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A THROUGH CALCULATION OF 1,100 MWe  
PWR LARGE BREAK LOCA BY THYDE-P  
(SAMPLE CALCULATION RUN 20)

November 1981

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A Through Calculation of 1,100 MWe PWR  
Large Break LOCA by THYDE-P

(Sample Calculation Run 20)

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THYDE-P is a code to analyze loss-of-coolant accidents (LOCA's) of the pressurized water reactor (PWR). In this report, the calculated results of THYDE-P sample calculation Run 20 is presented, which is a through BE (best estimated model) calculation of LOCA for a commercial 1,100 MWe class PWR plant.

Keywords: LOCA, PWR, THYDE-P Code, Code Verification,  
Through Calculation, Best Estimated Model

THYDE -P コードによる 1,100 MWe PWR

大破断事故の一貫計算

(サンプル計算 Run 20)

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THYDE -P コードは、加圧水型原子炉の冷却材喪失事故を解析するコードである。本報告書には、サンプル計算 20 の計算結果が載せてある。サンプル計算 20 は 1,100 MWe クラスの商用加圧水型原子力発電プラントの BE (最適予測) 計算による冷却材喪失事故の一貫計算である。

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## 1. Introduction

THYDE-P(1) is a computer code to analyze the transient thermal hydraulic response of a PWR plant to a postulated LOCA loss of coolant accident).

The present status of THYDE-P may be considered to be at the stage of verification, so that what is needed at present for THYDE-P development may be to conduct a systematic study by sample calculations.

Thus far, sample calculations 10(2), 30(3) and 40(4) have been reported. In these calculations, a large break in the cold leg is assumed to occur. Run 10 is calculated for a typical 4-loop commercial PWR, while Runs 30 and 40 for LOFT.

In this report, sample calculation Run 20 by the latest version (SV02L03) for a typical 4-loop PWR plant will be presented, which is characterized by

- (1) BE calculation,
- (2) two core channels calculation with single cross flow area,
- (3) discharge coefficient 0.6,
- (4) through calculation to end of reflooding,
- (5) locked rotor of centrifugal pumps,
- (6) ECC water enthalpy 30kcal/kg,
- (7) same heat transfer correlations in reflooding as in blowdown, and
- (8) double ended guillotine break at the cold leg

The geometrical plant data are almost identical with those used in RELAP4/MOD5 sample problem(5).

## 2. Code Modifications

The detail of the THYDE-P code was described in Ref.(1), part of which, however, had been revised since its publication. Among those, main items were;

- (1) solution technique at low pressure,
- (2) flow model at low pressure,
- (3) implicit solution of fuel rod temperature, and
- (4) improvement of CHF and heat transfer correlations.

### 2.1 Mass Equation

In Run 10 calculation, the mass and energy conservation equations were used in the form of

$$\dot{h} = \frac{1}{\rho h} \{ G_A h_A - G_E h_E - h(G_A - G_E) + I_A - I_E + Q \} \quad \text{--- (1)}$$

$$\left\{ 1 + \frac{1}{\rho} \frac{\partial \rho}{\partial h} (h - h_A) \right\} G_A - \left\{ 1 + \frac{1}{\rho} \frac{\partial \rho}{\partial h} (h - h_E) \right\} G_E - \frac{1}{\rho} \frac{\partial \rho}{\partial h} (I_A - I_E + Q) - L \frac{\partial \rho}{\partial h} \frac{dp}{dt} \quad \text{--- (2)}$$

where

$$I = \sum \rho_{fs} u_{gj} (h_{gs} - h_{fs})$$

They were obtained from

$$L \frac{\partial \rho}{\partial t} = G_A - G_E \quad \text{--- (3)}$$

and

$$L \frac{\partial \rho h}{\partial t} = G_A h_A - G_E h_E + I_A - I_E + Q \quad \text{--- (4)}$$

with the help of

$$d\rho = \frac{\partial \rho}{\partial h} dh + \frac{\partial \rho}{\partial p} dp \quad \text{--- (5)}$$

Since it was found in the course of sample calculation Run 10 that relationship(5) broke down at low pressure and low quality, it was decided to implement Eq. (3) instead of Eq. (2) in THYDE-P so that mass is strictly conserved.

### 2.2 Flow Model

When low enthalpy ECC water was injected into primary loop, depressurization of the system caused by a rapid condensation took place and the calculations by the homogeneous equilibrium models failed. In the refill-reflood phase of a large break LOCA analysis for a commercial PWR plant, the enthalpy of ECC water was 30kcal/kg, whereas the coolant enthalpy was about 500~600 kcal/kg, so that the situation was beyond equilibrium model.

In the present calculation, a non-equilibrium model for

density calculation was introduced to cover this non-equilibrium phenomenon by means of a relaxation equation;

$$\frac{d\rho}{dt} = \frac{\rho - \rho^*}{\tau} \quad \text{--- (6)}$$

$\rho$  ; equilibrium density  
 $\rho^*$  ; non-equilibrium density  
 $\tau$  ; delay parameter

### 2.3 Implicit Solution of Fuel Rod Temperature

The heat conduction equation within a fuel rod was given by

$$\rho c_p \frac{\partial T}{\partial t} = \frac{1}{r} \frac{\partial}{\partial r} (kr \frac{\partial T}{\partial r}) + \frac{\partial}{\partial z} (k \frac{\partial T}{\partial z}) + q \quad \text{--- (7)}$$

Radial mesh width was the order of  $10^{-3}$  m, while axial mesh width was the order of 1 m. So the equation was simplified ignoring the second term of the right hand side which was originally included in the equation, and was solved using a fully linear implicit method. By this improvement coupled with other implicit techniques for various parameters, calculations go ahead with a larger time step width.

### 2.4 Improvement of CHF and Heat Transfer Correlations

In the present version, various types of correlations for CHF and heat transfer were included in THYDE-P code, and the user can select appropriate correlations.

Presently available CHF correlations are;

- (1) Biasi correlation
- (2) GE correlation
- (3) B&W-2, Bennett, Modified Bennett correlation
- (4) Modified Zuber correlation

and available heat transfer correlations are;

for nucleate boiling condition

- (1) Jens-Lottes correlation
- (2) Thom correlation

for post-CHF forced convection condition

- (1) Groenevelt correlation and,
- (2) Dougall-Rosenow correlation

for post-CHF pool condition

- (1) Berenson correlation
- (2) Modified Bromley correlation
- (3) Bromley-Pomeranz correlation

### 3. Main Assumptions and Conditions for Run 20

#### 3.1 Run 20 Plant Representation

An overall description of the specified 4-loop 1,100MWe PWR is given in section 2 of Ref. (2).

The input data used in the present calculation are listed in Appendix A. In the following, the main inputs and assumptions for Run 20 are shown.

- (1) The specific enthalpy of ECC water in pumped injection and accumulator injection were assumed to be 30 kcal/kg.
- (2) The nodalization of the present calculation is shown in Fig.3-1.

Nodes from 23 to 28 represent average core channel and a hot core channel was nodalized into nodes from 29 to 34. Node 36 was added for a cross flow simulation. The lower plenum and the upper plenum were expressed by a single node (node 22 and node 37, respectively), and the downcomer was simulated also by a single node (node 21). The upperhead was simulated by linkage node 38. The steam generator and the accumulator were modeled as special nodes and were nodalized into nodes 46,47 and nodes 41,42, respectively.

- (3) The double-ended break were assumed to occur at junction 8 at 0.01 sec after the calculation started. The pressure at the break was assumed to drop to the containment pressure exponentially with a time constant 0.4 sec.
- (4) In the steady state adjustment, THYDE-P requires initial mass flux and enthalpy at one point of the primary loop. They were selected to be

$$G = 9.0 \times 10^3 \text{ kg/m}^2\cdot\text{sec}$$

$$h = 360 \text{ kcal/kg}$$

at point A of normal node 1.

The geometrical data and loss coefficients for each node are shown in Table 3-1 and Table 3-2. Fig.3-2 shows the distribution of node average pressure in the primary loop which was obtained following the procedure described in section 4 of Ref. (2).

- (5) Steam generator was modeled by the following data.

secondary system pressure	62atm
U-tube pitch	$3.0 \times 10^{-2} \text{ m}$
number of U-tubes of one unit	3265
specific enthalpy of feedwater	222 kcal/kg
feedwater mass flow rate	474.0 kg/sec
subcooled water level	4.0 m
void fraction of saturated region	0.95

## initial heat flux

node no.	heat flux(kcal/m <sup>2</sup> .sec)
14	65.65
15	49.24
16	41.03

The feedwater was cut off at 0.4 sec after LOCA initiated. In the present calculation, a relief valve could not be simulated.

- (6) The core was divided radially into two regions, i.e. average channel region and hot channel region. The hot channel simulated the most hottest assembly in the core, and its hot channel factor was assumed to be 1.30. Input data for the core nodes were;

reactor thermal power            3,479 MWt  
initial heat flux and number of fuel rods

	hot channel region		average channel region	
	node no.	heat flux (kcal/m <sup>2</sup> .sec)	node no.	heat flux (kcal/m <sup>2</sup> .sec)
initial heat flux	29	non-heated	23	non-heated
	30	203.0	24	156.0
	31	304.0	25	234.0
	32	304.0	26	234.0
	33	203.0	27	156.0
	34	non-heated	28	non-heated
number of rods	200		39170	

fuel length                            3.66 m  
plenum gas volume                    1.235 x 10<sup>-5</sup> m  
clad outer diameter                   1.0732 x 10<sup>-2</sup> m  
clad thickness                        6.187 x 10<sup>-4</sup> m  
pellet diameter                        9.3146 x 10<sup>-3</sup> m  
fuel rod pitch                         1.42 x 10<sup>-2</sup> m

(The last four values are those at a full power operating condition)

- (7) Input data for the pressurizer were;

cross-sectional area                    3.58 m<sup>2</sup>  
height                                    15.56 m  
subcooled water level                   9.0 m  
void fraction of saturated region      0.99  
stand pipe length                        0.1 m

- (8) ECC water was assumed to be injected into mixing junction 28 in the intact loop, and into mixing junction 34 in the broken loop. Input data for ECC were;

Accumulator  
 initial water volume 23.3 m<sup>3</sup>  
 initial nitrogen volume 10.0 m<sup>3</sup>  
 specific enthalpy of water 30 kcal/kg  
 initial pressure 44 atm

Pumped injection  
 specific enthalpy of water 30 kcal/kg  
 mass flow rate 220 kg/sec

for each loop.

Pumped injection was initiated after the break took place with a delay time of 25 sec.

- (9) No structural heat source or sink was assumed.  
 (10) No particular model for the container was provided except the temporary behavior of the container pressure which was an input function of time, i.e.;

time(sec)	0.0	7.5	15.0	30.0	1000.0
pressure(atm)	1.0	2.7	4.0	4.0	4.0

- (11) Reverse flow at the break point occurred when the pressure in the primary loop became below the containment pressure. The quality for this reverse flow was set to 0.001. The loss coefficient at the break point for a reverse flow was 20.0, so that the total amount of reverse flow turned out to be small compared with ECC water injected.
- (12) Because the flow area at the pump node was small compared with the other nodes, the pressure in the broken loop pump node dropped very rapidly in the early stage of blowdown. To avoid this unrealistically great pressure drop, the momentum flux term was omitted from the momentum equation only in the pump node.
- (13) Core bypass area was nodalized into a single node, so condensation at this node became so large in reflooding stage that flow tended to go into the bypass region rather than into the core nodes. To prevent this flow pattern, the loss coefficient of bypass node was increased gradually from 47.9 to 150.0 after 60.0 sec.
- (14) Loss coefficient of node 10 for a reverse flow was increased gradually from 0.1 to 10.0 to account for a form loss at the connection between the downcomer and the pipe after 21 sec.

### 3.2 CHF Correlations

In the present calculation, in the same way as RELAP-4 code<sup>(6)</sup>, B&W-2, Bennett, and Modified Bennett correlations were used as follows;

p > 1500	B&W-2 correlation
1500 > p > 1300	Interpolation between B&W-2 and Bennett
1300 > p > 1000	Bennett correlation
1000 > p > 725	Interpolation between Bennett and Modified Bennett
725 > p	Modified Bennett

in pool condition i.e.  $G < G_{min}$ , CHF was calculated by

$$\phi = (\phi_{CHF} - \phi_{min})G/G_{min} + \phi_{min}$$

$$\phi_{min} ; 67.9 \text{ kcal/m}^2\text{.sec (90000 Btu/ft}^2\text{.hr)}$$

$$G_{min} ; 271.2 \text{ kg/m}^2\text{.sec (20000 lbm/ft}^2\text{.hr)}$$

$$\phi_{CHF} ; \text{CHF value at } G=G_{min}$$

### 3.3 Heat Transfer Correlations

The heat transfer correlations used in the present calculation were summarized in Table 3-3 and Table 3-4. In the post CHF calculation, heat transfer correlations which assume film boiling conditions are likely to be underestimated compared with experimental data at low qualities. So transition boiling are assumed if quality becomes smaller than a threshold value in the following way.

#### (1) Forced convection transition boiling

$$h^{tr} = (h_0^{tr} - h_c^{tr}) \left(\frac{x}{x_T}\right)^2 + 2(h_c^{tr} - h_0^{tr}) \frac{x}{x_T} + h_0^{tr}$$

$h^{tr}$  ; H.T.C. in transition boiling

$h_0^{tr}$  ; H.T.C. by Mode 4 at  $x=0.0$

$h_c^{tr}$  ; H.T.C. by Mode 4 at  $x=x$

$x$  ; quality

$x_T$  ; threshold quality (=0.5)

#### (2) Pool transition boiling

$$h^{tr} = \phi_{CHF} / (T_W - T_B)$$

$h^{tr}$  ; H.T.C. in transition boiling

$\phi_{CHF}$  ; CHF value at present condition

$T_W$  ; wall temperature

$T_B$  ; bulk temperature

$x_T$  ; threshold quality (=0.1)

### 3.4 Critical Flow

Zaloudeck equation and Moody table are implemented for a subcooled condition and a saturated condition respectively. At the region,  $0.0 < x < 0.02$ , critical flows calculated by these two models are connected smoothly. The discharge coefficient for Moody model was 0.6.

### 3.5 Relaxation Parameters

In the THYDE-P the relaxation models are implemented to avoid discontinuity or rapid change which may be brought about by various mode transitions.

- (1) In order to ensure smooth variation in enthalpy with the flow direction change, we introduce parameters for each node such that

$$\frac{d\eta_i}{dt} = \frac{S - \eta_i}{\tau_i} \quad \text{--- (8)} \quad (\eta_i = A \text{ or } E)$$

where  $S = \begin{cases} 1 & \text{when } G < 0 \\ 0 & \text{when } G \geq 0 \end{cases}$   
 $\tau_i$  delay parameter (=0.05 sec)

Eq. (8) is calculated using a linear implicit method.

- (2) To ensure continuous transition of the heat transfer coefficient with mode changes,  $h^{tr}$  is smoothed by

$$\frac{dh^{tr}}{dt} = \frac{h_c^{tr} - h^{tr}}{\tau_2} \quad \text{--- (9)}$$

where  $h^{tr}$  ; effective H.T.C.  
 $h_c^{tr}$  ; H.T.C. calculated by correlations  
 $\tau_2$  ; delay parameter (=0.08 sec)

Eq. (9) is calculated using a linear implicit method.

- (3) As shown in Section 2.2, to avoid a rapid depressurization caused by a condensation, coolant density change is moderated as follows;

$$\frac{d\rho^*}{dt} = \frac{\rho - \rho^*}{\tau}$$

Since there is no theoretical basis to determine  $\tau$  until now, we have no choice but to set the values empirically in the way that  $\tau$  is proportional to node volume. In the present calculation,  $\tau$  was set as follows;



after 8 sec

1.0 sec for nodes 8,9,19,20 and nodes 41,44  
 0.08 sec for other nodes  
 1.0 sec for mixing junctions 28 and 34  
 0.08 sec for other mixing junctions

after 21 sec

1.0 sec for nodes 23 and 29  
 2.0 sec for nodes 24 28 and nodes 30 34  
 4.0 sec for nodes 17 20 and nodes 41 44  
 6.0 sec for nodes 5 7 and node 9  
 10.0 sec for nodes 10,22 and 35  
 12.0 sec for node 8  
 20.0 sec for node 21 (downcomer node)  
 0.1 sec for other nodes  
 4.0 sec for mixing junctions from 28 to 34  
 0.1 sec for other mixing junctions

### 3.6 Drift Flux Model

In the THYDE-P code, the relative velocity between gas and liquid is given by

$$u_{rel} = \frac{u_{gj}}{1-\alpha} \quad \text{--- (10)}$$

where  $U_{gj}$  is the drift velocity,

$$u_{gj} = u_{gj}^0 S_{\alpha}^2 = 1.41 \left[ \frac{g \sigma (P_{gs} - P_{ls})}{\rho_{fs}} \right]^{1/4} S_{\alpha}^2 \quad \text{--- (11)}$$

and  $S_{\alpha}$  is chosen to be of the form

$$S_{\alpha} = 1 - e^{-\frac{1-\alpha}{1-\alpha_c}} \quad \text{--- (12)}$$

so as to avoid that  $U_{rel}$  becomes unrealistically great as  $\alpha$  approaches unity.

As the energy equation is the form of (1) in section 2.1, the term 'I' plays an important role, when flow becomes stagnant i.e.  $G = 0$ . On the other hand, in the RELAP4 code<sup>(6)</sup> a vertical slip model is introduced to account for the energy transport effect by two phase flow, using the vertical slip velocity defined only by void fractions as follows;

$$V_{slip} = (10.0 + 4.0 \times \Delta \alpha) \alpha (1-\alpha) (1 - 1.25\alpha) \quad \text{--- (13)}$$

In the reflooding stage, flow in the core becomes nearly

stagnant and void fractions are great, so the drift flux model becomes important in energy transport calculation and the value of the drift velocity has a great influence on a quenching time.

In the reflooding stage, the system pressure becomes nearly equal to the containment pressure (in the present calculation 4.0 atm), so the drift velocity  $U_{dj}^0$  has almost the same value anywhere in the system (in the present calculation, about 0.21 m/sec). Plot data in Fig.3-3 are calculated using these values.

As be seen from Eqs. (11) and (12), the relative velocity in a high void region depends greatly on the value of  $\alpha_c$ .

In Fig.3-3  $U_{rel}$  for  $\alpha_c = 0.8, 0.97, 0.99$  are shown with  $V_{slip}$  calculated by Eq. (13). In the present calculation  $\alpha_c$  was set to 0.97, as will be discussed in section 4.10.

Table 3-1 Node Geometrical Data

Node No.	Description	Flow Area	Node Length	Node Volume
		A (m <sup>2</sup> )	L (m)	V (m <sup>3</sup> )
1	Broken loop hot leg	0.4266	5.240	2.235
2	SG inlet plenum	2.8953	1.665	4.821
3	SG U-tube	0.9952	5.000	4.976
4	SG U-tube	0.9952	5.460	5.434
5	SG U-tube	0.9952	10.460	10.410
6	SG outlet plenum	2.8953	1.665	4.821
7	Broken loop cold leg	0.4865	7.340	3.571
8	Pump	0.1917	12.412	2.379
9	Broken loop cold leg	0.3837	2.825	1.084
10	Broken loop cold leg	0.3837	3.130	1.201
11	Intact loop hot leg	1.2798	2.000	2.560
12	Intact loop hot leg	1.2798	3.240	4.147
13	SG inlet plenum	8.6859	1.665	14.462
14	SG U-tube	2.9856	5.460	14.928
15	SG U-tube	2.9856	5.460	16.301
16	SG U-tube	2.9856	10.460	31.229
17	SG outlet plenum	8.6859	1.665	14.462
18	Intact loop cold leg	1.4594	7.340	10.712
19	Pump	0.5750	12.412	7.137
20	Intact loop cold leg	1.1512	5.955	6.855
21	Downcomer	2.7435	7.248	19.885
22	Lower plenum	4.8578	6.075	29.511
23	non-Active core in average	4.3552	0.230	1.002
24	Active core in average	4.3552	0.800	3.484
25	Active core in average	4.3552	0.800	3.484
26	Active core in average	4.3552	0.800	3.484
27	Active core in average	4.3552	0.800	3.484
28	non-Active core in average	4.3552	0.230	1.002
29	non-Active core in hot	0.0222	0.230	5.11-3
30	Active core in hot	0.0222	0.800	1.78-2
31	Active core in hot	0.0222	0.800	1.78-2
32	Active core in hot	0.0222	0.800	1.78-2
33	Active core in hot	0.0222	0.800	1.78-2
34	non-Active core in hot	0.0222	0.230	5.11-3
35	Core bypass	0.2419	3.660	0.885
36	Core cross area	9.079-4	0.100	9.08-5
37	Upper plenum	9.2941	4.341	40.346
38	Upper head	3.8568	3.658	14.108
39	Pressurizer surge line	0.0661	15.00	0.992
40	Pressurizer surge line	0.0661	14.30	0.945
41	Pumped injection duct	0.2192	12.00	2.630
42	Pumped injection duct	0.0731	12.00	0.877
43	Accumulator duct	0.1161	120.00	13.932
44	Accumulator duct	0.0387	120.00	4.644

Table 3-2 Loss Coefficient

Node No.	K	K <sup>Af</sup>	K <sup>Ar</sup>	K <sup>Ef</sup>	K <sup>Er</sup>
1	0.023	0.043	0.084	0.0	0.0
2	0.021	3.73	1.97	0.0	0.0
3	0.011	0.033	0.048	0.0	0.0
4	0.008	0.0	0.0	0.0	0.0
5	0.017	0.0	0.0	0.033	0.048
6	0.030	0.0	0.0	3.73	1.97
7	0.019	0.042	0.077	0.0	0.0
8	0.303	0.273	0.367	0.203	0.203
9	0.028	0.0	0.0	0.0	0.0
10	0.019	0.0	0.0	0.0	0.0
11	0.024	0.043	0.083	0.0	0.0
12	0.028	0.0	0.0	0.0	0.0
13	0.013	3.73	1.97	0.0	0.0
14	0.025	0.033	0.048	0.0	0.0
15	0.021	0.0	0.0	0.0	0.0
16	0.027	0.0	0.0	0.033	0.048
17	0.029	0.0	0.0	3.73	1.97
18	0.022	0.042	0.077	0.0	0.0
19	0.303	0.273	0.367	0.203	0.203
20	0.011	0.0	0.0	0.0	0.0
21	1.032	0.0	0.0	0.0	0.0
22	1.398	0.0	0.0	0.0	0.0
23	6.416	0.74	0.74	0.0	0.0
24	6.161	0.0	0.0	0.0	0.0
25	6.072	0.0	0.0	0.0	0.0
26	6.218	0.0	0.0	0.0	0.0
27	6.045	0.0	0.0	0.0	0.0
28	6.737	0.0	0.0	0.0	0.0
29	5.295	1.284	2.482	0.0	0.0
30	5.642	0.0	0.0	0.0	0.0
31	5.570	0.0	0.0	0.0	0.0
32	5.574	0.0	0.0	0.0	0.0
33	4.662	0.0	0.0	0.0	0.0
34	6.104	0.76	0.34	0.0	0.0
35	47.857	0.77	0.83	0.87	0.78
36	12.354	0.0	0.0	0.0	0.0
37	2.08-2	0.0	0.0	0.0	0.0
38	5.0	1.491+4	1.491+4	0.0	0.0
39	5.0	0.41	0.87	0.0	0.0
40	5.0	0.0	0.0	0.0	0.0
41	10.0	0.0	0.0	0.0	0.0
42	10.0	0.0	0.0	0.0	0.0
43	10.0	0.109	0.049	0.0	0.0
44	10.0	0.109	0.049	0.0	0.0

Table 3-3 Heat Transfer Correlations

Mode	Conditions		Correlations
	Coolant Condition	Other Conditions	
1	subcooled	$T_w < T_{sat}$	Dittus-Boelter
2	subcooled	$T_w > T_{sat}$	Interpolation between Mode1 and Mode3
3	saturated	$\phi < \phi_{CHF}$	Thom
4	saturated	$\phi > \phi_{CHF}$	( see Table 3.4 )
5	superheated	$Re < 3000$	Forced convection
6	superheated	$3000 < Re < 5000$	Interpolation between Mode5 and Mode7
7	superheated	$Re > 5000$	McEligot

Table 3-4 Heat Transfer Correlations in Mode-4

Mode	Conditions	Correlations
4-1	$G > G_{min}, x > x_{c1}$	Groenevelt and Dougall-Rhosenow
4-2	$G > G_{min}, x < x_{c1}$	Forced convection transition boiling
4-3	$G < G_{min}, x > x_{c2}$	Modified Bromley
4-4	$G < G_{min}, x < x_{c2}$	Pool transition boiling

$T_w$  ; Wall temperature

$G_{min}$  ; Minimum mass flux (=271.2 kg/m<sup>2</sup>sec)

$x_{c1}$  ; Threshold quality (=0.5)

$x_{c2}$  ; Threshold quality (=0.1)

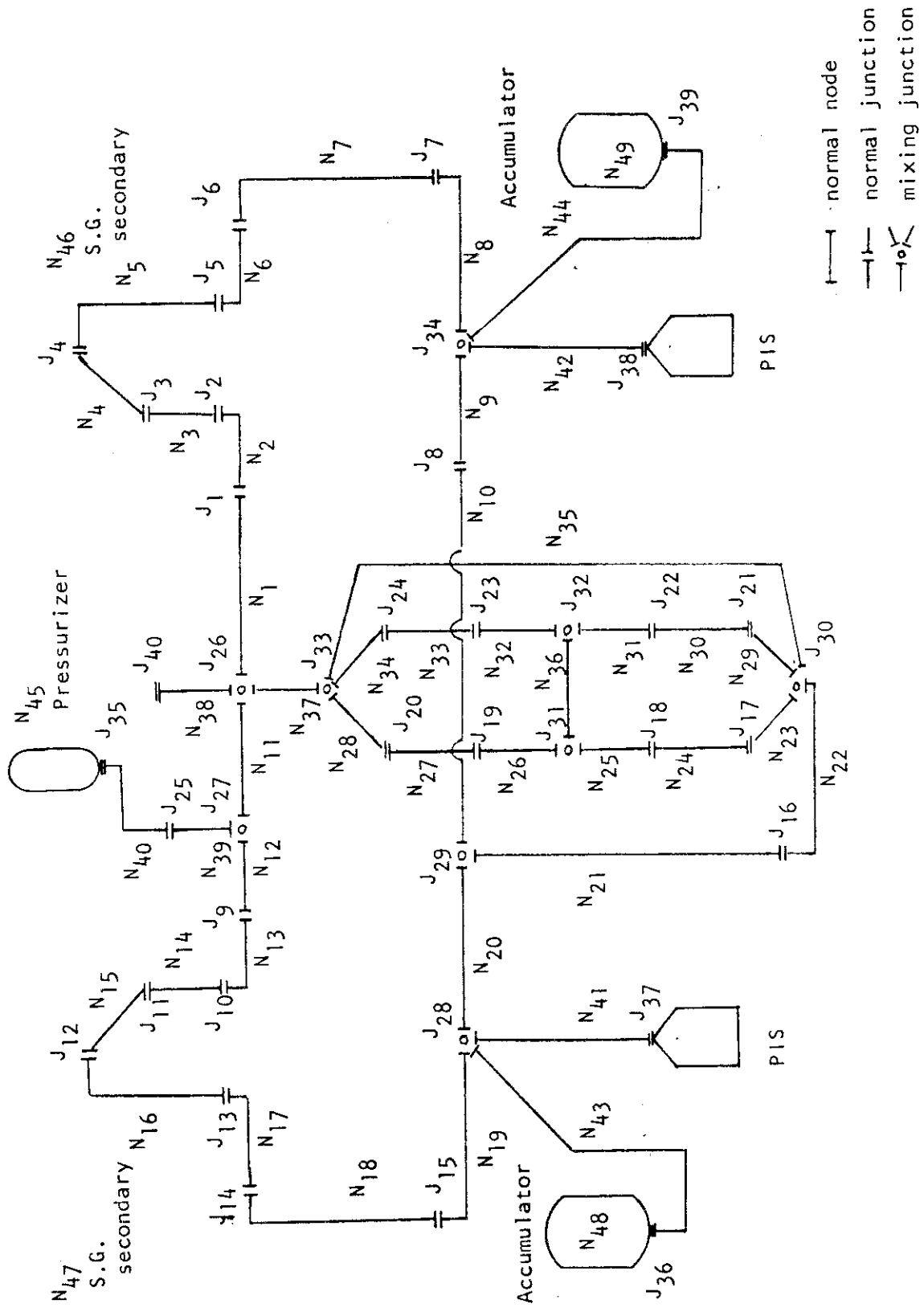


Fig. 3-1 Nodalization for 4-loop PWR

No.	Description	Length (m)	Pressure (MPa)
1	hot leg	5.24	16.06
2	SG inlet plenum	1.665	16.10
3	SG U-tube	20.92	16.01
4	SG outlet plenum	1.665	15.99
5	crossover leg	7.34	15.96
6	pump	12.41	15.97
7	cold leg	5.96	16.42
8	downcomer	7.248	16.48
9	lower plenum	1.948	16.50
10	core	3.66	16.31
11	upper plenum	1.64	16.12

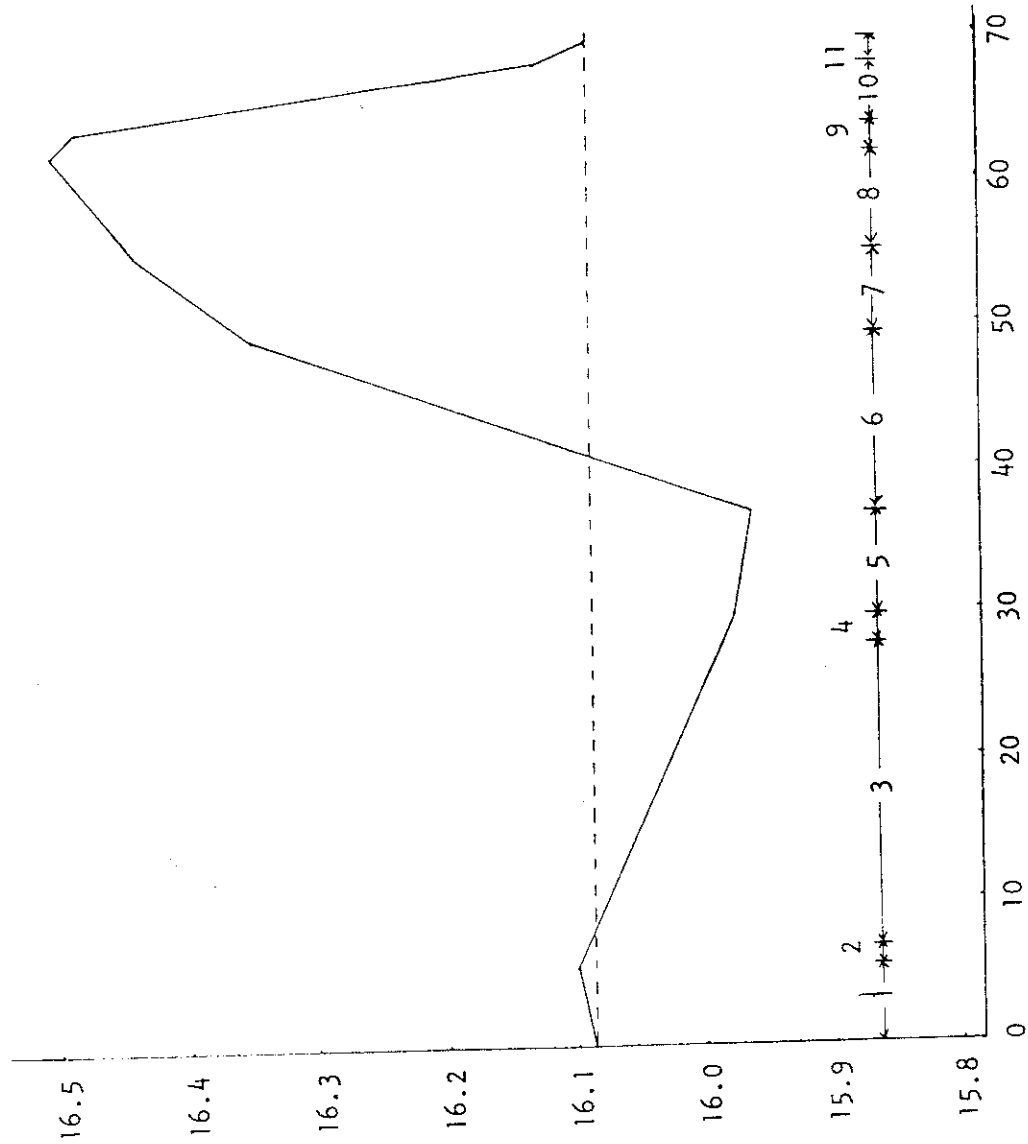


Fig. 3-2 Steady State Pressure Distribution

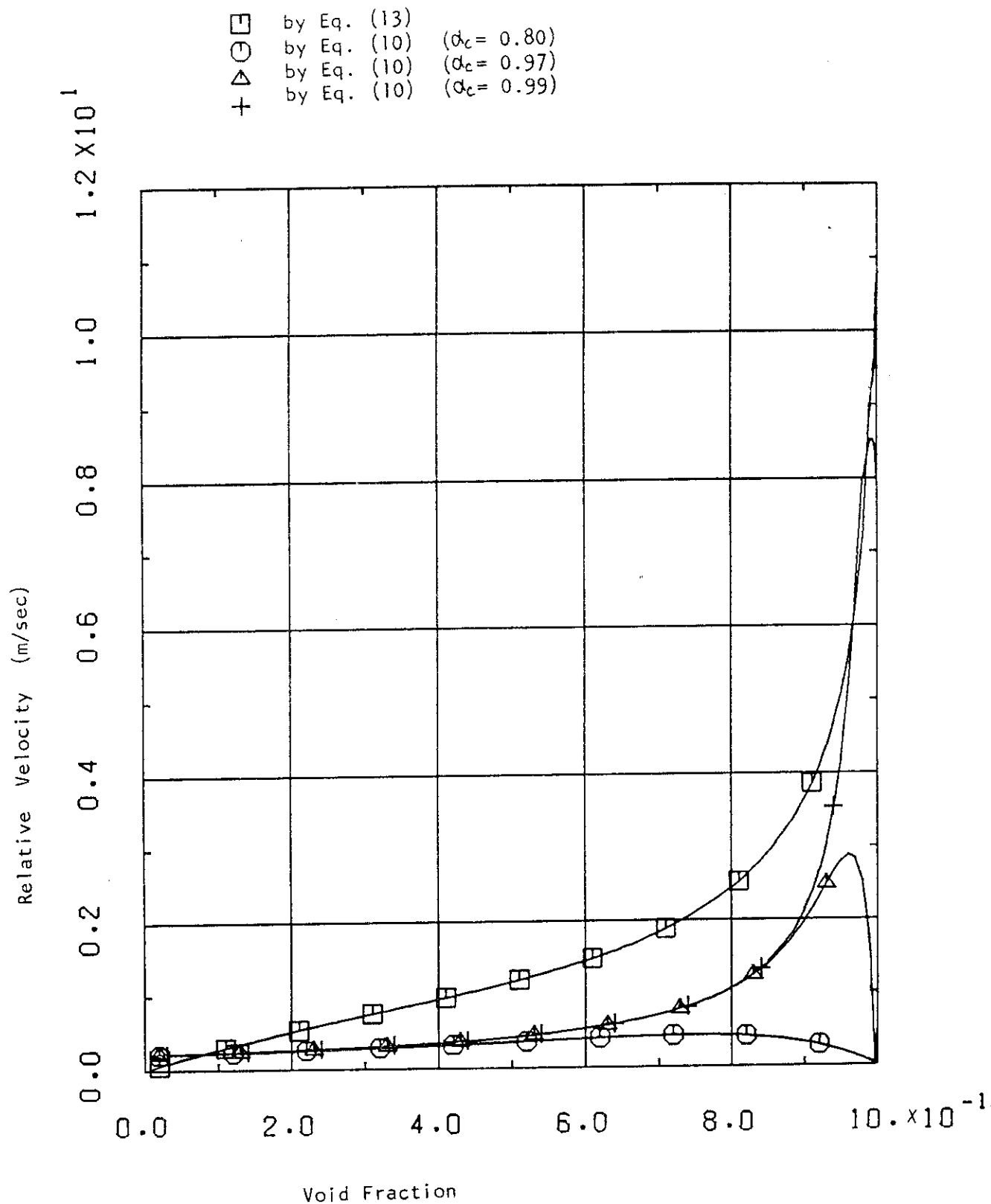


Fig. 3-3 Relative Velocity



#### 4. Calculated Results

In this section, we will show the calculated results of Run 20 along with the discussions on them.

The maximum time step width was set to 32 ms after 20sec, and the total CPU time required for Run 20 by a FACOM M-200 computer was about 1.5 hours.

##### 4.1 Overall Description

The chronology of events is shown in Table 4-1. If we define the end of blowdown to be the time when the system pressure stops decreasing and the injected ECC water starts to flow down the downcomer, then it was 38 sec after the break occurred. The accumulator in the intact loop started injection at 16 sec and a duct node (node 20) was filled with subcooled water at 25 sec, but almost all of injected ECC water flowed out directly through the break until 38 sec. After ECC water penetrated the downcomer, an upward flow persisted in the heated section of the core by 49 sec which we define to be the time of the start of the reflooding. The accumulator was exhausted at 62 sec, but the P.I. kept injecting low enthalpy water. The driving force of the P.I. together with the static head of water in the downcomer supplied subcooled water to the heated section of the core and eventually quenching took place. After 140 sec all heated nodes in the average channel quenched, so that the average quench velocity was about 3.5 cm/sec.

The system pressure was shown in Fig.4-1-1. At 0.05 sec after the break occurred, the system was depressurized to 14 Mpa and bubbles appeared at the core nodes. After 0.1 sec, the upper plenum and the hot leg region became saturated successively so the system depressurized more gradually to the containment pressure. Except the period of ECC water penetration through the downcomer and the lower plenum, the system pressure was kept constantly at a little bit above the containment pressure after 45 sec.

##### 4.2 Fuel and Core

In Figs.4-2-1 to 4-2-4, the clad surface temperatures in the average channel and in the hot channel are shown. The fuel center temperatures for two channels are shown in Fig.4-2-5 and in Fig.4-2-6, respectively.

The clad surface temperatures reached the peak temperature after 10 sec, and then they dropped owing to a reverse flow in the core. After 25 sec the core flow became stagnant again so the surface temperature rose gradually till 50 sec when reflooding began. After reflooding started, the surface temperature began to decrease and the lowest part of the heated section of the average core became quenched at 74 sec. After that, the core quenched from lower part to upper part successively and all of the heated section of the average core became quenched by 140 sec. On the other hand, in the hot channel region quenching delayed compared with the average channel.

The period during which film boiling was dominant was rather long after reflooding started (see Figs.4-2-7 and 4-2-8). By taking a node average quality as the arithmetic mean of  $x_A$  and  $x_E$  instead of the outlet quality, an average quality would decrease more rapidly and as a result, the heat transfer mode would shift to the pool transition mode more earlier.

Fig.4-2-7 and Fig.4-2-8 show the behavior of the heat transfer coefficient in the average channel along with the heat transfer modes designated by

- 2 ; subcooled boiling
- 3 ; nucleate boiling
- 4 ; film boiling
- 4' ; transition boiling
- 5 ; forced convection by superheated steam

In Figs.4-2-9 to 4-2-12, the mass flux in the core inlet and in the core outlet are shown. Fig.4-2-13 shows the heat generation rate in fuels. Fig.4-2-14 and Fig.4-2-15 show the behavior of the fuel gap pressure and the heat transfer coefficient in the gap, respectively. The gap pressure was controlled mainly by the temperature in the gas plenum. The heat transfer coefficient in the gap was almost constant during the transient, so the initial value would become an important factor for the calculation of fuel temperatures. Fig.4-2-16 shows qualities in the average channel, which indicates that qualities oscillate around 0.1, because the heat transfer modes change between the film boiling condition and the pool transition boiling condition at the threshold quality of 0.1.

Figs.4-2-17 and 4-2-18 show enthalpies in the core inlet and in the core outlet. Fig.4-2-19 shows a core cross flow. By comparing the core inlet flow with the core outlet flow in the hot channel region, it is seen that a cross flow flattens the flow distributions in the upper region of the core. This fact coincides with our physical intuition.

#### 4.3 Downcomer and Lower plenum

Figs.4-3-1 and 4-3-2 show the mass flux in the downcomer and in the lower plenum, respectively. Figs.4-3-3 and 4-3-4 show qualities in the downcomer and in the lower plenum. By 44 sec,  $G_{21}^A$  started to increase and at 48 sec  $G_{21}^E$  became positive. After that, the difference between  $G_{21}^A$  and  $G_{21}^E$  became small indicating that the downcomer was filled with liquid.

As for the flow in the lower plenum, in the early stage of blowdown from 1 sec to 3 sec the value of  $G_{22}^A - G_{22}^E$  was negative, indicating that flashing occurred in the node, but water flowed out of the system through the downcomer rather than into the core. The reason is supposed to be that the upperhead reached saturation earlier than the lower plenum on account of the high initial enthalpy, and that the water from the upperhead flowed down the core during this period.

From 46 sec till 72 sec, the value of  $G_{22}^A - G_{22}^E$  remained positive and a moderate condensation occurred in the lower plenum. This is because a large delay parameter for the density calculation was used in the downcomer node.

#### 4.4 Break

Fig.4-4-1 shows break flows  $G_{10}^E$  and  $G_9^A$ . In Fig.4-4-2 enthalpies at the break point are shown. It should be noted that  $G_{10}^E$  took on a positive value at the initial steady state. As soon as the break occurred,  $G_{10}^E$  decelerated and began to reverse its direction. But the other break flow  $G_9^A$  remained mostly positive throughout the LOCA.

From 30 sec till 47 sec, the break flow  $G_{10}^E$  increased and its enthalpy was below 100 kcal/kg. It shows that the ECC water bypassed over the downcomer to the break during this period.

Fig.4-4-4 shows the pressure at the break,  $P_{10}^E$  and  $P_9^A$ , which indicate that break flows  $G_{10}^E$  and  $G_9^A$  returned to inertial flows at 26 sec and 30 sec, respectively.

#### 4.5 Pressurizer

At 1.9 sec, region 2 (the lower region of the pressurizer) became saturated. At 22.5 sec, the level of region 2 reached the top of the stand pipe so that the coolant in region 1 started to flow out of the pressurizer. Fig.4-5-1 shows the surge flow  $G_{39}^A$ , whose rate of change varied at 1.9 sec and 22.5 sec, corresponding to the times when region 2 became saturated and when its level reached the top of the stand pipe. Fig.4-5-2 shows  $P_{39}^A$  and  $P_{40}^E$ , the former of which has the same tendency as those in the primary loop, and the latter is pressure in the pressurizer. The difference between  $P_{40}^E$  and  $P_{39}^A$  can be regarded as the driving force for the flow in the pressurizer duct.

Fig.4-5-3 shows the mass flux in the intact loop hot leg. The surge water from the pressurizer flowed into the core from 10 sec till 25 sec and it cooled the core effectively.

Fig.4-5-4 shows the water level in the pressurizer. At 22.5 sec it vanished. From 76 sec the water level increased to about 3.0 m, but the integrated mass of this flow was about 50 kg and had no effect on core cooling.

#### 4.6 Pumped Injection

Figs.4-6-1, 4-6-2 and 4-6-3 show the behavior of the P.I. system. When the P.I. system was actuated at 25 sec,  $P_{41}^E$  deviated from  $P_{41}^A$  as shown in Fig.4-6-2, and their difference accounted for the driving force of P.I. flow shown in Fig.4-6-1. Fig.4-6-3 shows  $h_{41}^E$  which gradually approached to the ECC water enthalpy of 30 kcal/kg after the actuation of P.I.

The mass flow rate in the intact loop P.I. duct was 660 kg/sec, and about one third of it flowed into the core after the termination of the accumulator injection. So the P.I. system played an important role on core cooling.

#### 4.7 Accumulator

The accumulator was actuated at 16 sec and it was terminated at 62 sec with a time constant of 3.0 sec. Fig.4-7-1 shows  $P_{43}^A$  and  $P_{43}^E$ . The former is practically the system pressure, while

the latter the pressure of the accumulator. The difference between  $P_{43}^A$  and  $P_{43}^E$  is the driving force of the accumulator. Fig.4-7-2 shows  $G_{43}^E$ . Fig.4-7-3 shows  $h_{43}^E$ , which started to decrease at 16 sec to 30 kcal/kg and after the termination of the accumulator injection increased due to flow reversal at the injection duct.

Fig.4-7-4 shows enthalpies at 19E and 20A, which tend to deviate from each other with the actuation of A.I. and P.I. When A.I. terminated at 62 sec, the quality at 20A increased because of the superheated steam flow through The S.G., but a steady flow to the core continued due to the low enthalpy water from P.I. system.

#### 4.8 Steam Generator

Fig.4-8-1 shows the S.G. feedwater flows in the broken and intact loops. Figs.4-8-2 and 4-8-3 show the heat transfer coefficient between nodes 47 and 14, and between nodes 46 and 3, respectively. Fig.4-8-4 shows the heat inputs to nodes 3 and 14 from the corresponding steam generators, both of which became heat sources to the primary flow at 17 sec. Fig.4-8-5 shows the pressure in the S.G. secondary system. The pressure in the broken loop S.G. reached 8.2 MPa, but a relief flow was not taken into account.

#### 4.9 Pump

Fig.4-9-1 shows the relative pump speed. It was shown that the rotor was locked when the break occurred and pump speed decreased exponentially with a time constant 0.05 sec.

Fig.4-9-2 shows pump heads in each loop.

#### 4.10 Difference due to Drift Velocity

In Fig.4-10-1, the clad surface temperatures calculated on the condition that  $\phi_c = 0.97$  (see Eq. (12) in section 3.6) are shown, and in Fig.4-10-2 the clad surface temperatures for  $\phi_c = 0.8$  are shown. In Figs.4-10-3 and 4-10-4, heat transfer coefficients for each case are shown.

In the case where  $\phi_c$  was set to 0.97, the lowest part of the heated section in the average channel became quenched at 74 sec, while in the case where  $\phi_c$  was set to 0.80, the quenching time was delayed. The reason is that when  $\phi_c$  is set to small value such as 0.80 the drift velocity becomes nearly zero at the quench front where a large amount of bubbles are formed, and energy transfer by rise effect of these bubbles can not be taken into account.

At the quench front, heat from fuel rods is transferred to coolant by forming bubbles, and these bubbles rise faster than liquid. To evaluate a relative velocity between vapour and liquid at the quench front is difficult, though, in the present calculation this velocity was limited to about 2.5 m/sec (when  $\phi_c = 0.97$ ) as the maximum velocity.

Table 4-1 Chronology of Events Table

Time (sec)	Events
0.01	Break took place and pump was tripped.
0.221	Upperhead became saturated.
0.40	SG feed water was tripped.
1.40	Lower plenum became saturated.
16.0	Accumulator in intact loop started injection.
22.5	Pressurizer water level reached stand pipe.
25.01	Pumped injection started.
38.0	ECC water penetrated the downcomer. (start of refill)
47.0	Bypass ended.
49.0	Subcooled water reached the bottom of core. (start of reflood)
74.0	Bottom core node in average channel quenched.
140.0	All nodes in average channel quenched.

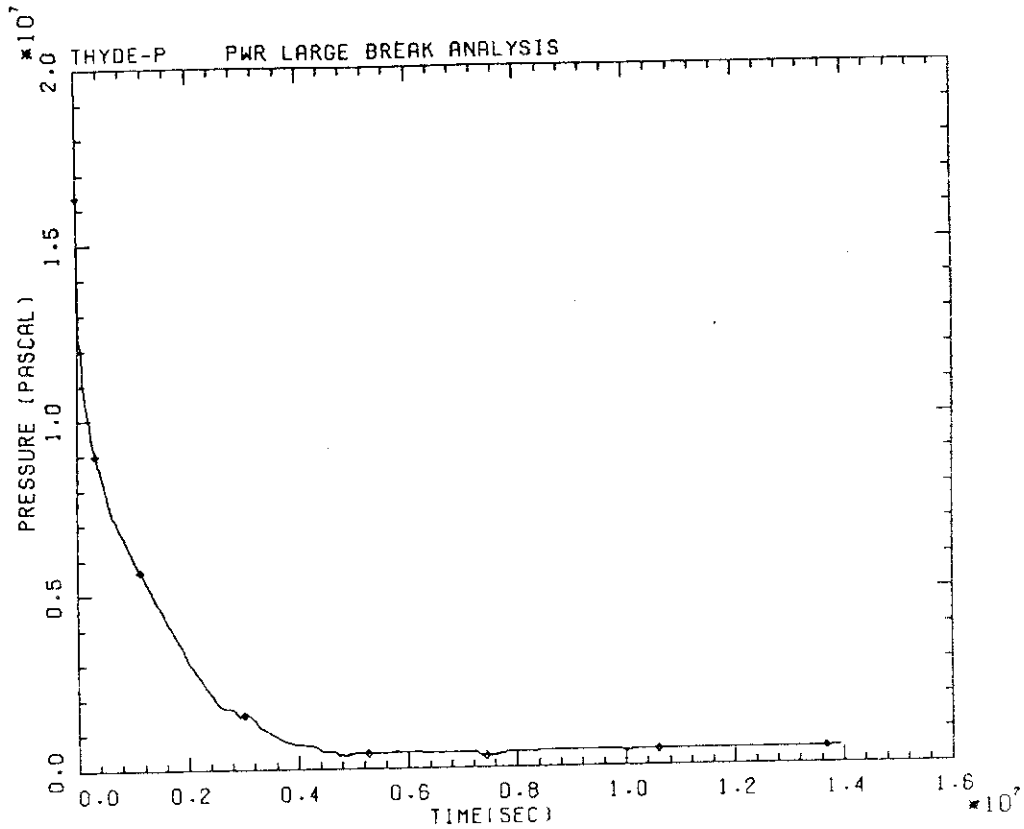


FIG. 4-1-1 PRESSURE IN CORE (NODE 25)

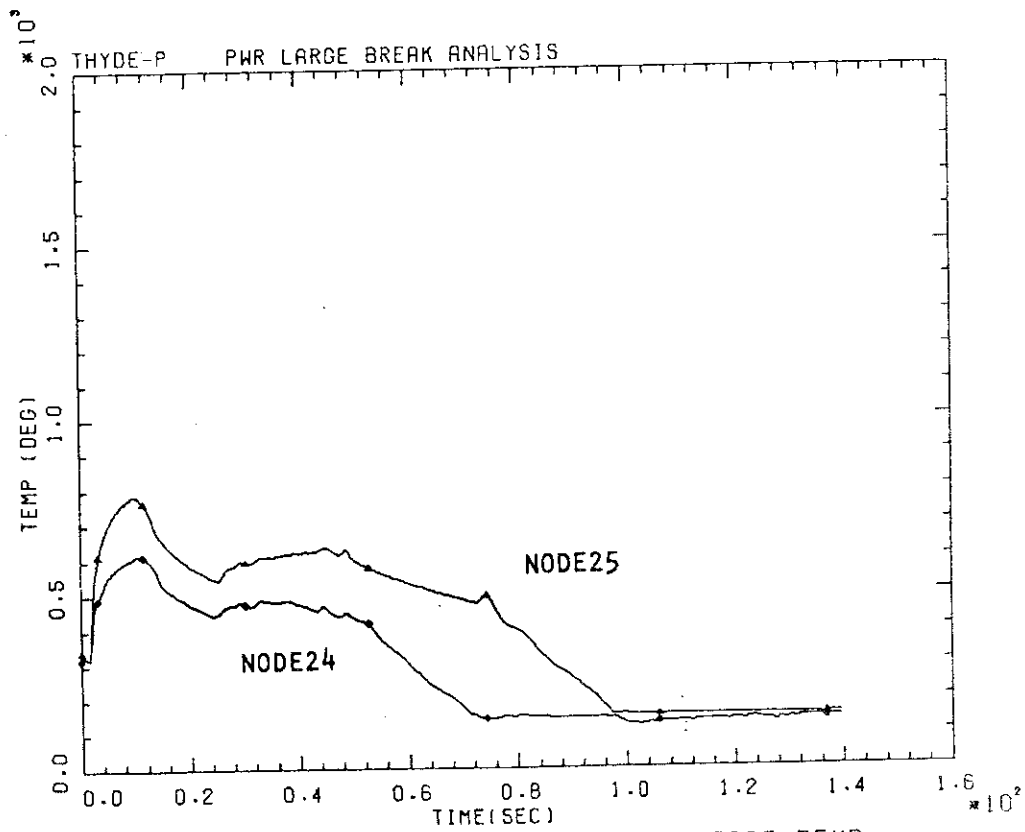


FIG. 4-2-1 AVERAGE CHANNEL SURFACE TEMP.

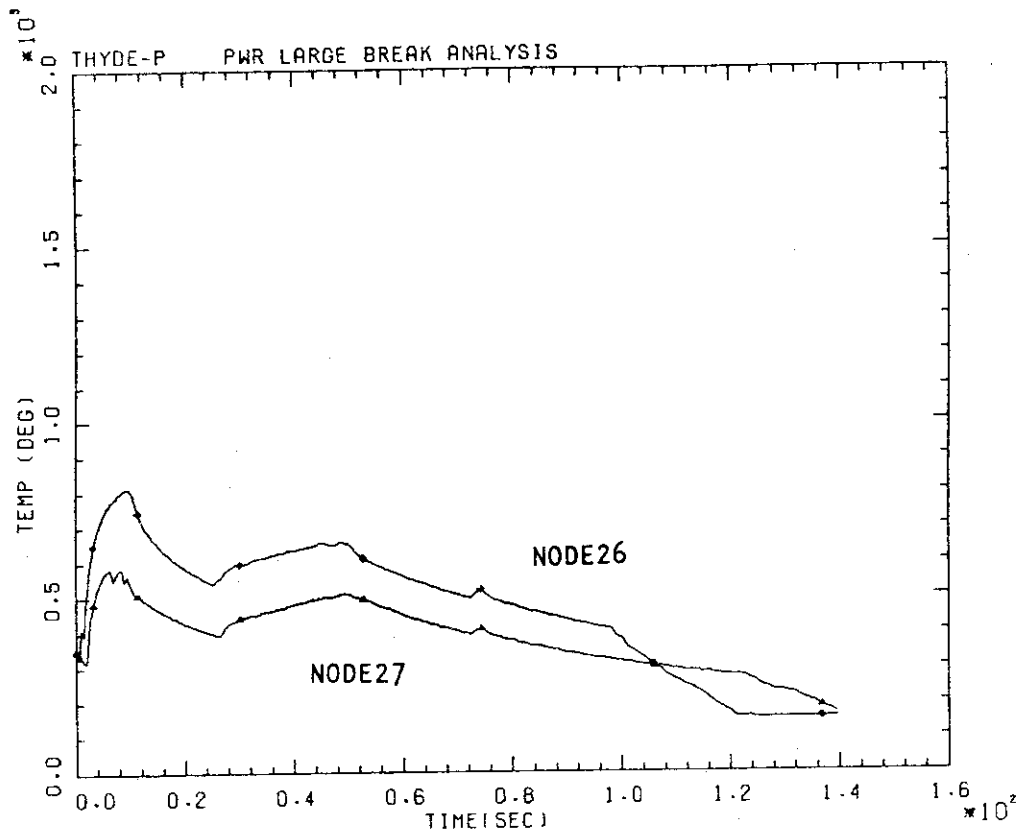


FIG. 4-2-2 AVERAGE CHANNEL SURFACE TEMP.

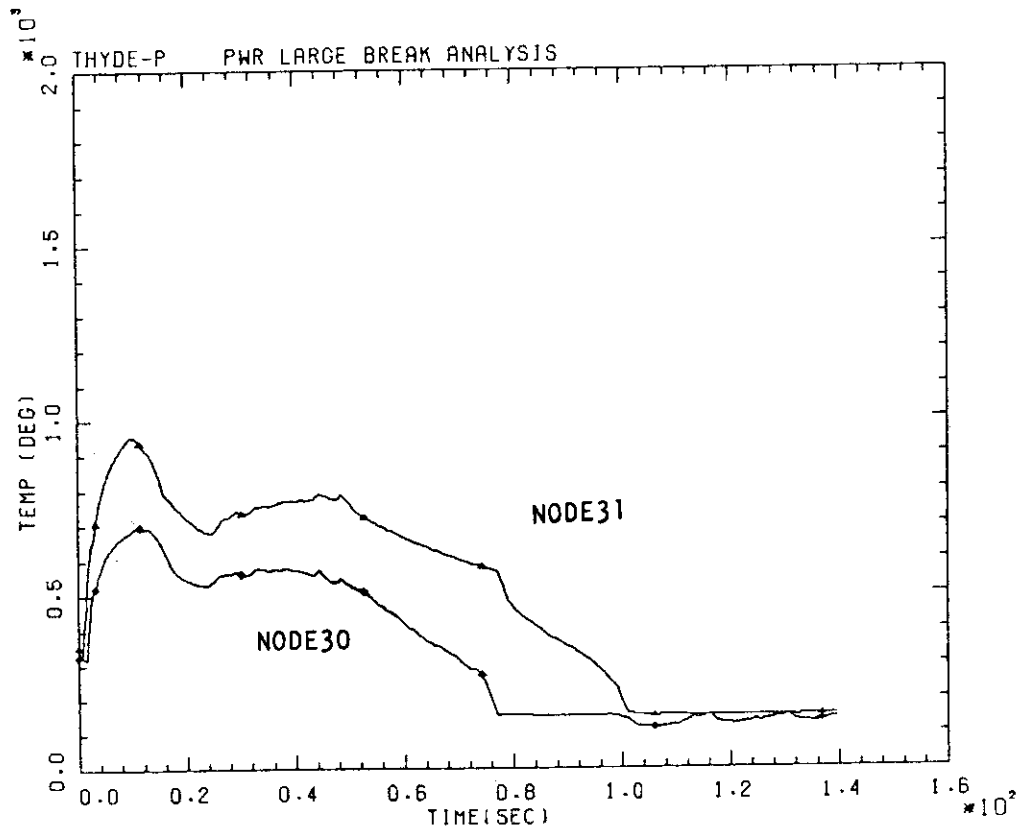


FIG. 4-2-3 HOT CHANNEL SURFACE TEMP.

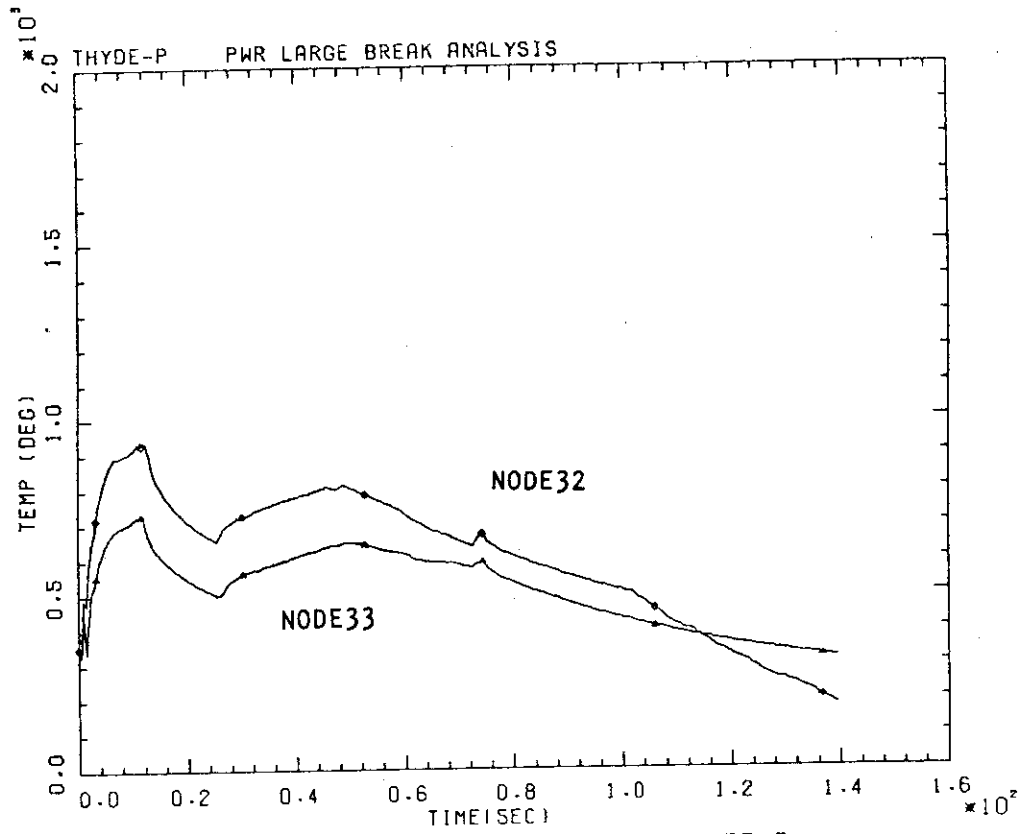


FIG. 4-2-4 HOT CHANNEL SURFACE TEMP.

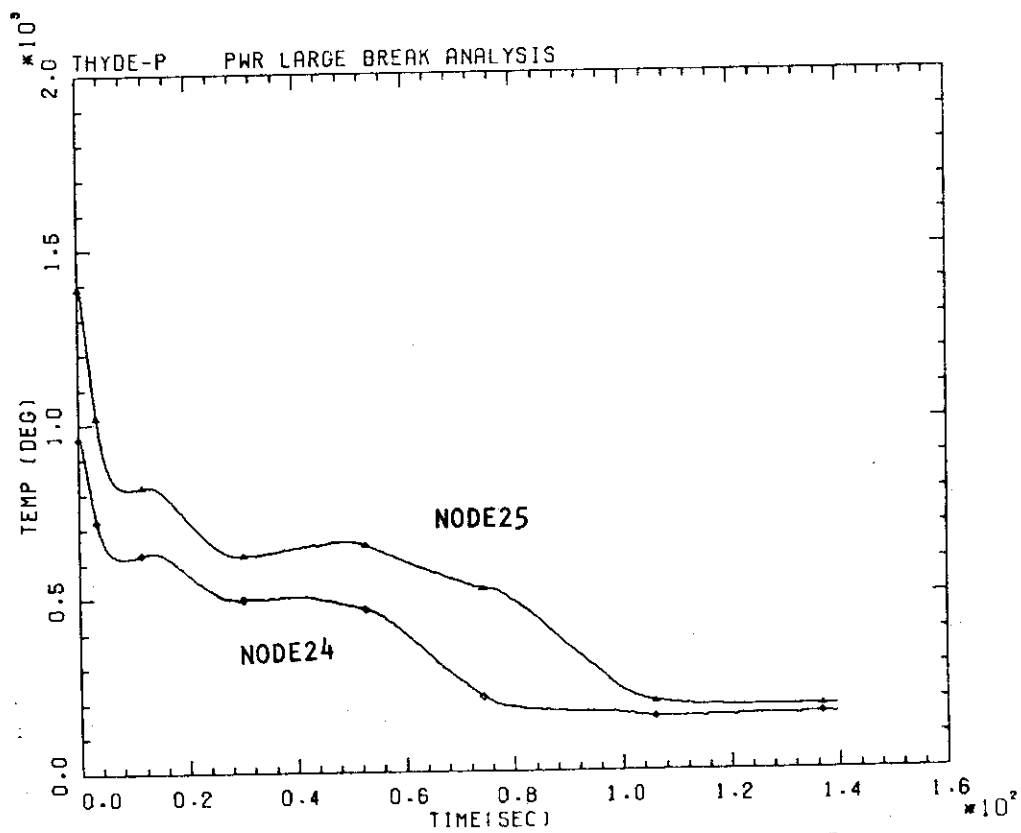


FIG. 4-2-5 AVERAGE CHANNEL CENTER TEMP.



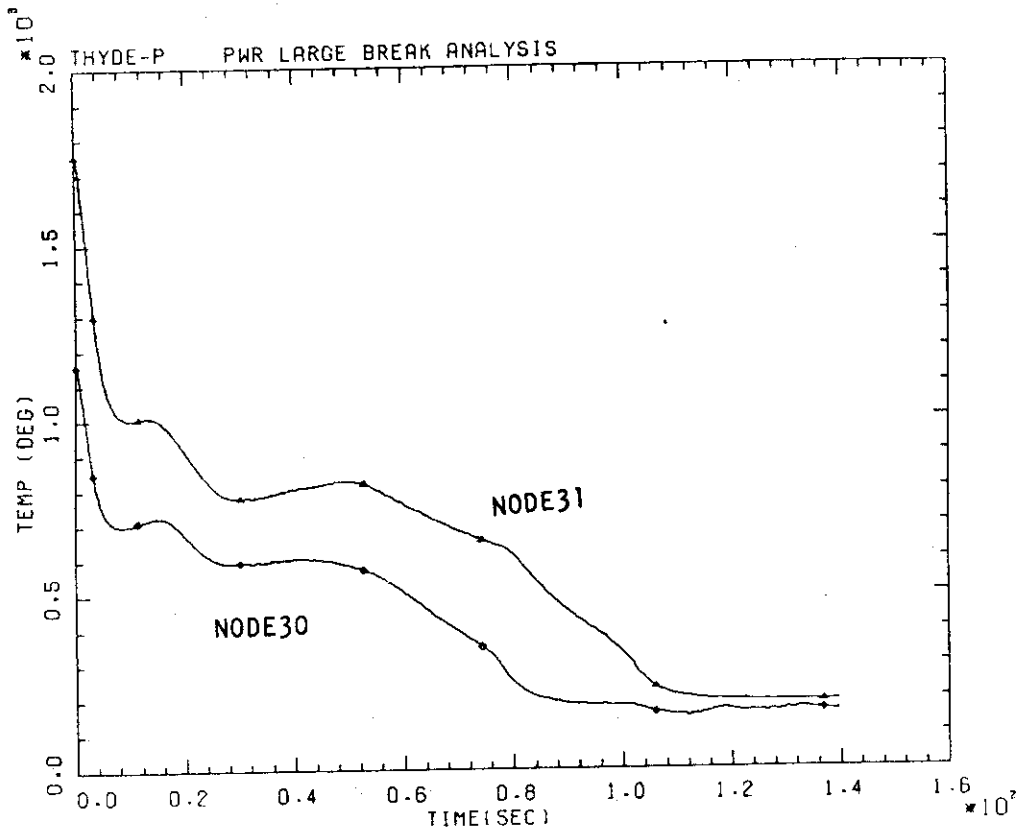


FIG. 4-2-6 HOT CHANNEL CENTER TEMP.

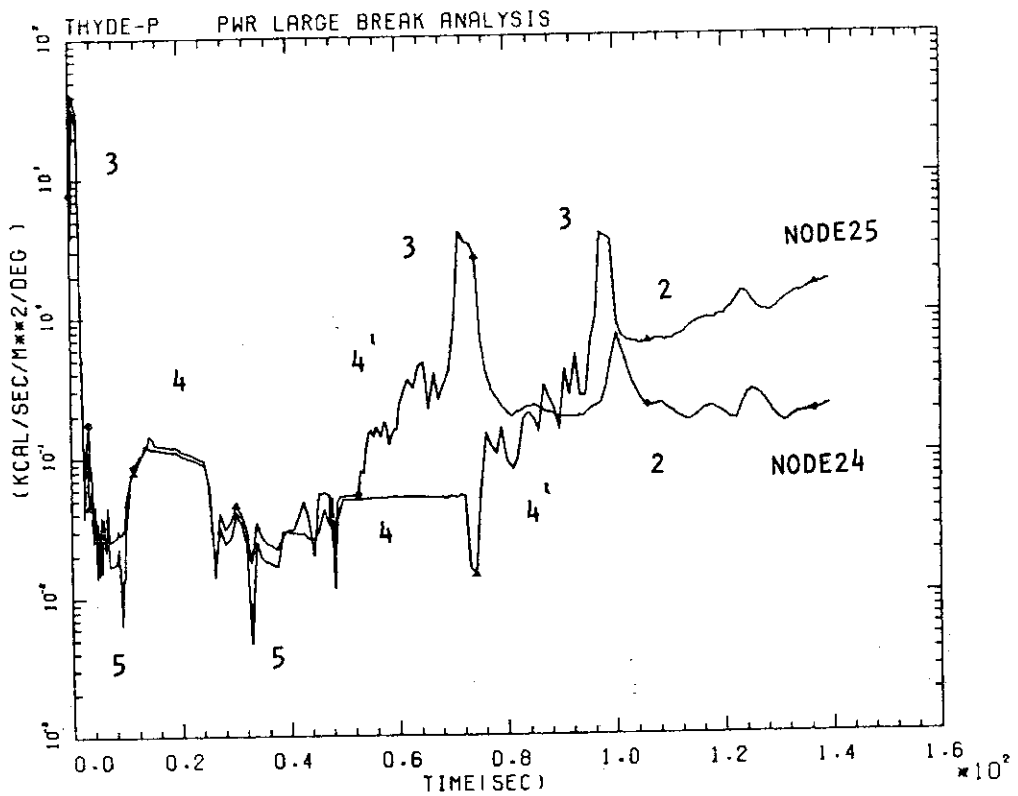


FIG. 4-2-7 H.T.C. AT AVERAGE CHANNEL

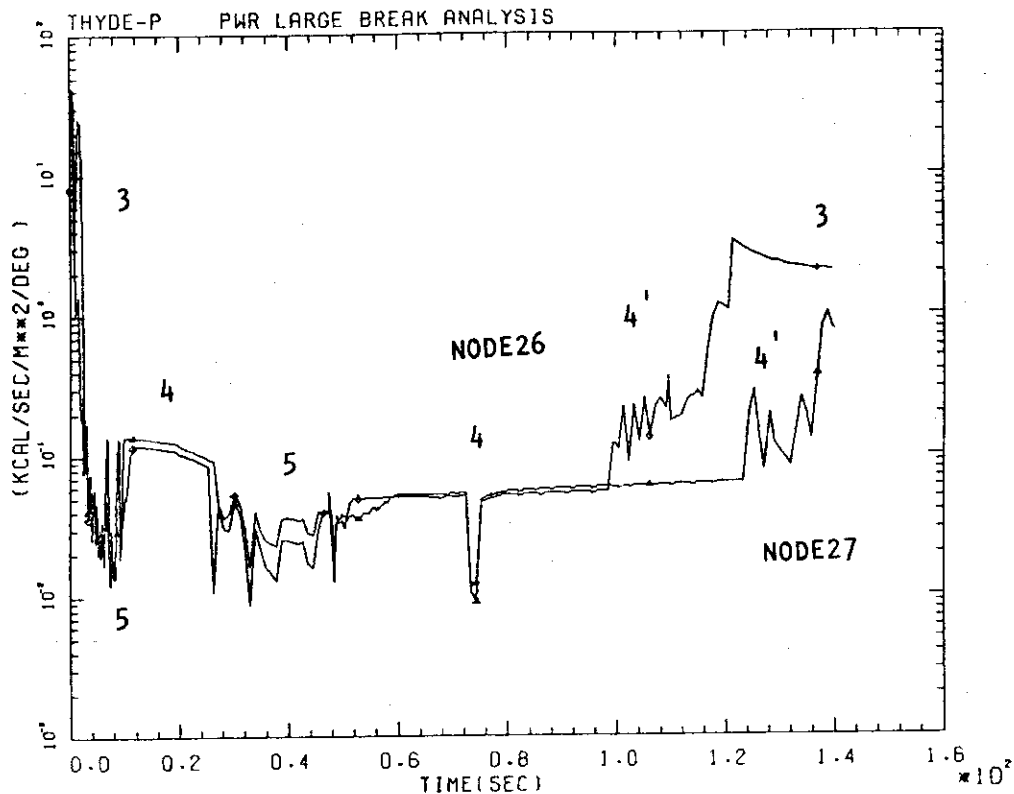


FIG. 4-2-8 H.T.C. AT AVERAGE CHANNEL

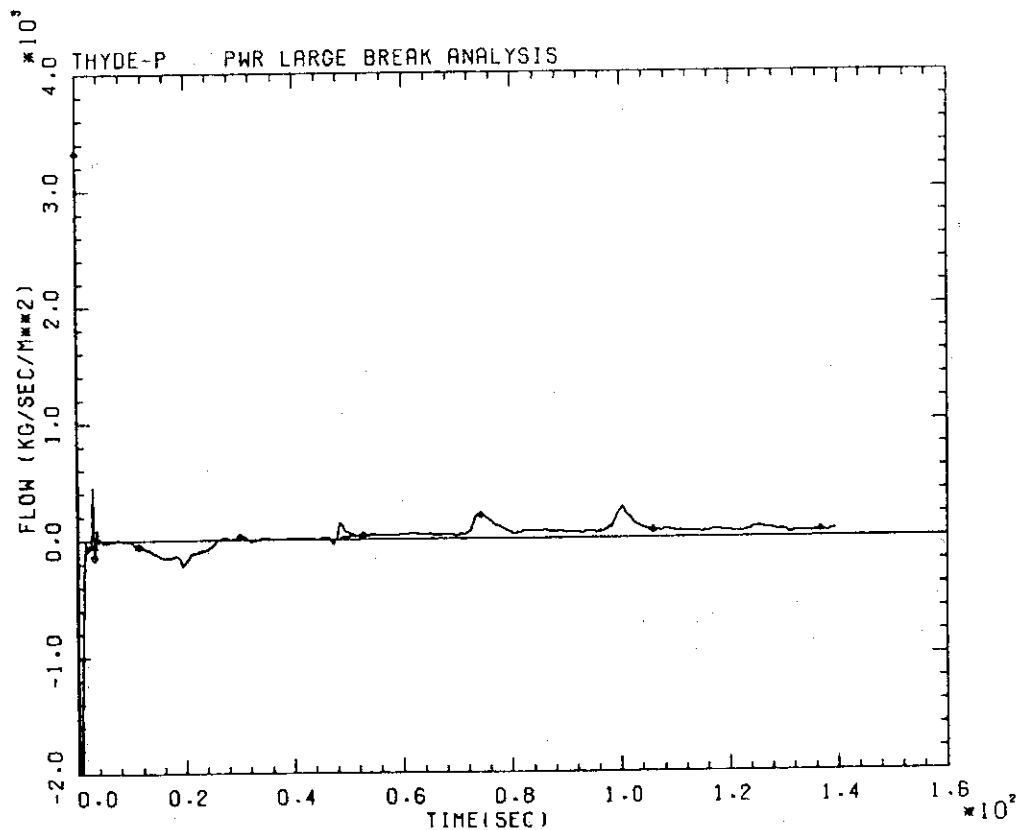


FIG. 4-2-9 CORE INLET FLOW (NODE 23)

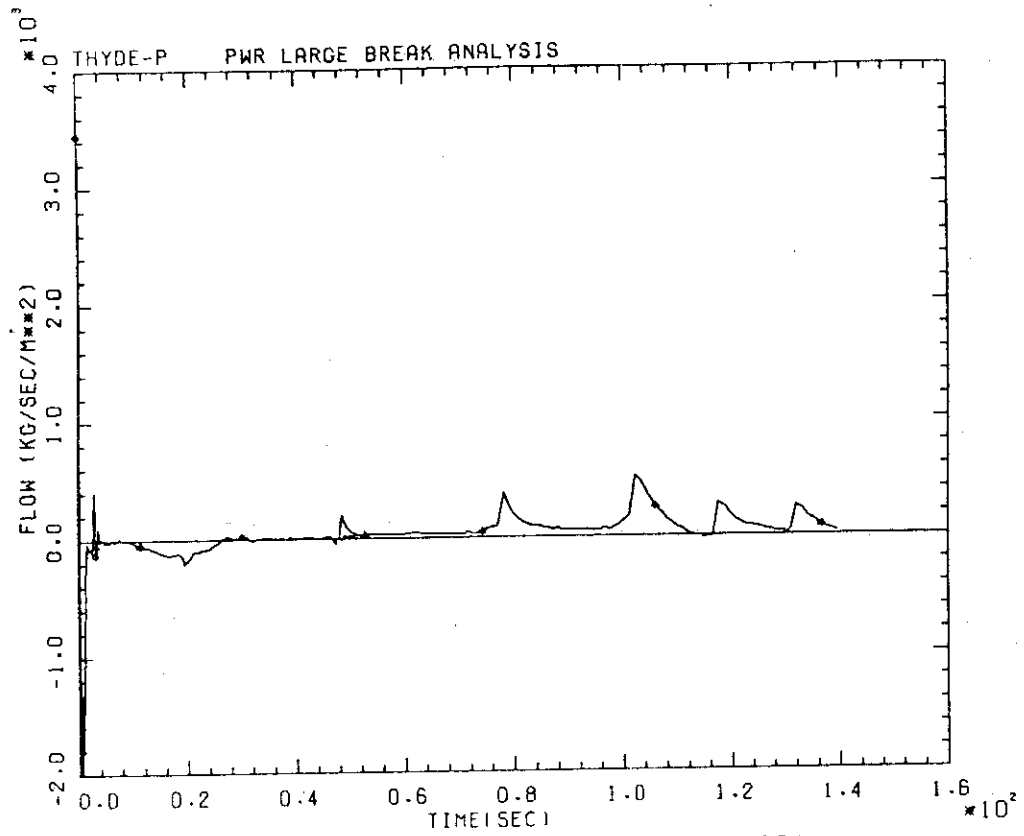


FIG. 4-2-10 CORE INLET FLOW (NODE 29)

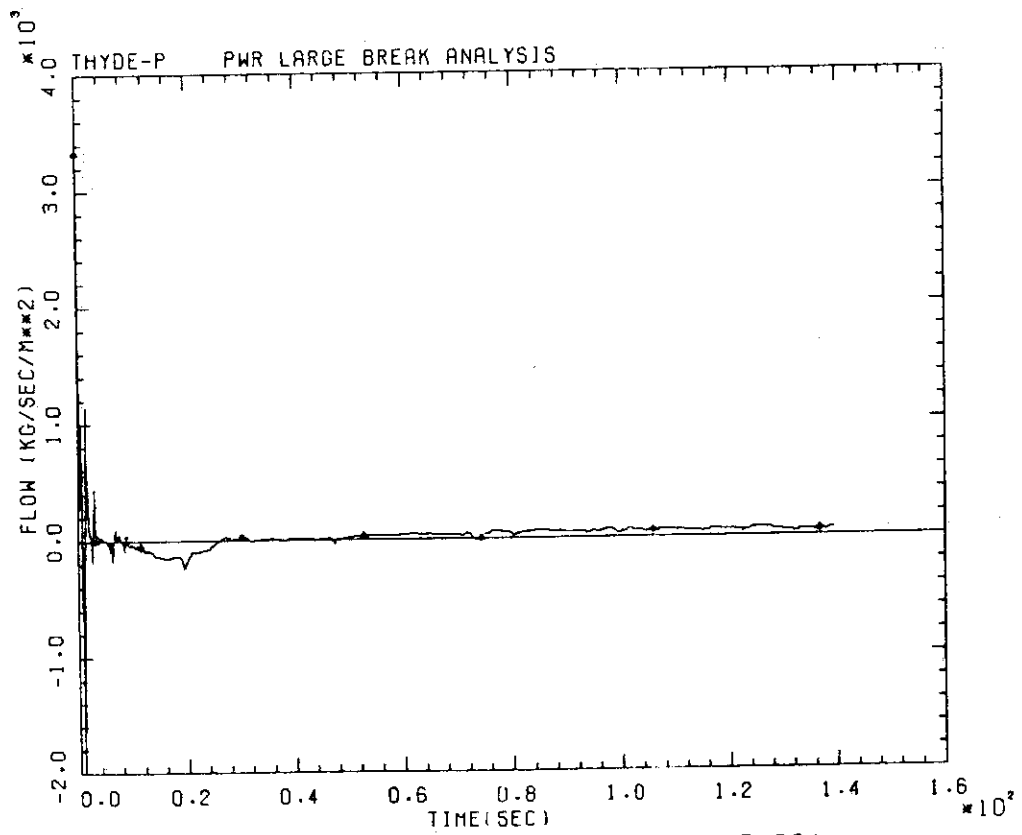


FIG. 4-2-11 CORE OUTLET FLOW (NODE 28)

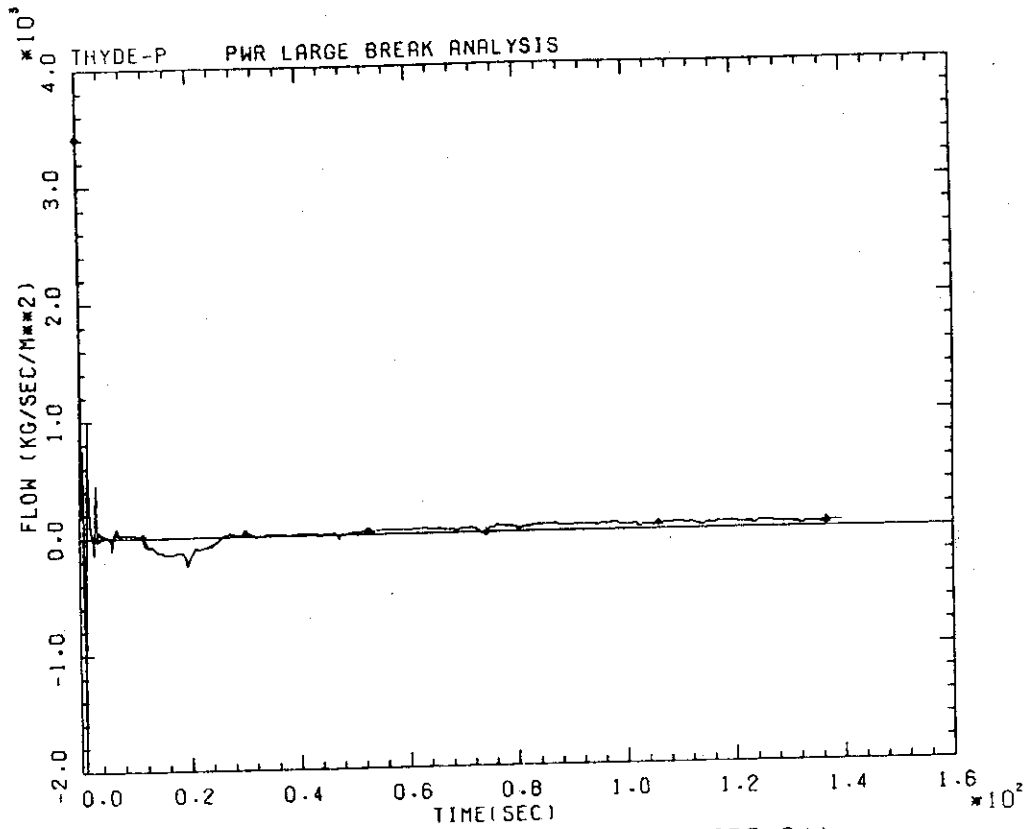


FIG. 4-2-12 CORE OUTLET FLOW (NODE 34)

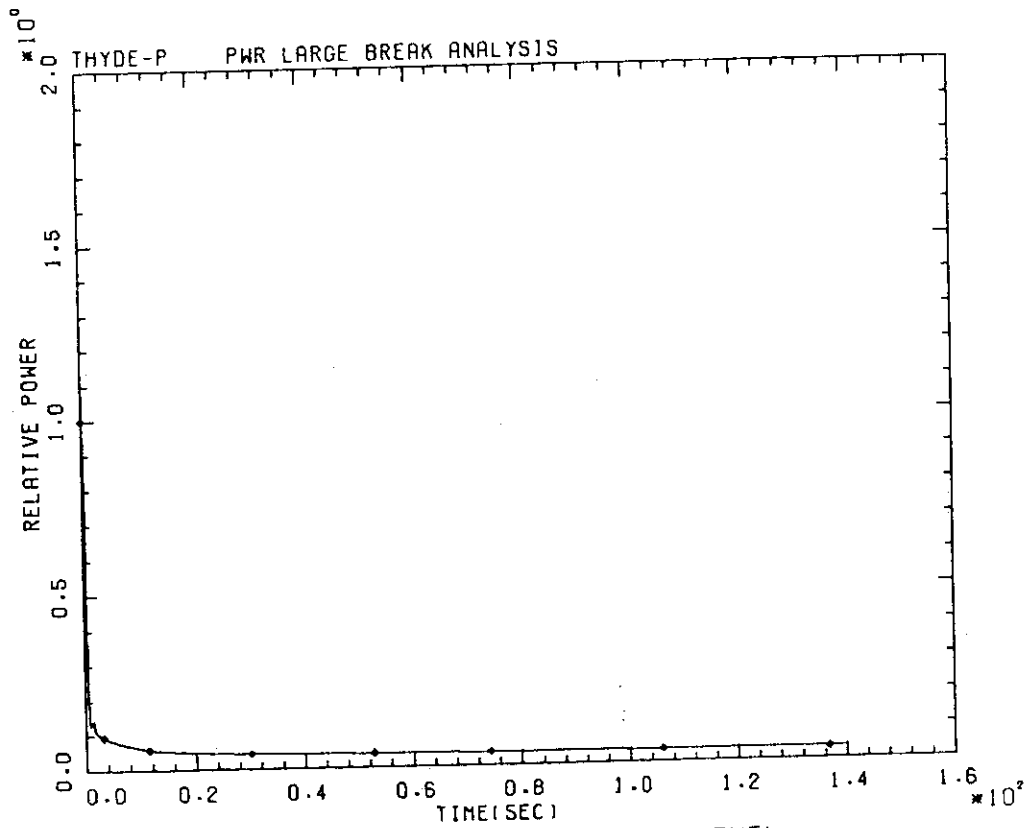


FIG. 4-2-13 HEAT GENERATION IN FUEL

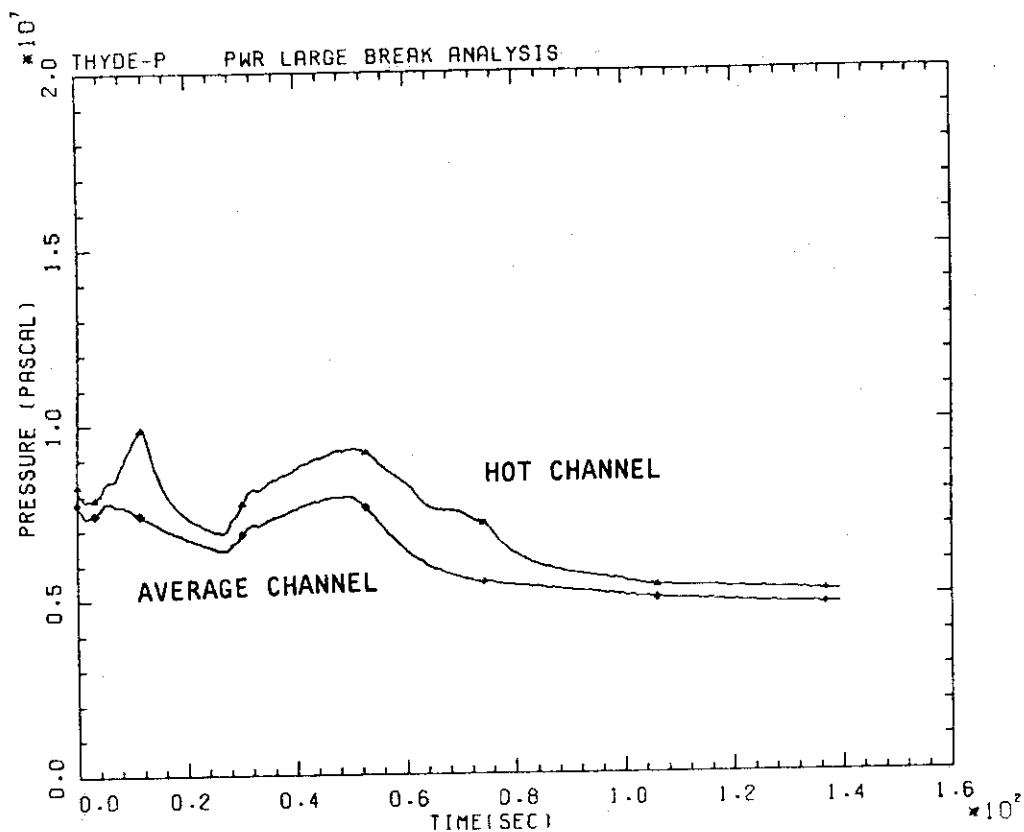


FIG. 4-2-14 GAP PRESSURE

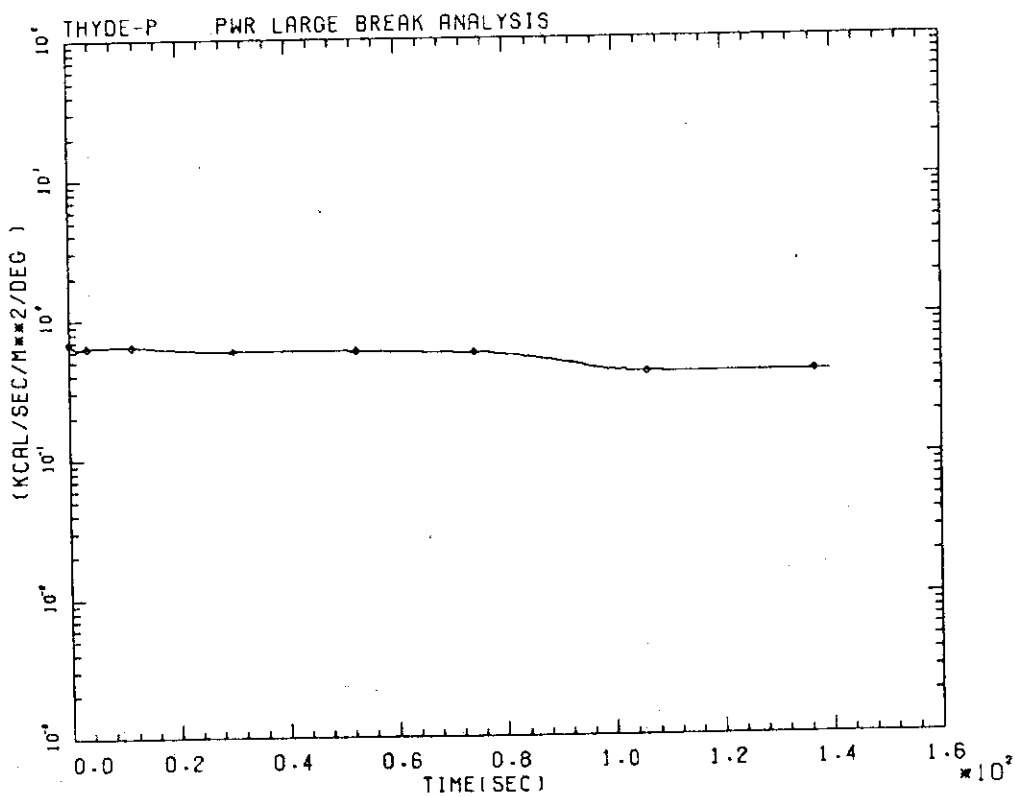


FIG. 4-2-15 H.T.C. AT GAP

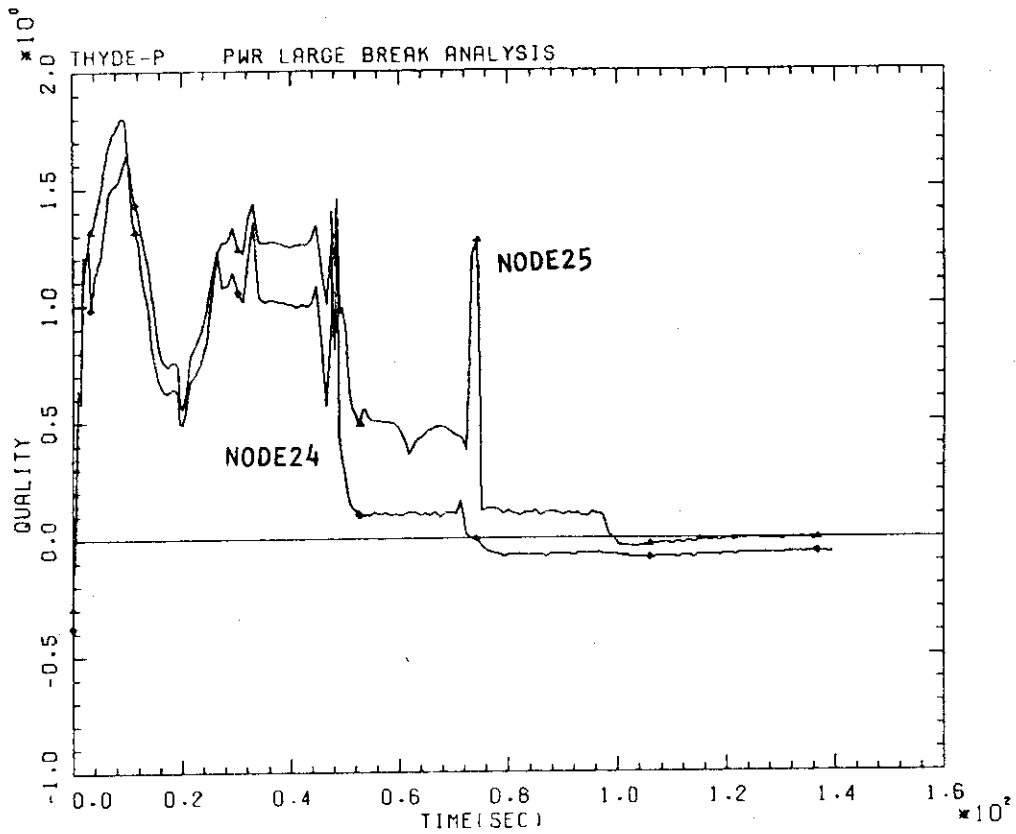


FIG. 4-2-16 QUALITY AT AVERAGE CHANNEL

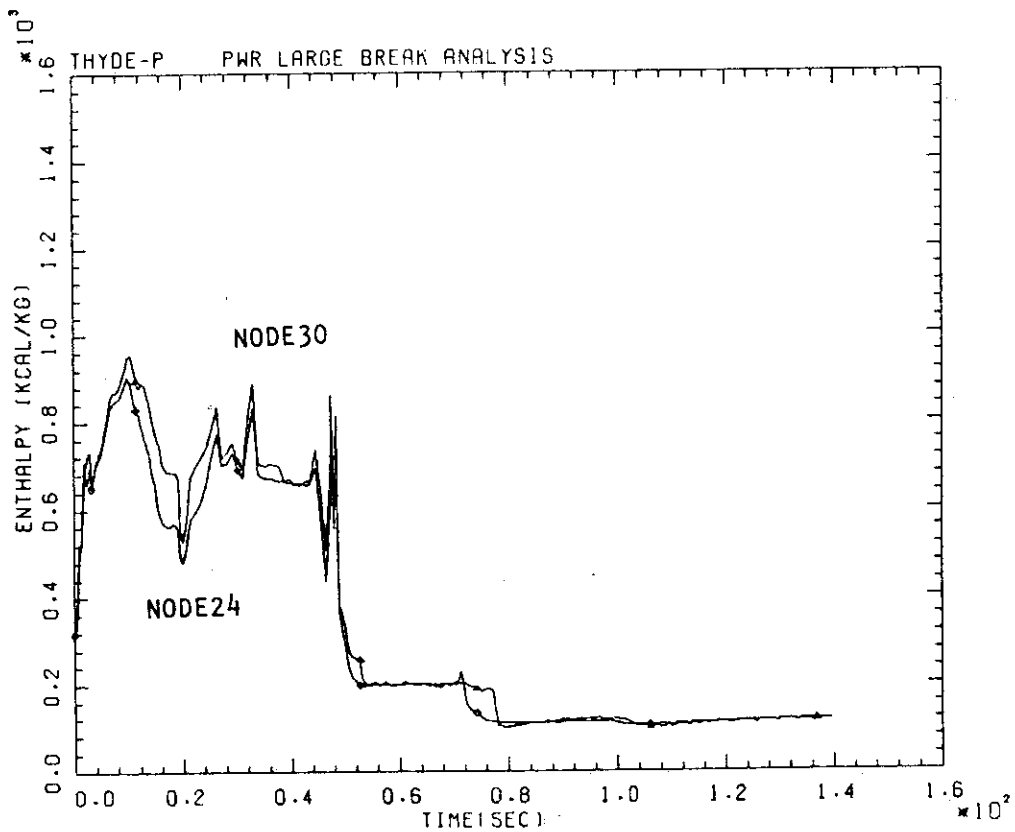


FIG. 4-2-17 ENTHALPY AT CORE INLET

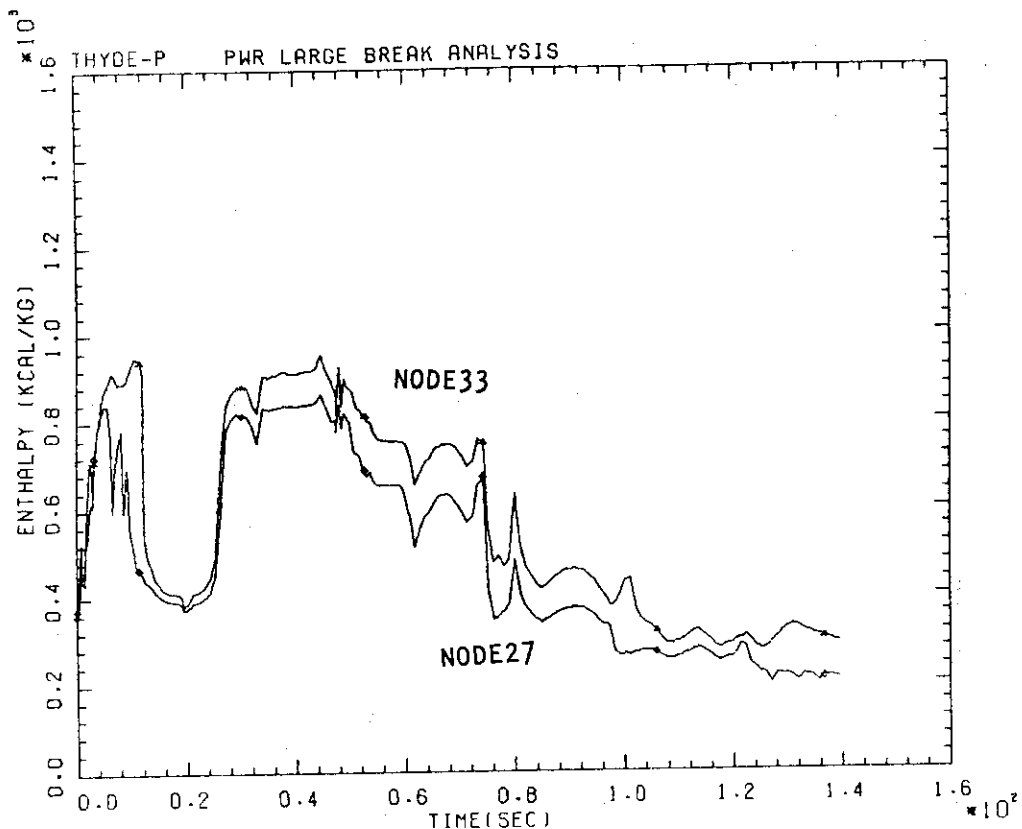


FIG. 4-2-18 ENTHALPY AT CORE OUTLET

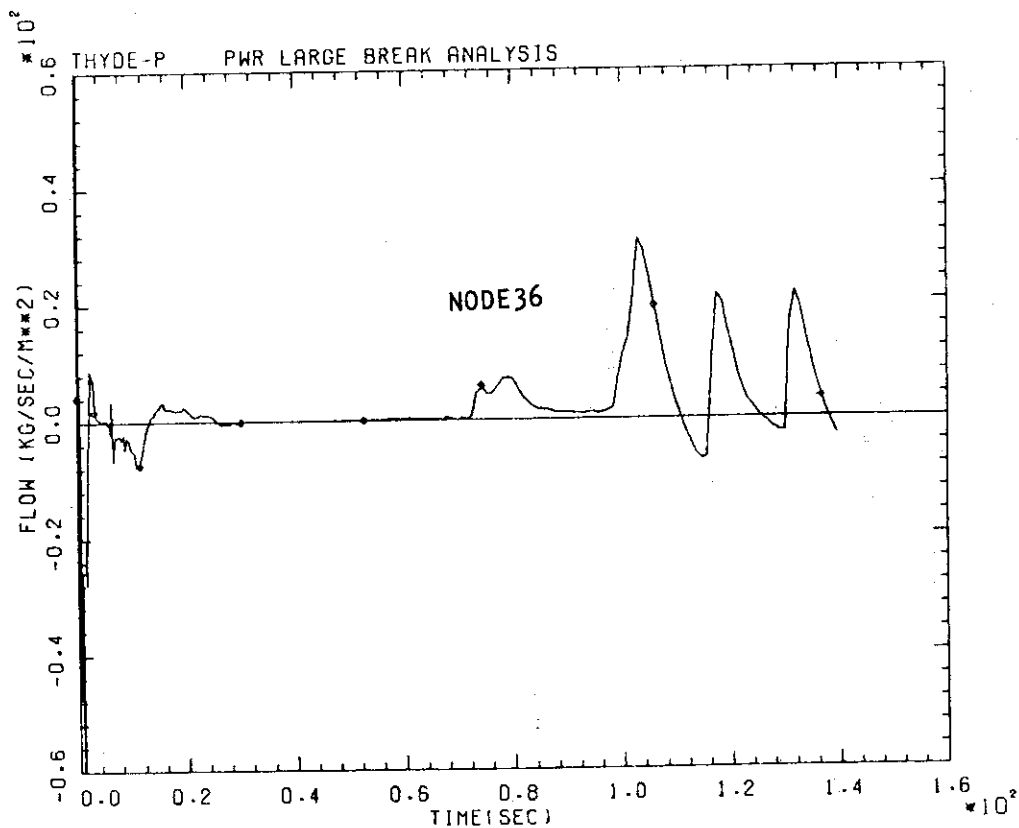


FIG. 4-2-19 CORE CROSS FLOW

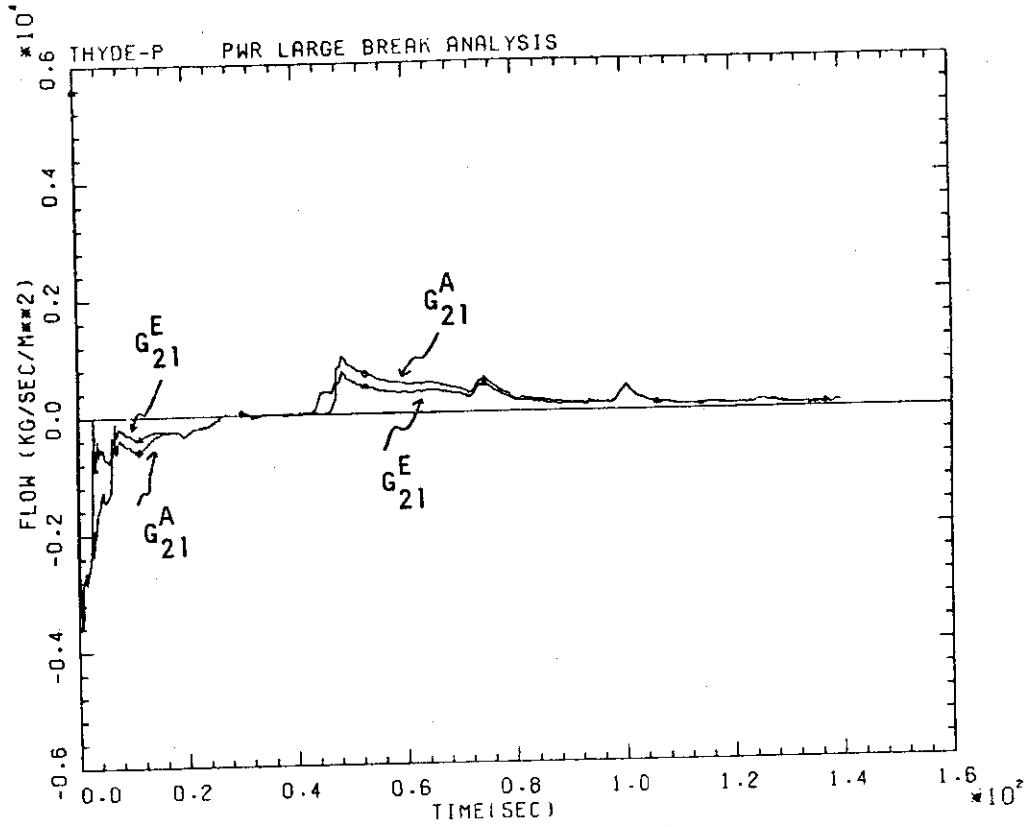


FIG. 4-3-1 FLOW IN DOWNCOMER

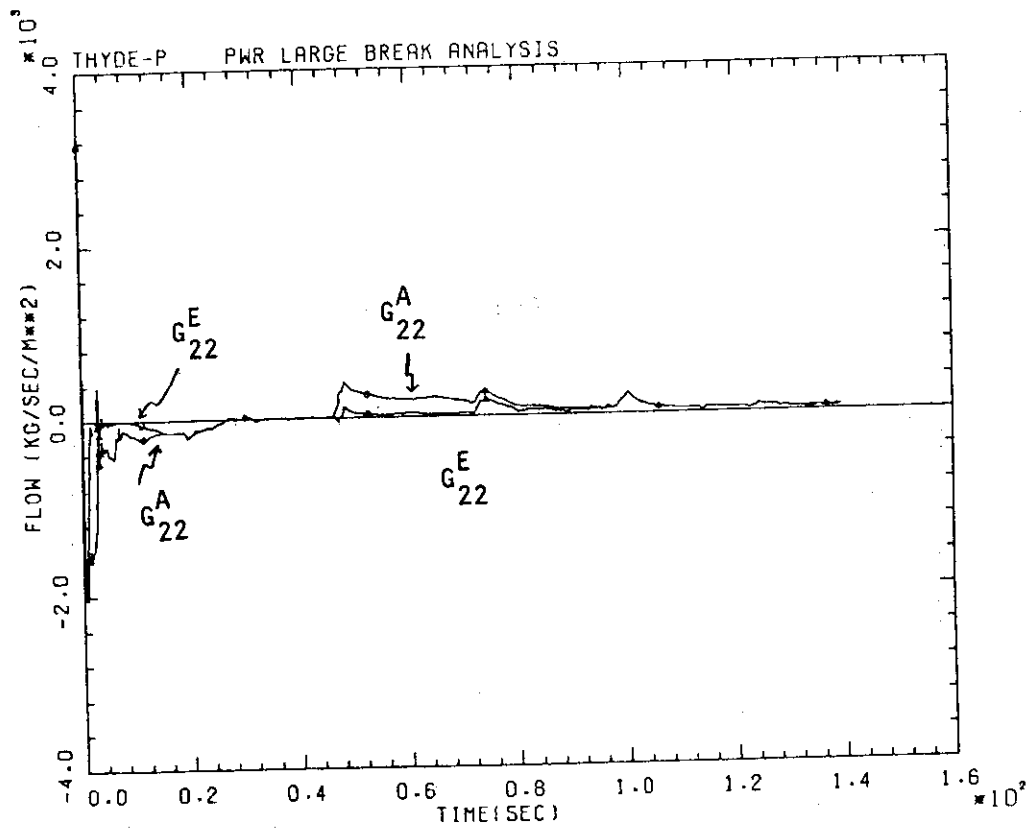


FIG. 4-3-2 FLOW IN LOWER PLENUM



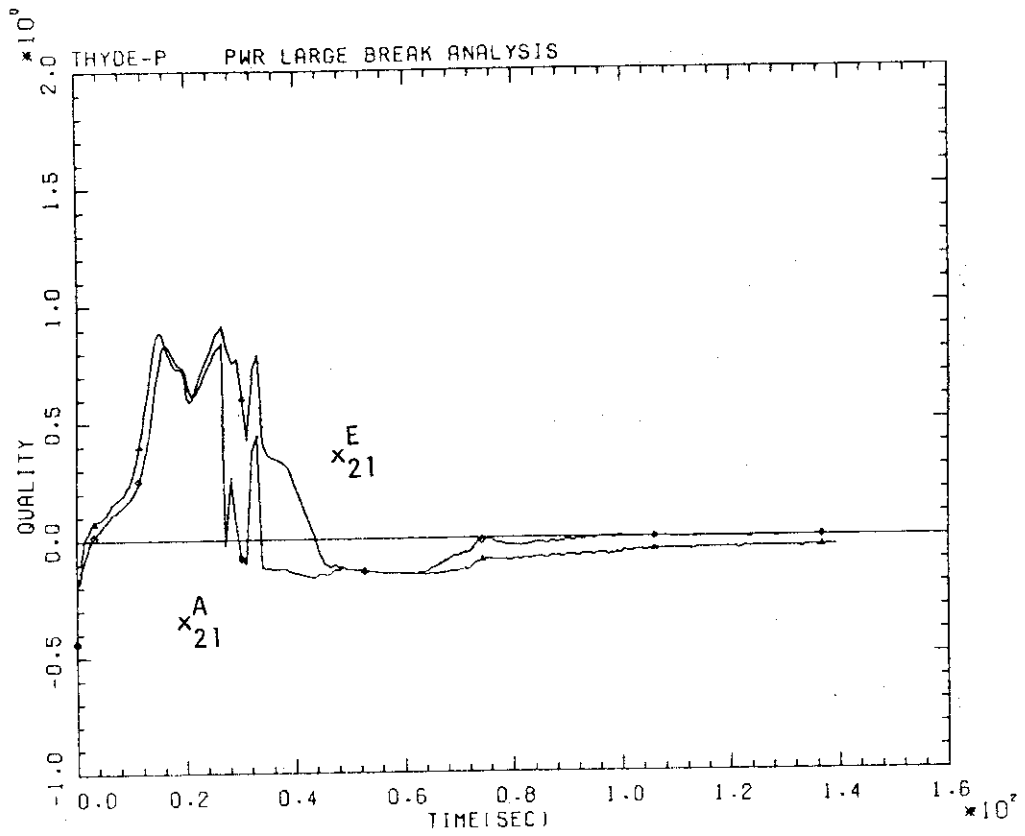


FIG. 4-3-3 QUALITY IN DOWNCOMER

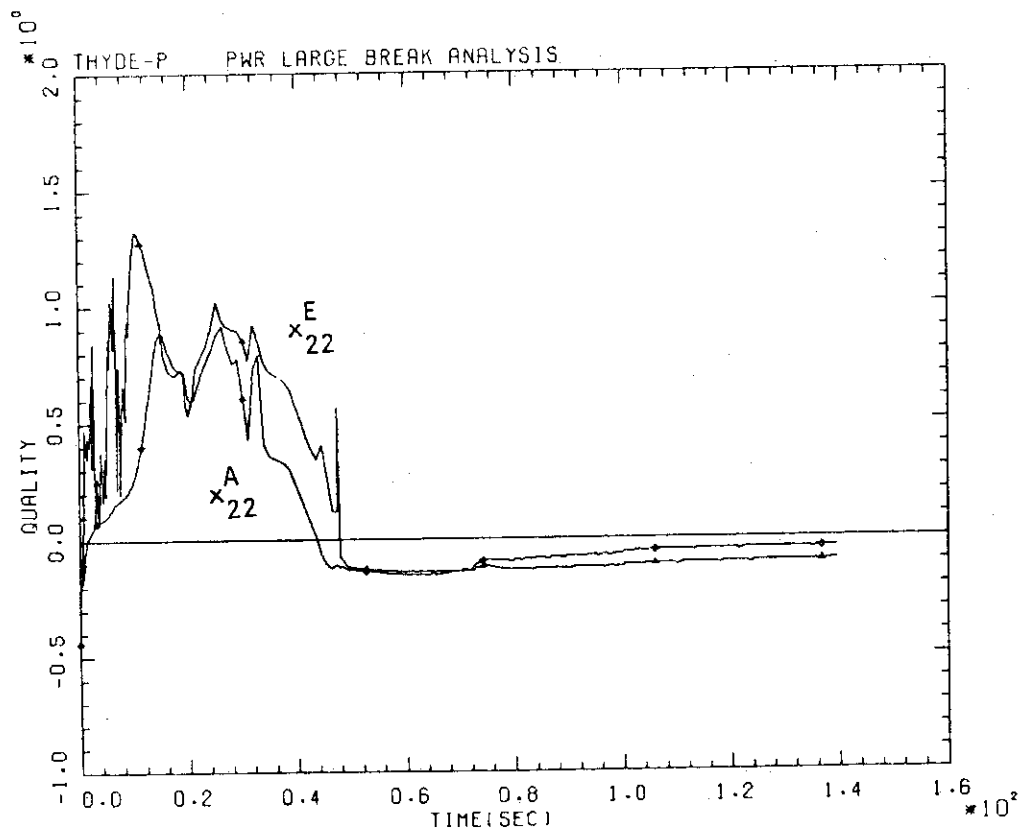


FIG. 4-3-4 QUALITY IN LOWER PLENUM

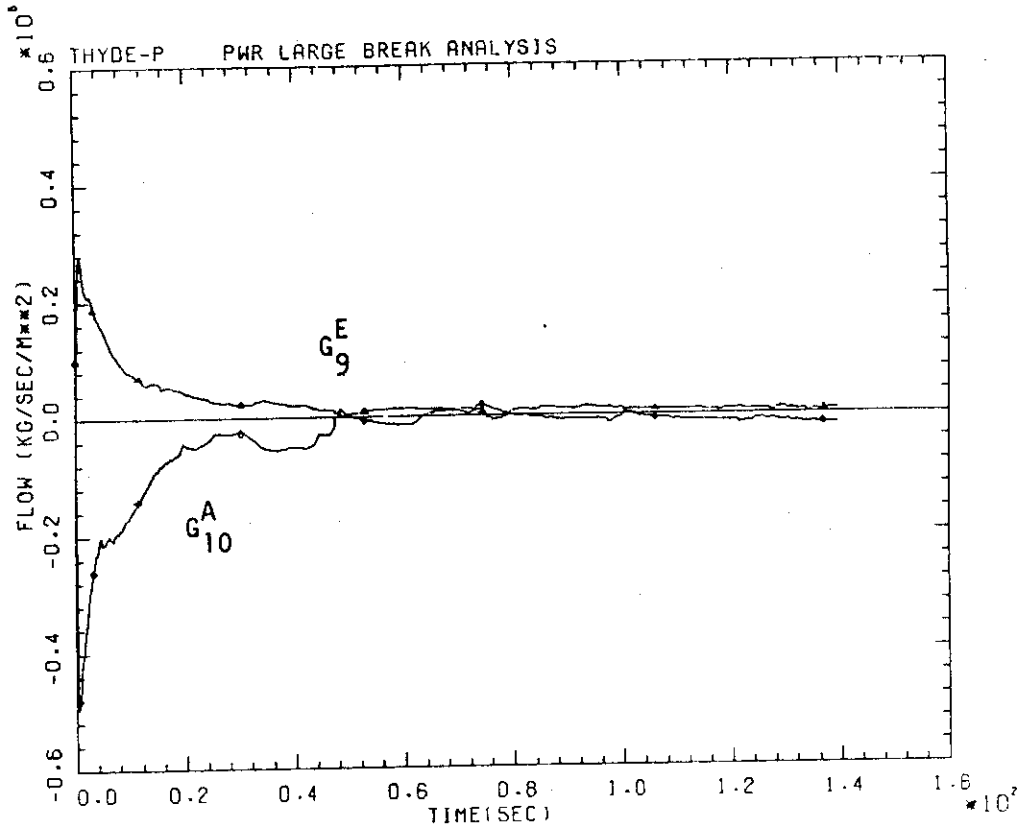


FIG. 4-4-1 BREAK FLOW

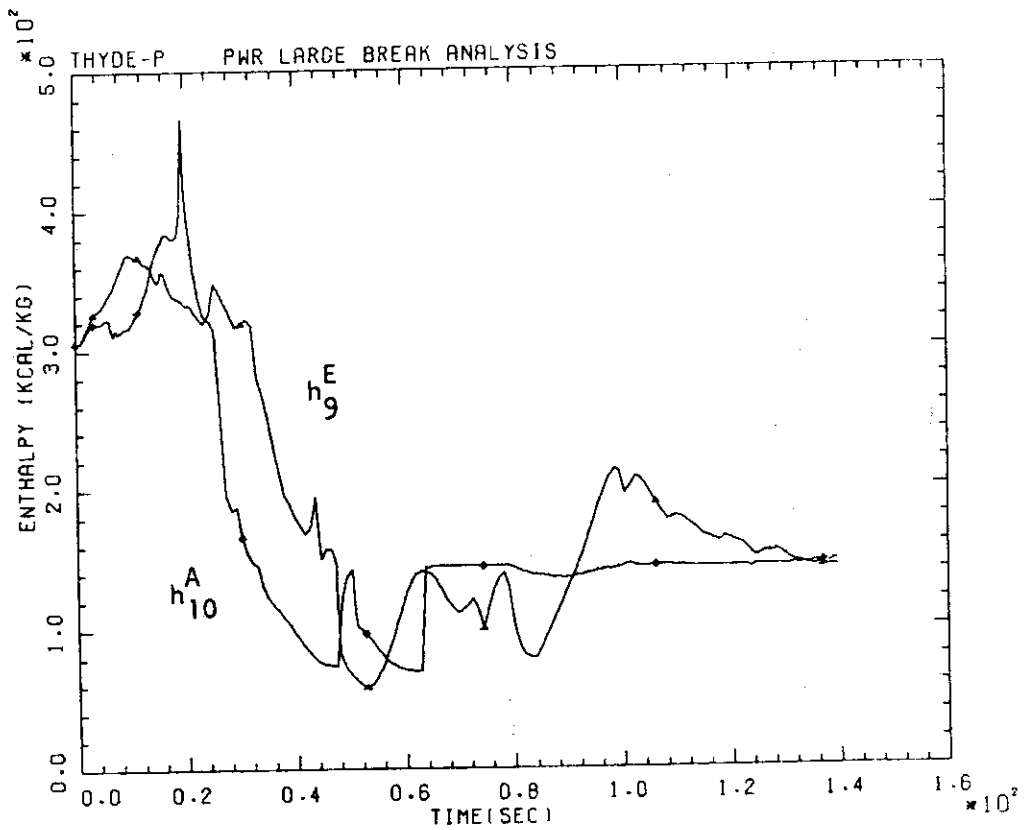


FIG. 4-4-2 ENTHALPY AT BREAK POINT

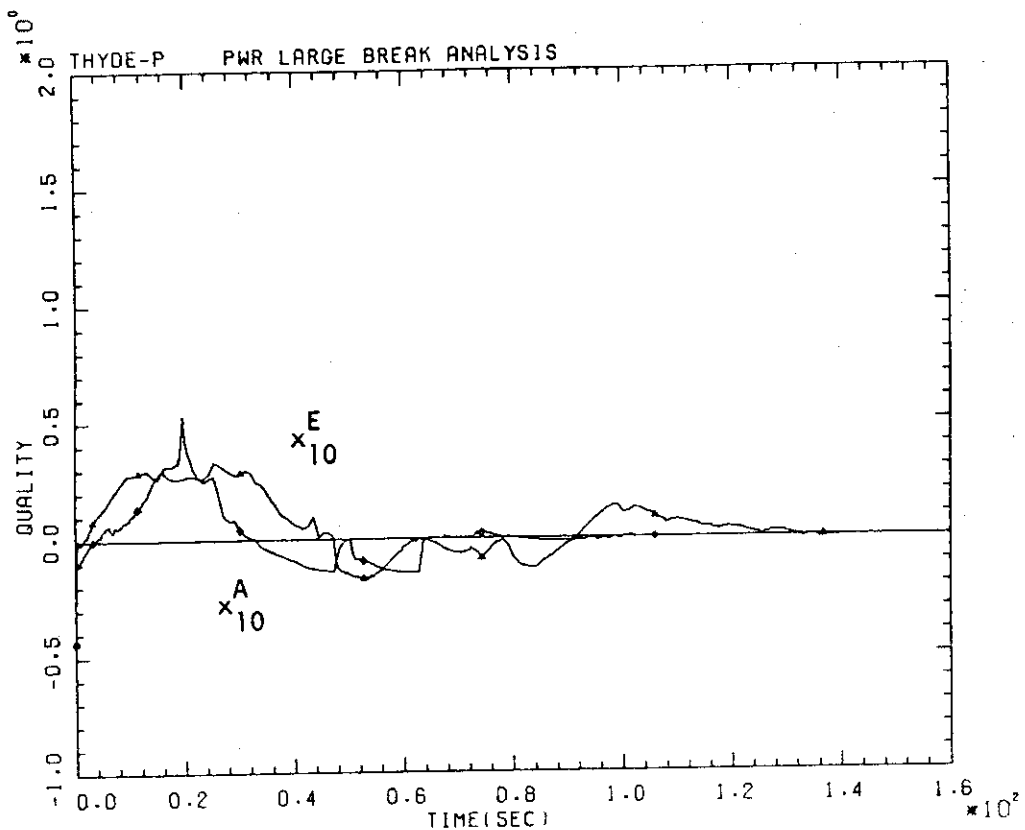


FIG. 4-4-3 QUALITY AT BREAK POINT

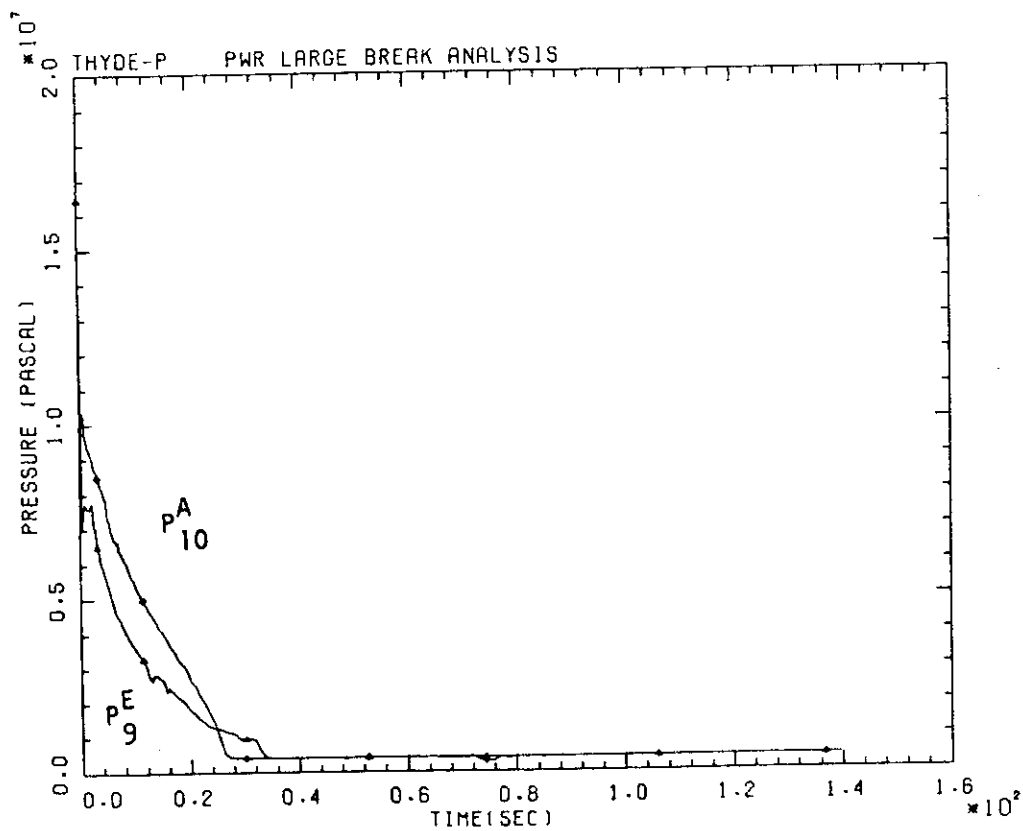


FIG. 4-4-4 PRESSURE AT BREAK POINT

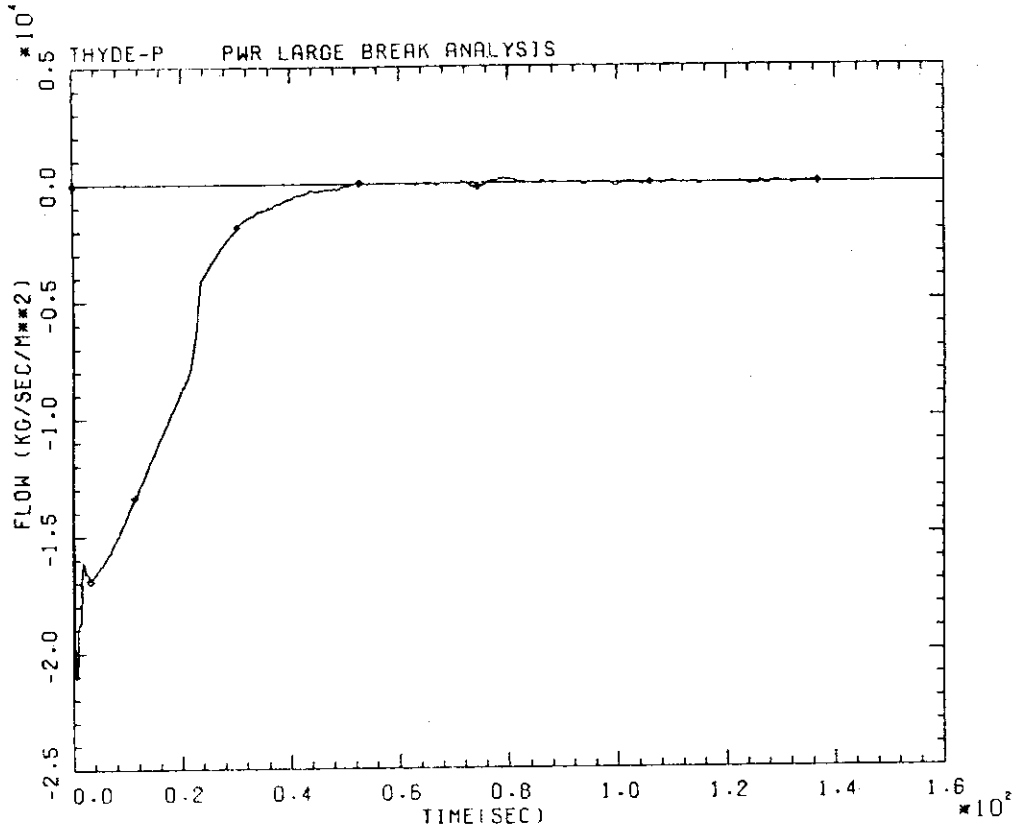


FIG. 4-5-1 PRESSURIZER SURGE FLOW

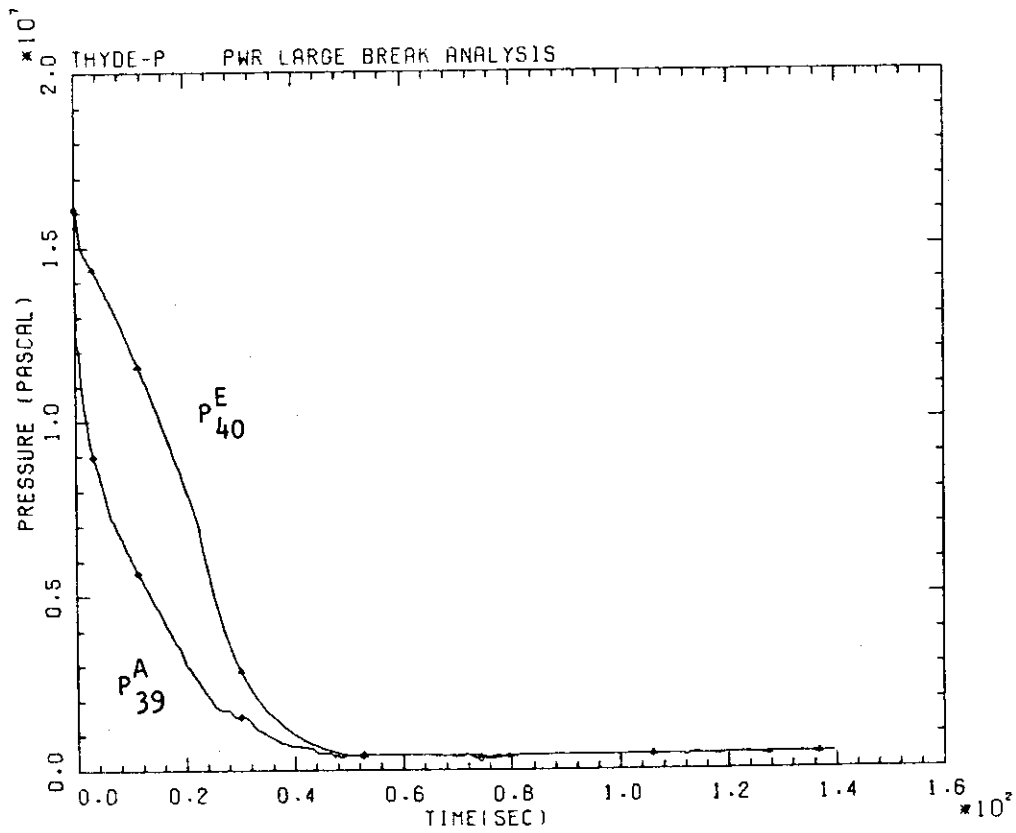


FIG. 4-5-2 PRESSURE IN PRZ. DUCT

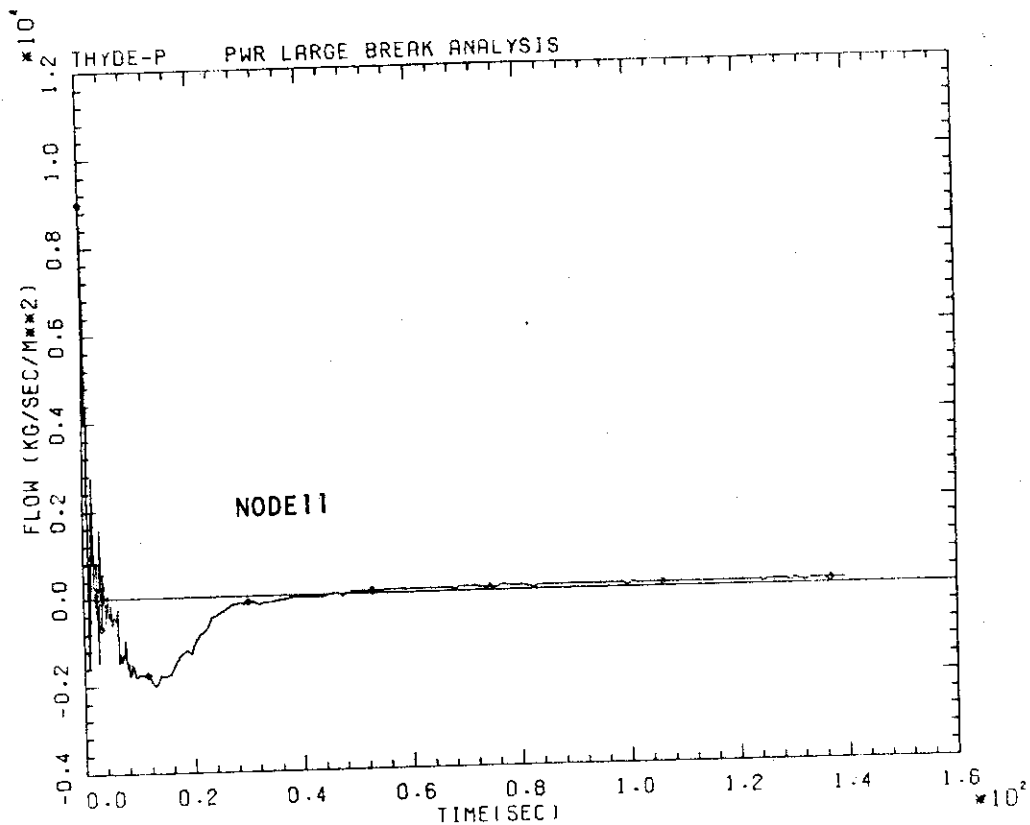


FIG. 4-5-3 FLOW IN INTACT LOOP HOT REG

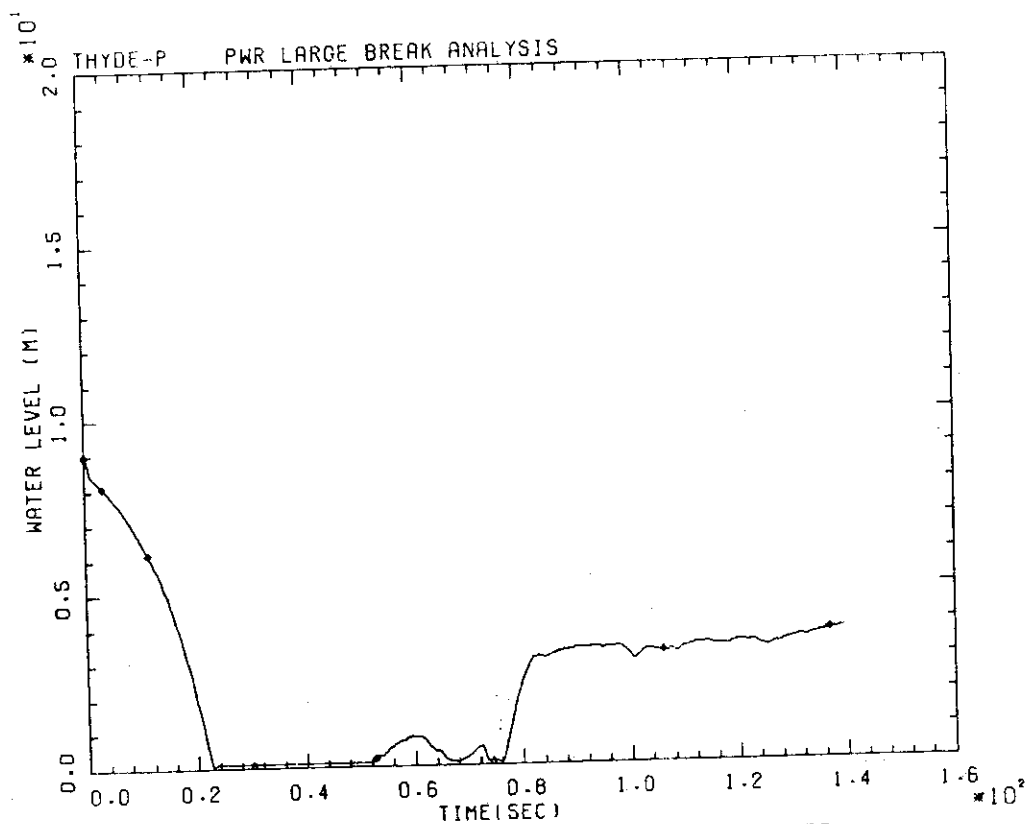


FIG. 4-5-4 WATER LEVEL IN PRESSURIZER

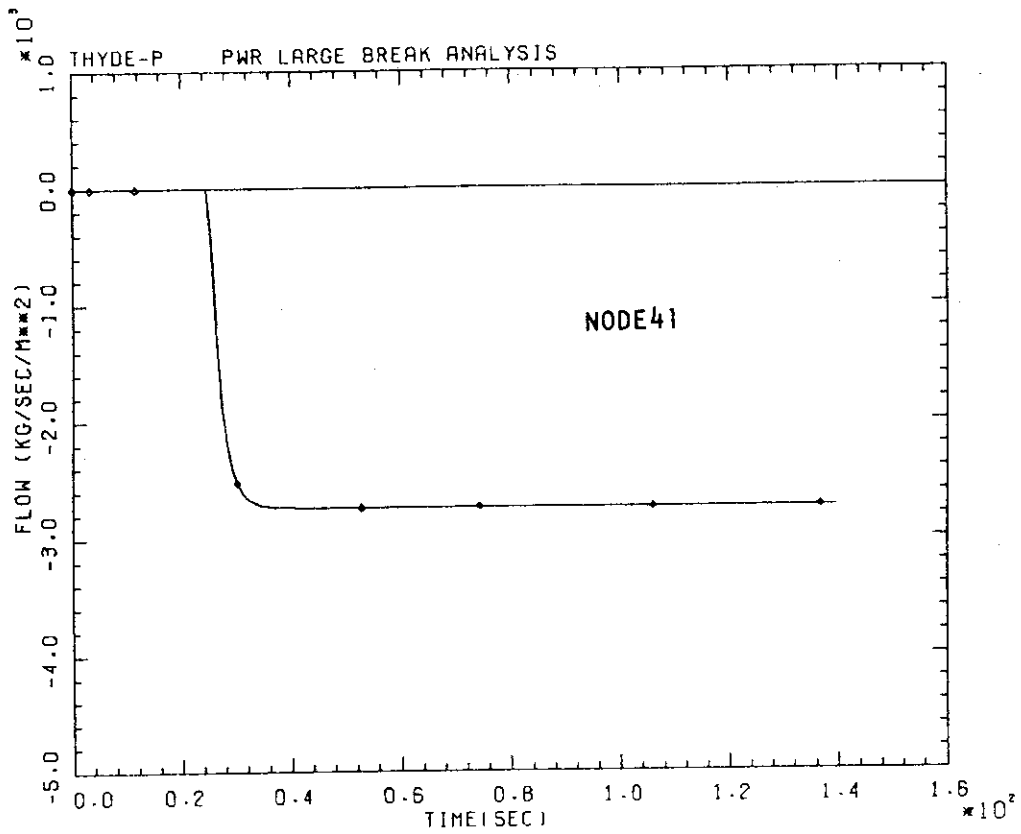


FIG. 4-6-1 FLOW IN P.I. DUCT

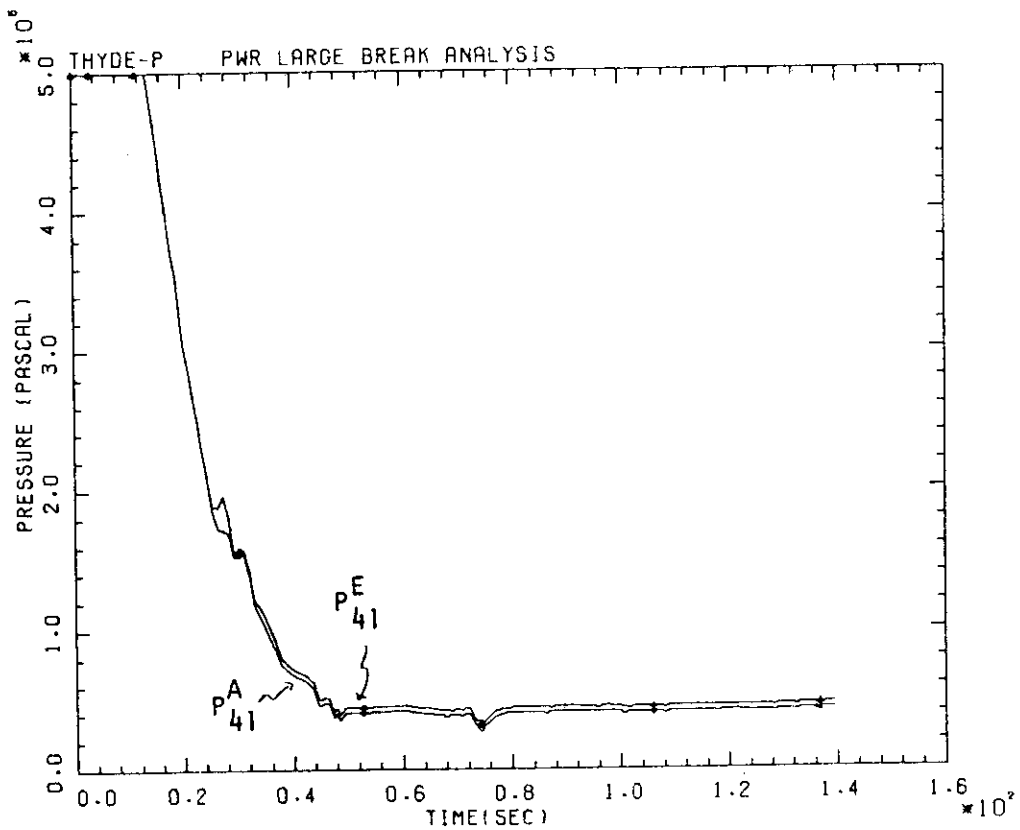


FIG. 4-6-2 PRESSURE IN P.I. DUCT

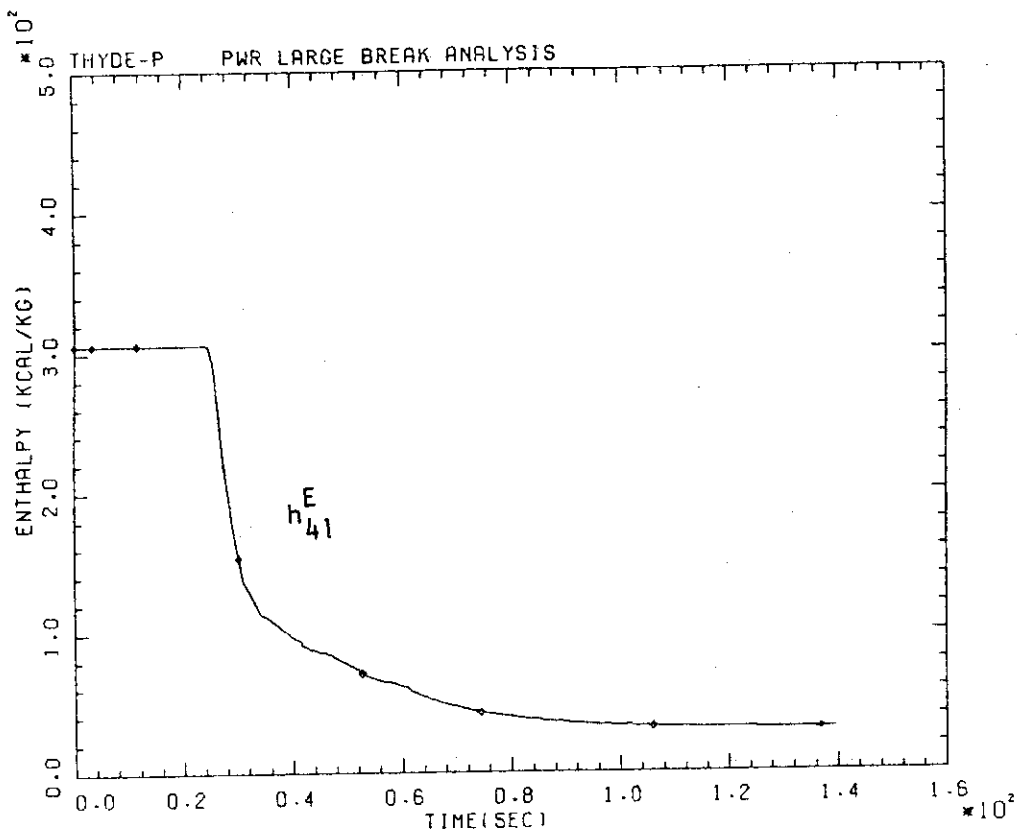


FIG. 4-6-3 ENTHALPY IN P.I. DUCT

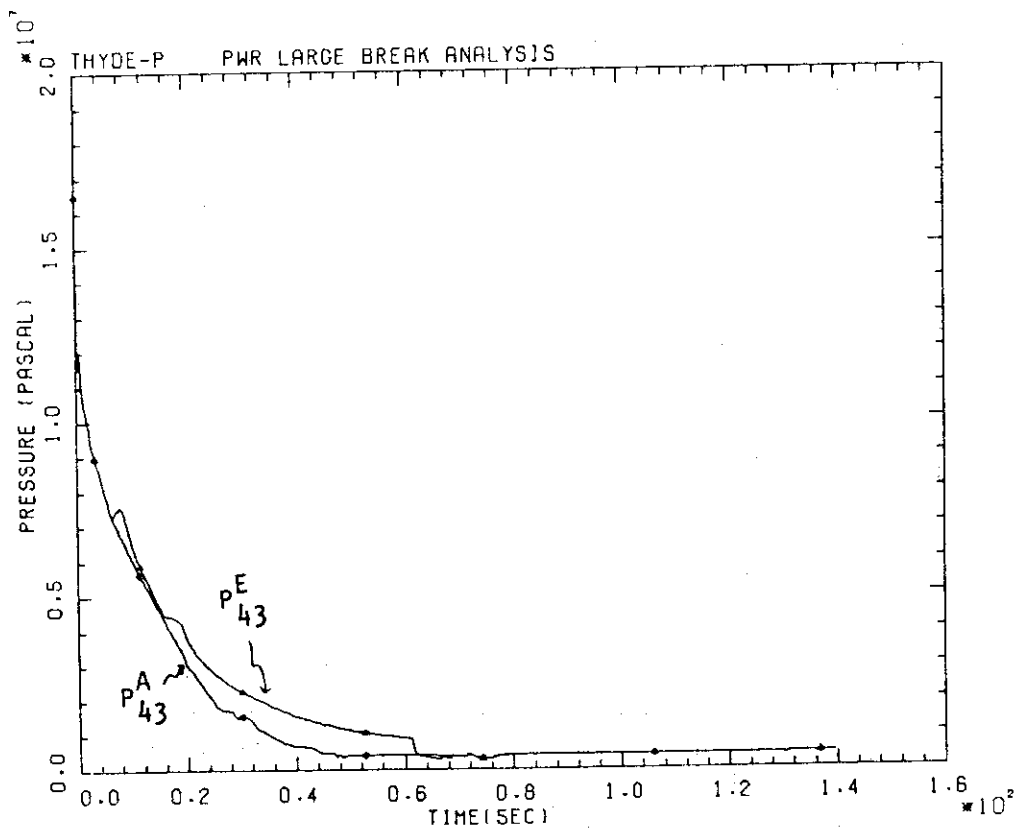


FIG. 4-7-1 PRESSURE IN A.I. DUCT

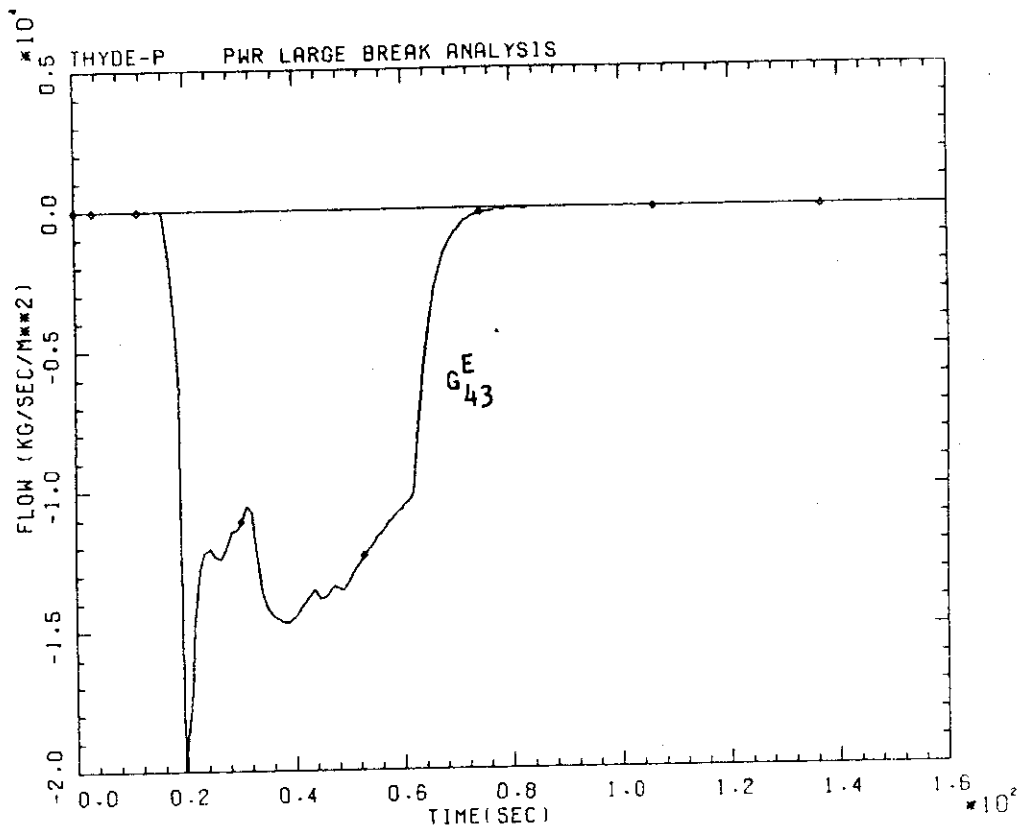


FIG. 4-7-2 FLOW IN A.I. DUCT

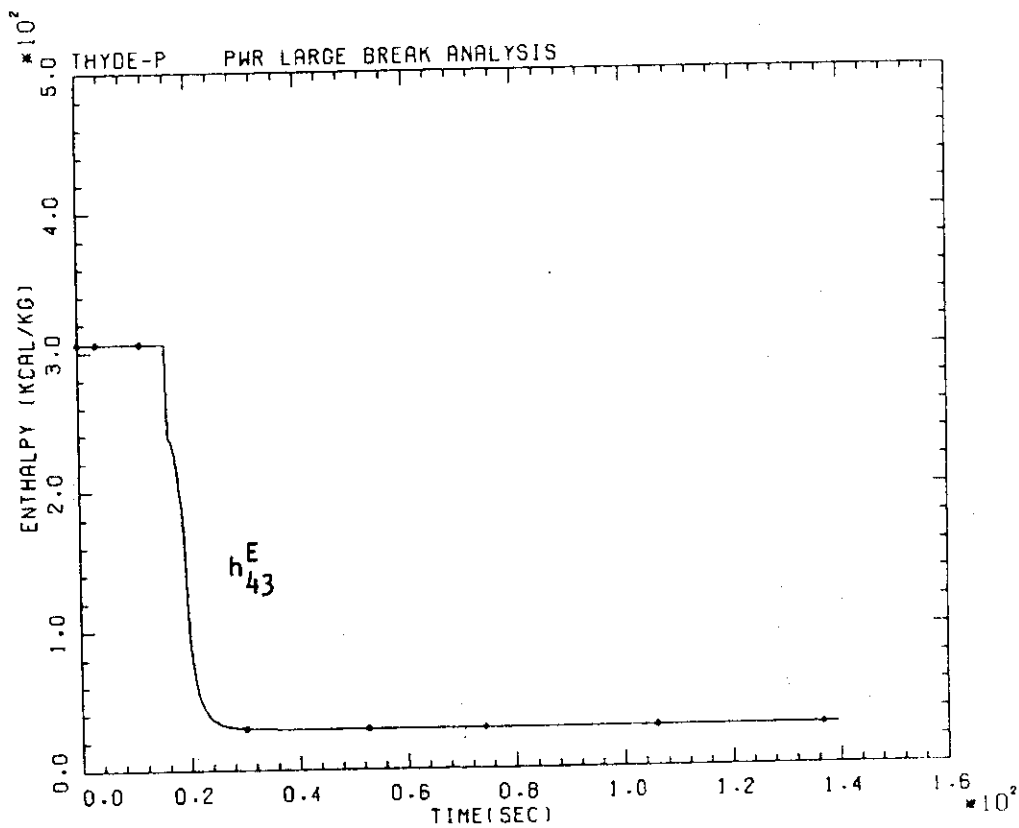


FIG. 4-7-3 ENTHALPY IN A.I. DUCT



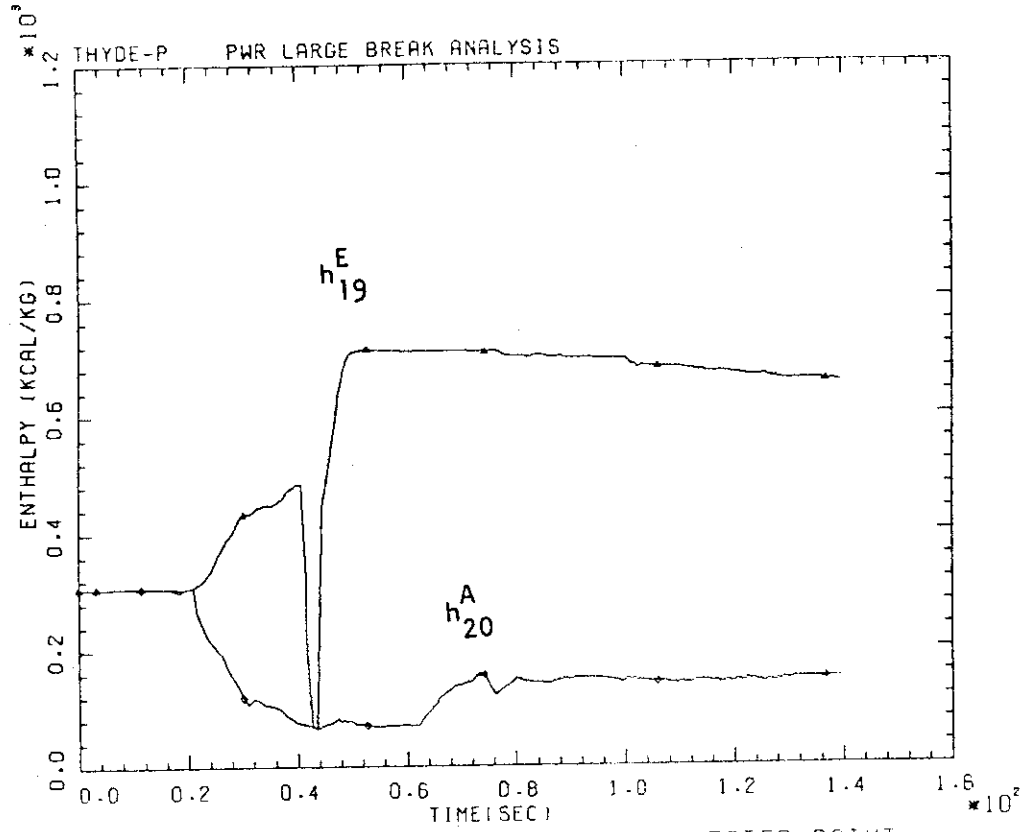


FIG. 4-7-4 ENTHALPY IN ECC INJECTED POINT

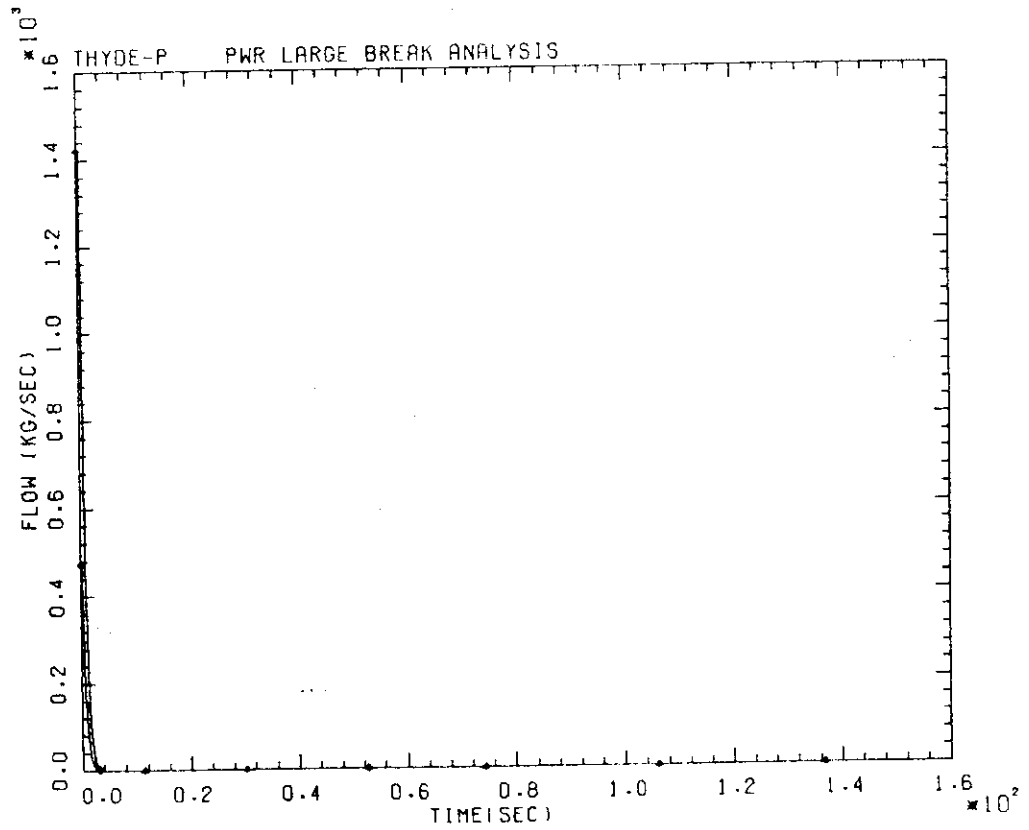


FIG. 4-8-1 S.G. FEED WATER

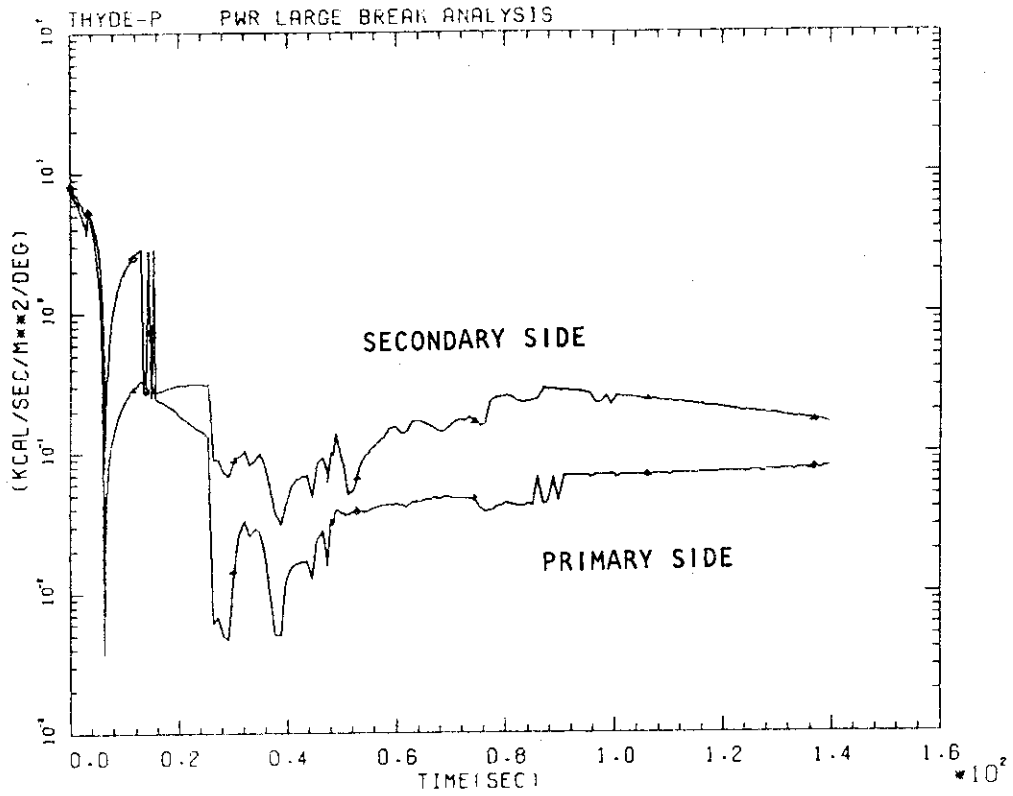


FIG. 4-8-2 H.T.C. IN INTACT LOOP S.G.

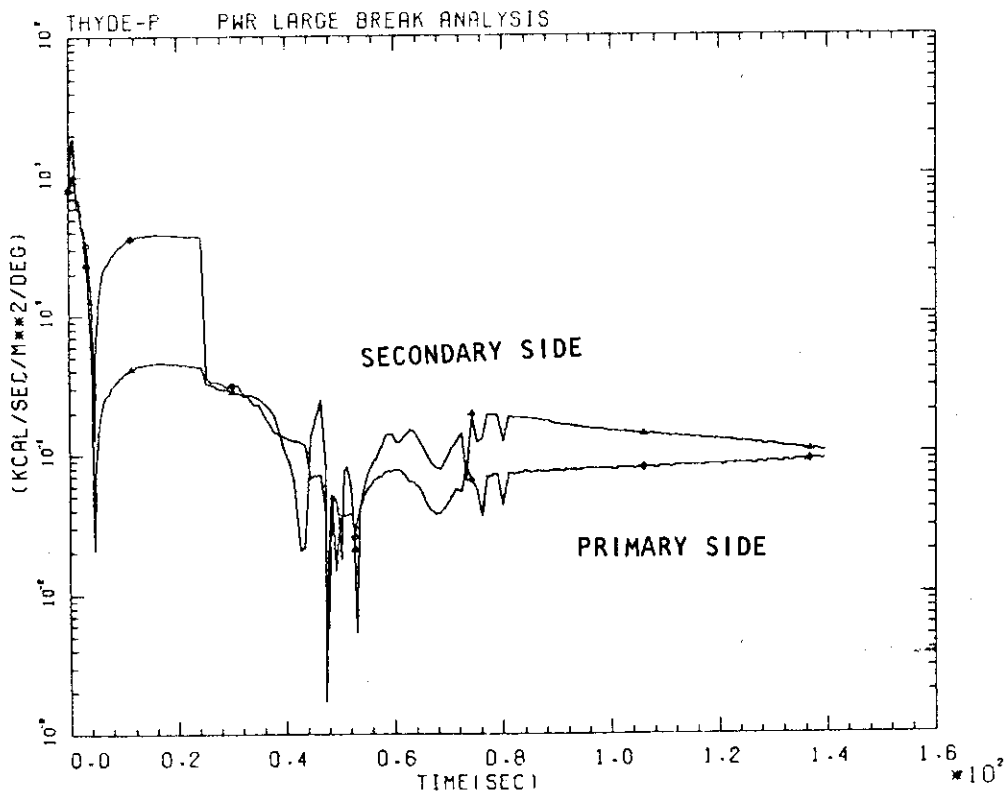


FIG. 4-8-3 H.T.C. IN BROKEN LOOP S.G.

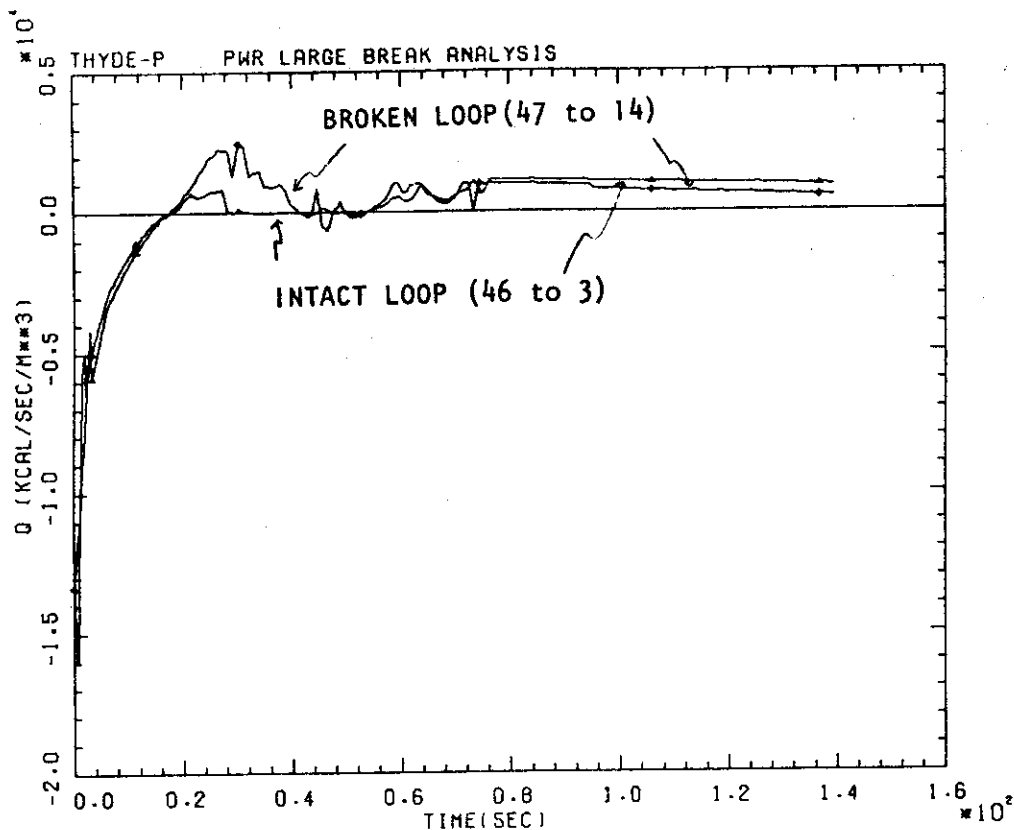


FIG. 4-8-4 HEAT FLOW IN S.G.

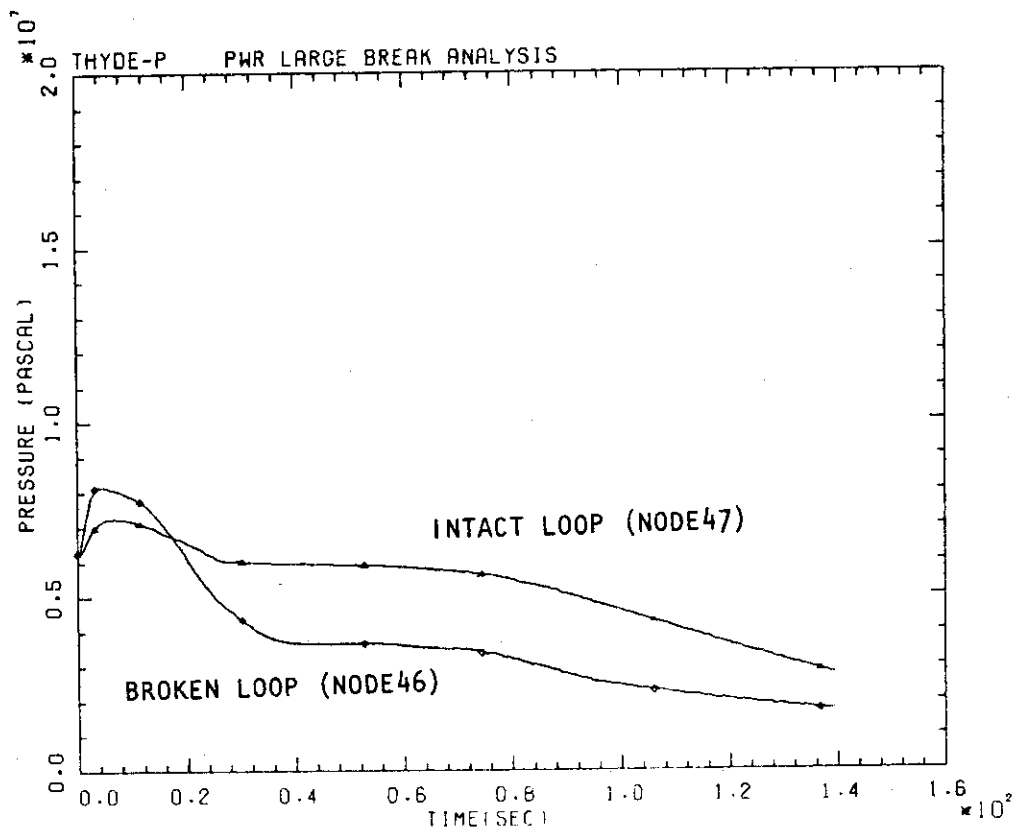


FIG. 4-8-5 PRESSURE IN S.G. SECONDARY

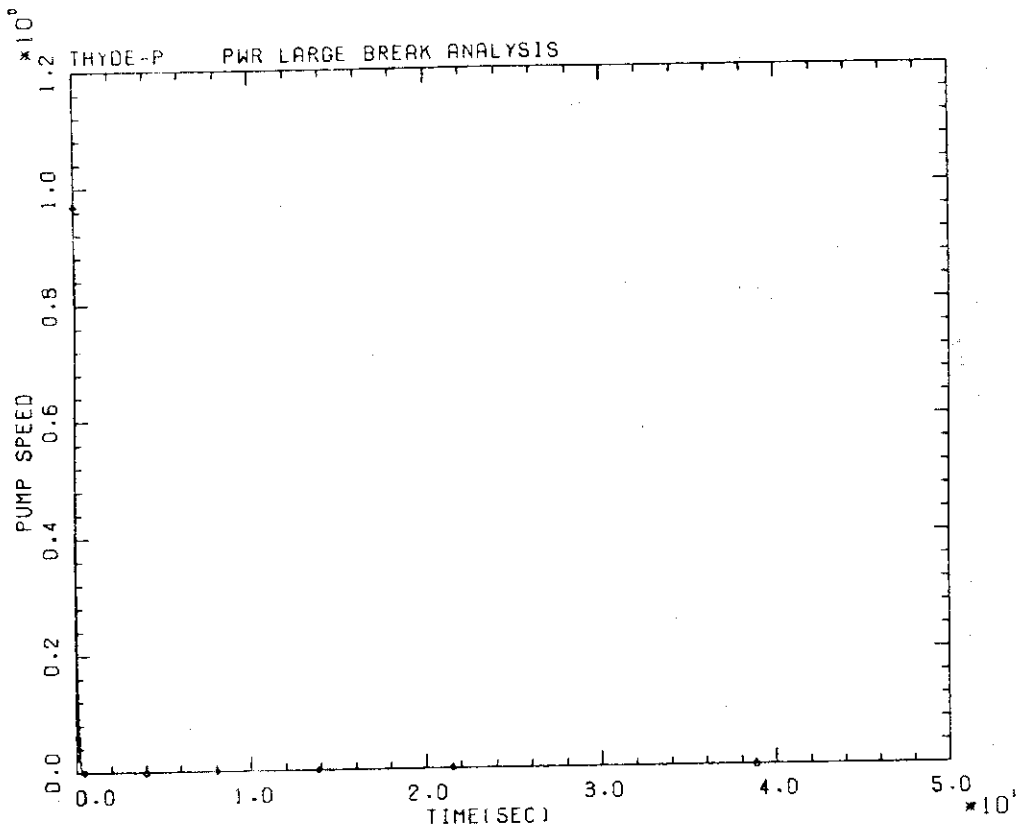


FIG. 4-9-1 PUMP SPEED

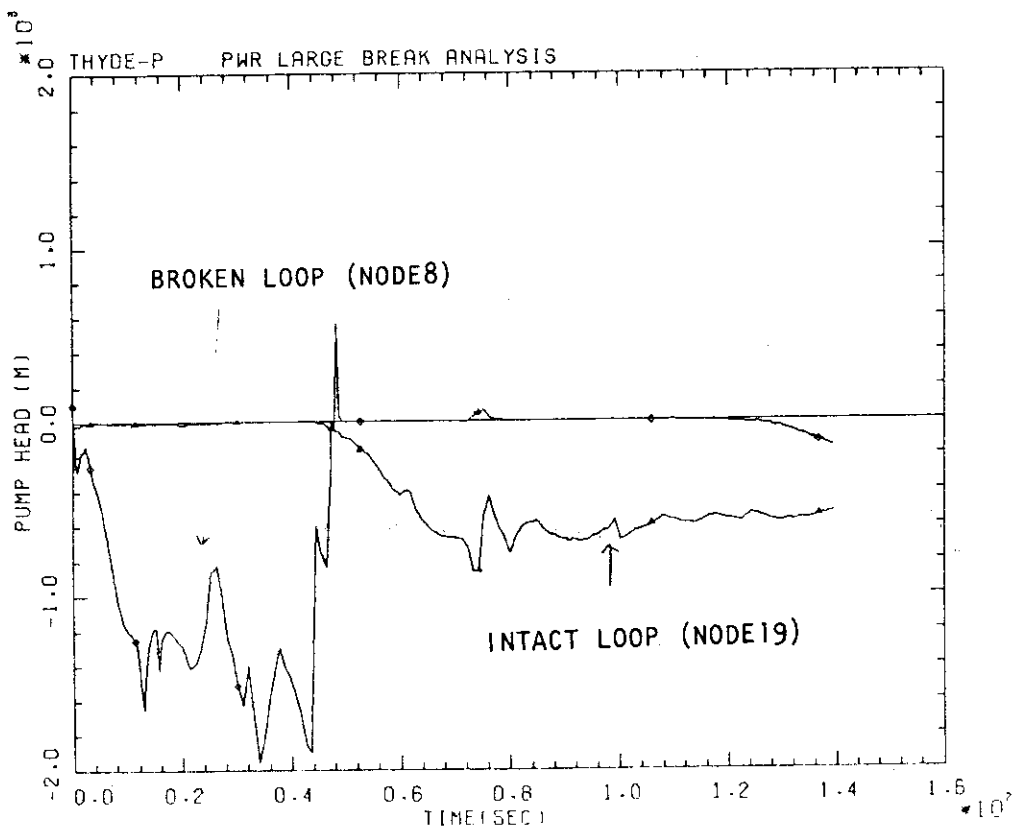


FIG. 4-9-2 PUMP HEAD

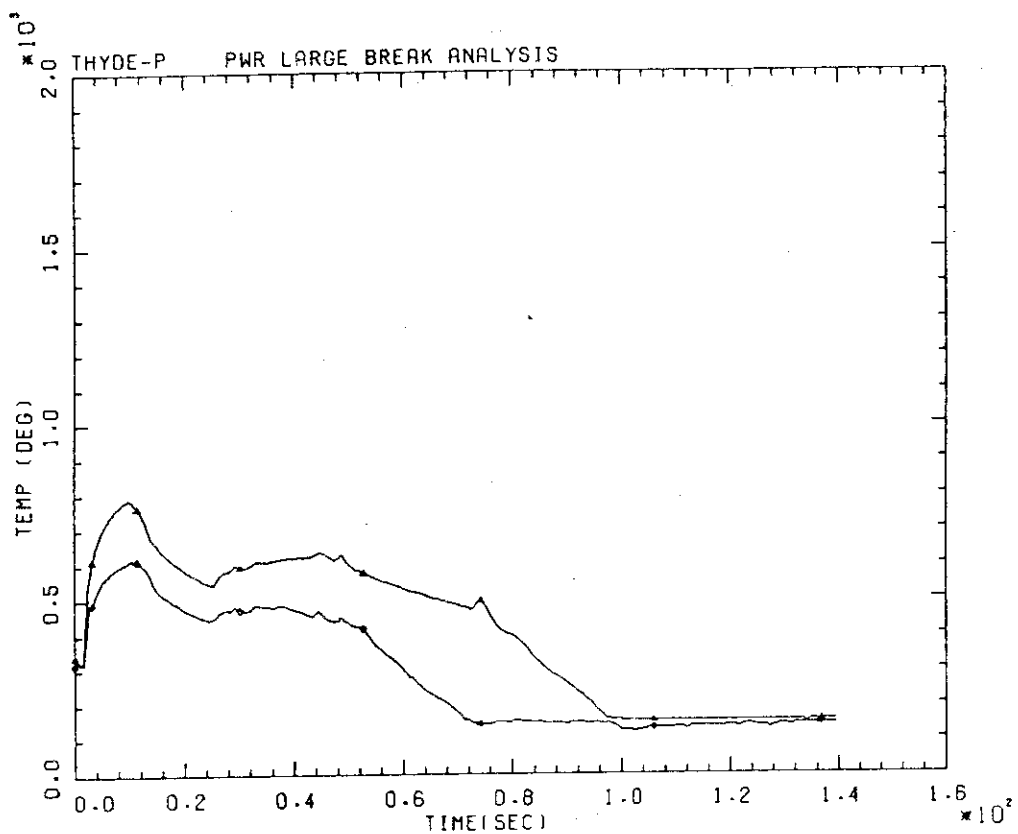


FIG. 4-10-1 AVERAGE CHANNEL SURFACE TEMP.

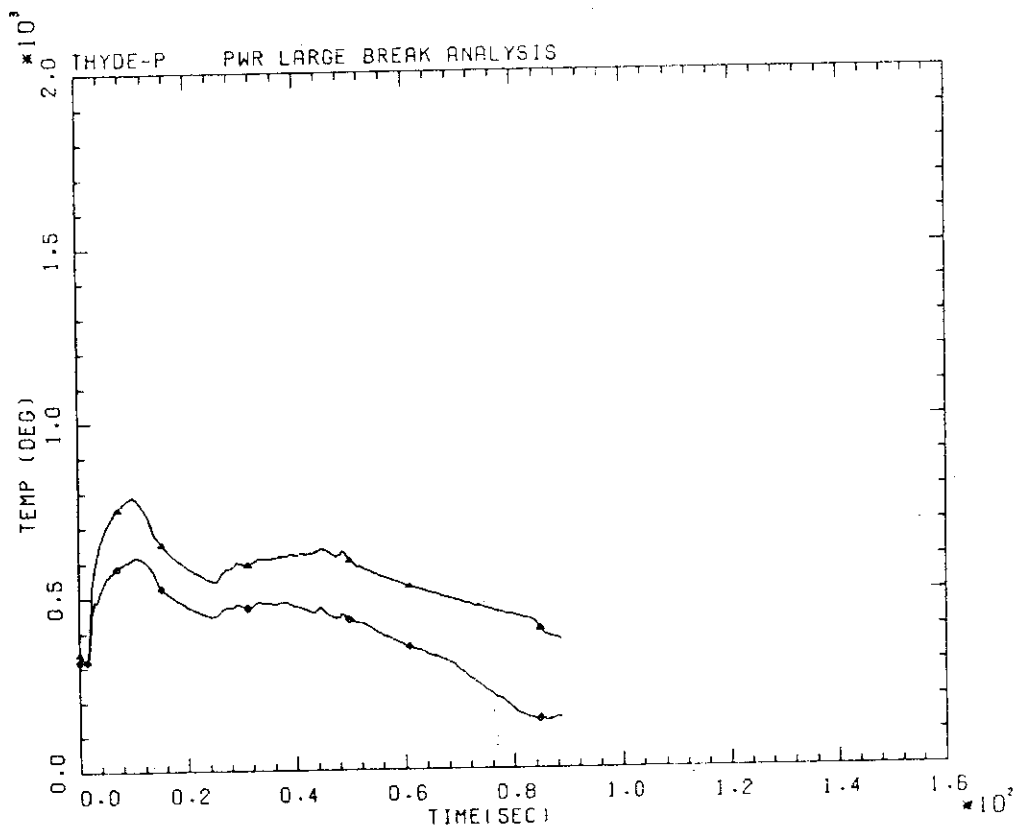


FIG. 4-10-2 AVERAGE CHANNEL SURFACE TEMP.

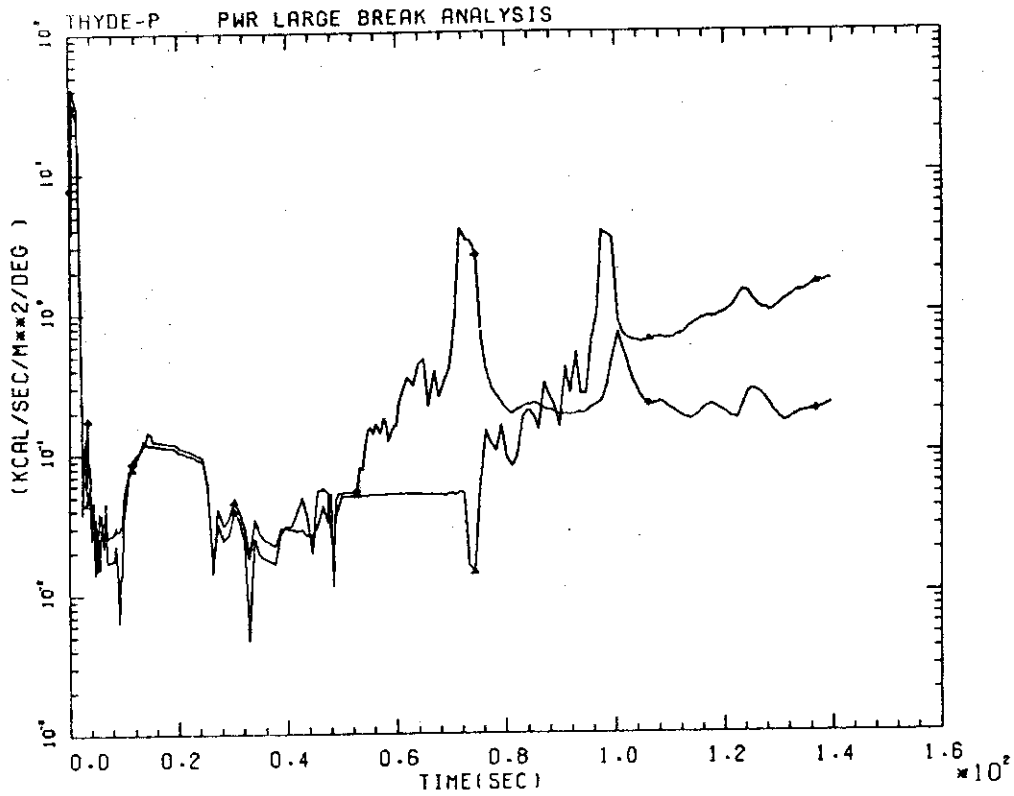


FIG. 4-10-3 H.T.C. IN AVERAGE CHANNEL

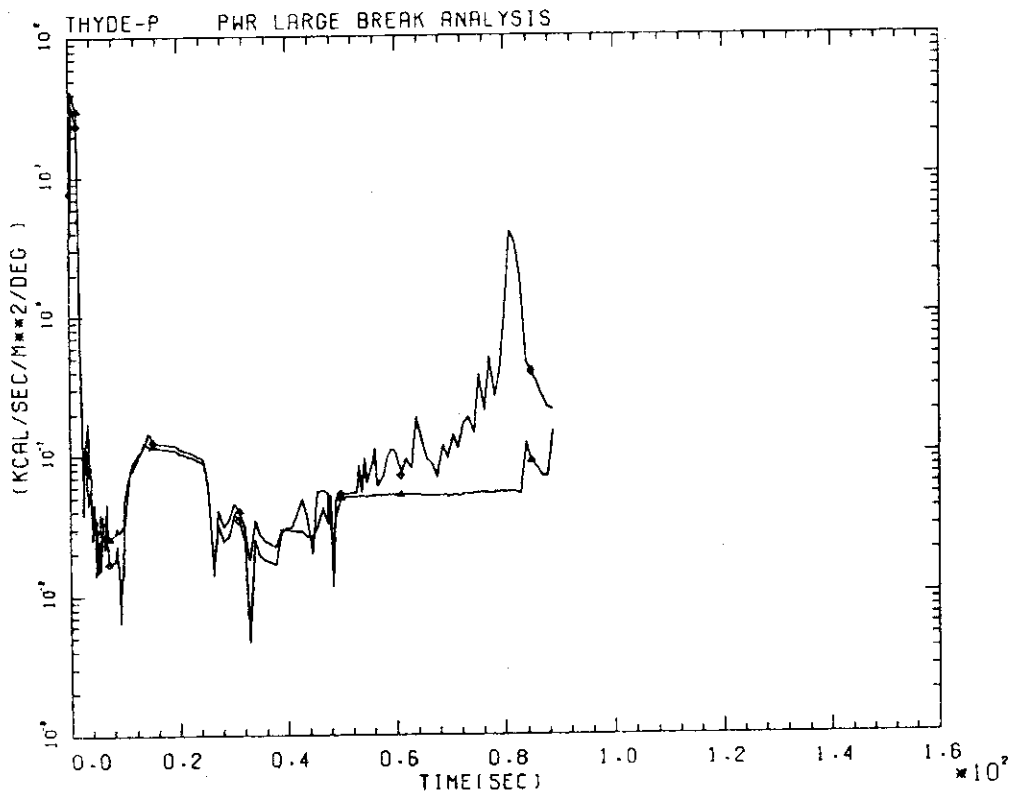


FIG. 4-10-4 H.T.C. IN AVERAGE CHANNEL

## 5. Concluding Remarks

This work was a through LOCA calculation for a commercial 1,100 MWe class PWR plant from the occurrence of a double-ended break till complete fuel quenching due to core reflood.

Therefore, the calculated chronology of events discussed in section 4 is of great interest. The assumptions and conditions for Run 20 are shown in section 3. Their effects on the calculated results should be examined in comparison with other sample calculations by THYDE-P under different assumptions.

The discharge coefficient for critical flow at break points and the delay parameters for non-equilibrium effects have a big influence on the calculation of clad surface temperature and on the quenching time of the core.

It was shown in the present calculation that surge flow from the pressurizer, drift flux model during reflooding phase, pumped injection flow rate and its enthalpy play important roles on core cooling.

## Acknowledgments

The authors would like to express thier thanks to Mr. K Sato, Chief of Reactor Safety Code Development Labolatory, for his valuable suggestions to this work.

The authors are also very grateful to the members of the laboratory for thier useful discussions.

## References

- (1) Y. Asahi, 'Description of the THYDE-P Code (Preliminary Report of Methods and Models)', JAERI-M7751, 1978.
- (2) Y. Asahi and M. Hirano, 'Verification Study of LOCA Analysis Code THYDE-P (Sample Calculation Run 10)', JAERI-M8560, 1979.
- (3) M. Hirano and Y. Asahi, 'Through Analysis of LOFT L2-2 by THYDE-P Code', JAERI-M9535, 1981.
- (4) M. Hirano, 'Through Analysis of LOFT L2-3 by THYDE-P Code', to be published.
- (5) T.R. Chalton, et al., 'RELAP4/MOD5 Users Manual Volume 3 (Checkout Applications)', ANCR-NUREG-1335 (Vol.3), 1976.
- (6) K.R. Katsuma, et al., 'RELAP4/MOD5 Users Manual Volume 1 (RELAP4/MOD5 Description)', ANCR-NUREG-1335 (Vol.1), 1976.

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- (4) M. Hirano, 'Through Analysis of LOFT L2-3 by THYDE-P Code', to be published.
- (5) T.R. Chalton, et al., 'RELAP4/MOD5 Users Manual Volume 3 (Checkout Applications)', ANCR-NUREG-1335 (Vol.3), 1976.
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## References

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- (2) Y. Asahi and M. Hirano, 'Verification Study of LOCA Analysis Code THYDE-P (Sample Calculation Run 10)', JAERI-M8560, 1979.
- (3) M. Hirano and Y. Asahi, 'Through Analysis of LOFT L2-2 by THYDE-P Code', JAERI-M9535, 1981.
- (4) M. Hirano, 'Through Analysis of LOFT L2-3 by THYDE-P Code', to be published.
- (5) T.R. Chalton, et al., 'RELAP4/MOD5 Users Manual Volume 3 (Checkout Applications)', ANCR-NUREG-1335 (Vol.3), 1976.
- (6) K.R. Katsuma, et al., 'RELAP4/MOD5 Users Manual Volume 1 (RELAP4/MOD5 Description)', ANCR-NUREG-1335 (Vol.1), 1976.

Appendix A. Input Data List

```

-- 1000 MWE PWR BLOWDOWN ANALYSIS ( WITH HOT CHANNEL) 81.05.13 --
/
/ **** DIMENSION DATA ****
BB01
 0 0 9 3 16 49 40 9 2 2 2 2 3 6 5 3 0 2
/
/ **** MINOR EDIT DATA ****
BB02
PRE-08 PRA-12 GLA-23 GLA-29 GLE-35 GLE-36 GLA-37 GLA-38 PRA-26
/
/ **** TIME STEP CONTROL DATA ****
BB03
SB0301
 0.2 0.2 100.
SB0304
 20 3 50 0 1.0E-3 1.0E-6 0.3 0.1
SB0305
 30 3 50 0 8.0E-3 1.0E-6 90.0 0.1
SB0308
 30 3 50 0 4.E-3 1.E-6 2000.0 0.1
/
/ **** TRIP CONTROLL DATA ****
BB04
SB0480
 1 0 1 0 1000.0 0.0
SB0481
 5 46 1 0 0.4 0.0
SB0482
 5 47 1 0 0.4 0.0
SB0483
 2 8 1 0 0.01 0.0
SB0484
 2 19 1 0 0.01 0.0
SB0485
 3 0 1 0 0.01 0.0
SB0486
 4 1 1 0 25.01 0.0
SB0487
-4 1 1 0 1000.0 0.
SB0488
 4 2 1 0 25.01 0.0
SB0489
-4 2 1 0 1000.0 0.
SB0492
 6 1 -3 1 240.0 0.005
SB0493
 6 2 -3 1 250.0 0.0
SB0494
 6 3 -3 1 360.0 0.0
SB0495
-6 1 3 1 350.0 0.0
SB0496
-6 2 3 1 305.0 0.0
SB0497
-6 3 3 1 380.0 0.00
/
/ **** FLOW AJUST DATA ****
BB05
 1 9000.0 360.0

```

```

00000010
00000020
00000030
00000040
00000050
00000060
00000070
00000080
00000090
00000100
00000110
00000120
00000130
00000140
00000150
00000160
00000170
00000180
00000190
00000200
00000210
00000220
00000230
00000240
00000250
00000260
00000270
00000280
00000290
00000300
00000310
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00000440
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00000460
00000470
00000480
00000490
00000500
00000510
00000520
00000530
00000540
00000550
00000560
00000570
00000580
00000590

```

/											00000600
/	****	NODE	DATA	****							00000610
BB06											00000620
SB0601											00000630
1	1	26	1	0	1	158.4538	0.737	0.	5.24	0.0	00000640
						0.043	0.084	0.0	0.0		00000650
SB0602											00000660
2	1	1	2	0	1	158.9708	1.92	0.	1.665	1.665	00000670
						3.73	1.97	0.0	0.0		00000680
SB0603											00000690
3	7	2	3	1	3265	158.7624	0.0197	0.	5.0	5.0	00000700
						0.033	0.048	0.0	0.0		00000710
SB0604											00000720
4	7	3	4	1	3265	158.1581	0.0197	0.	5.46	5.46	00000730
						0.0	0.0	0.0	0.0		00000740
SB0605											00000750
5	7	4	5	1	3265	157.4898	0.0197	0.	10.46	-10.46	00000760
						0.0	0.0	0.033	0.048		00000770
SB0606											00000780
6	1	5	6	0	1	157.7862	1.92	0.	1.665	-1.665	00000790
						0.0	0.0	3.73	1.97		00000800
SB0607											00000810
7	1	6	7	0	1	157.4466	0.787	0.	7.34	-3.54	00000820
						0.042	0.077	-1.	-1.		00000830
SB0608											00000840
8	8	7	34	0	1	154.7227	0.494	0.	12.4117	3.54	00000850
						-1.	-1.	0.2029	0.2027		00000860
SB0609											00000870
9	1	34	8	0	1	162.0607	0.699	0.	2.825	0.0	00000880
						0.0	0.0	0.0	0.0		00000890
SB0610											00000900
10	1	8	29	0	1	162.0332	0.699	0.	3.13	0.0	00000910
						0.0	0.0	0.0	0.0		00000920
SB0611											00000930
11	1	26	27	0	3	158.4538	0.737	0.	2.0	0.	00000940
						0.043	0.083	0.0	0.0		00000950
SB0612											00000960
12	1	27	9	0	3	158.4334	0.737	0.	3.24	0.	00000970
						0.0	0.0	0.0	0.0		00000980
SB0613											00000990
13	1	9	10	0	3	158.9528	1.92	0.	1.665	1.665	00001000
						3.73	1.97	0.0	0.0		00001010
SB0614											00001020
14	7	10	11	1	9795	158.7445	0.0197	0.	5.0	5.0	00001030
						0.033	0.048	0.0	0.0		00001040
SB0615											00001050
15	7	11	12	1	9795	158.1387	0.0197	0.	5.46	5.46	00001060
						0.0	0.0	0.0	0.0		00001070
SB0616											00001080
16	7	12	13	1	9795	157.4691	0.0197	0.	10.46	-10.46	00001090
						0.0	0.0	0.033	0.048		00001100
SB0617											00001110
17	1	13	14	0	3	157.7645	1.92	0.	1.665	-1.665	00001120
						0.0	0.0	3.73	1.97		00001130
SB0618											00001140
18	1	14	15	0	3	157.4249	0.787	0.	7.34	-3.54	00001150
						0.042	0.077	-1.	-1.		00001160
SB0619											00001170
19	8	15	28	0	3	154.6997	0.494	0.	12.4117	3.54	00001180
						-1.	-1.	0.2029	0.2027		00001190

SB0620	20	1	28	29	0	3	162.0373	0.699	0.	5.955	0.	00001200
							0.0	0.0	0.0	0.0		00001210
												00001220
SB0621	21	4	29	16	0	1	162.4638	1.869	0.	7.248	-7.248	00001230
							0.0	0.0	0.0	0.0		00001240
												00001250
SB0622	22	5	16	30	0	1	162.9140	2.487	0.	6.075	1.948	00001260
							0.0	0.0	0.0	0.0		00001270
												00001280
SB0623	23	2	30	17	0	39170	162.6047	1.0	0.	0.23	0.23	00001290
							0.74	0.74	0.0	0.0		00001300
												00001310
SB0624	24	2	17	18	1	39170	162.1173	1.0	0.	0.80	0.80	00001320
							0.0	0.0	0.0	0.0		00001330
												00001340
SB0625	25	2	18	31	1	39170	161.5410	1.0	0.	0.80	0.80	00001350
							0.0	0.0	0.0	0.0		00001360
												00001370
SB0626	26	2	31	19	1	39170	160.9517	1.0	0.	0.80	0.80	00001380
							0.0	0.0	0.0	0.0		00001390
												00001400
SB0627	27	2	19	20	1	39170	160.3296	1.0	0.	0.80	0.80	00001410
							0.0	0.0	0.0	0.0		00001420
												00001430
SB0628	28	2	20	33	0	39170	159.7062	1.0	0.	0.23	0.23	00001440
							0.0	0.0	0.0	0.0		00001450
												00001460
SB0629	29	2	30	21	0	200	162.6047	1.0	0.	0.23	0.23	00001470
							1.284	2.482	0.0	0.0		00001480
												00001490
SB0630	30	2	21	22	1	200	162.1173	1.0	0.	0.80	0.80	00001500
							0.0	0.0	0.0	0.0		00001510
												00001520
SB0631	31	2	22	32	1	200	161.5410	1.0	0.	0.80	0.80	00001530
							0.0	0.0	0.0	0.0		00001540
												00001550
SB0632	32	2	32	23	1	200	160.946155	1.0	0.	0.80	0.80	00001560
							0.0	0.0	0.0	0.0		00001570
												00001580
SB0633	33	2	23	24	1	200	160.3296	1.0	0.	0.80	0.80	00001590
							0.0	0.0	0.0	0.0		00001600
												00001610
SB0634	34	2	24	33	0	200	159.7062	1.0	0.	0.23	0.23	00001620
							0.76	0.34	0.0	0.0		00001630
												00001640
SB0635	35	3	30	33	0	1	162.6047	0.555	0.	3.66	3.66	00001650
							0.77	0.83	0.87	0.78		00001660
												00001670
SB0636	36	1	32	31	0	200	161.029550	0.034	0.	0.1	0.	00001680
							0.0	0.0	0.0	0.0		00001690
												00001700
SB0637	37	1	33	26	0	1	159.17209	3.44	0.	4.341	1.64	00001710
							0.0	0.0	0.0	0.0		00001720
												00001730
SB0638	38	13	26	40	0	1	5.0	2.216	0.	3.658	2.073	00001740
							1.491E4	1.491E4	0.0	0.0		00001750
												00001760
SB0639	39	13	27	25	0	1	5.0	0.29	0.	15.0	1.7	00001770
							0.41	0.87	0.0	0.0		00001780
												00001790

SB0640										00001800
40 13 25 35 0 1	5.0	0.29	0.	14.3	1.6					00001810
	0.0	0.0	0.0	0.0						00001820
SB0641										00001830
41 13 28 37 0 3	10.0	0.305	0.	12.0	0.0					00001840
	0.0	0.0	0.0	0.0						00001850
SB0642										00001860
42 13 34 38 0 1	10.0	0.305	0.	12.0	0.0					00001870
	0.0	0.0	0.0	0.0						00001880
SB0643										00001890
43 13 28 36 0 3	10.0	0.222	0.	120.0	0.0					00001900
	0.109	0.049	0.0	0.0						00001910
SB0644										00001920
44 13 34 39 0 1	10.0	0.222	0.	120.0	0.0					00001930
	0.109	0.049	0.0	0.0						00001940
/										00001950
/ **** JUNCTION DATA ****										00001960
BB07										00001970
1 1 0.0										00001980
2 1 0.0										00001990
3 1 0.0										00002000
4 1 0.0										00002010
5 1 0.0										00002020
6 1 0.0										00002030
7 1 0.0										00002040
8 1 0.0										00002050
9 1 0.0										00002060
10 1 0.0										00002070
11 1 0.										00002080
12 1 0.0										00002090
13 1 0.0										00002100
14 1 0.0										00002110
15 1 0.0										00002120
16 1 0.0										00002130
17 1 0.0										00002140
18 1 0.0										00002150
19 1 0.0										00002160
20 1 0.0										00002170
21 1 0.										00002180
22 1 0.										00002190
23 1 0.										00002200
24 1 0.										00002210
25 1 0.										00002220
26 2 1.027										00002230
27 4 0.049										00002240
28 4 0.351										00002250
29 3 0.531										00002260
30 4 0.1										00002270
31 4 0.01										00002280
32 4 0.01										00002290
33 4 0.05										00002300
34 4 0.117										00002310
35 6 0.										00002320
36 5 0.										00002330
37 7 0.										00002340
38 7 0.										00002350
39 5 0.										00002360
40 8 0.										00002370
/										00002380
/ **** MIXING JUNCTION DATA ****										00002390

```

BB08
SB0801
 26 3 1 11 38 0 0.25 0.75 0. 0.
SB0802
 27 2 12 39 0 0 1.0 0.0 0. 0.
SB0803
 28 3 20 41 43 0 1.0 0.0 0.0 0.0
SB0804
 29 1 21 0 0 0 1.0 0.0 0.0 0.0
SB0805
 30 3 23 29 35 0 0.945 0.005 0.05 0.0
SB0806
 31 1 26 0 0 0 1.0 0.0 0. 0.
SB0807
 32 2 32 36 0 0 0.99 0.01 0. 0.
SB0808
 33 1 37 0 0 0 1.0 0.0 0.0 0.0
SB0809
 34 3 9 42 44 0 1.0 0.0 0.0 0.0
/
/ **** PUMPED INJECTION DATA ****
BB09
SB0901
 1 37 30.0
 2 1
 0.0 666.0 1000.0 666.0
SB0902
 2 38 30.0
 2 1
 0.0 222.0 1000.0 222.0
/
/ **** PUMP DATA ****
BB10
SB1001
 8 1 0 1185.0 5.58 4.33E4 105.0 749.0 1150.0 3460.0 0.5 0.0
 0.05
SB1002
 19 1 0 1185.0 5.58 4.33E4 105.0 749.0 1150.0 3460.0 0.5 0.0
 0.05
/
/ **** PUMP DATA TABLE ****
BB11
SB1101
 1
 14
-1.0 1.56 -0.85 1.33 -0.80 1.28 -0.72 1.30
-0.62 1.35 -0.50 1.36 -0.34 1.34 -0.21 1.29
-0.11 1.23 0.0 1.22 0.25 1.16 0.50 1.13
 0.75 1.07 1.0 0.98
 14
-1.0 0.18 -0.85 0.34 -0.80 0.40 -0.72 0.48
-0.62 0.556 -0.50 0.67 -0.34 0.77 -0.21 0.84
-0.11 0.89 0.0 0.95 0.25 1.16 0.50 1.35
 0.75 1.62 1.0 1.94
 11
-1.0 0.18 -0.75 -0.13 -0.50 -0.32 -0.32 -0.40
-0.16 -0.42 0.0 -0.39 0.16 -0.28 0.32 0.16
 0.50 0.01 0.75 0.40 1.0 0.98
 11
-1.0 1.56 -0.75 1.12 -0.50 0.90 -0.32 0.82

```

-0.16	0.76	0.0	0.71	0.16	0.71	0.32	0.76	00003000	
0.50	0.90	0.75	1.33	1.0	1.94			00003010	
14								00003020	
-1.0	0.70	-0.90	0.70	-0.80	0.68	-0.70	0.63	00003030	
-0.60	0.53	-0.50	0.47	-0.40	0.46	-0.30	0.45	00003040	
-0.20	0.45	0.0	0.48	0.25	0.55	0.50	0.66	00003050	
0.75	0.83	1.0	1.02					00003060	
14								00003070	
-1.0	-1.42	-0.90	-1.32	-0.80	-1.23	-0.70	-1.14	00003080	
-0.60	-1.07	-0.50	-0.99	-0.40	-0.91	-0.30	-0.84	00003090	
-0.20	-0.77	0.0	-0.64	0.25	-0.49	0.50	-0.34	00003100	
0.75	-0.20	1.0	-1.10					00003110	
13								00003120	
-1.0	-1.42	-0.8	-1.12	-0.6	-0.82	-0.5	-0.68	00003130	
-0.4	-0.55	-0.2	-0.28	0.0	-0.08	0.11	0.0	00003140	
0.25	0.12	0.50	0.33	0.75	0.61	0.92	0.82	00003150	
1.0	1.02							00003160	
13								00003170	
-1.0	0.70	-0.8	0.5	-0.6	0.4	-0.5	0.39	00003180	
-0.4	0.38	-0.2	0.33	0.0	0.28	0.11	0.25	00003190	
0.25	0.22	0.50	0.14	0.75	0.03	0.92	0.01	00003200	
1.0	-0.10							00003210	
/ 2								00003220	
/ 0.0	1.0	1000.0	0.5					00003230	
/ 2								00003240	
/ -1.0	-50.0	1.0	50.0					00003250	
12								00003260	
-1.0	-1.15	-0.9	-1.24	-0.6	-2.8	-0.5	-2.9	00003270	
-0.4	-2.7	0.0	0.0	0.12	0.85	0.2	1.1	00003280	
0.5	1.02	0.7	1.0	0.9	0.95	1.0	1.0	00003290	
4								00003300	
-1.0	0.0	0.0	0.0	0.5	-0.8	1.0	-1.46	00003310	
7								00003320	
-1.0	0.0	0.0	0.0	0.1	-0.02	0.2	0.0	00003330	
0.3	0.1	0.9	0.78	1.0	1.0			00003340	
12								00003350	
-1.0	-1.15	-0.8	-0.5	-0.6	-0.2	-0.4	0.03	00003360	
-0.2	0.04	0.0	0.1	0.2	0.15	0.4	0.12	00003370	
0.6	0.05	0.8	-0.5	0.9	-0.9	1.0	-1.46	00003380	
0								00003390	
0								00003400	
0								00003410	
0								00003420	
0								00003430	
13								00003440	
0.0	0.0	0.05	0.0	0.1	0.025	0.15	0.075	0.2	0.18
0.3	0.475	0.4	0.625	0.5	0.74	0.6	0.82		
0.7	0.87	0.8	0.84	0.9	0.72	1.0	0.08		
11									
0.0	0.0	0.1	0.0	0.20	0.13	0.3	0.24		
0.4	0.31	0.5	0.33	0.6	0.3	0.7	0.23		
0.8	0.16	0.9	0.08	1.0	0.0				
6 6									
0.0	0.2	0.4	0.6	0.8	1.0				
0.0	0.0	0.0	0.0	0.0	0.0	0.0			
0.2	0.0	3.065E-5	7.7239E-5	1.3263E-4	1.946E-4	2.6207E-4			
0.4	0.0	4.866E-5	1.2261E-4	2.1053E-4	3.0996E-4	4.1602E-4			
0.6	0.0	6.376E-5	1.6066E-4	2.7587E-4	4.0485E-4	5.4514E-4			
0.8	0.0	7.7239E-5	1.9463E-4	3.3419E-4	4.9044E-4	6.6037E-4			
1.0	0.0	8.9628E-5	2.2585E-4	3.878E-4	5.691E-4	7.6631E-4			

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/ **** ACCUMULATOR DATA ****
BB12
SB1201
  48 36 70. 30. 30.0 44.
  0.9 3.0
SB1202
  49 39 23.3 10. 30.0 44.
  0.9 1.0
/
/ **** BREAK POINT DATA ****
BB13
  8 0.01 0.4 0.8 0.6 0.6 0.8 0.6 0.6
  6
  0.0 1.0 7.5 2.7 15. 4.0 30. 4.0 60. 4.0 1000. 4.0
/
/ **** PRESSURIZER DATA ****
BB14
  45 35 11 3.58 15.56 9.0 0.99 0.1
  1.7 385.0
  50.0 1.0 0.1 0.0 0.0
  0.915 0.915 0.915 1.525 3.05 4.58
  0.564 0.67 0.619
  2
  0. 1.0 1.0 1.0 1000. 1.0 1.0 1.0
/
/ **** STEAM GENERATOR DATA ****
BB15
SB1501
  46 3265 3 5 3 1
  5.5 18.9 0.7 0.5 3.0E-2 1.0E-2 10.4 4.0 222.1 474.5
  0.1 0.95 62.0
  2.0 11.0
  -40. -30. -25.
  0.001 80. 0.5 0.5 0.5
  3
  0.0 1.0 1.0 1.0 1.0 1.0 1.0 1.0 1000.0 1.0 1.0 0.0
SB1502
  47 9795 14 16 3 1
  16.5 18.9 2.1 0.5 3.0E-2 1.0E-2 10.4 4.0 222.1 1423.5
  0.1 0.95 62.0
  2.0 11.0
  -40.0 -30.0 -25.0
  0.003 80. 0.5 0.5 0.5
  3
  0.0 1.0 1.0 1.0 1.0 1.0 1.0 1.0 1000.0 1.0 1.0 0.0
/
/ **** CORE DATA ****
BB16
/ --- AVERAGE CHANNEL -----
SB1601
  1
  39170 23 28 0 3 1 2 2
  9000.0 5.3658E-3 0.6187E-3 4.6573E-3 1.42E-2 0.6 1.0E-4
  0.0124 0.0212E-02 0.0305 0.1402E-02
  0.111 0.1254E-02 0.301 0.2529E-02
  1.13 0.0736E-02 3.00 0.0269E-02
  5.0 0.6 4.91E-04 3.41E-06 1.2 1.54E03
  0. 156. 234. 234. 156. 0.
  1.6122E-07 6.42E-07 7.56E-07 7.56E-07 6.42E-07 1.622E-07
  1.6122E-07 6.42E-07 7.56E-07 7.56E-07 6.42E-07 1.622E-07
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00004110
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00004130
00004140
00004150
00004160
00004170
00004180
00004190

```



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1.6122E-07 6.42E-07 7.56E-07 7.56E-07 6.42E-07 1.622E-07 00004200
1.6122E-07 6.42E-07 7.56E-07 7.56E-07 6.42E-07 1.622E-07 00004210
/ --- HOT CHANNEL --- 00004220
SB1602 00004230
2 00004240
200 29 34 0 3 1 2 2 00004250
9000.0 5.3665E-3 0.6187E-3 4.6682E-3 1.42E-2 0.6 1.0E-4 00004260
0. 203.0 304.0 304.0 203.0 0. 00004270
1.6122E-07 6.42E-07 7.56E-07 7.56E-07 6.42E-07 1.622E-07 00004280
1.6122E-07 6.42E-07 7.56E-07 7.56E-07 6.42E-07 1.622E-07 00004290
1.6122E-07 6.42E-07 7.56E-07 7.56E-07 6.42E-07 1.622E-07 00004300
1.6122E-07 6.42E-07 7.56E-07 7.56E-07 6.42E-07 1.622E-07 00004310
/ 00004320
/ **** REACTIVITY DATA **** 00004330
BB17 00004340
3 00004350
0. 0. 0.5 -5. 1. -25. 00004360
5 00004370
20. 0.5 300. 0. 1500. -0.5 2500. -1. 4500. -5. 00004380
5 00004390
0.0 0.0 1.0 -0.1 1.5 -0.2 2.0 -3.0 1000. -8.0 00004400
/ 00004410
/ ****METAL WATER REACTION DATA **** 00004420
BB18 00004430
1.54E03 0.775E-04 2.29E04 00004440
/ 00004450
/ **** FUEL GAP DATA **** 00004460
BB19 00004470
0.0301 0.0 1.235E-5 0.0 0.0 0.0 0.0 0.6 0.6 0.0 00004480
0.9495 0.0157 0.0028 0.0 0.032 0.0 0.0 00004490
/ 00004500
/ **** BURST DATA **** 00004510
BB21 00004520
2 2 5.0E7 6.96E-08 2.87E4 2.86E-03 1.15E0 1.528E0 00004530
1.49E-07 2.0E-08 1.25E-16 1.85E-01 8.0E09 3.3E-03 00004540
0.1 00004550
/ 00004560
/ **** OTHER DATA **** 00004570
BB22 00004580
0. 1.4 1.4 0. 00004590
BEND 00004600
5 00004610
0 0 0 0 0 0.0 00004620
0. 1.0-7 0. 0. 0. 0. 00004630
0 00004640
0 0.0 00004650
0 0.0 00004660

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