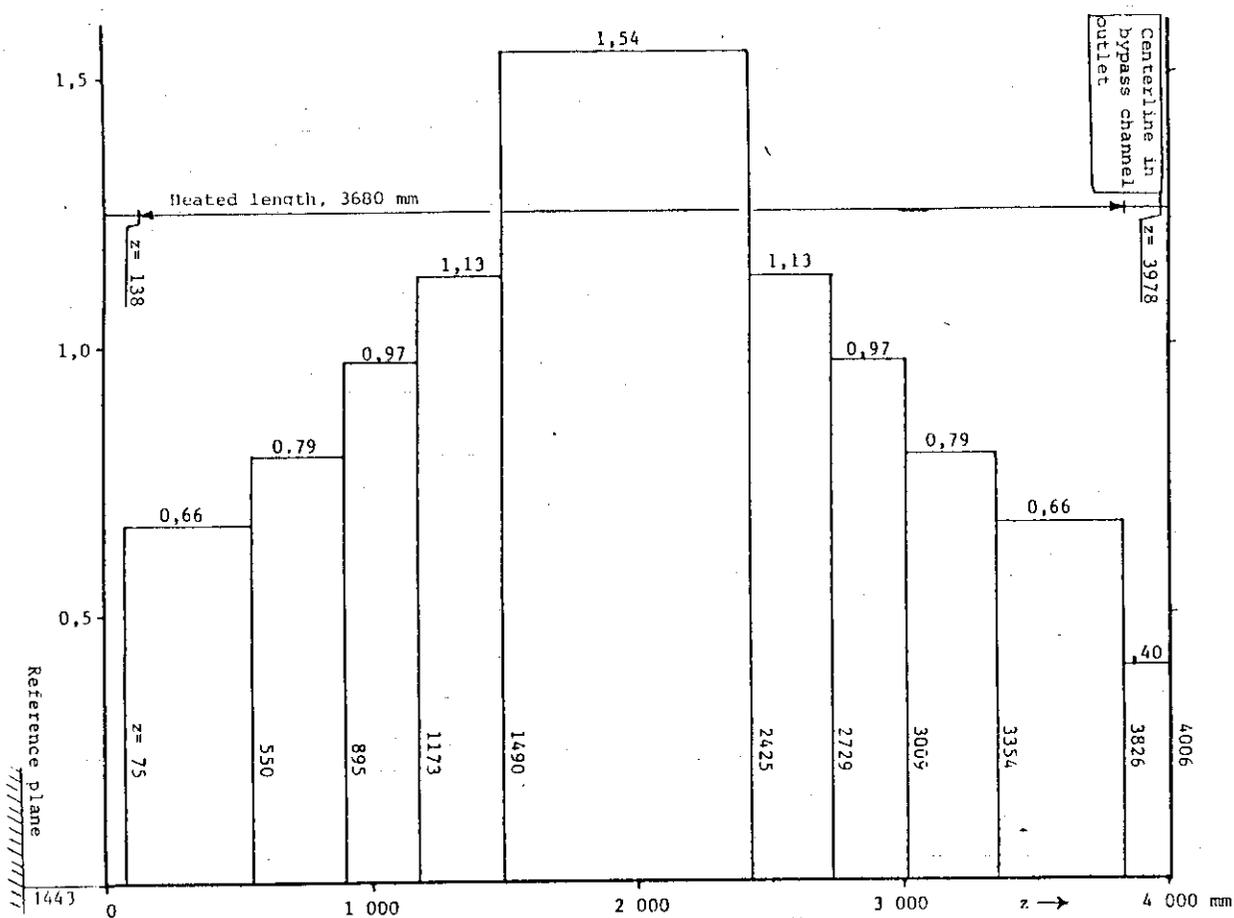


Fig. 2.1.8 Axial power distribution of bypass channel

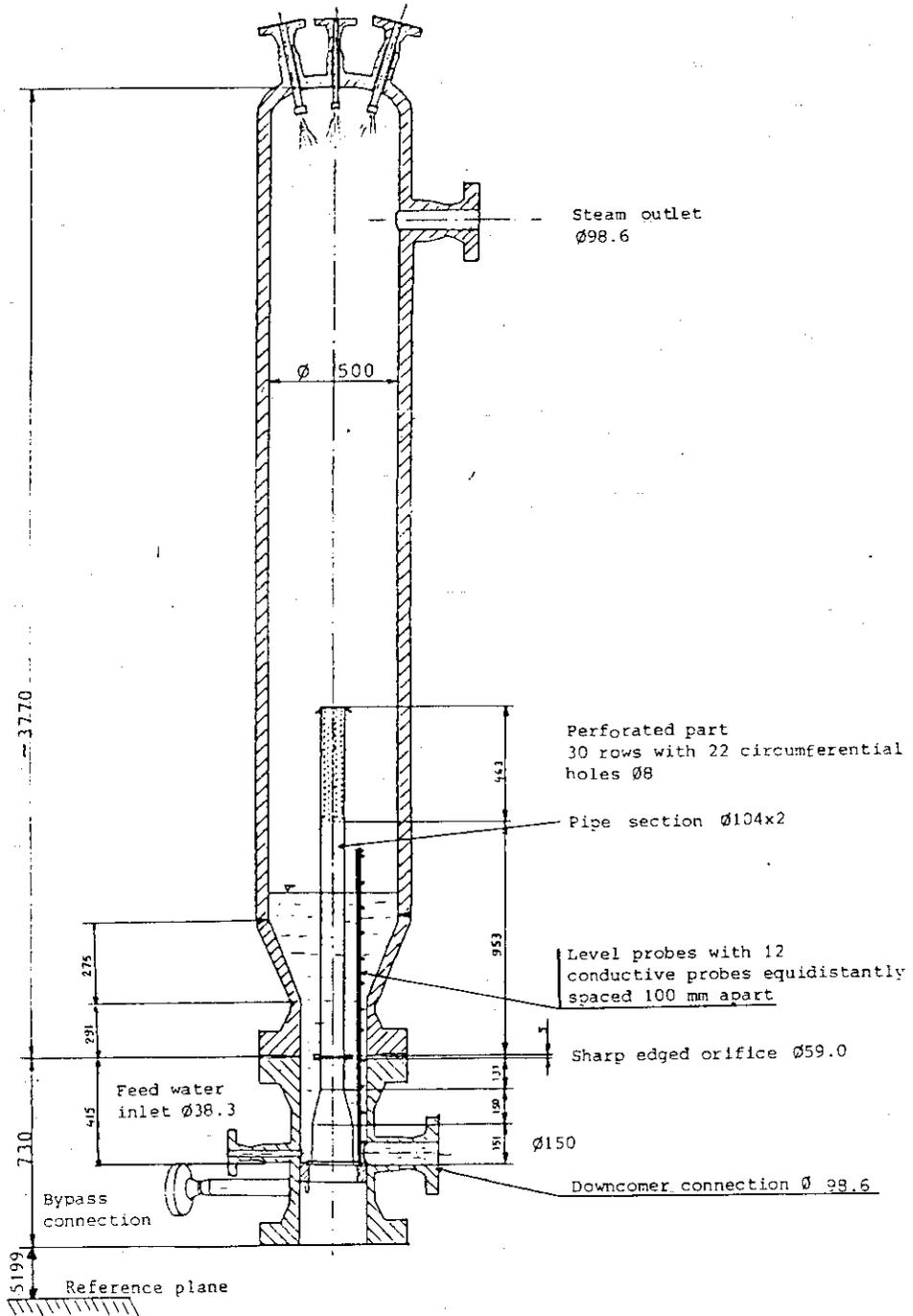


Fig. 2.1.9 Structure above test section

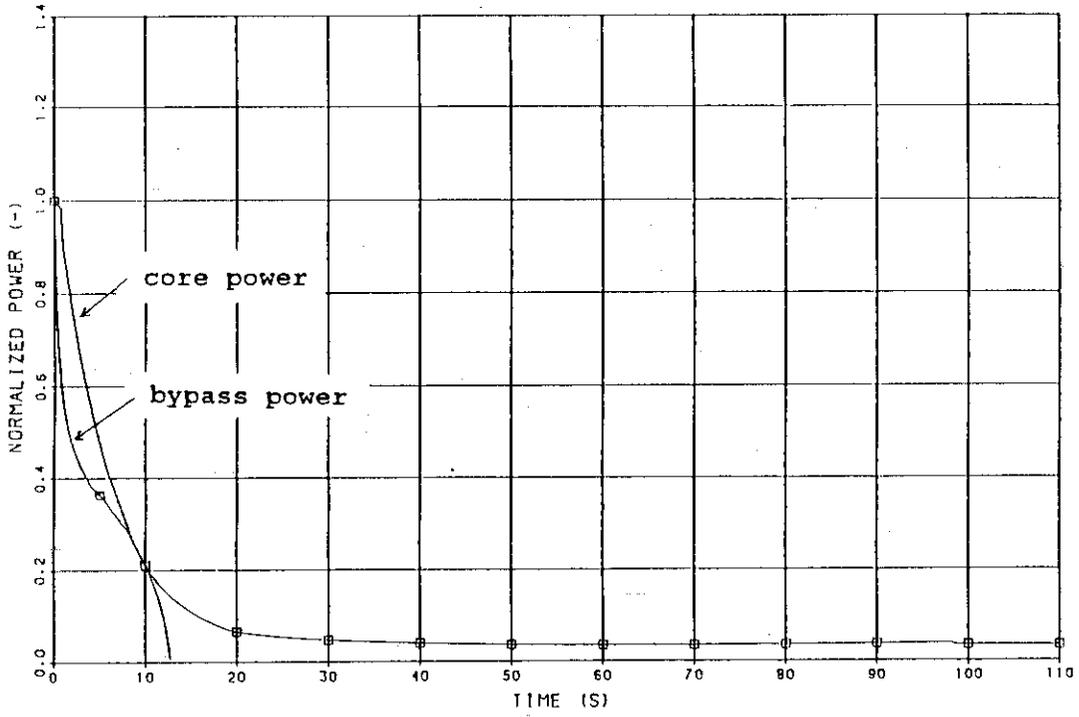


Fig. 2.3.1 Planned core and bypass power histories

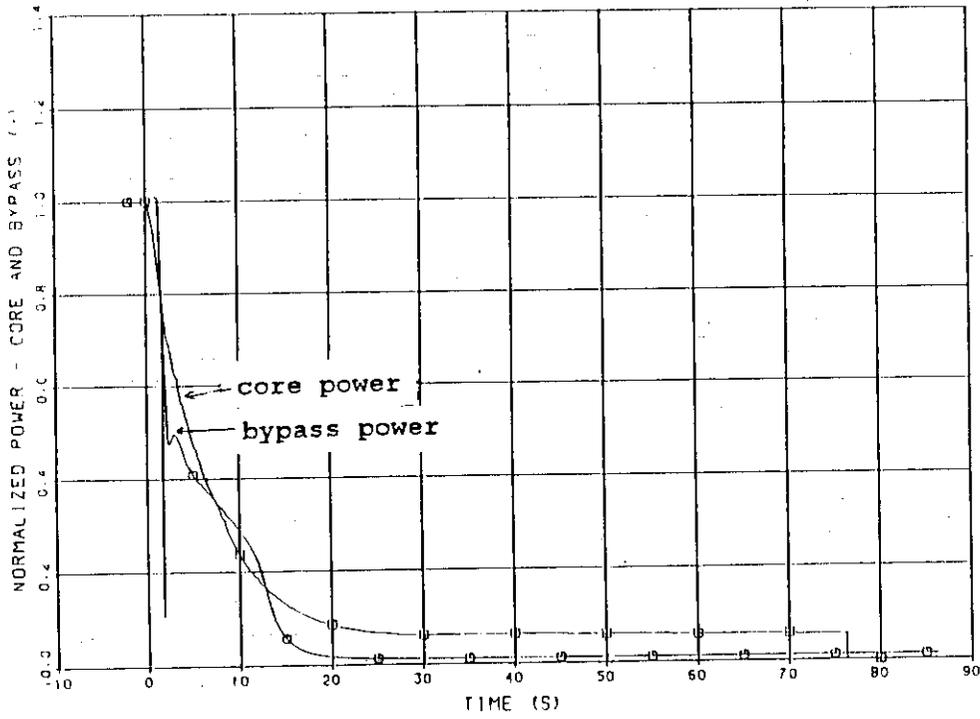


Fig. 2.3.2 Measured core and bypass power histories

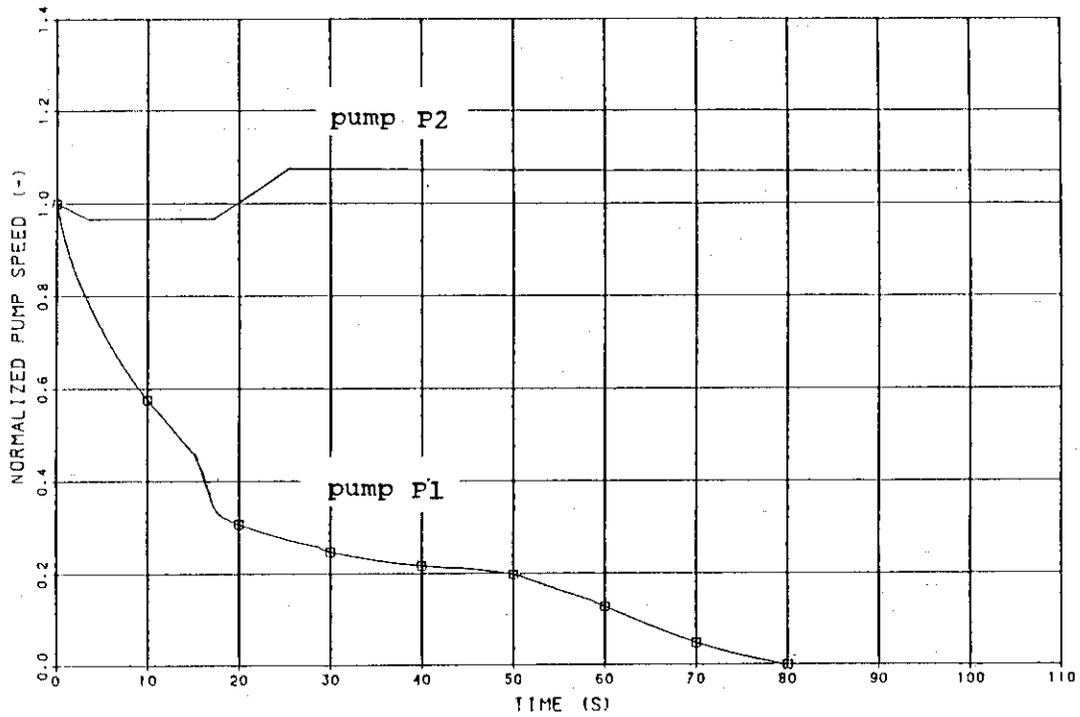


Fig. 2.3.3 Planned pump speed histories

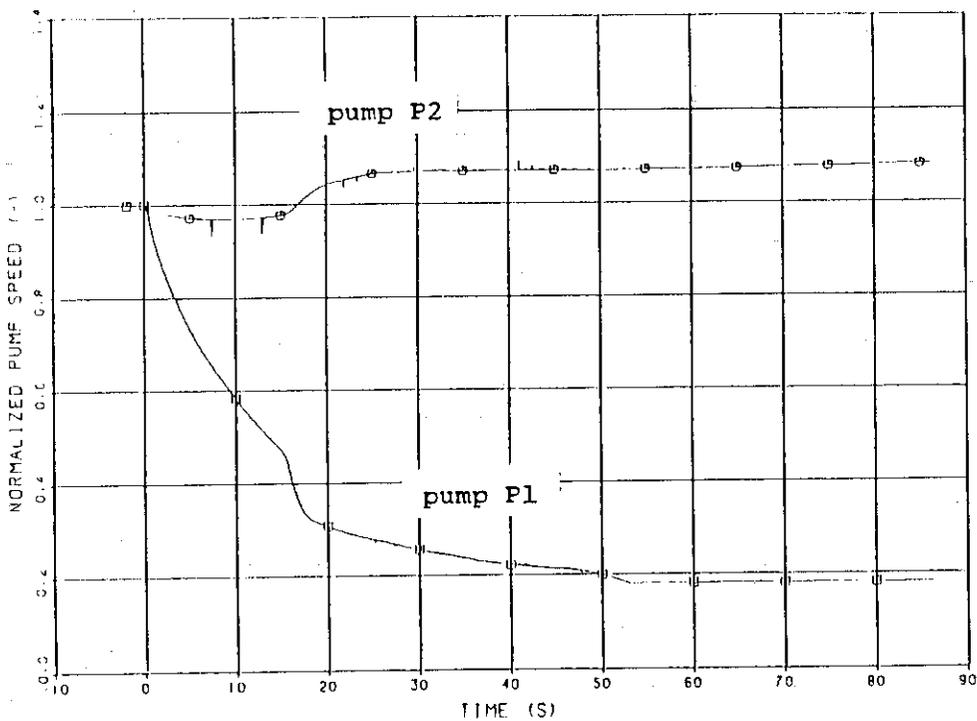


Fig. 2.3.4 Measured pump speed histories

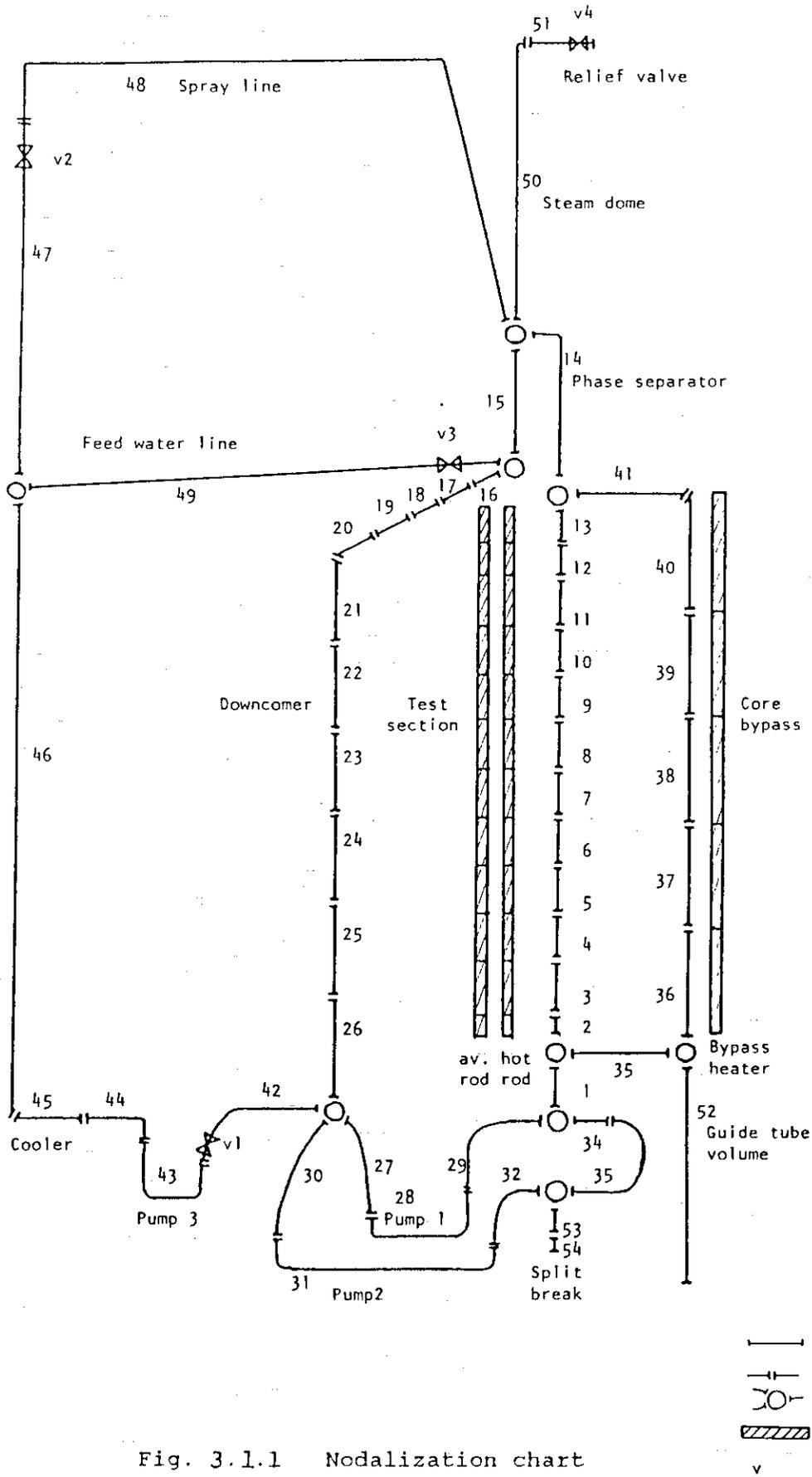


Fig. 3.1.1 Nodalization chart

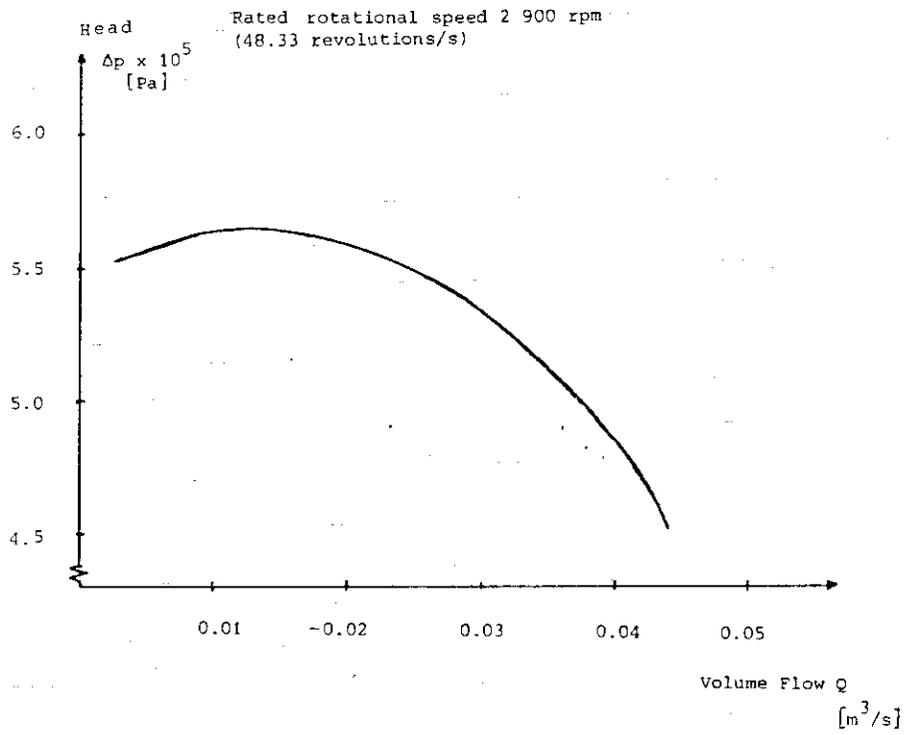


Fig. 3.5.1 Pump characteristic curve for pump P1

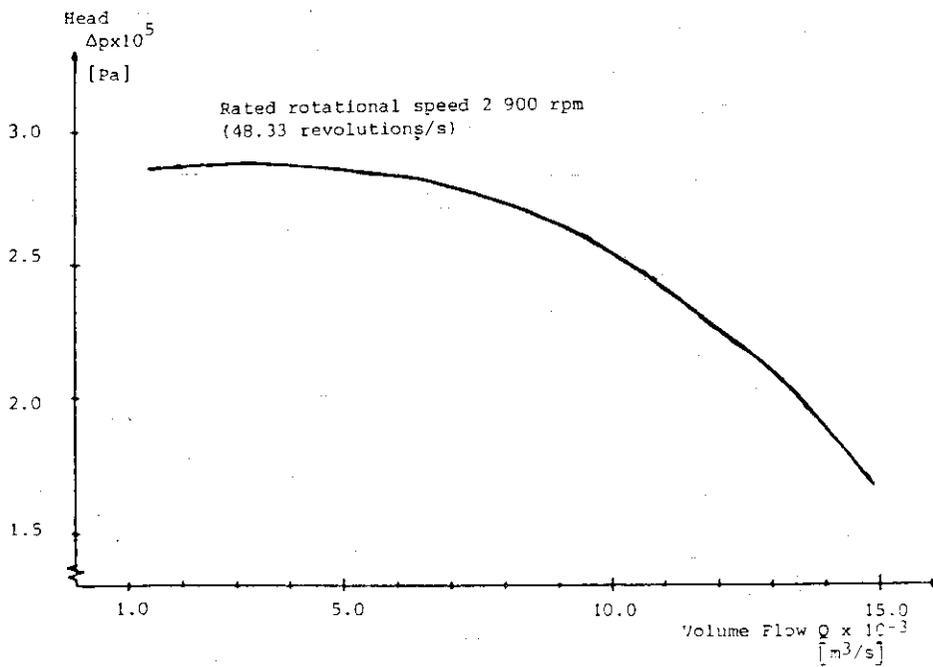


Fig. 3.5.2 Pump characteristic curve for pump P2

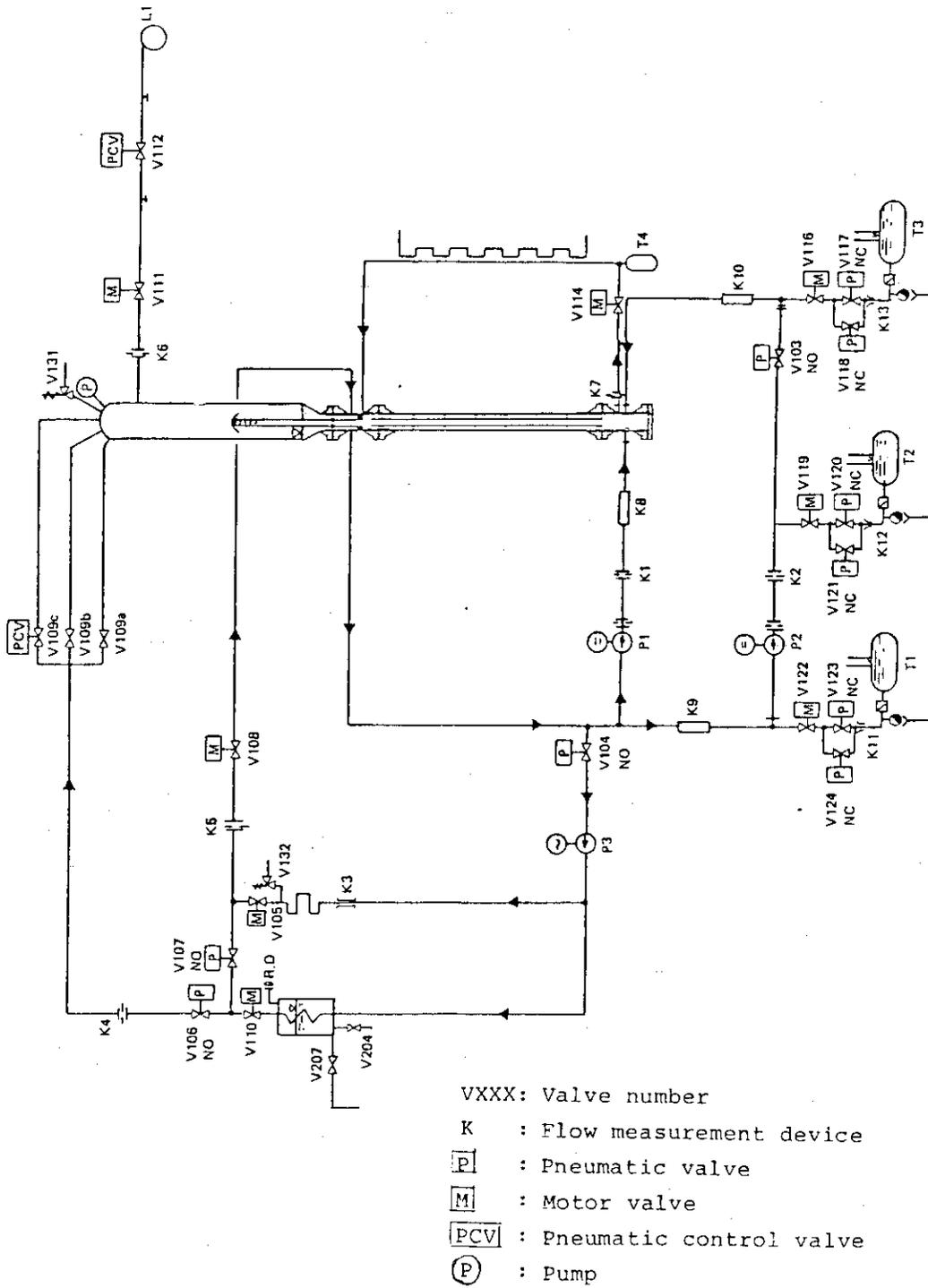
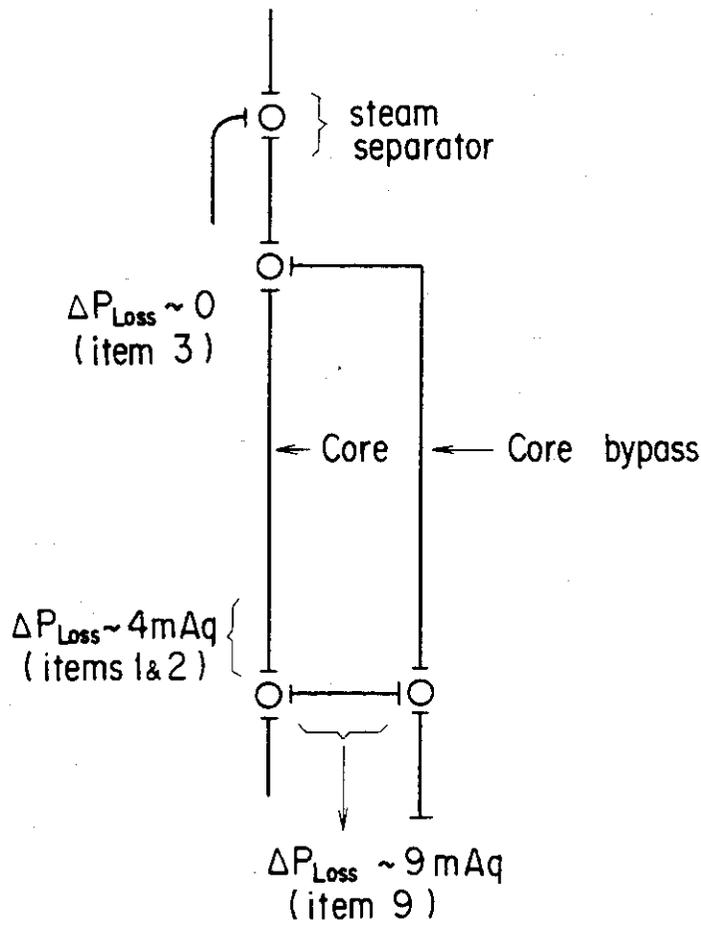


Fig. 3.6.1 Flow diagram of FIX II loop



Remarks

- 1) See Table 3.6.1 for item numbers
- 2) ΔP_{Loss} : pressure drop (m of water head)

Fig. 3.6.2 Pressure losses at core and bypass

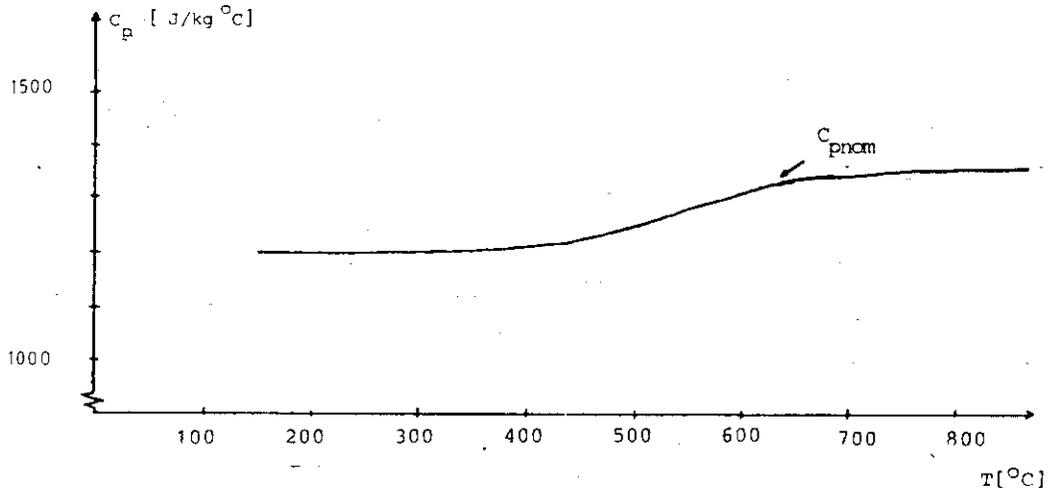


Fig. 3.8.1 Specific heat of fuel rod simulator filler material

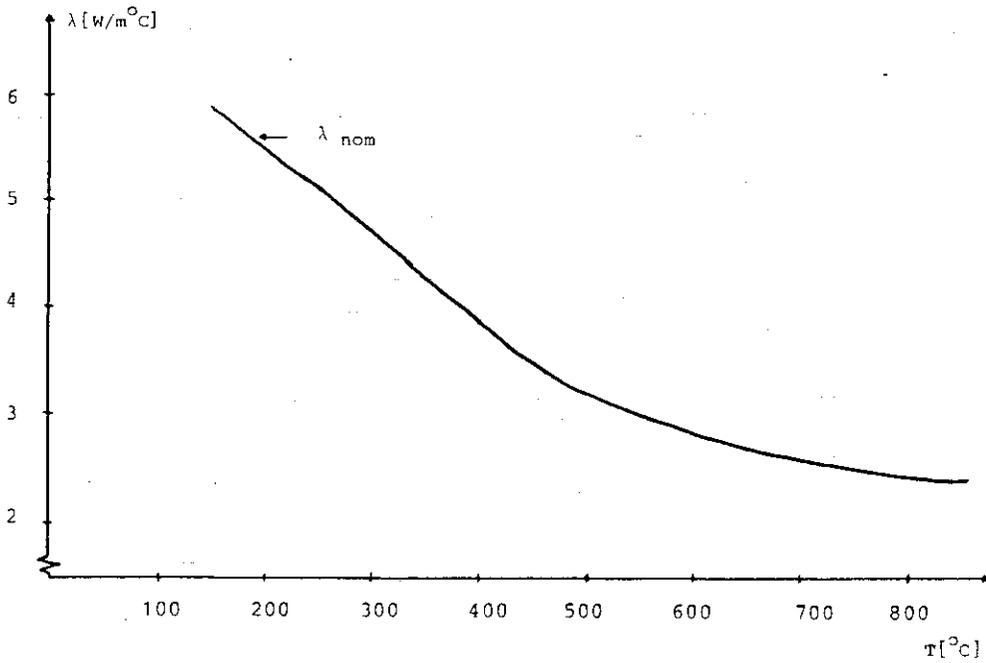


Fig. 3.8.2 Thermal conductivity of fuel rod simulator filler material

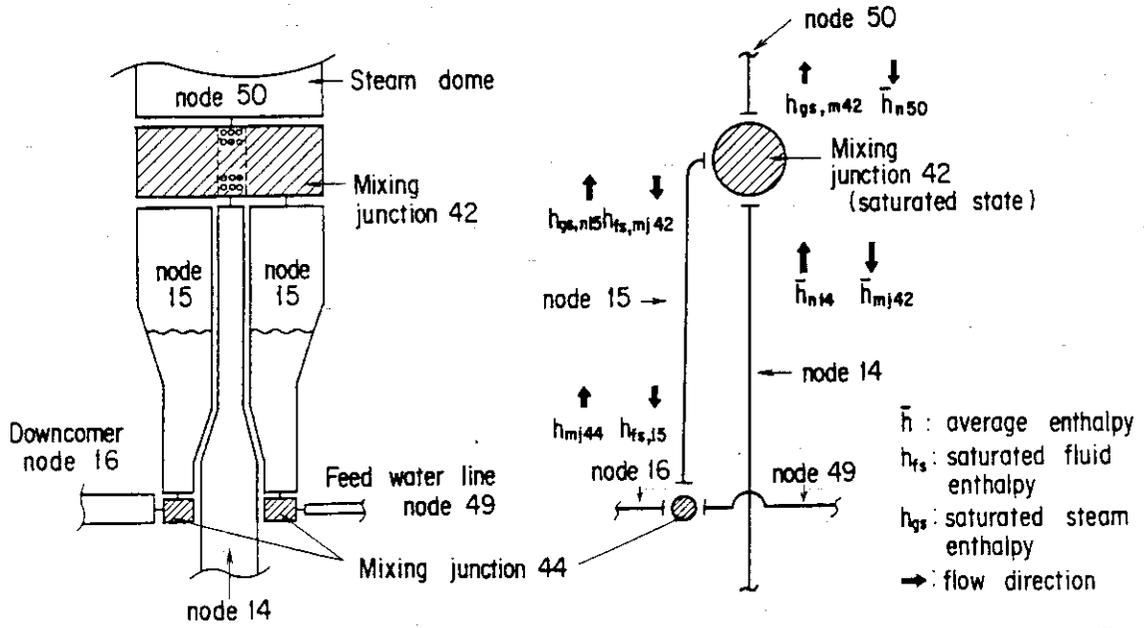


Fig. 3.9.1 Schematic diagram of phase separation

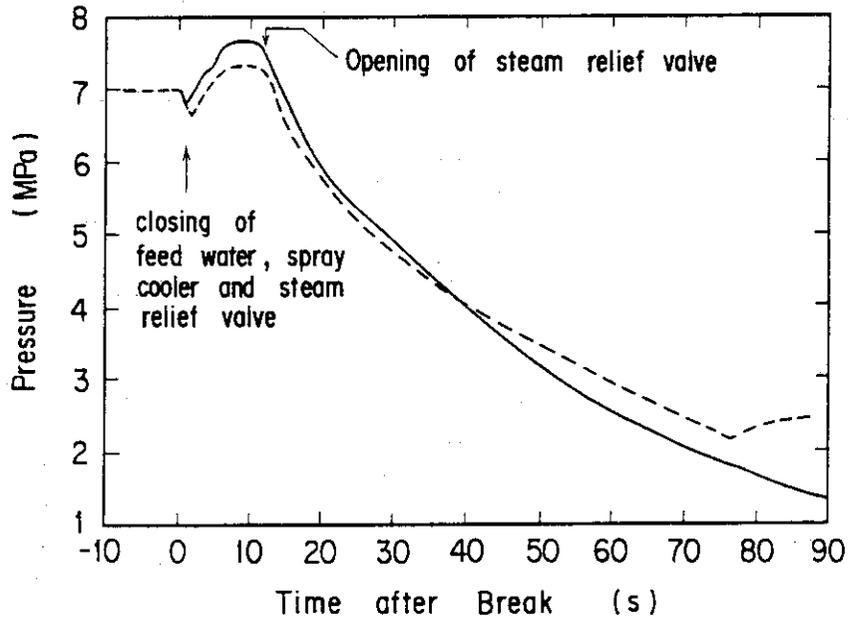


Fig. 4.1.1 Pressure at steam dome

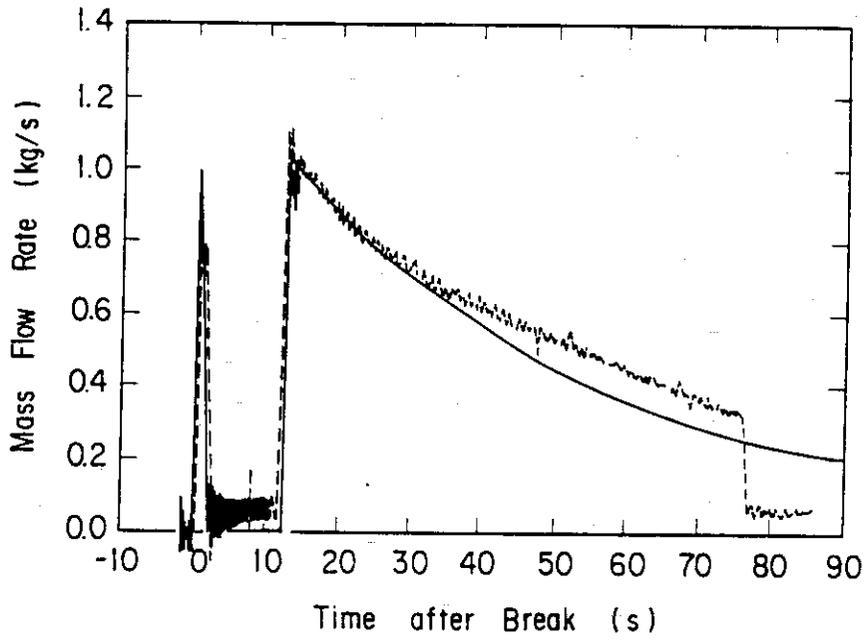


Fig. 4.1.2 Mass flow rate through steam relief valve

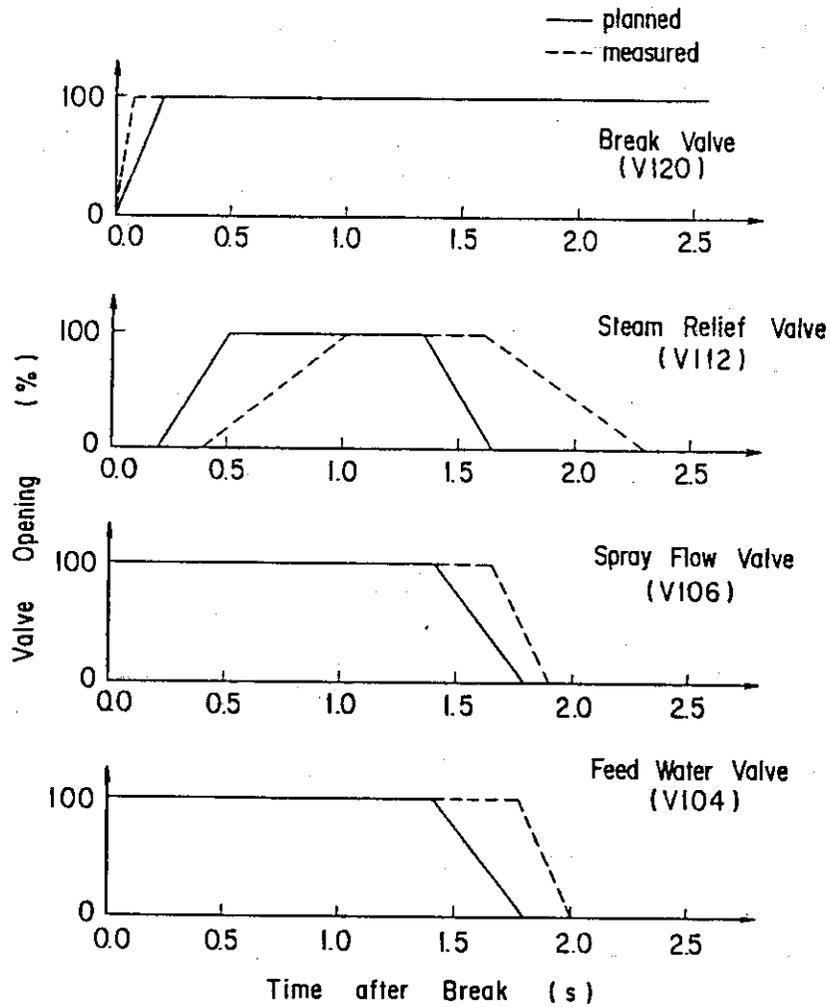


Fig. 4.1.3 Schematic diagram of valve actions

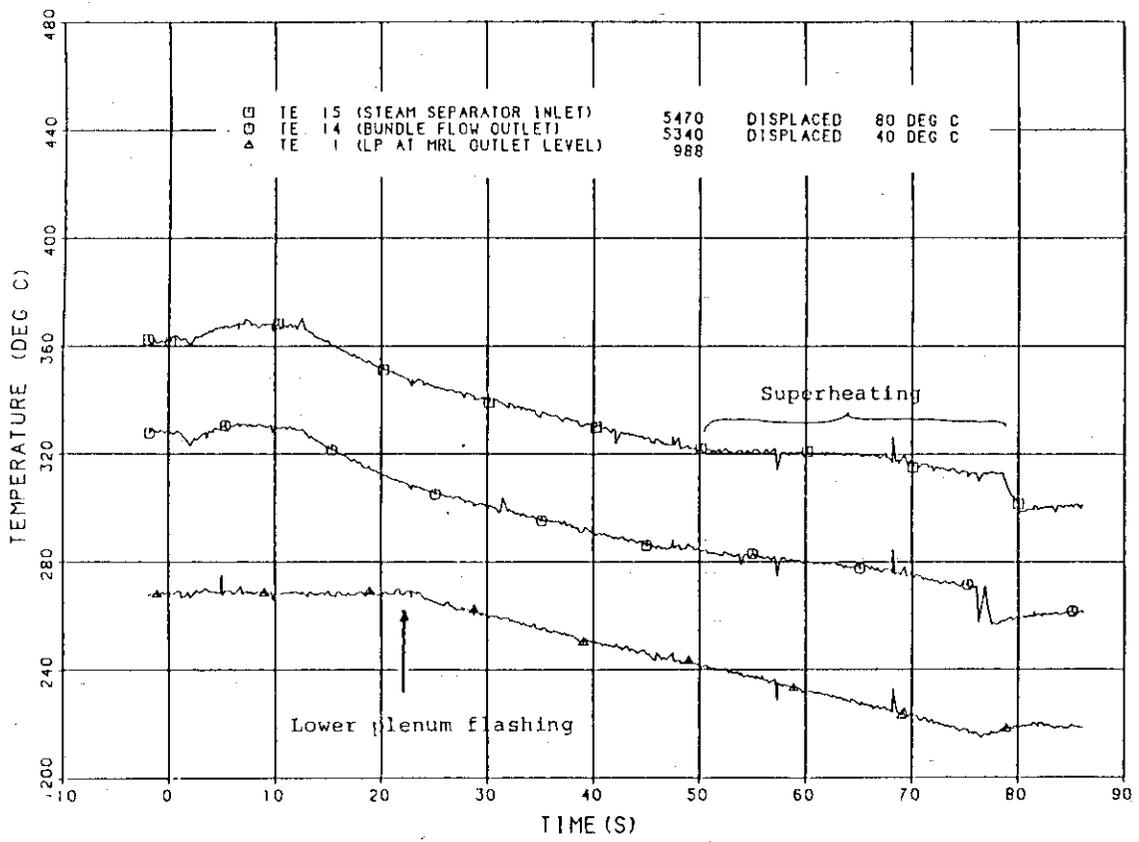


Fig. 4.1.4 Measured fluid temperatures at loer plenum and upper plenum

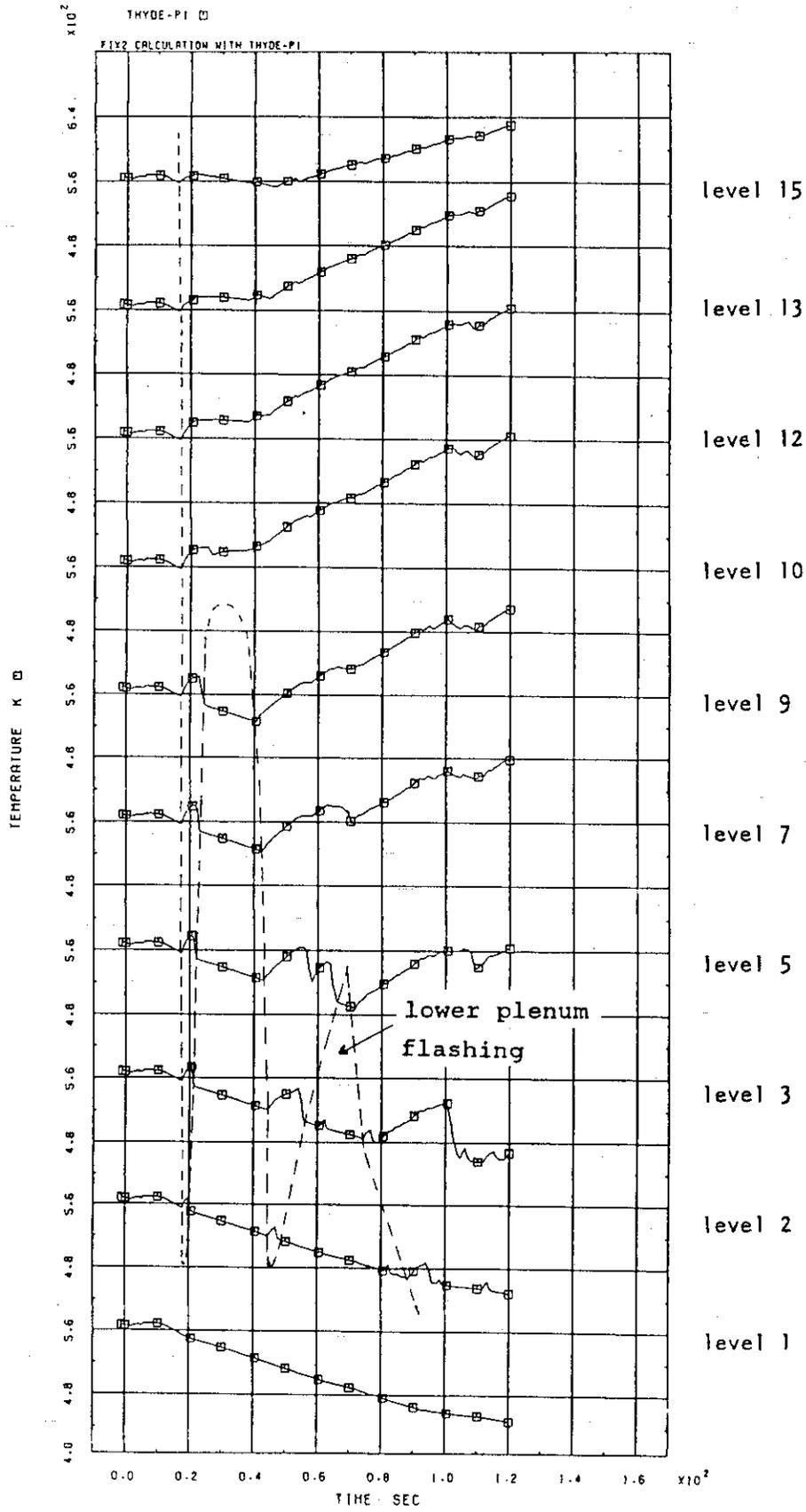


Fig. 4.2.1 Vertical distribution of calculated cladding surface temperature

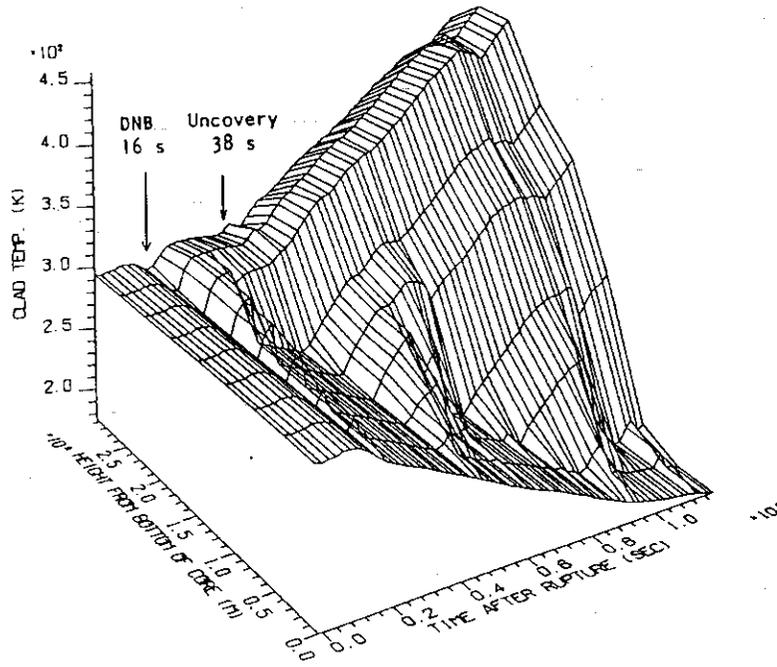


Fig. 4.2.2 Three-Dimensional view of calculated cladding surface temperature

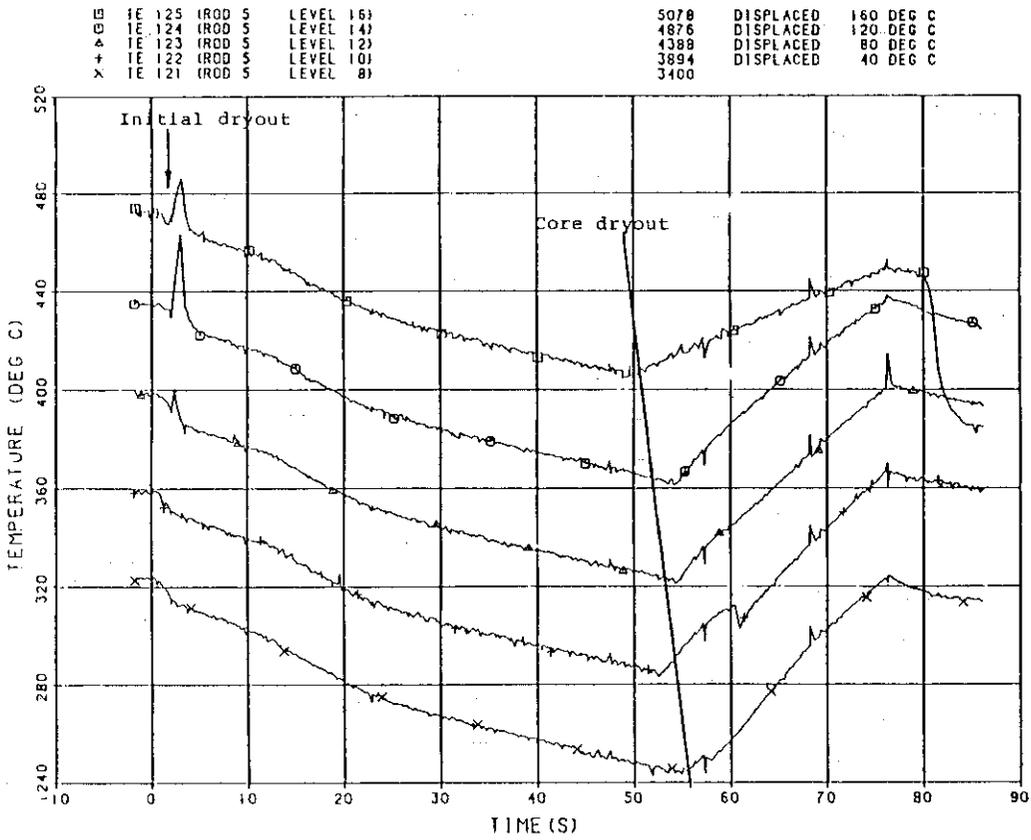


Fig. 4.2.3 Experimental cladding surface temperatures in upper part part of core

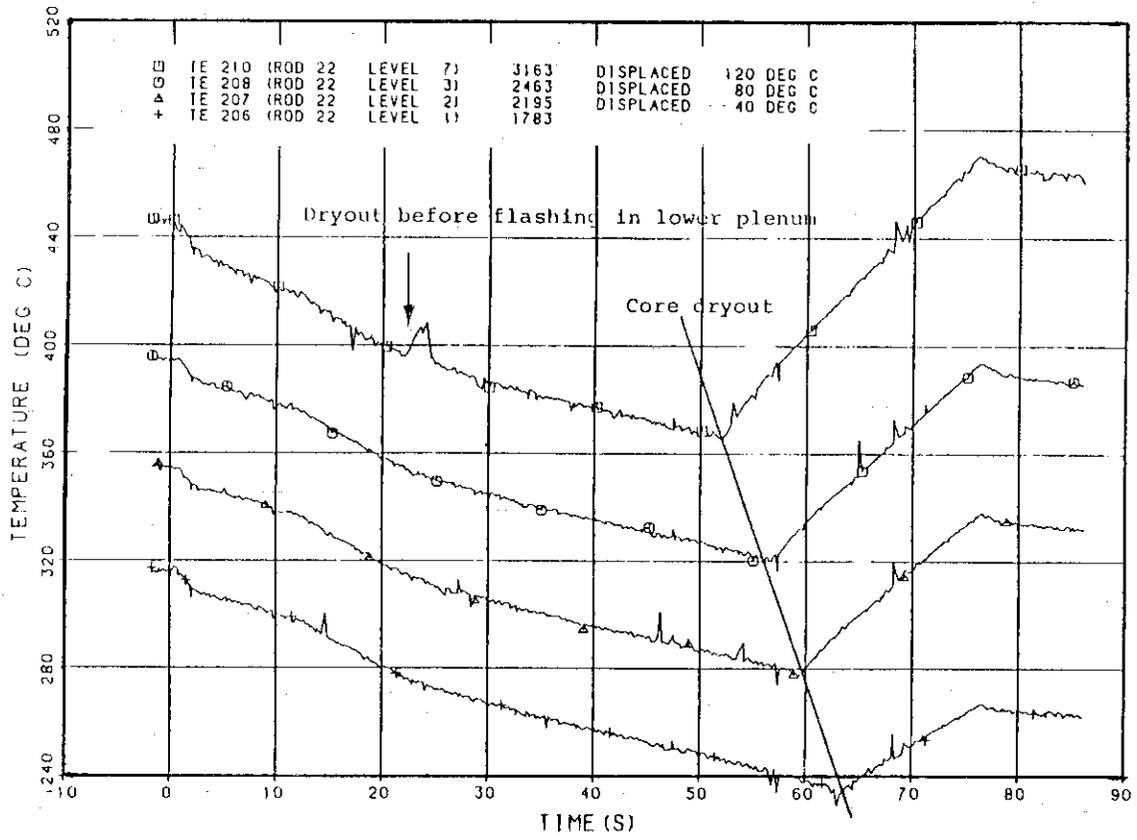


Fig. 4.2.4 Experimental cladding surface temperatures in lower part of core

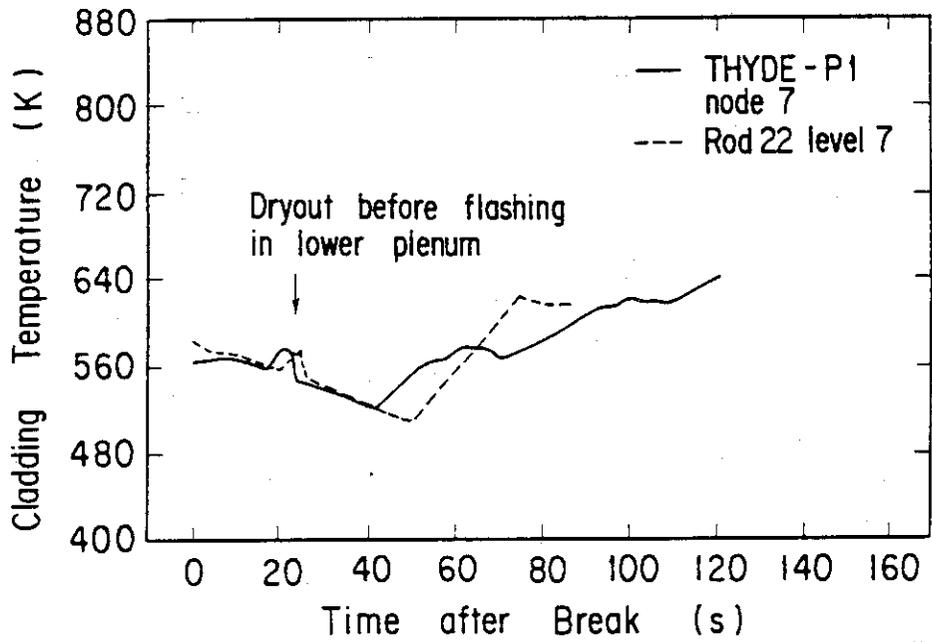


Fig. 4.2.5 Effect of lower plenum flashing on cladding surface temperature

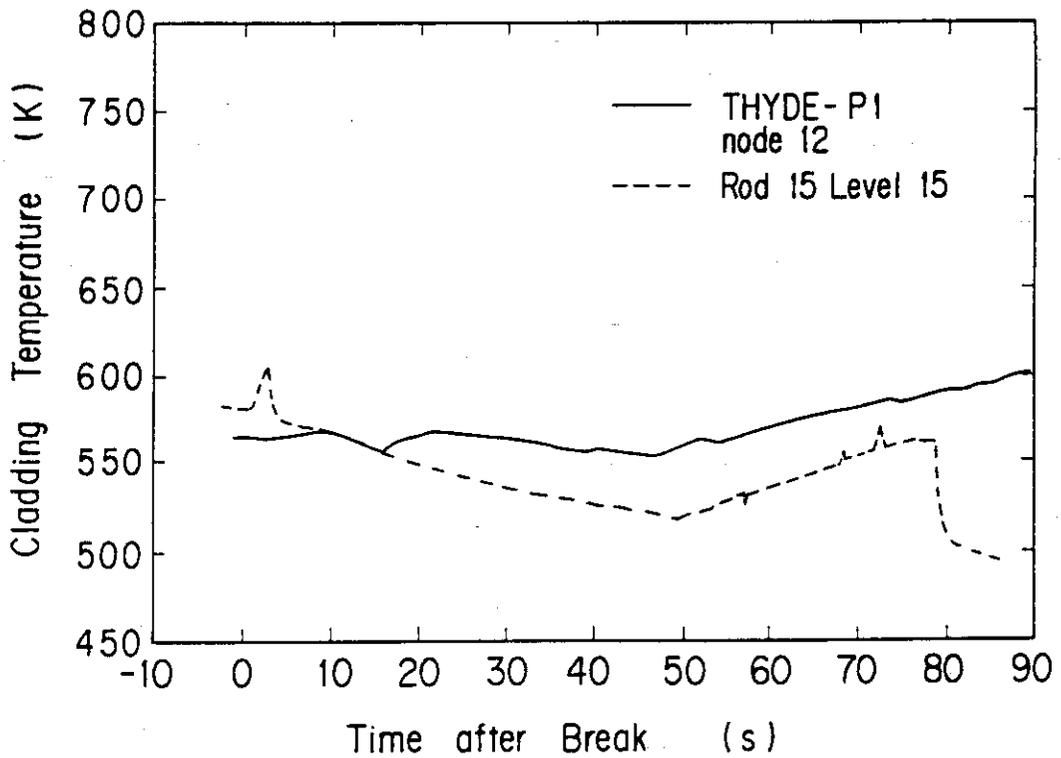


Fig. 4.2.6 Cladding surface temperature at level 1

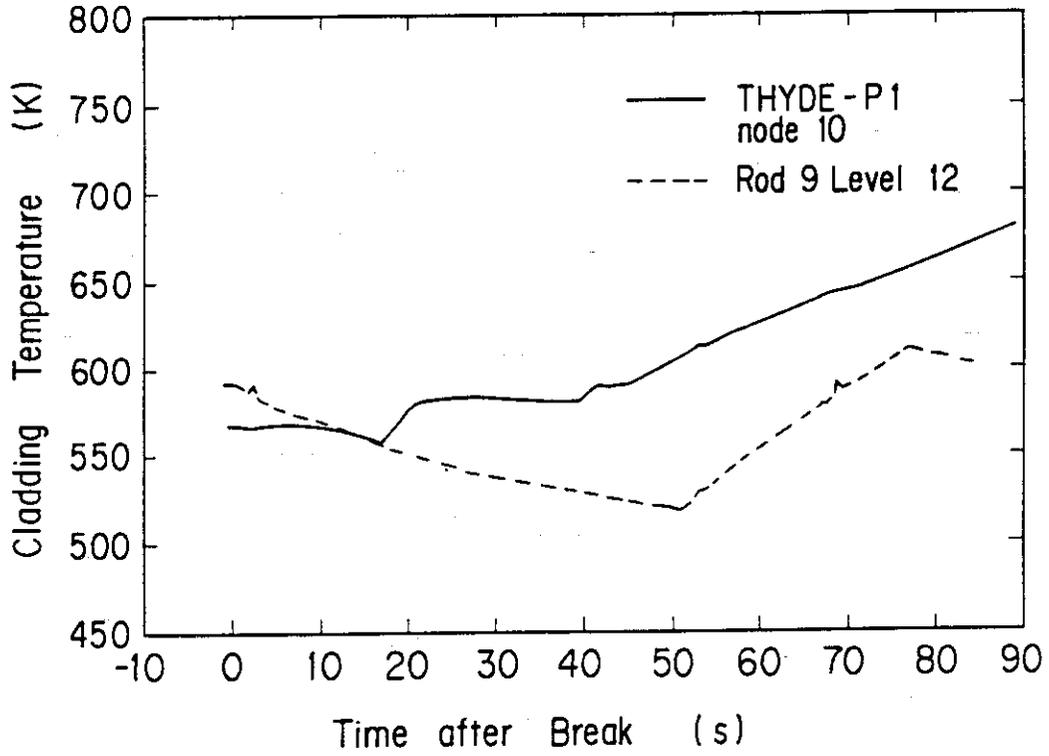


Fig. 4.2.7 Cladding surface temperature at level 3

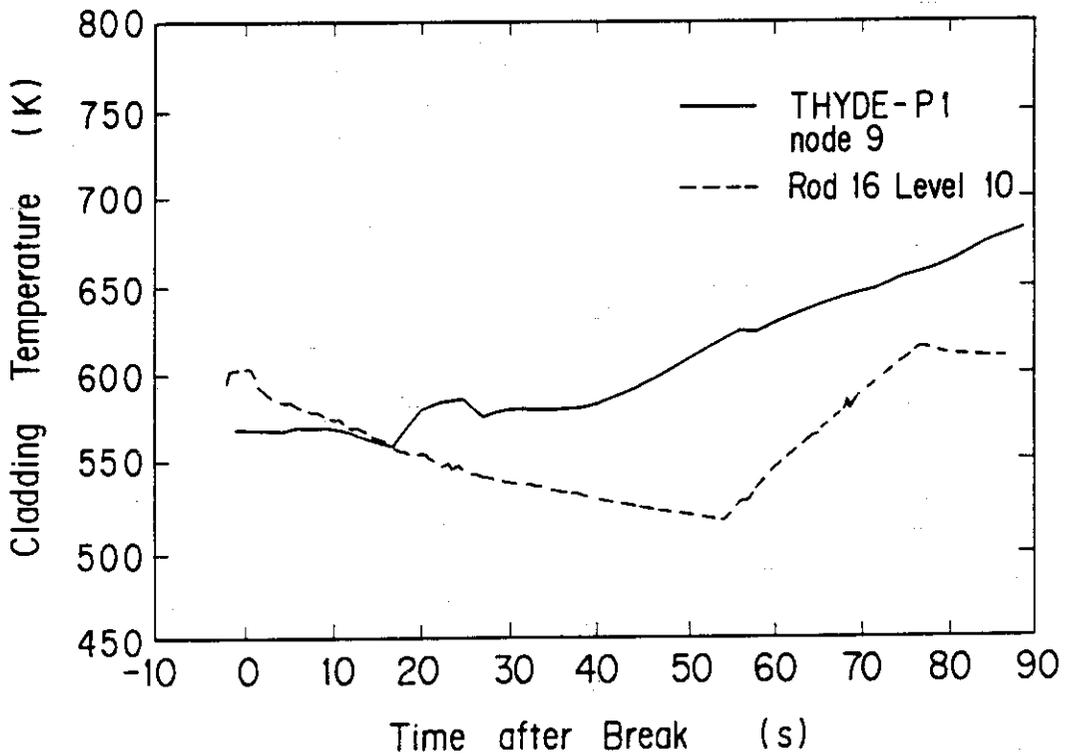


Fig. 4.2.8 Cladding surface temperature at level 5

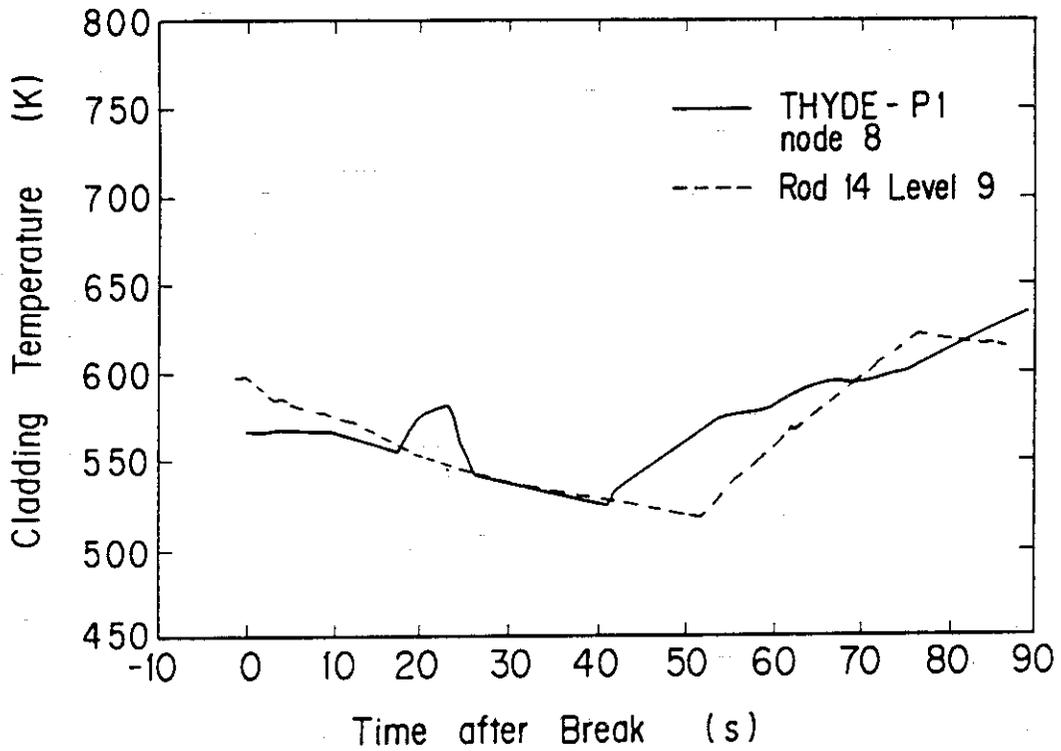


Fig. 4.2.9 Cladding surface temperature at level 7

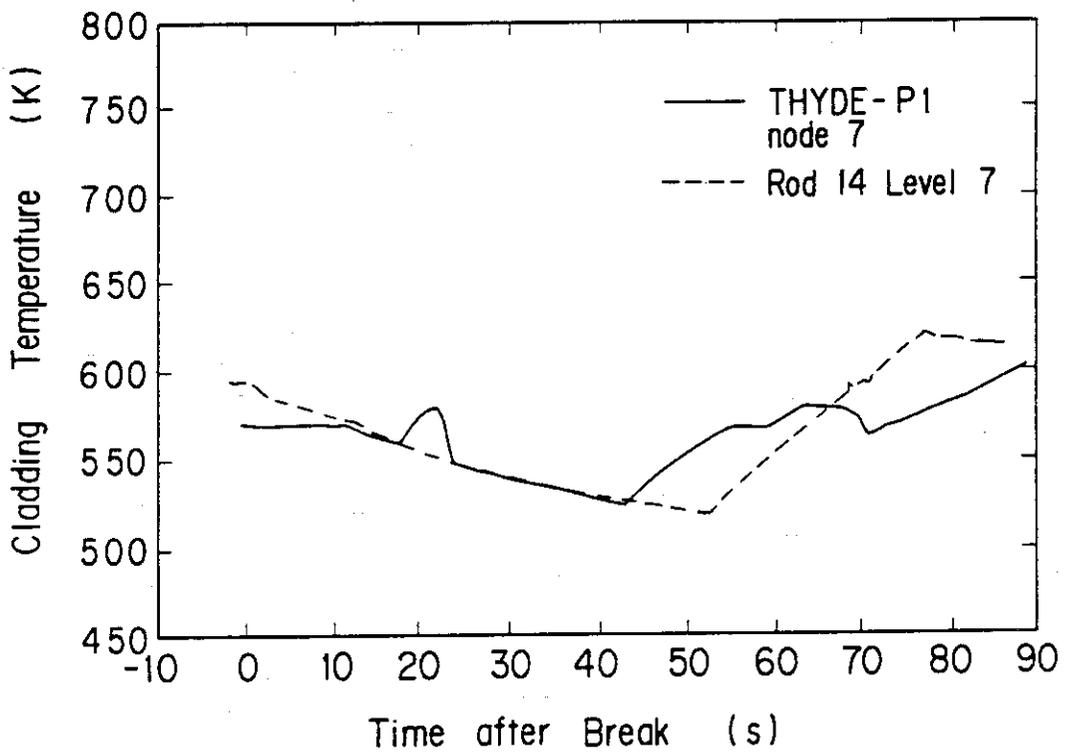


Fig. 4.2.10 Cladding surface temperature at level 9

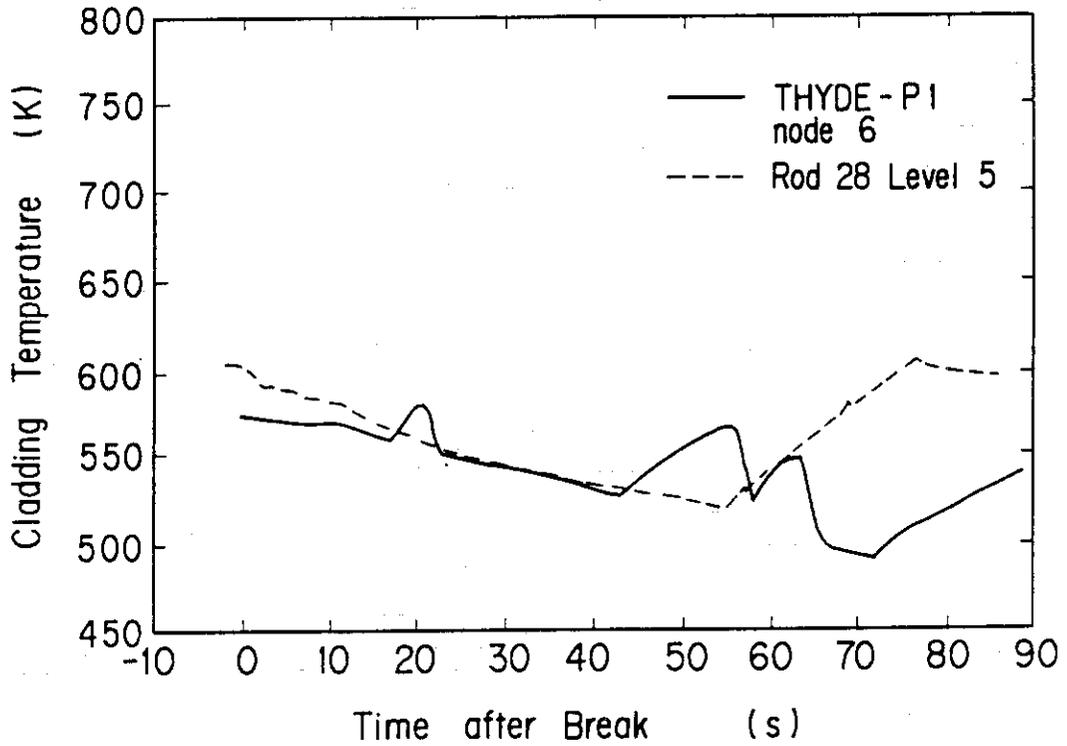


Fig. 4.2.11 Cladding surface temperature at level 10

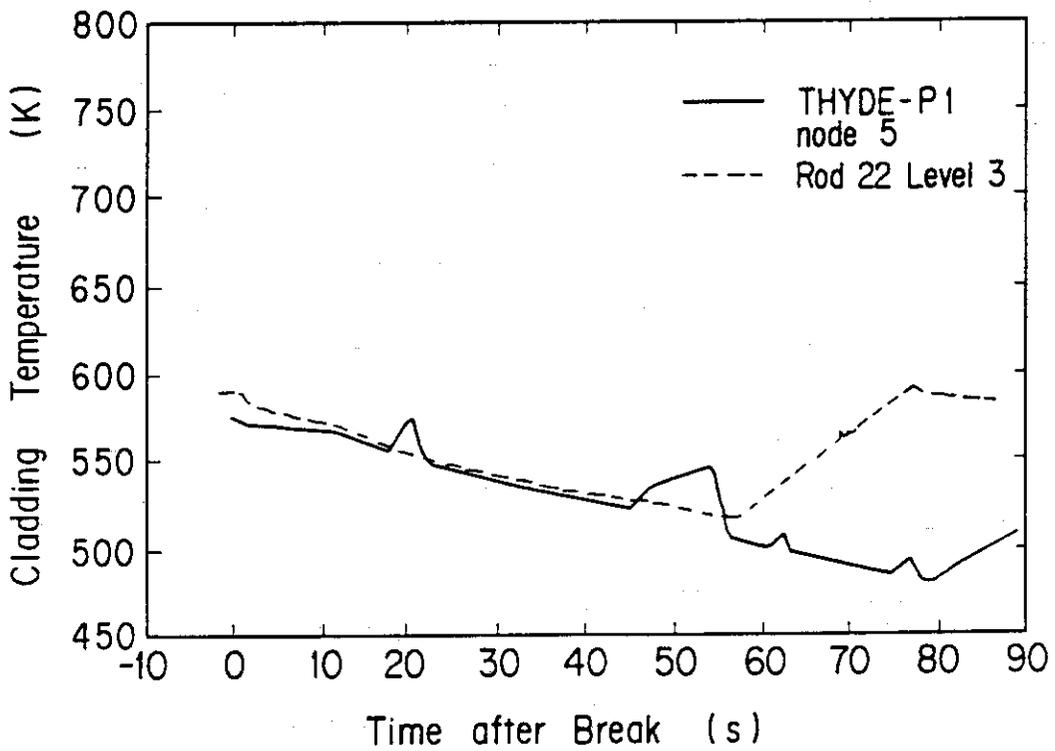


Fig. 4.2.12 Cladding surface temperature at level 12

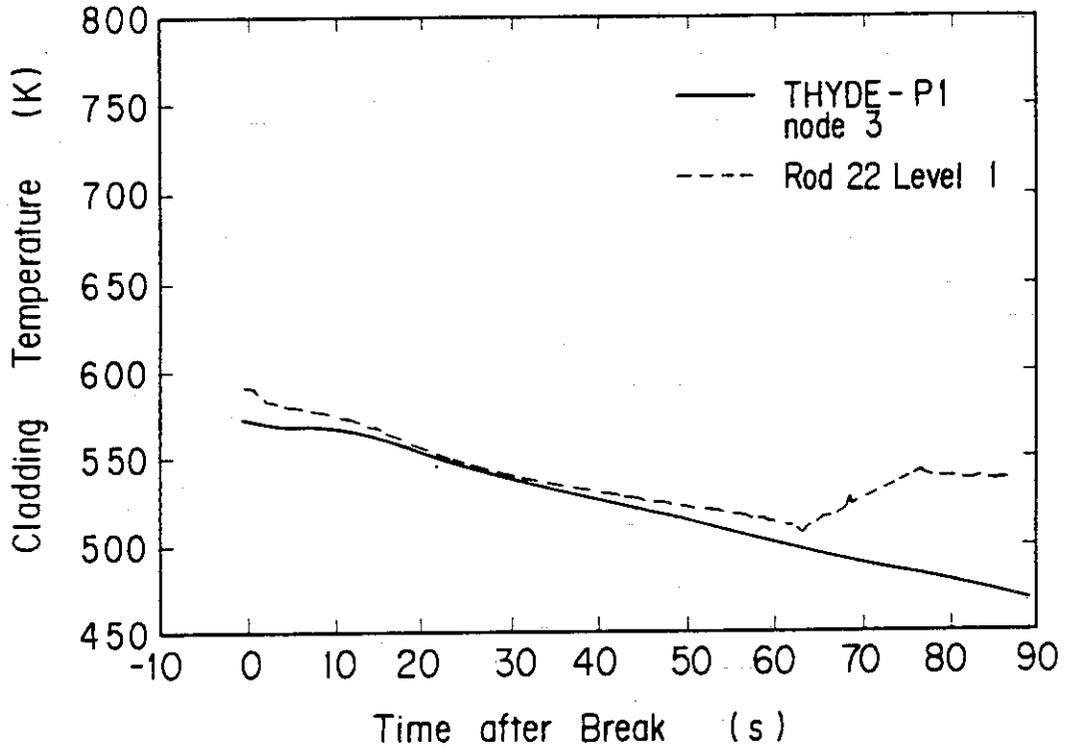


Fig. 4.2.13 Cladding surface temperature at level 15

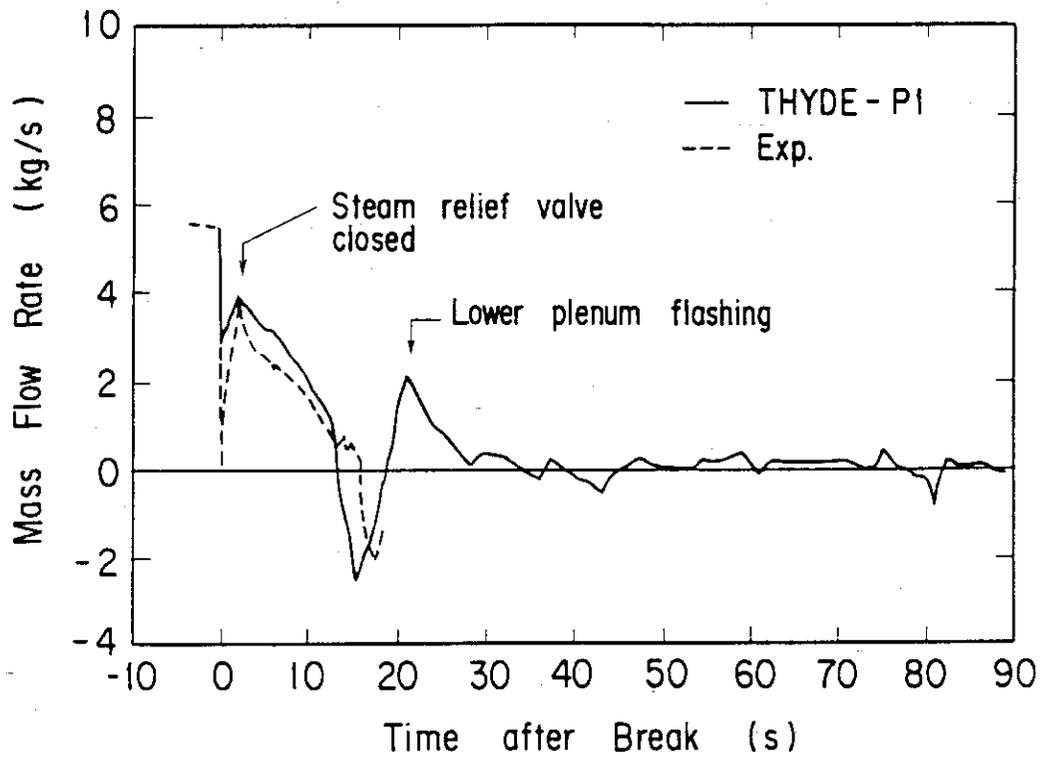


Fig. 4.3.1 Bundle inlet mass flow rate

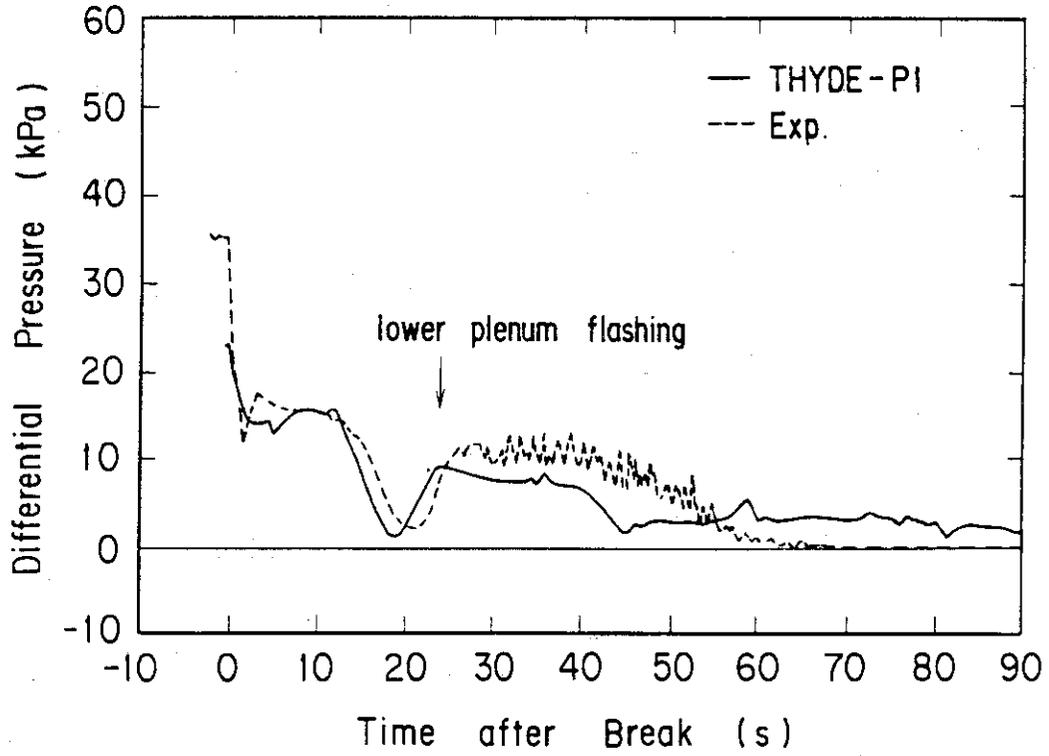


Fig. 4.3.2 Differential pressure through core

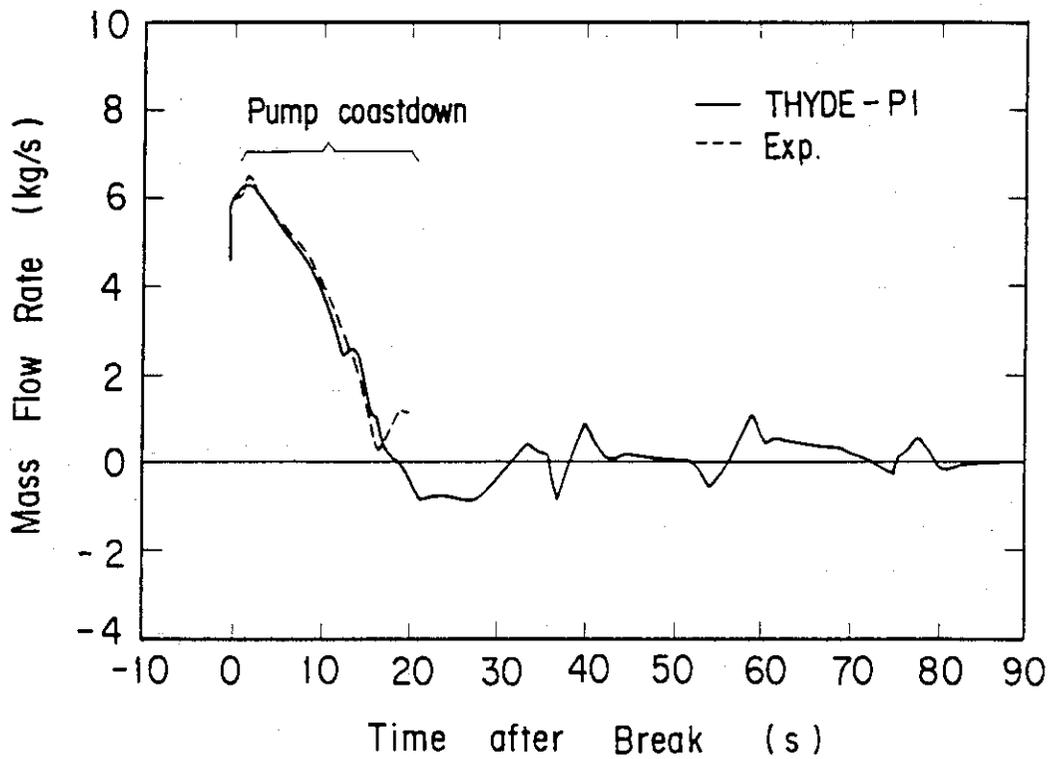


Fig. 4.3.3 Mass flow rate through pump P1

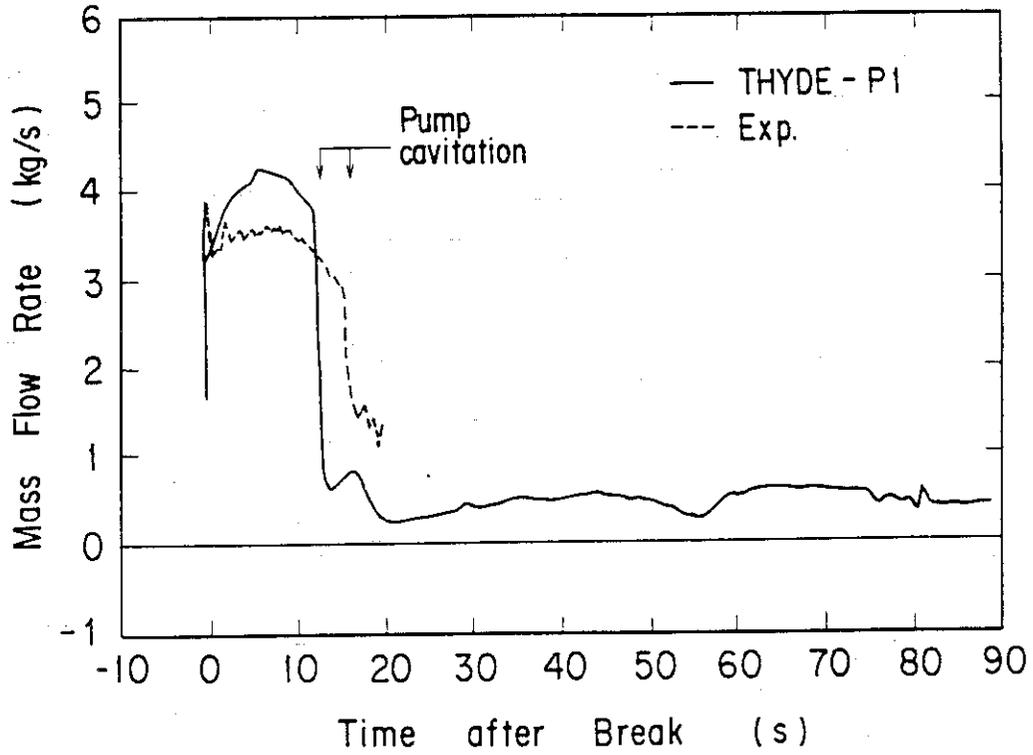


Fig. 4.3.4. Mass flow rate through pump P2

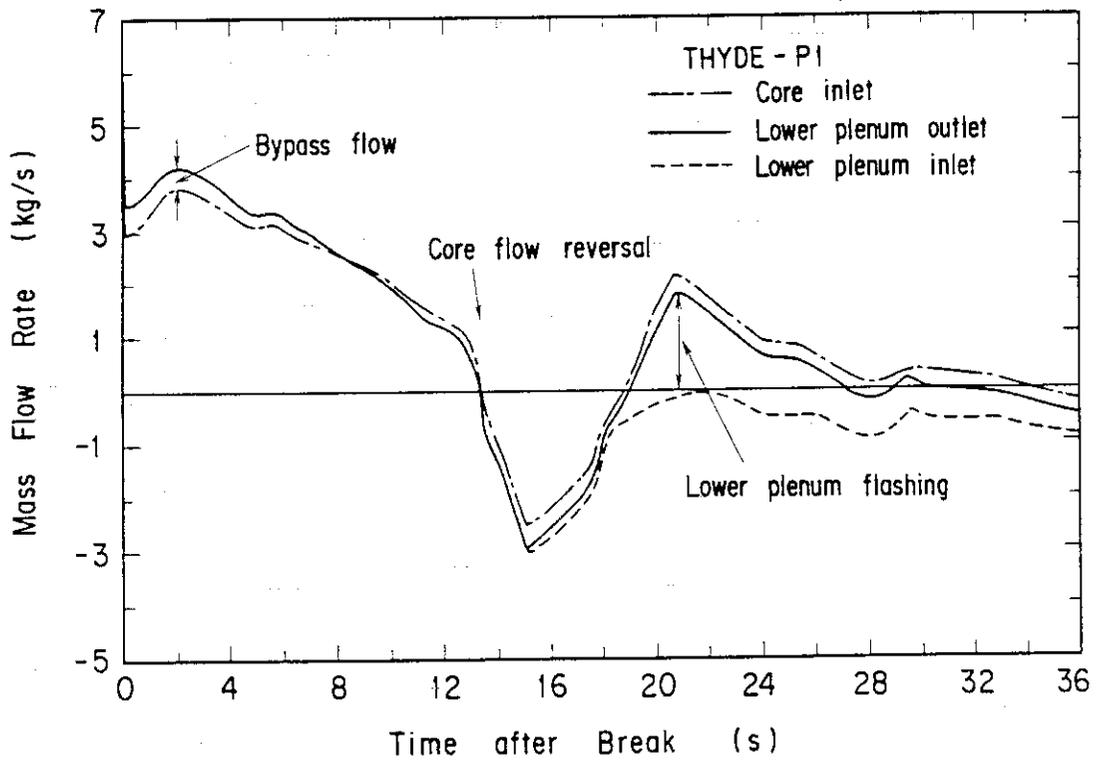
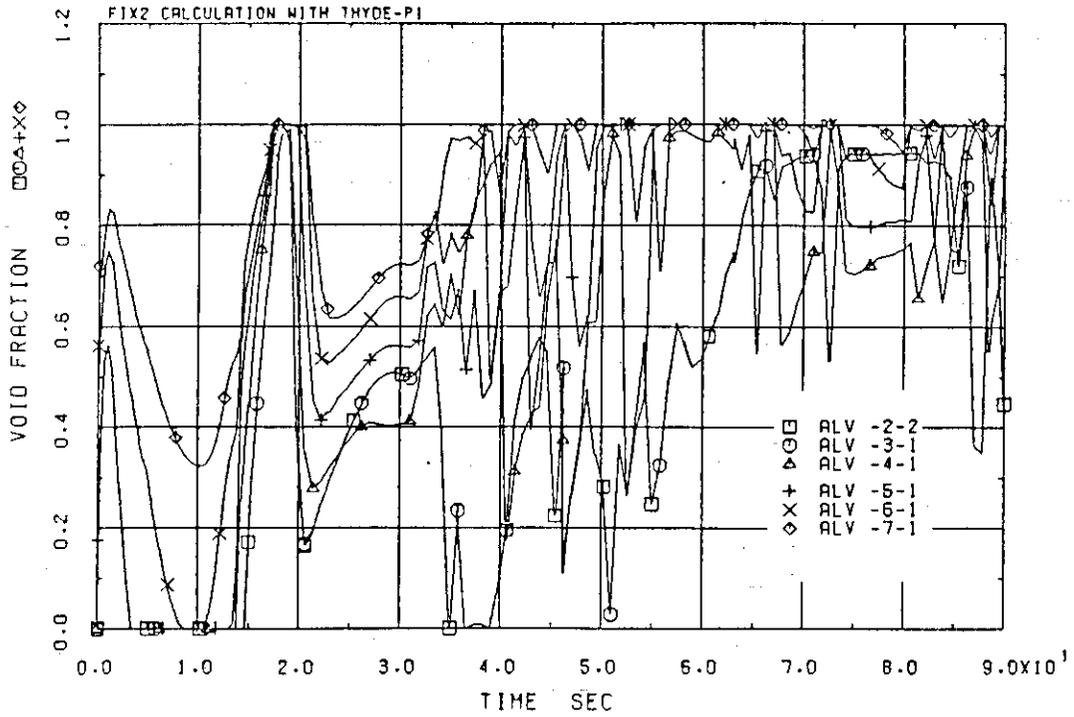
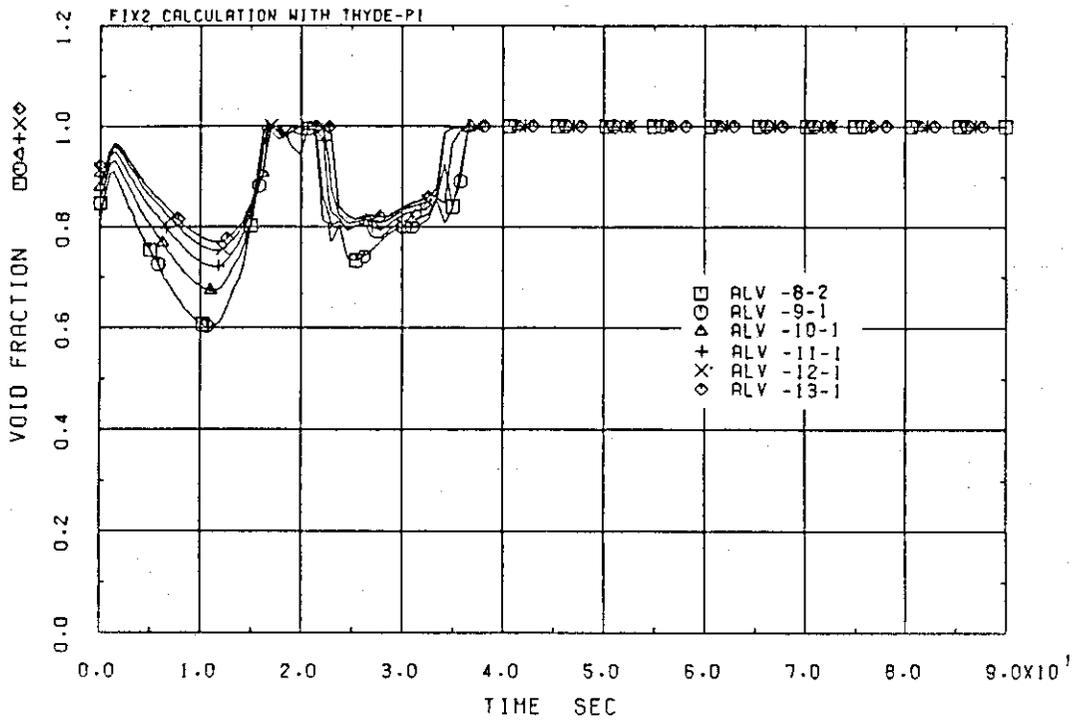


Fig. 4.3.5. Calculated mass flow rates at core inlet and both inlet and outlet of lower plenum



(a) Lower part of core



(b) Upper part of core

Fig. 4.3.6 Calculated void fractions at core

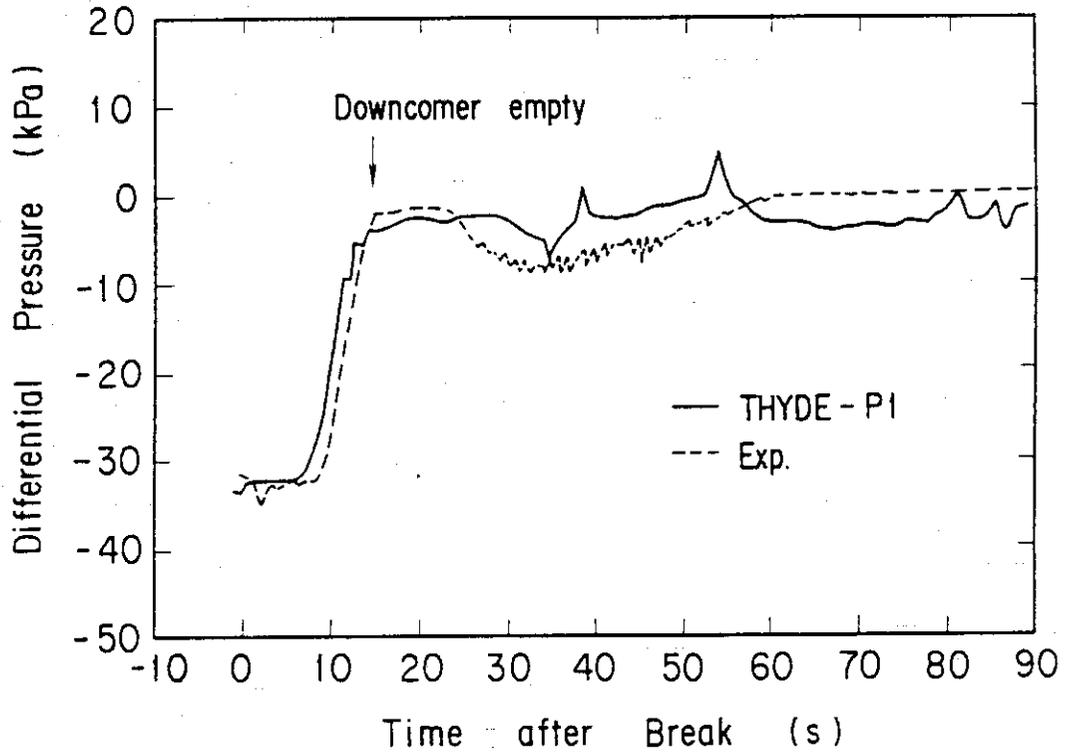


Fig. 4.4.1 Differential pressure through downcomer

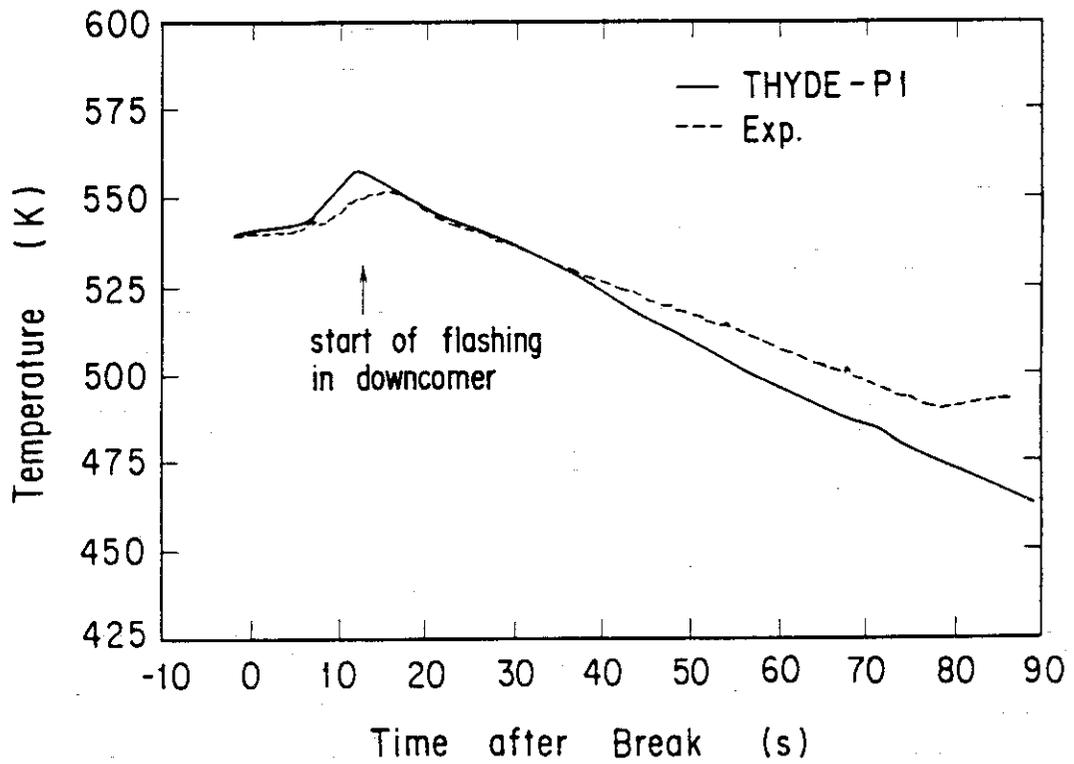


Fig. 4.4.2 Fluid temperature at downcomer inlet

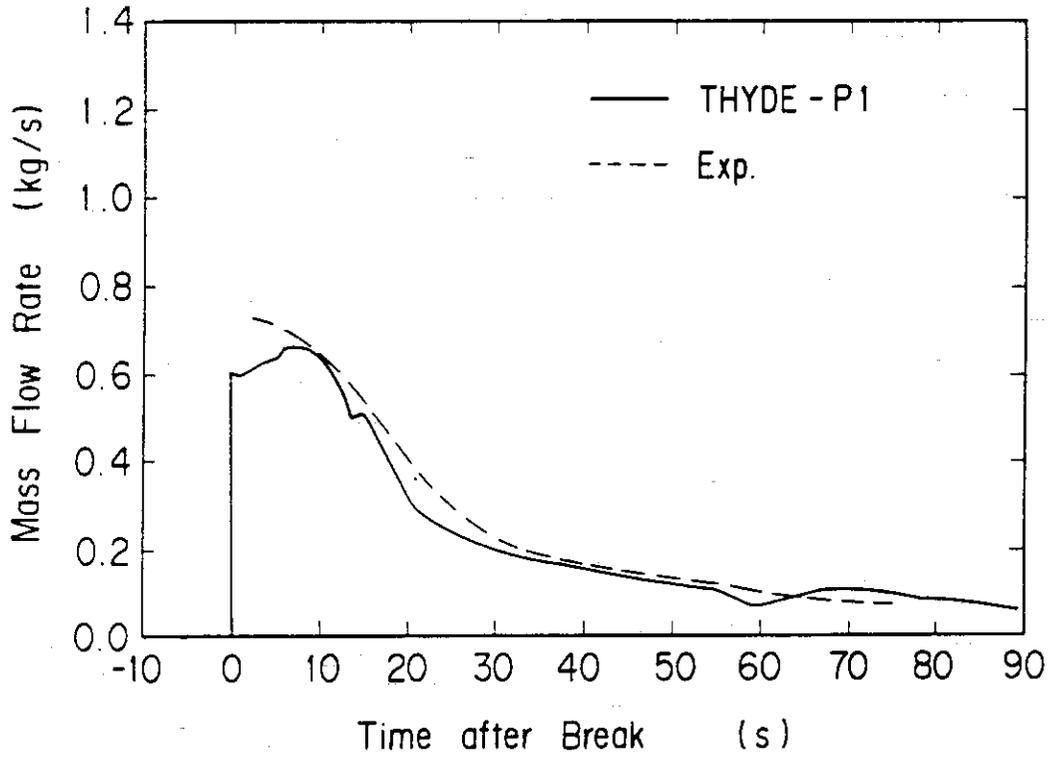


Fig. 4.5.1 Break flow

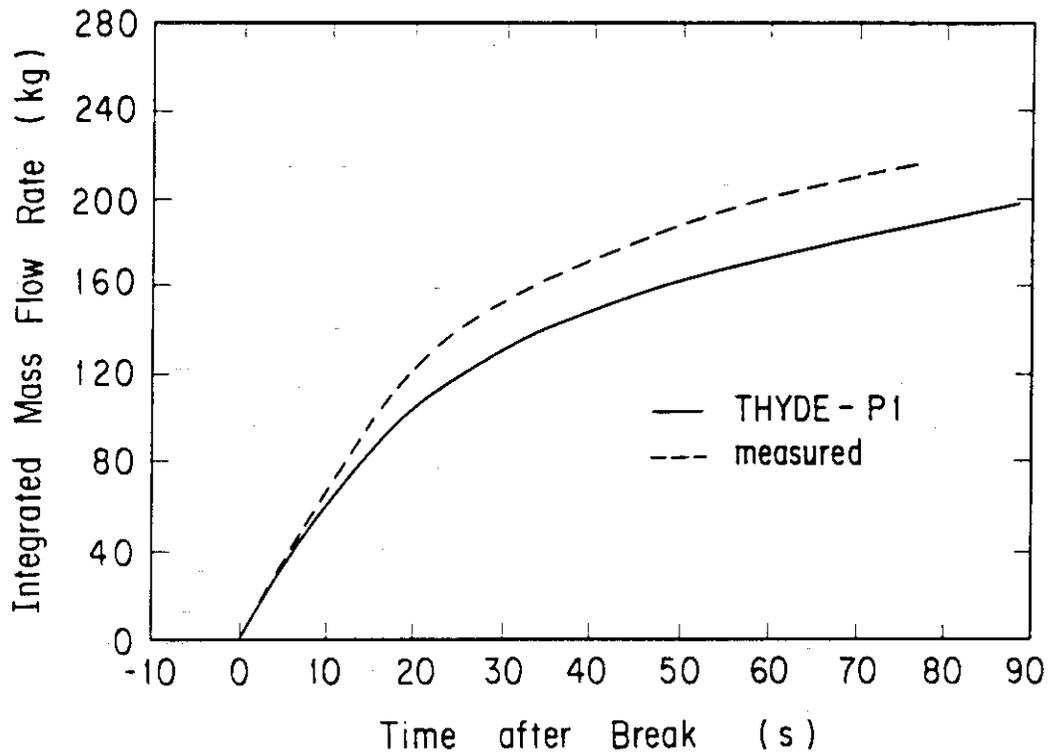


Fig. 4.5.2 Integrated break mass flow

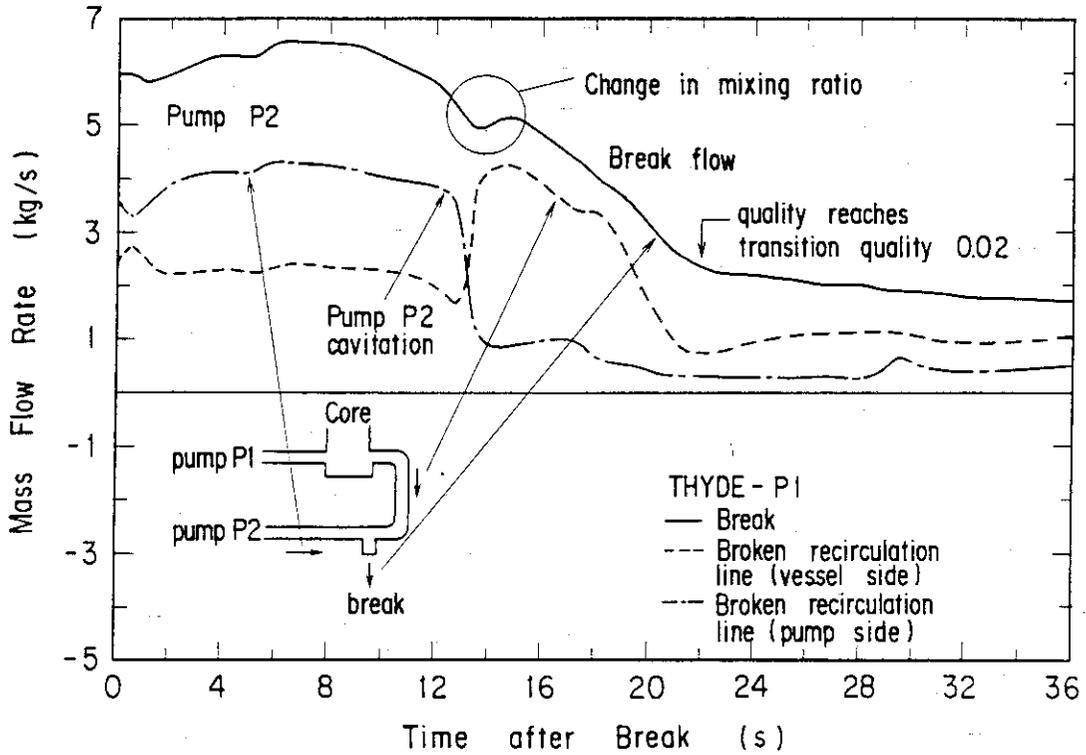


Fig. 4.5.3 Calculated mass flow rates at break, vessel side and pump side of broken recirculation line

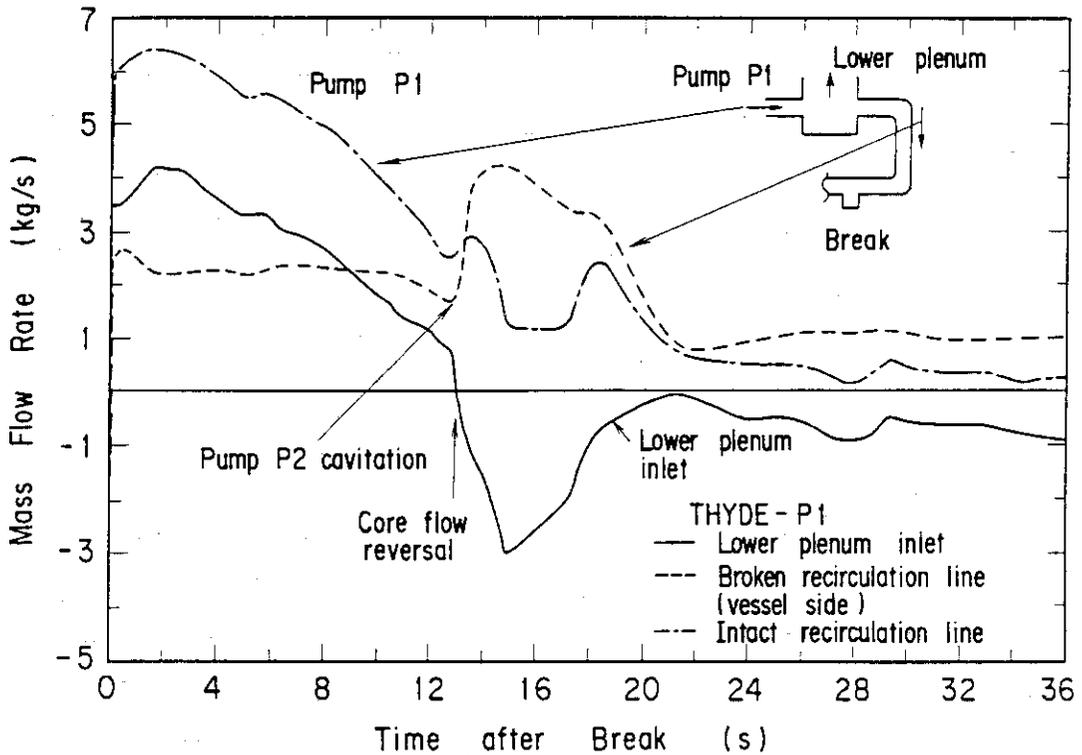
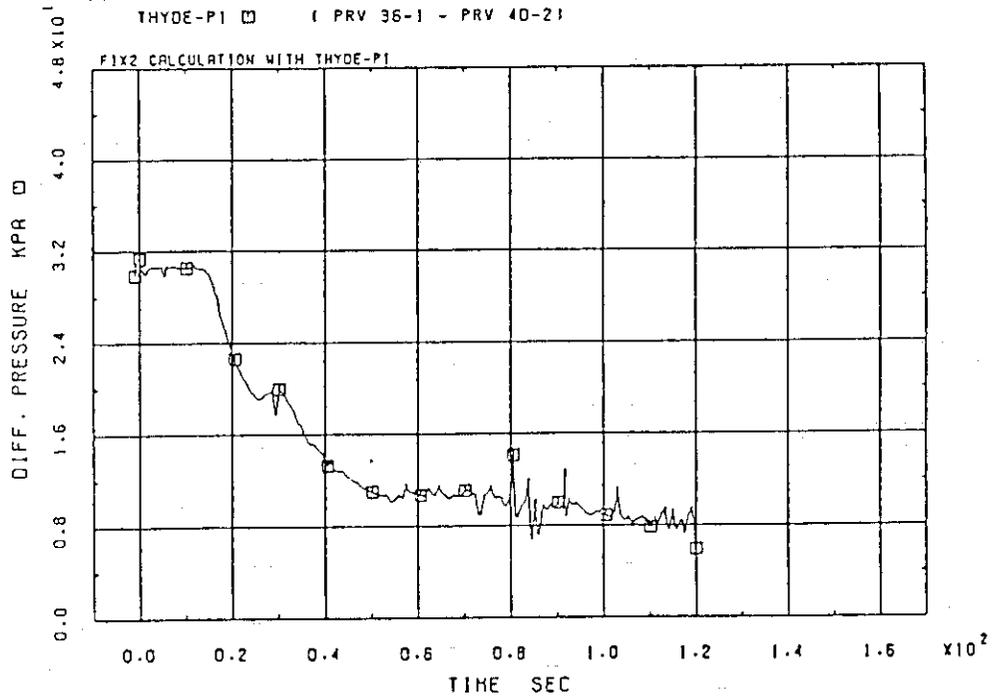
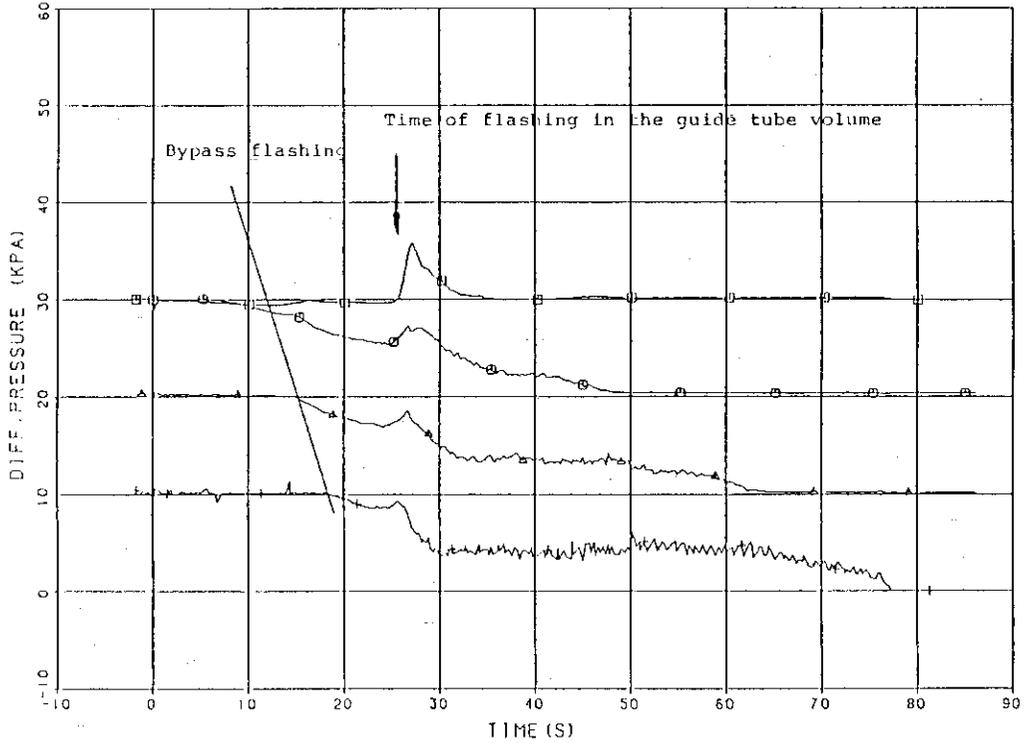


Fig. 4.5.4 Calculated mass flow rates at lower plenum inlet, vessel side of broken recirculation line and intact recirculation line



(a) Calculated

□	DPI 23 (BYPASS CHANNEL LEVEL 3 -UPPER PLENUM)	DISPLACED 30 KPA
○	DPI 22 (BYPASS CHANNEL LEVEL 2 -BYPASS CHANNEL LEVEL 3)	DISPLACED 20 KPA
△	DPI 21 (BYPASS CHANNEL LEVEL 1 -BYPASS CHANNEL LEVEL 2)	DISPLACED 10 KPA
+	DPI 20 (BYPASS CHANNEL BOTTOM -BYPASS CHANNEL LEVEL 1)	



(b) Measured

Fig. 4.6.1 Differential pressure through bypass

Appendix A List of input data

```

***** THYDE-P INPUT DATA ***** PAGE 1

--- FIX-II CALCULATION WITH THYDE-P1 --- FIX2.DATA(FIX2) 83/04/2200000100
/
/ PROBLEM DIMENSION DATA
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5 1 5 4 0 24 2
/
/ MINOR EDIT DATA
BB02
GLE-51 GLE-49 GLE-54 GLA-02 GLA-36 GLE-15 GLE-13 GLE-35 PRA-50
/
/ TIME STEP CONTROL DATA
SB03
SB0301
0.2 0.2 100.
SB0302
20 20 50 0 8.0E-3 1.0E-6 0.99 0.1
SB0303
20 20 50 0 4.0E-3 1.0E-6 1.2 0.1
SB0304
25 30 50 0 16.0E-3 1.0E-6 20.0 0.1
SB0305
25 30 50 0 32.0E-3 1.E-6 1000.0 0.1
/
/ TRIP CONTROL DATA
BB04
SB0401
1 0 1 0 1.E4 0.0 / END OF PROBLEM
SB0402
2 28 1 0 1.0 0.0 / PUMP 1 TRIP
SB0403
2 31 1 0 1. 0.0 / PUMP 2 TRIP
SB0404
2 43 1 0 1. 0.0 / PUMP 2 TRIP
SB0405
3 0 1 0 1.E4 0.0 / SCRAM
SB0406
5 55 1 0 5. 0.0 / SG ISCLATION
/
/ STEADY STATE MASS FLUX AND ENTHALPY
BB05
1 150.63 280.
/
/ CONTROL VOLUME DATA
BB06
SB0601
/1 1 39 40 0 1 70.4303 0.2252 0. 0.3680 0.3000
1 1 39 40 0 1 70.3303 0.2252 0. 0.7680 0.3000
0. 0. 0. 0. 0.
SB0602
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0. 0. 56. 56. 0.
SB0603
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SB0604
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SB0606
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SB0607

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JAERI-M 85-115

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							0.	0.	0.	0.	0.	00015000
SB0638												
38	1	28	29	0	1	69.870840	0.0949	0.	0.9350	0.9350		00015100
							0.	0.	0.	0.	0.	00015200
SB0639												
39	1	29	30	0	1	69.802854	0.0949	0.	0.9290	0.9290		00015300
							0.	0.	0.	0.	0.	00015400
SB0640												
40	1	30	31	0	1	69.735975	0.0949	0.	0.6520	0.6520		00015500
							0.	0.	0.	0.	0.	00015600
SB0641												
41	1	31	41	0	1	69.658476	0.0485	0.	1.1600	0.0		00015700
							0.	0.	0.	0.	0.	00015800
SB0642												
42	1	44	32	0	1	69.4301	0.0802	0.	3.0700	0.		00015900
							0.	0.	0.	0.	0.	00016000
SB0643												
43	8	32	33	0	1	69.3947	0.1	0.	0.75	-0.7500		00016100
							0.	0.	0.	0.	0.	00016200
SB0644												
44	1	33	34	0	1	70.4520	0.0722	0.	2.4500	0.0580		00016300
							0.	0.	0.	0.	0.	00016400
SB0645												
45	7	34	35	1	139	70.4576	0.0080	0.	6.8140	4.000		00016500
							0.	0.	0.	0.	0.	00016600
SB0646												
46	1	35	47	0	1	69.9494	0.0718	0.	6.9000	4.7020		00016700
							0.	0.	0.	0.	0.	00016800
SB0647												
47	1	47	36	0	1	69.5548	0.0590	0.	2.4900	2.4920		00016900
							0.	0.	0.	0.	0.	00017000
SB0648												
48	1	36	42	0	2	69.2898	0.0427	0.	7.3500	1.8350		00017100
							0.	0.	0.	0.	0.	00017200
SB0649												
49	1	47	43	0	1	69.4936	0.0467	0.	6.2700	0.1560		00017300
							0.	0.	0.	0.	0.	00017400
SB0650												
50	13	42	37	0	1	0.	0.5000	0.	2.3600	2.3480		00017500
							0.	0.	0.	0.	658.00	00017600
SB0651												
51	13	37	49	0	1	0.	0.0809	0.	1.8900	0.		00017700
							0.	0.	12.2	12.2	658.00	00017800
SB0652												
52	13	46	48	0	1	0.	0.1635	0.	1.2200	-1.2210		00017900
							0.	0.	0.	0.	0.	00020000
SB0653												
53	13	45	38	0	1	0.	0.07360	0.	1.4800	0.		00020100
												00020200
												00020300
												00020400

S80654						0.	0.	0.	0.	0.	00020500
/54 13 38 50 0 1					0.	0.0120	0.	0.2000	0.		00020600
54 13 38 50 0 1					0.	0.07360	0.	0.2000	0.		00020700
						0.	0.	0.	0.	0.	00020800
/											00020900
/ JUNCTION DATA											00021000
8807											00021100
1 1 0.0											00021200
2 1 0.0											00021300
3 1 0.0											00021400
4 1 0.0											00021500
5 1 0.0											00021600
6 1 0.0											00021700
7 1 0.0											00021800
8 1 0.0											00021900
9 1 0.0											00022000
10 1 0.0											00022100
11 1 0.0											00022200
12 1 0.0											00022300
13 1 0.0											00022400
14 1 0.0											00022500
15 1 0.0											00022600
16 1 0.0											00022700
17 1 0.0											00022800
18 1 0.0											00022900
19 1 0.0											00023000
20 1 0.0											00023100
21 1 0.0											00023200
22 1 0.0											00023300
23 1 0.0											00023400
24 1 0.0											00023500
25 1 0.0											00023600
26 1 0.0											00023700
27 1 0.0											00023800
28 1 0.0											00023900
29 1 0.0											00024000
30 1 0.0											00024100
31 1 0.0											00024200
32 1 0.0											00024300
33 1 0.0											00024400
34 1 0.0											00024500
35 1 0.0											00024600
36 1 0.0											00024700
37 1 0.0											00024800
38 1 0.0											00024900
39 4 0.0045											00025000
/ 40 4 0.0153											00025100
40 4 0.0010											00025110
41 4 0.0020											00025200
42 4 0.0909											00025300
43 4 0.0010											00025400
44 4 0.0028											00025500
45 4 0.0010											00025600
46 4 0.0010											00025700
47 4 0.0010											00025800
48 8 0.0											00025900
49 8 0.0											00026000
50 8 0.0											00026100
/											00026200
/ MIXING JUNCTION DATA											00026300
8803											00026400
S80801											00026500
39 1 1 0 0 0 1.						.0	.0	.0	.0		00026600
S80802											00026700
40 2 2 35 0 0 .89						.11	.0	.0	.0		00026800
S80803											00026900
41 1 14 0 0 0 1.						.0	.0	.0	.0		00027000
S80804											00027100

JAERI-M 85-115

42	2	15	50	0	0	1.	.0	.0	.0			00027200		
S80805												00027300		
43	1	16	0	0	0	1.	.0	.0	.0			00027400		
S80807												00027500		
44	3	27	30	42	0	.303	.101	.596	.0			00027600		
S80808												00027700		
45	2	23	53	0	0	1.	.0	.0	.0			00027800		
S80809												00027900		
46	2	36	52	0	0	1.	.0	.0	.0			00028000		
S80810												00028100		
47	2	47	49	0	0	.667	.333	.0	.0			00028200		
/												00028300		
/ PUMP DATA												00028400		
8810												00028500		
S81001												00028600		
28	1	0	2900.	0.0383	81.7	50.0	1000.	1566.	1.E3	.0	.0	2.	00028700	
S81002													00028800	
31	2	0	2900.	0.0106	13.2	25.0	1000.	2035.	1.E3	.0	.0	2.	00028900	
S81003													00029000	
43	1	0	2900.	0.0106	13.2	30.0	1000.	2035.	1.E3	.0	.0	2.	00029100	
/													00029200	
/ PUMP CHARACTERISTIC CURVES													00029300	
8811													00029400	
S81101													00029500	
1	/												00029600	
9													00029700	
-1.0	2.00			-0.95	1.81			-0.62	1.52			-0.50	1.39	00029800
0.0	1.181			0.27	1.18			0.47	1.16			0.64	1.13	00029900
1.0	1.0													00030000
2														00030100
-1.0	-1.0			1.0	1.0									00030200
6														00030300
-1.0	-1.0			0.0	-0.56			0.41	-0.16			-0.51	-0.05	00030400
0.76	0.48			1.0	1.0									00030500
6														00030600
-1.0	2.0			-0.79	1.53			-0.63	1.25			-0.30	0.93	00030700
0.0	0.72			1.0	1.0									00030800
2														00030900
-1.0	2.0			1.0	1.0									00031000
2														00031100
-1.0	1.0			1.0	0.0									00031200
2														00031300
-1.0	-1.0			1.0	1.0									00031400
2														00031500
-1.0	2.0			1.0	0.0									00031600
16														00031700
-1.0	-1.16			-0.90	-1.24			-0.80	-1.77			-0.70	-2.36	00031800
-0.6	-2.79			-0.5	-2.91			-0.40	-2.67			-0.25	-1.69	00031900
-0.1	-0.50			0.0	0.0			0.10	0.83			0.20	1.09	00032000
0.5	1.02			0.7	1.01			0.90	0.94			1.00	1.0	00032100
2														00032200
-1.0	0.0			1.0	0.0									00032300
9														00032400
-1.0	0.0			0.0	0.0			0.10	-0.04			0.20	0.0	00032500
0.3	0.1			0.40	0.21			0.80	0.67			0.90	0.80	00032600
1.0	1.0													00032700
11														00032800
-1.0	-1.16			-0.90	-0.78			-0.80	-0.50			-0.70	-0.31	00032900
-0.6	-0.17			-0.50	-0.08			-0.35	0.0			-0.20	0.05	00033000
-0.1	0.08			0.0	0.11			1.0	0.0					00033100
0														00033200
0														00033300
0														00033400
0														00033500
11														00033600
0.0	0.0			0.1	0.0			0.15	0.05			0.24	0.08	00033700
0.3	0.96			0.4	0.98			0.60	0.97			0.80	0.90	00033800
0.90	0.80			0.96	0.50			1.0	0.0					00033900

2								00034000
0.0	0.0	1.0	0.0					00034100
2	2							00034200
	0.0	1.0						00034300
0.0	0.0	0.0						00034400
1.0	0.0	0.0						00034500
SB1102								00034600
2	/ PUMP2							00034700
10								00034800
-1.0	2.00	-0.95	1.81	-0.62	1.52	-0.50	1.39	00034900
0.0	1.136	0.20	1.135	0.40	1.12	0.60	1.10	00035000
0.8	1.055	1.0	1.0					00035100
2								00035200
-1.0	-1.0	1.0	1.0					00035300
8								00035400
-1.0	-1.0	0.0	-0.56	0.20	-0.385	0.40	-0.18	00035500
0.5	-0.055	0.6	0.145	0.80	0.565	1.0	1.0	00035600
6								00035700
-1.0	2.0	-0.79	1.53	-0.63	1.25	-0.30	0.93	00035800
0.0	0.72	1.0	1.0					00035900
2								00036000
-1.0	2.0	1.0	1.0					00036100
2								00036200
-1.0	1.0	1.0	0.0					00036300
2								00036400
-1.0	-1.0	1.0	1.0					00036500
2								00036600
-1.0	2.0	1.0	0.0					00036700
16								00036800
-1.0	-1.16	-0.90	-1.24	-0.80	-1.77	-0.70	-2.36	00036900
-0.6	-2.79	-0.5	-2.91	-0.40	-2.67	-0.25	-1.69	00037000
-0.1	-0.50	0.0	0.0	0.10	0.83	0.20	1.09	00037100
0.5	1.02	0.7	1.01	0.90	0.94	1.00	1.0	00037200
2								00037300
-1.0	0.0	1.0	0.0					00037400
9								00037500
-1.0	0.0	0.0	0.0	0.10	-0.04	0.20	0.0	00037600
0.3	0.1	0.40	0.21	0.80	0.67	0.90	0.80	00037700
1.0	1.0							00037800
11								00037900
-1.0	-1.16	-0.90	-0.78	-0.80	-0.50	-0.70	-0.31	00038000
-0.5	-0.17	-0.50	-0.08	-0.35	0.0	-0.20	0.05	00038100
-0.1	0.08	0.0	0.11	1.0	0.0			00038200
0								00038300
0								00038400
0								00038500
0								00038600
11								00038700
0.0	0.0	0.1	0.0	0.15	0.05	0.24	0.08	00038800
0.3	0.96	0.4	0.96	0.60	0.97	0.80	0.90	00038900
0.90	0.80	0.96	0.50	1.0	0.0			00039000
2								00039100
0.0	0.0	1.0	0.0					00039200
2	2							00039300
	0.0	1.0						00039400
0.0	0.0	0.0						00039500
1.0	0.0	0.0						00039600
/								00039700
/ BREAK DATA								00039800
SB13								00039900
50	1.0	0.1	.0	.0	.0			00040000
			.8	.6	.6	3.8E3	3.8E3	00040100
2								00040200
0.	4.	1000.	4.					00040300
/								00040400
/ STEAM GENERATOR DATA								00040500
SB15								00040600

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SB1501
55 139 45 45 1 1
5.5 18.9 0.7 0.5 0.02 0.004 10.4 1.0 30. 1.3
0.1 0.9999 1. 0.2 0.1
-7.0
0.001 30. 0.5 0.5 0.5
3
0.0 1.0 1.0 1.0 1.0 1.0 1.0 1.0 1000.0 1.0 1.0 0.0
/
/ CORE DATA
B216
/ AVERAGE HEATER ROD
SB1501
1
35 2 13 12 0 1 2 2 3
0.006125 0.016896 0.
2 3 4 5 6 7 8 9 10 11 12 13
0.071 0.368 0.368 0.368 0.368 0.368 0.368 0.368 0.368 0.368 0.368 0.368 0.182
/0. 109.9 154.1 170.7 180.5 180.5 170.7 170.7 154.1 129.5 72.3 0.
0. 119.7 169.3 187.6 198.4 198.4 187.6 187.6 169.3 142.3 79.4 0.
/ MATERIAL INDEX
/ 1 2 3 4 5
1 1 1 1 2
0. 0. 0. 0. 1.
/
/ HOTTEST HEATER ROD
SB1502
2
1 2 13 12 0
0.006125 0.016896 0.
2 3 4 5 6 7 8 9 10 11 12 13
0.071 0.368 0.368 0.368 0.368 0.368 0.368 0.368 0.368 0.368 0.368 0.368 0.182
/0. 112.8 159.4 179.7 186.9 186.9 176.7 176.7 159.4 134.0 74.8 0.
0. 123.9 175.2 194.2 205.4 205.4 194.2 194.2 175.2 147.3 82.2 0.
/ MATERIAL INDEX
/ 1 2 3 4 5
1 1 1 1 2
0. 0. 0. 0. 1.
/ PHYSICAL PROPERTIES OF HEATER ROD MATERIALS
BB17
/ NORMALIZED CORE POWER
11
0.0 1.0 1.0 1.0 2.0 0.98 2.7 0.8
4.5 0.6 7.0 0.4 11.5 0.2 16.5 0.1
21.0 0.065 31.0 0.05 1.84 0.035
/----- DENSITY (MGO) -----
2
0.0 3.53 2000.0 3.53
/----- DENSITY (INCONEL 600) -----
2
0.0 1.0 2000.0 1.0
/----- SPECIFIC HEAT CAPACITY (MGO) -----
6
0.0 0.287
100.0 0.287 450.0 0.291 650.0 0.318 800.0 0.323
1000.0 0.323
/----- VOLUMETRIC HEAT CAPACITY (INCONEL 600) -----
20
-17.3 523.5 93.3 919.9 148.8 953.6 204.4 979.9
250.0 1000.8 315.6 1018.3 371.1 1034.5 426.7 1051.0
482.2 1058.4 537.8 1093.3 593.3 1122.7 648.9 1160.4
704.4 1208.1 760.0 1267.9 815.6 1341.5 871.1 1431.1
926.7 1528.4 962.2 1665.4 1037.5 1814.3 1093.3 1986.5
/----- THERMAL CONDUCTIVITY (MGO) -----
7
2
0.0 100.0 450.0 500.0
700.0 800.0 1000.0
00040700
00040800
00040900
00041000
00041100
00041200
00041300
00041400
00041500
00041600
00041700
00041800
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00047400

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14(3.E3)
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0.00062 0.00062 0.00057 0.00057 0.00057 0.00057 00047600
0.00062 0.00062 0.00057 0.00057 0.00057 0.00057 00047700
/----- THERMAL CONDUCTIVITY (INCONEL 600) ----- 00047800
20 2 00047900
-17.3 92.3 148.8 204.4 00048000
250.0 315.5 371.1 426.7 00048100
482.2 537.8 593.3 648.9 00048200
704.4 760.0 815.6 871.1 00048300
926.7 982.2 1037.8 1093.3 00048400
40(1.0) 00048500
0.003484 0.003484 0.003715 0.003715 0.003925 0.003925 0.004105 0.004105 00048600
0.004299 0.004299 0.004506 0.004506 0.004721 0.004721 0.004946 0.004946 00048700
0.005180 0.005180 0.005419 0.005419 0.005663 0.005663 0.005911 0.005911 00048800
0.006153 0.006153 0.006411 0.006411 0.006660 0.006660 0.006902 0.006902 00048900
0.007143 0.007143 0.007379 0.007379 0.007606 0.007606 0.007821 0.007821 00049000
/ HEAT SLAB CONTROL DATA 00049100
BB21 00049200
1 1 00049300
/ 00049400
/ HEAT SLAB DATA 00049500
BB22 00049600
SB2201 00049700
1 2 1 1 0 36 0.475 0. 0.005 69.0 .0 .0 00049800
1 1 5 1 0.006 2.23E4 00049900
/ 1 1 5 1 0.006 1.98E4 00050000
SB2202 00050100
2 2 1 1 0 37 0.940 0. 0.006 100.2 .0 .0 00050200
1 1 5 1 0.006 3.24E4 00050300
/ 1 1 5 1 0.006 2.88E4 00050400
SB2203 00050500
3 2 1 1 0 38 0.935 0. 0.006 161.2 .0 .0 00050600
1 1 5 1 0.006 5.20E4 00050700
/ 1 1 5 1 0.006 4.62E4 00050800
SB2204 00050900
4 2 1 1 0 39 0.929 0. 0.006 100.2 .0 .0 00051000
1 1 5 1 0.006 3.24E4 00051100
/ 1 1 5 1 0.006 2.88E4 00051200
SB2205 00051300
5 2 1 1 0 40 0.625 0. 0.006 61.5 .0 .0 00051400
1 1 5 1 0.006 1.99E4 00051500
/ 1 1 5 1 0.006 1.77E4 00051600
/ 00051700
/ SLAB MATERIAL DATA 00051800
BB23 00051900
SB2301 00052000
1 00052100
STAINLESS STEEL ( 18CR 2NI ) 00052200
/ DENSITY 00052300
2 00052400
20.0 7820.0 1000.0 7820 00052500
/ SPECIFIC HEAT 00052600
2 00052700
20.0 0.118 1000.0 0.118 00052800
/ THERMAL CONDUCTIVITY 00052900
6 00053000
20.0 0.35 100.0 0.28 200.0 .40 400.0 0.47 00053100
600.0 0.55 1000.0 0.70 00053200
/ 00053300
BB24 00053400
SB2401 00053500
1 00053600
/ NORMALIZED BYPASS HEATER POWER 00053700
11 00053800
0.0 1.0 1.0 1.0 1.2 0.8 1.75 0.6 00053900
2.5 0.5 4.5 0.4 8.4 0.3 11.3 0.2 00054000
13.0 0.1 13.8 0.0 1.E4 0.0 00054100

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/ VALVE DATA

BE25

SB2501

2 42 0.06 0.06 2.78 1.E5 0. 0. 0. 0.

/ 2 42 0.08 0.08 2.35 1.E5 0. 0. 0. 0.

SB2502

2 47 0.08 0.08 2.78 1.E5 0. 0. 0. 0.

/ 2 47 0.08 0.08 2.35 1.E5 0. 0. 0. 0.

SB2503

2 49 0.08 0.08 2.78 1.E5 0. 0. 0. 0.

/ 2 49 0.08 0.08 2.35 1.E5 0. 0. 0. 0.

SB2504

-2 51 0.06 0.06 1.E4 1.E5 0. 0. 0. 0.

/

/ MISCELLANEOUS DATA

B926

0. 1.4 1.4 0.

BEND

6

0 0 0 0 0 0.

0. 1.E-4 0. -1.E4 -1.E-4 1.E-4

0

0 0.08

0 0.08

00054200
 00054300
 00054400
 00054500
 00054510
 00054600
 00054700
 00054710
 00054800
 00054900
 00054910
 00055000
 00055100
 00055200
 00055300
 00055400
 00055500
 00055600
 00055700
 00055800
 00055900
 00056000
 00056100
 00056200

Appendix B Description and Geometrical Data of components
in FIX II Loop

Item no	Description	Figure no reference	Lowest level** (m)	Highest level** (m)	Diameter or hydr. diameter* (m)	Material thickness (mm)	Min flow area (cm ²)	Flow length (m)	Volume (dm ³)	Comments
1	Lower plenum	2:5	0.835	1.603	0.264	30	322.1	0.768	30.6	
2	Lower plenum baffle inlet	2:5	0.986	1.310	0.0815*	3/30	158.4	0.324	4.52	Section with- in baffle
3	Bundle inlet section	2:6	1.501	1.582	0.0136*	3-20	64	0.081	0.76	
4	Test section	2:5	1.582	5.326	0.0136*	2	60.34	3.744	22.7	Up to top plate
5	Upper plenum, lower part	2:5	5.330	5.444	0.120*	30	530	0.114	6.0	
6	Upper plenum, upper part	2:5	5.444	5.509	0.184	68	240	0.065	1.56	
7	Steam separator	2:15	5.509	7.357	0.150/0.100	2	78.5	1.848	16.7	
8	Spray condenser	2:15	5.514	9.705	0.264/0.5	53	361	4.191	678.0	
9	Downcomer, upper horizontal part	2:3	5.094	5.564	0.0986/ 0.1095	15.9	76	3.42	31.4	One 90°-elbow at end incl
10	Downcomer, vertical part, to pump P1	2:3	0.365	5.094	0.0737/ 0.1095	15.9	42.6	5.22	48.7	One 90°-elbow at pump inlet

Item no	Description	Figure no reference	Lowest level** (m)	Highest level** (m)	Diameter or hydr. diameter* (m)	Material thickness (mm)	Min flow area (cm ²)	Flow length (m)	Volume (dm ³)	Comments
11	Pump P2 suction line from downcomer	2:3	0.295	1.115	0.0492/ 0.0737	7.6	19.0	2.29	7.69	TWO T-connections plus one 90°-elbow
12	Pump P1	2:3	0.365	0.648	-	-	-	-	10.1	
13	Pump P2	2:3	0.295	0.523	-	-	-	-	5.0	
14	Intact recirculation line pressure side	2:3	0.648	1.201	0.0737/ 0.0986	7.6	42.6	6.15	28.8	One 65°-elbow, two 90°-elbows plus pump exit restriction orifice (see table 2:2)
15	Broken recirculation line, pump P2 to split break T-connection	2:3	0.523	0.940	0.0492	5.5	19.0	1.41	2.8	One 90° - elbow and pump exit restriction plus meas. orif (see Table 2:2)
16	As above, from break T-connection to break line from lower plenum	2:3	0.940	1.201	0.0492	5.5	19.0	3.06	5.82	Three 90° - elbows
17	Broken rec. line back to lower plenum	2:3	1.201	1.201	0.0737/ 0.0986	7.6	42.6	1.42	7.57	One 25° - turn

Item no	Description	Figure no re-ference	Lowest level** (m)	Highest level** (m)	Diameter or hydr.diameter* (m)	Material thickness (mm)	Min flow area (cm ²)	Flow length (m)	Volume (dm ³)	Comments
18	Bypass channel lower horizontal part	2:3	1.311	1.311	0.0248/ 0.0477	6.3	4.8	1.16	1.41	Measuring orifice (see Table 2:2)
19	Guide tube simulator volume	2:3/ 2:13	0.090	1.311	0.0777/ 0.226	12	47.4	1.22	32.3	See Fig 2:13 for details
20	Bypass channel vertical part	2:3/ 2:13	1.311	5.449	0.972/* 0.0673	8.6	70.7	4.14	29.3	Heater included
21	Bypass channel upper horizontal part	2:3/ 2:13	5.421	5.421	0.0477	5.5	17.9	1.16	2.14	
22	Split break line, from T-connection to break flow restrictor K12	2:1/ 2:3	0.940	0.940	0.0737	7.6	42.6	1.48	6.29	Flow restrictor see Fig 2:19 and Table 2:2
22a	As above, to tank T2	2:1/ 2:3	0.940	0.940	0.0737/ 0.1317	4	42.6	0.850	9.7	
23	Guillotine break line from pump P2 suction line to break flow restrictor K11	2:1/ 2:3	1.115	1.115	0.0649/ 0.0737	7.6	33.1	2.44	9.09	Flow restrictor see Fig 2:19 and Table 2:2
24	As above, to tank T1	2:1/ 2:3	1.115	1.115	0.1071	3.6	90.1	3.24	29.2	

Item no	Description	Figure no reference	Lowest level** (m)	Highest level** (m)	Diameter or hydr. diameter* (m)	Material thickness (mm)	Min flow area (cm ²)	Flow length (m)	Volume (dm ³)	Comments
25	Guillotine break line from T-connection before valve V116 to break flow restrictor K13	2:1/ 2:3	1.201	1.201	0.0649/ 0.0737	5	33.1	1.43	4.7	Flow restrictor see Fig. 2:19 and Table 2:2
26	As above, to tank T3	2:1/ 2:3	1.201	1.201	0.0661/ 0.1317	4	34.3	1.44	14.9	
27	Coolant flow line from down comer to pump P3	2:1/ 2:3	0.365	1.415	0.071	9.5	39.6	3.07	15.5	Two 90° - el-bows
28	Pump P3	2:1/ 2:3	0.365	0.648	-	-	-	-	10.1	
29	From pump P3 to evaporation cooler	2:1/ 2:3	0.648	0.165	0.071	9.5	39.6	2.46	10.08	Three 90° - el-bows
30	Evaporation cooler	2:1	0.706	0.706	139x0.008*	1	0.50	947.2	47.6	139 U-tubes
31	Outlet from evap cooler to feedwater line	2:1/ 2:3	0.706	5.408	0.071	9	39.6	6.90	27.9	

Item no	Description	Figure no reference	Lowest level** (m)	Highest level** (m)	Diameter or hydr. diameter* (m)	Material thickness (mm)	Min flow area (cm ²)	Flow length (m)	Volume (dm ³)	Comments
32	Feedwater line	2:1/ 2:3	5.408	5.564	0.0383/ 0.0492	5.4	11.5	6.27	10.74	Two 90° - elbows measuring orifice (see Table 2:2)
33	Spray water lines, up to V109	2:1/ 2:3	5.408	7.9	0.0294/ 0.0492	5.0	12.2	2.49	6.81	Including connections to the three separate lines. Measuring orifice, see Table 2:2
34	Spray lines(3) from V109 to spray condenser	2:1/ 2:3	7 900	~9.5	2*0.0492 1*0.0243	5.5	42.6	8.1 7.1 7.4	32.4	Three 90°-elbows. Volume for three lines
35	Steam line to V111	2:1/ 2:3	9.198	9.198	0.0737/ 0.0986	7.6	42.6	1.89	9.72	Measuring orifice, see Table 2:2

*) Hydraulic diameter is given for non-circular cross-sections

***) Level reference plane is the bottom plate

Appendix C Brief Description of Code Characteristics And
Information on Computer Resources Utilized in
Present Calculation

Table C.1 shows the brief description of the computer code characteristics. The detailed information of the utilized computer resources in the present calculation is shown in Table C.2. These tables have been made for the submittal to the ISP No.15.

Table C.1
Computer Code Characteristics

- 1 Participant Identification
 - JAPAN 'B'
- 2 Name and Version of Computer Code
 - THYDE-P1 (SV03L03)
- 3 Change in Computer Code not Included in Basic Code Description
 - Relief valve modeling for FIX-II calculation
 - Phase separator modeling
 - (THYDE-P1 has originally been developed for PWR LOCA analyses.)
- 4 Classification of Hydrodynamic Model
 - Drift flux model
 - Relaxation model considering thermal non-equilibrium
- 5 Slip-Model
 - Drift flux model
 - $V_{gj} = 1.14 \left(\frac{\sigma g (\rho_{fs} - \rho_{gs})}{\rho_{fs}^2} \right)^{1/4}$

6 Critical Flow Model

- Modified Zaloudek for subcooled state
- Moody for saturated state ($C_d=0.6$)
- Transition quality : 0.02

7 Interphase Drag Model

See item 4

8 Wall Friction Model

- Single phase friction factor :
 - $16/Re$ for laminar flow
 - Blasius formula for turbulent flow
- Two-phase multiplier : Thom

9 Flashing/Condensation Model

See item 4

10 Heat Transfer Models

- Present option selected :
 - Dittus-Boelter for subcooled forced convection
 - Thom for nucleate boiling
 - Bromley-Pomerantz for low flow film boiling
 - Groenevelt and Dougall-Rohsenow for high flow film boiling
 - McEligot for superheated steam forced convection
 - Interpolation for transition boiling

11 CHF Correlation

- Present option selected :
 - Biasi for forced flow condition (high flow)
 - Zuber for pool flow condition (low flow)

Table C.2

Details of the utilized computer resources in the calculation

1 Participant Identification

JAPAN 'B'

2 Computer code

THYDE-P1 (SV03LC3)

3 Number of nodes

54

4 Number of junctions

50

5 Number of heat slabs

- Core : 24 (12 for average rod and 12 for hottest rod)
- Core bypass heater : 5
- Cooler : 1

6 Computer utilized

FACOM M-380

7 Computer time required for the transient

CPU time : 2260 s

8 Number of calculated time steps

7800 steps

9 Remarks

- Ambient heat loss is not taken into consideration :
 - Initial core power : 3.5 MW
 - Initial core bypass power : 62.5 kW
 - No heat slab except core, core bypass and cooler

Appendix D Pump Characteristic Curves Used in Present Calculation

In this appendix, the pump characteristic curves used in the present calculation are presented. These data are taken from Ref. (13) in the text.

Homologous pump curves for pump P1 used in Studsvik's RELAP 5 calculations

Curve type	α/ν or ν/α	h/ν^2 or h/α^2
HAN (1)	0	1.181
	0.27	1.180
	0.47	1.160
	0.64	1.130
	1.00	1.000
HVN (2)	0.0	-0.56
	0.41	-0.16
	0.51	-0.05
	0.76	0.48
	1.00	1.00
HAD (3)	-1.00	2.00
	-0.95	1.81
	-0.62	1.52
	-0.50	1.39
	0.00	1.181
HVD (4)	-1.00	2.00
	-0.79	1.53
	-0.63	1.25
	-0.30	0.93
	0.00	0.72

Torque curves are not needed (speed given).

Homologous pump curves for pump P2 used in Studsvik's RELAP 5 calculations

Curve type	α/v or v/α	h/v^2 or h/α^2
HAN (1)	0	1.136
	0.2	1.135
	0.4	1.120
	0.6	1.100
	0.8	1.055
	1.0	1.000
HVN (2)	0.0	-0.56
	0.2	-0.385
	0.4	-0.180
	0.5	-0.055
	0.6	0.145
	0.8	0.565
	1.0	1.000
HAD (3)	-1.00	2.00
	-0.95	1.81
	-0.62	1.52
	-0.50	1.39
	0.00	1.136
HVD (4)	-1.00	2.00
	-0.79	1.53
	-0.63	1.25
	-0.30	0.93
	0.00	0.72

Torque curves are not needed (speed given).

Head multiplier for pumps P1 and P2 used in Studsvik's RELAP 5 calculations

α	M (α)
0.08	0.00
0.10	0.00
0.15	0.05
0.24	0.08
0.30	0.96
0.40	0.98
0.60	0.97
0.80	0.90
0.90	0.80
0.96	0.50
1.00	0.00

Dimensionless head ration difference data (Semiscale)
for pumps P1 and P2

Curve type	x	y
HAN (1)	0.0	0.00
	0.1	0.83
	0.2	1.09
	0.5	1.02
	0.7	1.01
	0.9	0.94
	1.0	1.0
HVN (2)	0.0	0.0
	0.1	-0.04
	0.2	0.00
	0.3	0.10
	0.4	0.21
	0.8	0.67
	0.9	0.80
HAD (3)	-1.0	-1.16
	-0.9	-1.24
	-0.8	-1.77
	-0.7	-2.36
	-0.6	-2.79
	-0.5	-2.91
	-0.4	-2.67
	-0.25	-1.69
	-0.1	-0.50
	0.0	0.00
HVD (4)	-1.0	-1.16
	-0.9	-0.78
	-0.8	-0.50
	-0.7	-0.31
	-0.6	-0.17
	-0.5	-0.08
	-0.35	0.00
	-0.2	0.05
	-0.1	0.08
	0.0	0.11