

# **JNC Contribution to the Benchmark Problem on Thermal Transient Strength Evaluation of a Welded Vessel**

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2000

**JNC contribution to the Benchmark Problem  
on thermal transient strength evaluation of a welded vessel  
(Research Report)**

Naoto Kasahara\*

**Abstract**

Fatigue and creep-fatigue strength of welded joints are lower than base metals, when applied to elevated temperature components subjected to cyclic thermal transient loading. CEA and JNC have developed design evaluation procedures for considering strength reduction of weldments in elevated temperature components. It was planned to compare both procedures based on the same benchmark problems under EJCC contract. One of benchmarks provided by CEA is fatigue and creep-fatigue evaluation of welded plates due to reverse bending at 550°C. Another problem by JNC is creep-fatigue evaluation of a welded vessel due to cyclic thermal transient loading. Point of view of the later problem is comparison of total strength evaluation of base metal and welded joints against actual loading conditions. This report describes details of the TTS experiments and defines the JNC benchmark problem.

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## 溶接容器の熱過渡強度評価に関するJNCベンチマーク問題

### (研究報告書)

笠原 直人<sup>\*</sup>

### 要 旨

高温構造設計ではクリープ疲労損傷が主要破損モードとして想定される。特に溶接部は、繰り返し熱過渡荷重が加わる実機条件下において母材より強度が低下するため留意が必要である。このため、CEAとJNCは高温機器における溶接部の強度低減を設計で考慮するための評価法を整備してきている。両者の評価法を相互比較するため、日欧高速炉協定に基づく国際協力によりベンチマーク問題を設定した。一つはCEAから出題されたもので、550℃の温度で繰り返し曲げ荷重を受ける溶接平板の疲労およびクリープ疲労強度評価に関するものである。もう一つはJNCから出題したもので、繰り返し熱過渡荷重を受ける溶接容器のクリープ疲労強度評価に関するものである。

後者は大洗のTTS試験装置で実施された試験に関するものであり、着眼点は実機荷重に対する母材と溶接部の総合強度評価に関する裕度と適用性の確認にある。本報告書ではTTSによる試験概要の説明を行うと共にベンチマーク問題を定義する。

尚、本内容は1999年9月から2000年8月までの期間にCEAカダラッシュ研究所にて実施した業務の一部である。

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## **1 INTRODUCTION**

Fatigue and creep-fatigue strength of welded joints in components are generally lower than base metal under elevated temperature operative conditions. In order to take strength reduction of welded joints into account for elevated temperature structural design, design codes are required to provide rational evaluation methods for welded joints.

CEA and JNC have developed design evaluation procedures for considering strength reduction of weldments. Under EJCC framework, intercomparison of both procedures is planned through application to the same benchmark problems. For benchmark, CEA and JNC have submitted two complementary problems. One of benchmarks, which was provided by Dr. Laurent LE BER of CEA, is fatigue and creep-fatigue evaluation of welded plates due to reverse bending at 550°C[1]. Another problem, proposed by JNC is creep-fatigue evaluation of a welded vessel due to cyclic thermal transient loading. The objective of the later problem is comparison of creep-fatigue evaluation methods of base metals and welded joints on actual components due to cyclic thermal transients. This report describes details of the welded vessel experiment and defines the benchmark problem.



## 2. THERMAL TRANSIENT TEST OF A WELDED VESSEL MODEL

A thermal transient strength test was conducted on a welded structure model by using a sodium test facility [2]. The test model is a vessel type structure, which has an outer container and an inner vessel as Fig.1. 1055 cycles of thermal transients were applied by alternate flow of hot (600°C) and cold sodium (250°C) which passed through the annulus space between the outside container and the inner vessel. During each cycle, creep damage was accumulated by 2 hours of holding time in 600°C.

As for materials, the outside container and half of the structure of the inner vessel are made of SUS304 (Japanese Type304SS), and the remainder half is made of 316FR (Japanese 316L with medium nitrogen for FBRs), which is low carbon nitrogen stainless steel for Liquid Metal Fast Reactors as in Fig.2. Circumferential and longitudinal welded joints are incorporated in both SUS304 and 316FR portions.

A photograph of the vessel model and an outline of the sodium test facility are as in Fig.3 and Fig.4.

Fig.5 describes the dimensions of the test model, where the outside container of the vessel model is 2210mm high and 980mm in diameter with 25 mm thickness wall, and the inner vessel is 456mm inner diameter with 20 mm thickness wall. The inner vessel has a restraint plate with 25mm wall thickness to make stress gradient on the inner vessel.

Fig.6 and Fig.7 show initiated cracks on the surface of the inner vessel observed by the liquid flaw detection test (PT) after 1055 cycles of thermal transients. In the 316FR division of inner vessel, few small cracks were found only at welded joints, while many cracks were observed at both base metals and welded joints in the 304SS parts. Here, the cracks at welded joints were observed to be deeper than that of the nearby base metal.

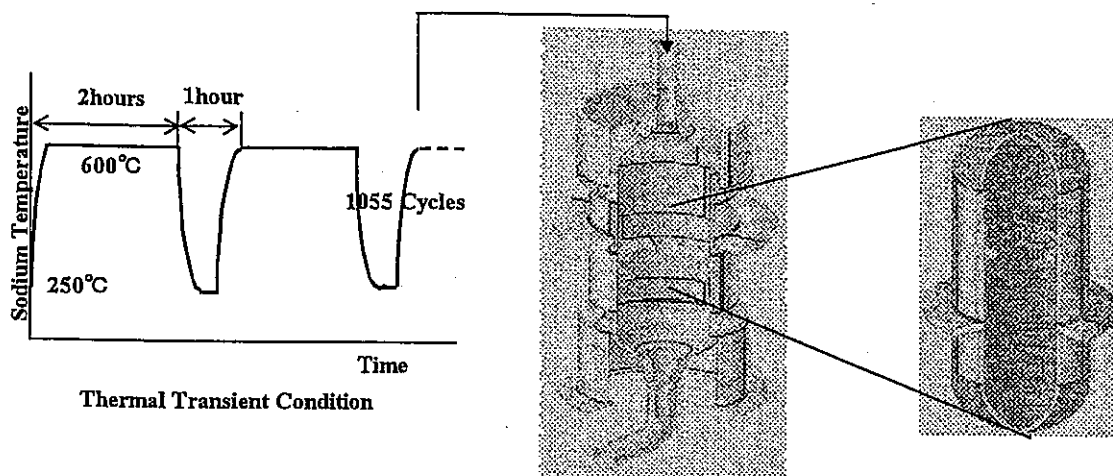


Fig.1 Thermal transient strength test of welded vessel model

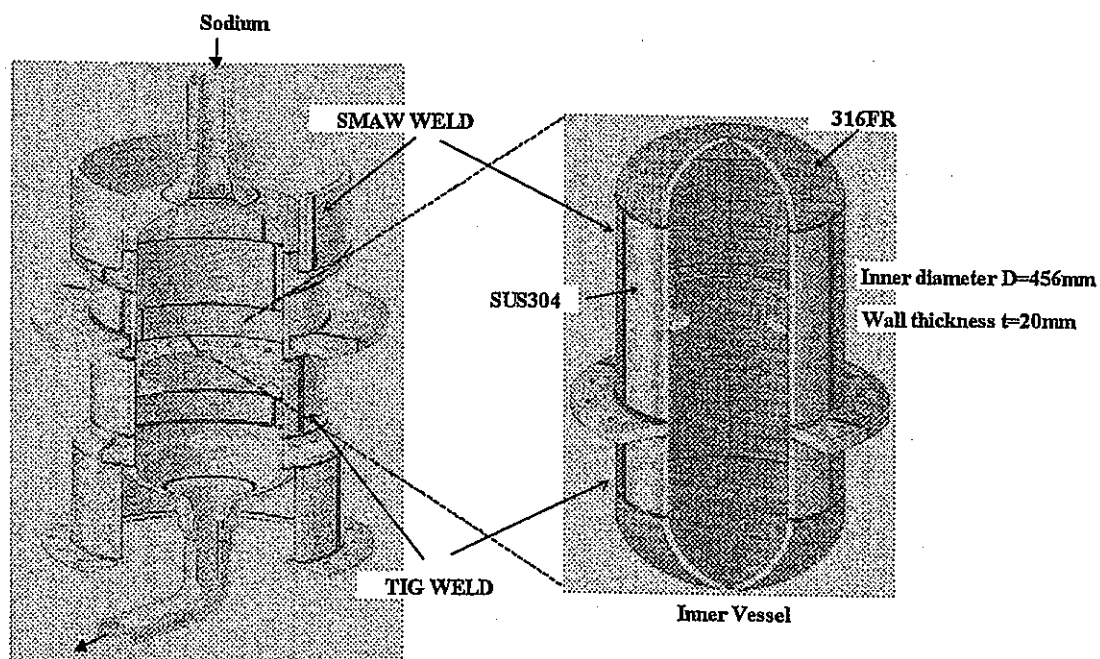


Fig.2 Welded joints in the welded vessel model

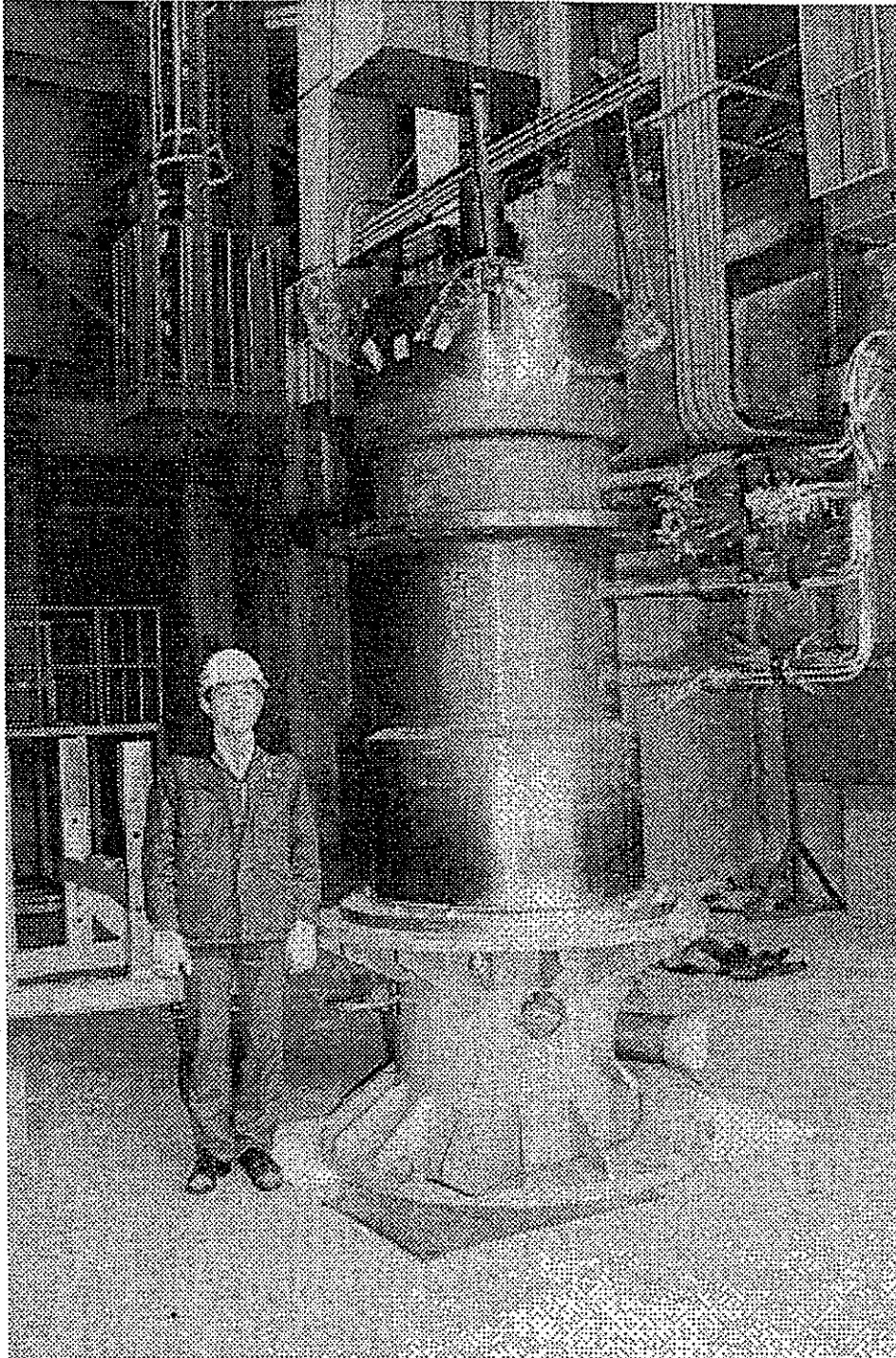


Fig.3 Welded vessel model

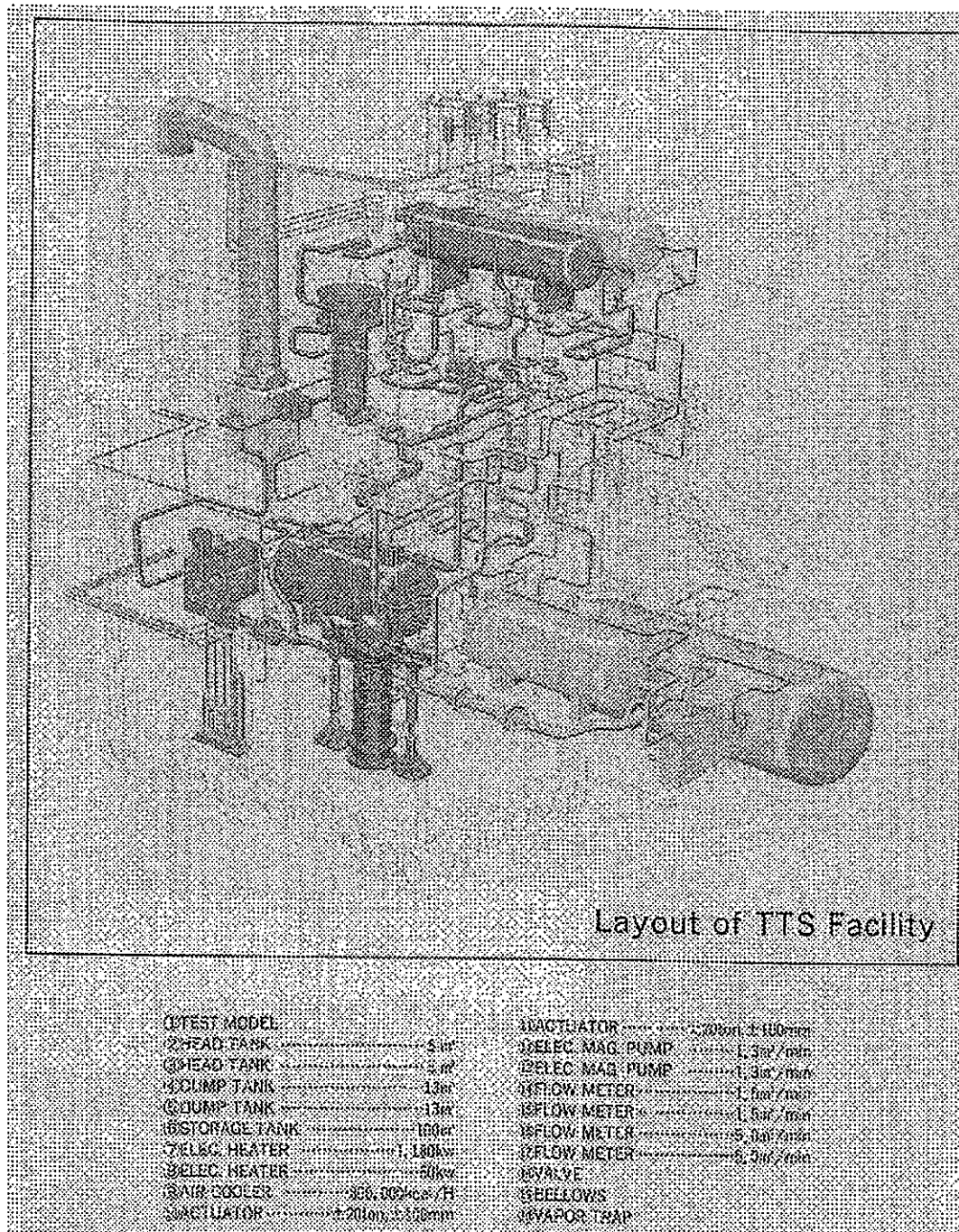


Fig.4 Thermal transient test facility for structure (TTS)

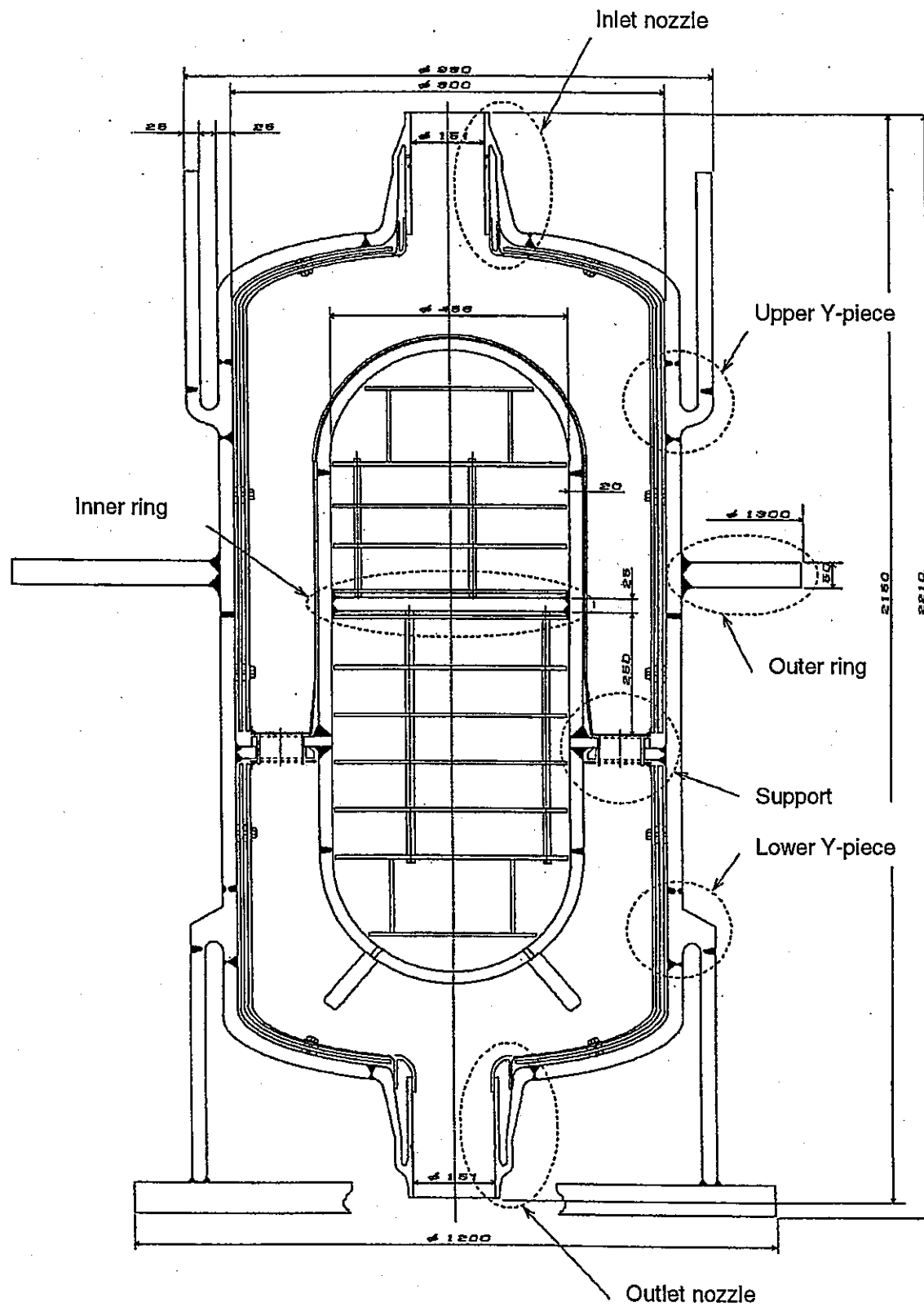


Fig.5 Dimension of welded vessel model

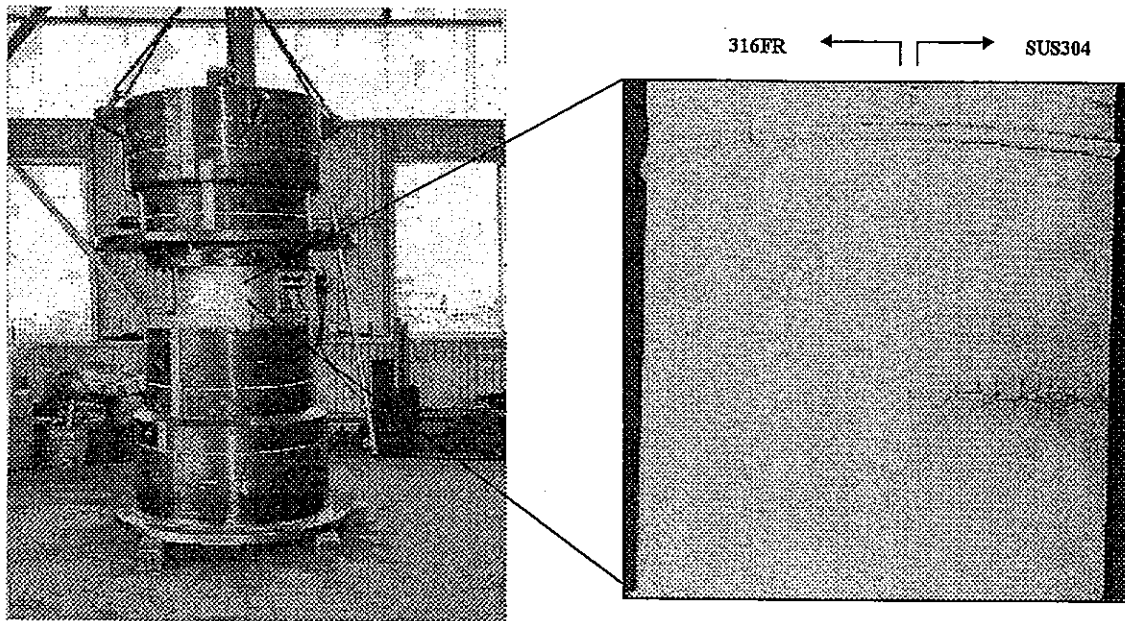


Fig.6 Thermal transient test result of inner vessel (316FR and SUS304)

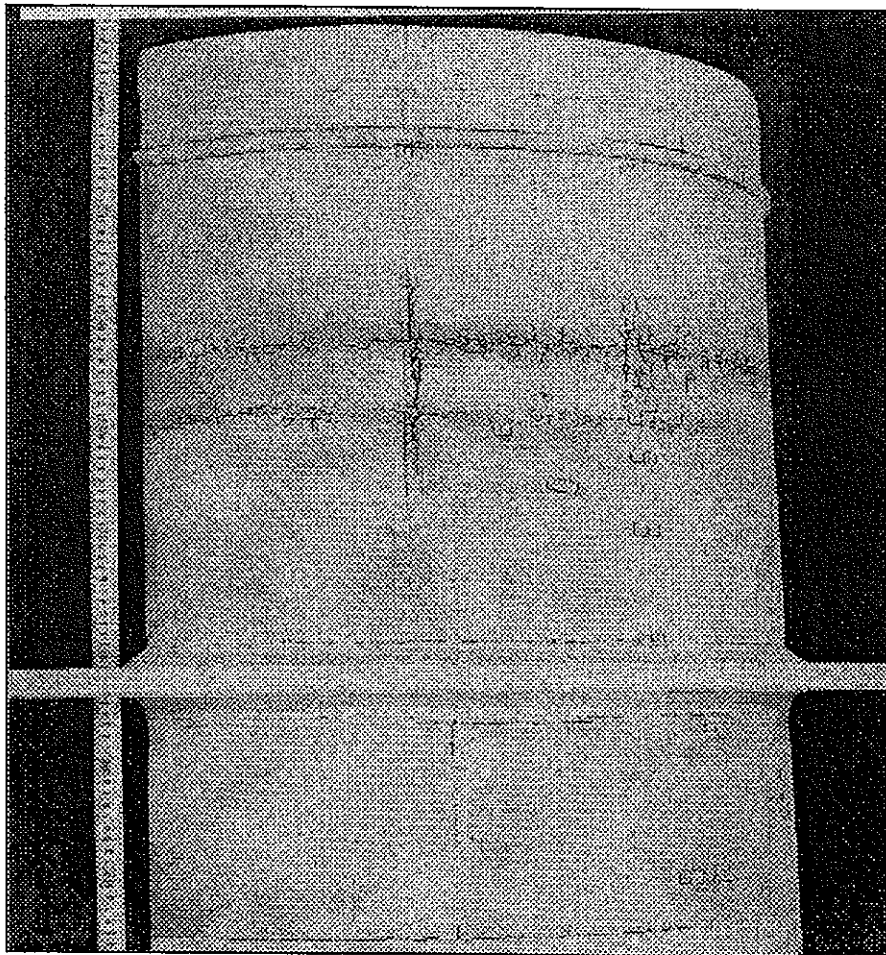


Fig.7 Thermal transient test result of inner vessel (SUS304)



### 3. SCOPE OF A BENCHMARK PROBLEM

In order to define a scope of the problem as the comparison of creep-fatigue evaluation procedures and to eliminate influences from differences of materials and structural analysis results, this benchmark program provides common material properties and structural analysis results as in Fig.8.

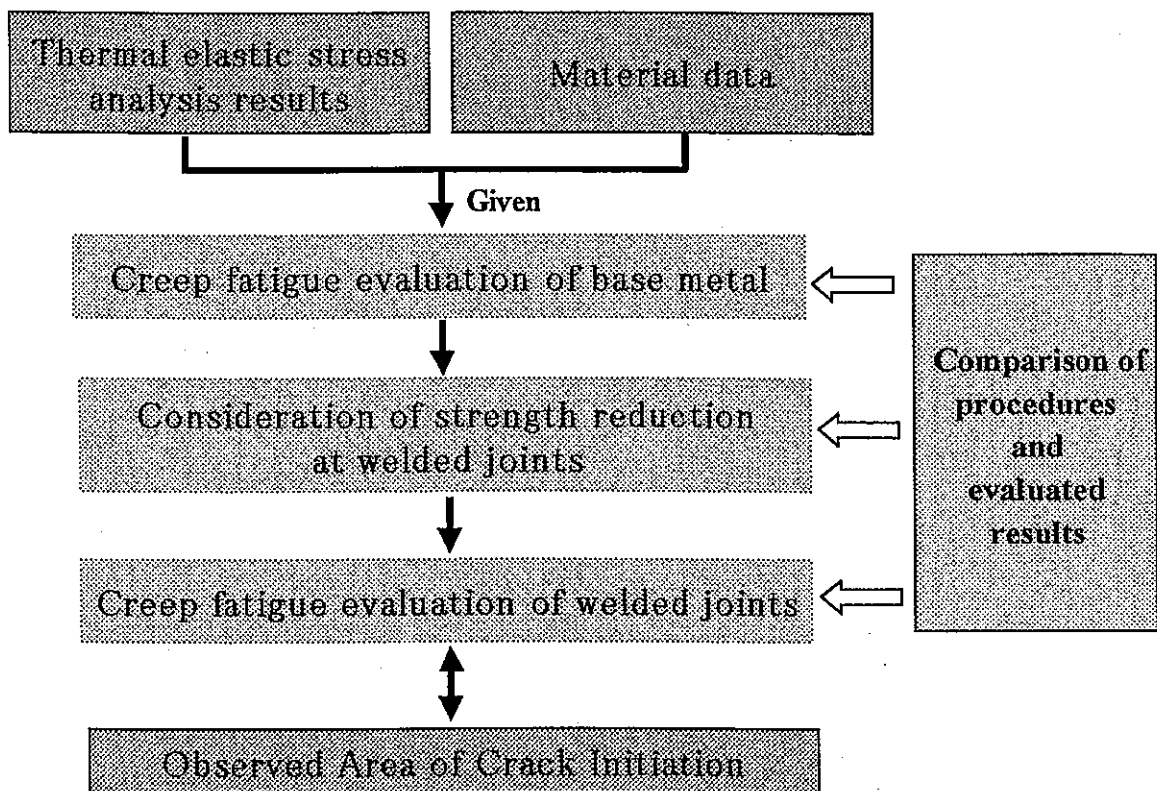


Fig.8 Scope of benchmark problem

Following material properties of SUS304 and 316FR are provided in Appendix1 and Appendix2.

- Fatigue curve
- Monotonic stress-strain curve
- Stress range-strain range relationship
- Creep strain

Above characteristics are average one of base metal obtained by regression analysis of material test results.

Material properties of a weld metal for SUS304 (Type308SS) are describes in literatures[3][4]. A fatigue curve of weld metal for SUS304 is equivalent to one of base metal. Yield stress of a weld metal for SUS304 is lower than base metal after sufficient cyclic loadings.

Properties of weld metal of 316FR are explained in a reference[5]. A fatigue curve is approximately the same as base metal. Difference of Yield stress between weld and base metals is smaller in 316FR than in SUS304.

Target area of benchmark in the welded vessel model is defined as 524mm length on the surface of the inner vessel as in Fig.9. This portion is made of SUS304 base metal, flash grained welded joint of SUS304, 316FR base metal, and flash grained welded joint of 316FR. All of these material parts have the same geometrical configuration.

For evaluation of structural strength against thermal transients, thermal stress analyses under thermal transient test conditions are required. To avoid complexities of structural analyses, a stress classification table obtained from thermal elastic analysis based on measured temperature data is provided in Fig.10 and by Appendix3. Both European and Japanese participants would apply the elastic route to evaluate creep-fatigue strength, based on the same stress classification table.



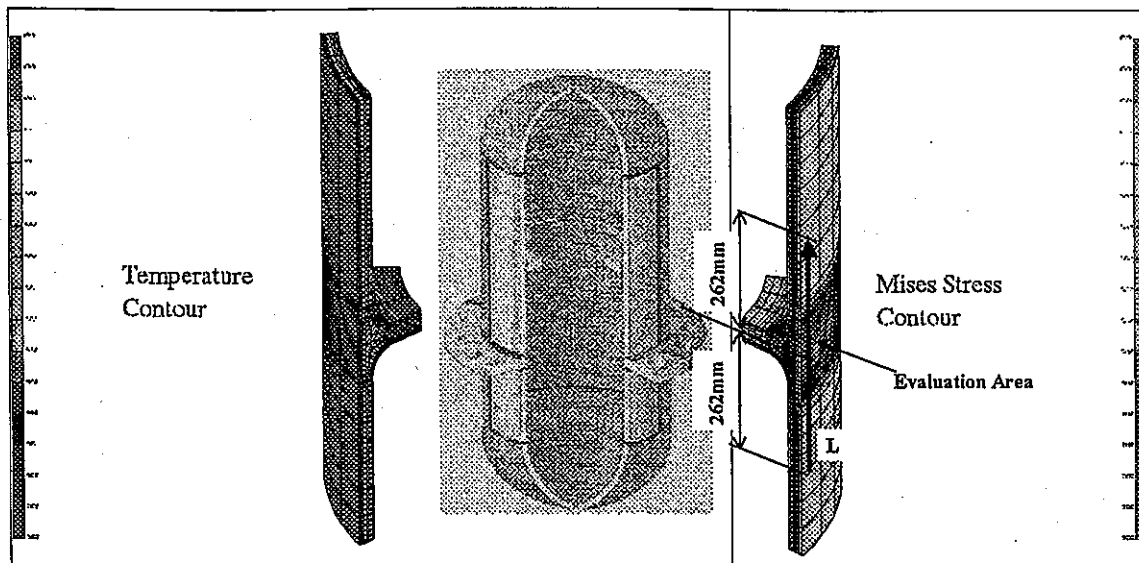


Fig.9 Thermal and thermal elastic analysis results

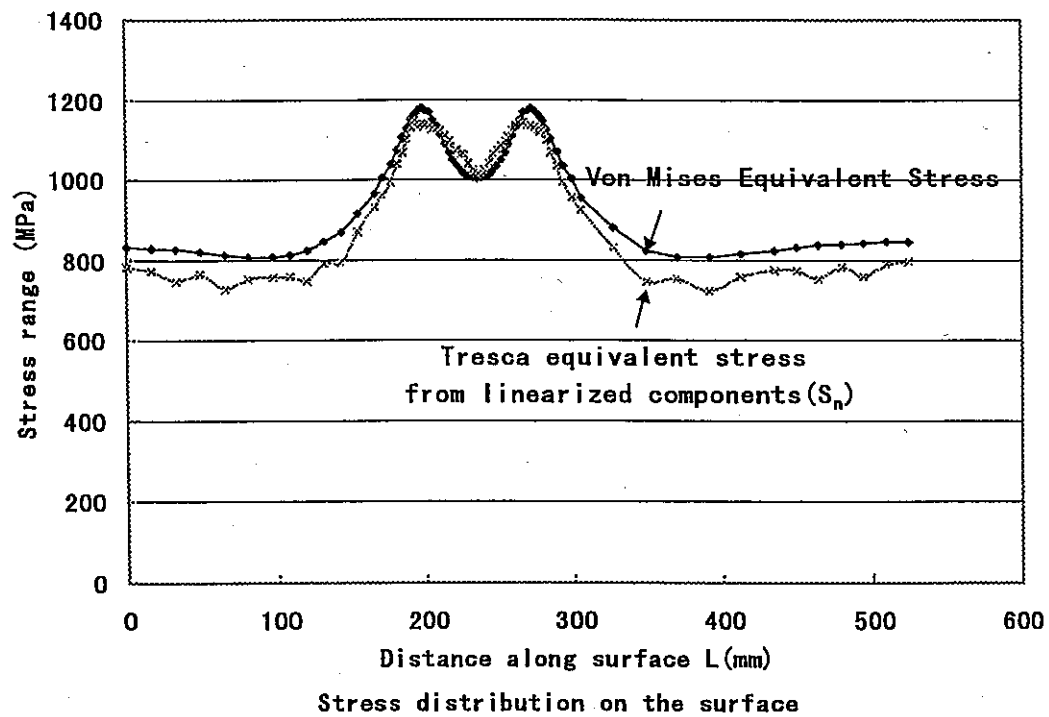


Fig.10 Thermal stress distribution along surface of inner vessel

#### **4. BENCHMARK**

The objective of this benchmark program is to evaluate creep-fatigue strength of the inner vessel in the welded vessel model which is made of SUS304, flash grained welded joint of SUS304, 316FR base metal, and flash grained welded joint of 316FR.

Both European and Japanese participants are expected to make a report with following items for SUS304, flash grained welded joint of SUS304, 316FR base metal, and flash grained welded joint of 316FR.

1. Creep-fatigue evaluation procedure
2. Criteria used for failure analysis
3. Total strain range
4. Fatigue strength reduction factor of welded joints
5. Fatigue damage factors
6. Creep damage factors

#### **5. EXPERIMENTAL DATA**

Photographs and sketches of initiated cracks on both 304SS and 316FR area of the inner vessel after 1055 cycles are provided in Appendix 4, which clarifies distribution and depth of cracks.

## 6. JNC CREEP-FATIGUE EVALUATION

### 6.1. CREEP-FATIGUE EVALUATION PROCEDURE

JNC creep-fatigue evaluation procedure for weldments is explained in literatures[6][7]. Concerning design factor, elastic follow-up parameter  $q$  for structural discontinuity is different from SOUFFLE test. Since geometries of SOUFFLE test specimens are flat plates without structural discontinuities, elastic follow-up parameter  $q$  is one. The inner vessel of a TTS welded vessel model is also a smooth cylinder, however, the surface becomes bi-axial field under severe thermal transients. Elastic follow-up parameter  $q$  of cylinders due to bi-axial bending stress was found to be  $q=1.67$  by previous studies[8][9] and its value was adopted.

When parameter  $q_w$  and  $\gamma_y$  are equal to one, the creep-fatigue evaluation procedure for weldments becomes a procedure for base metal[8].

### 6.2. FATIGUE EVALUATION RESULTS

Distributions of total strain range on the surface of the inner vessel were estimated as in Fig.11 and Fig.12.

From fatigue curves of materials (304SS and 316FR) and calculated strain range, fatigue damage factors were evaluated by *Miner's* rule as in Fig.13 and Fig.14.

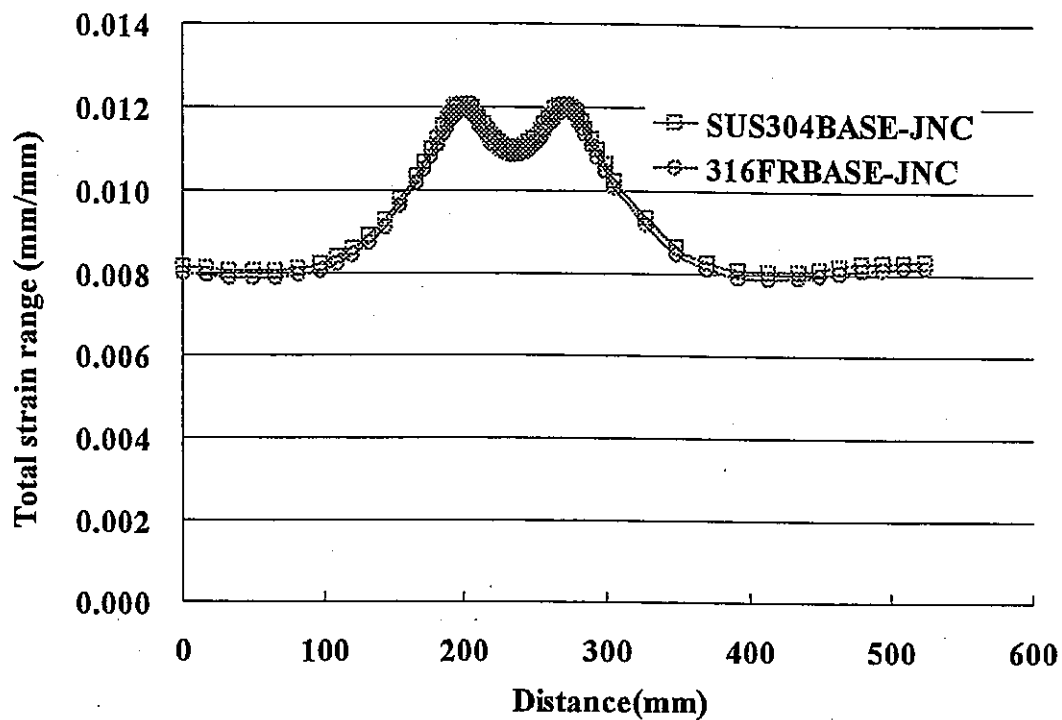


Fig.11 Distribution of total strain range on the surface of an inner vessel (Base metal)

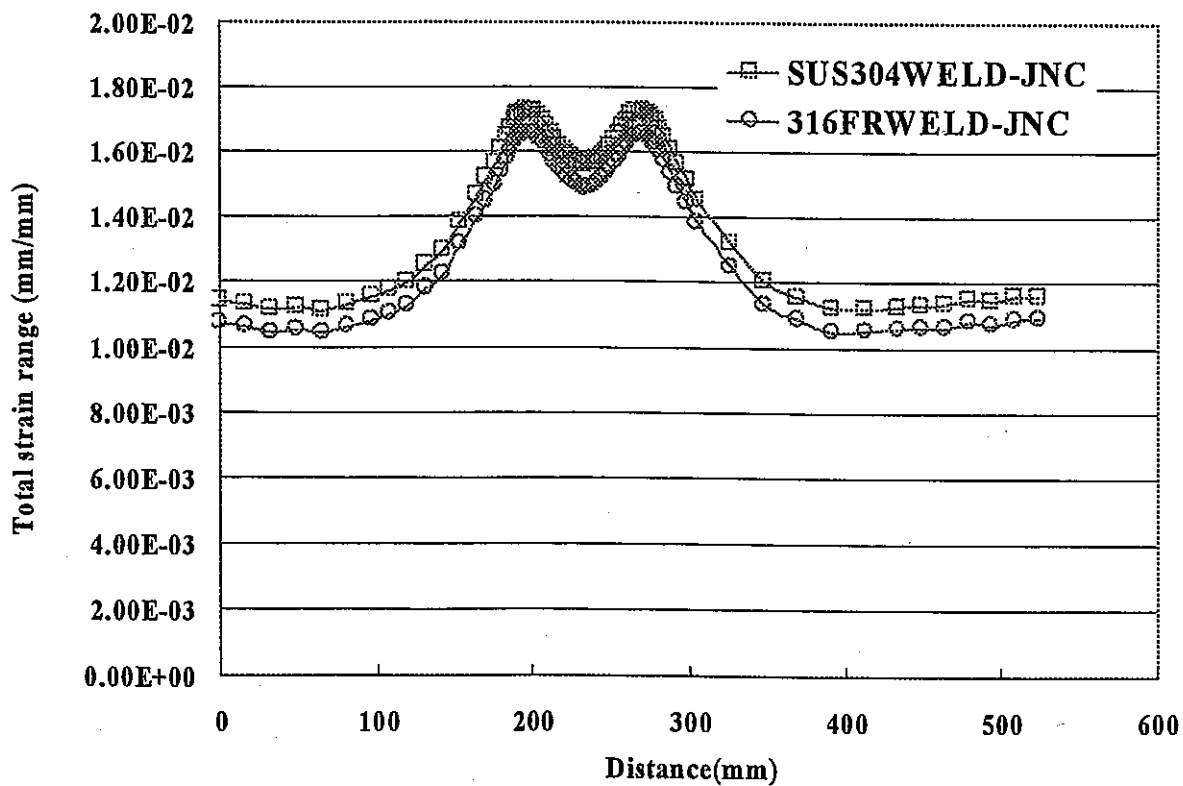


Fig.12 Distribution of total strain range on the surface of an inner vessel (Welded joint)

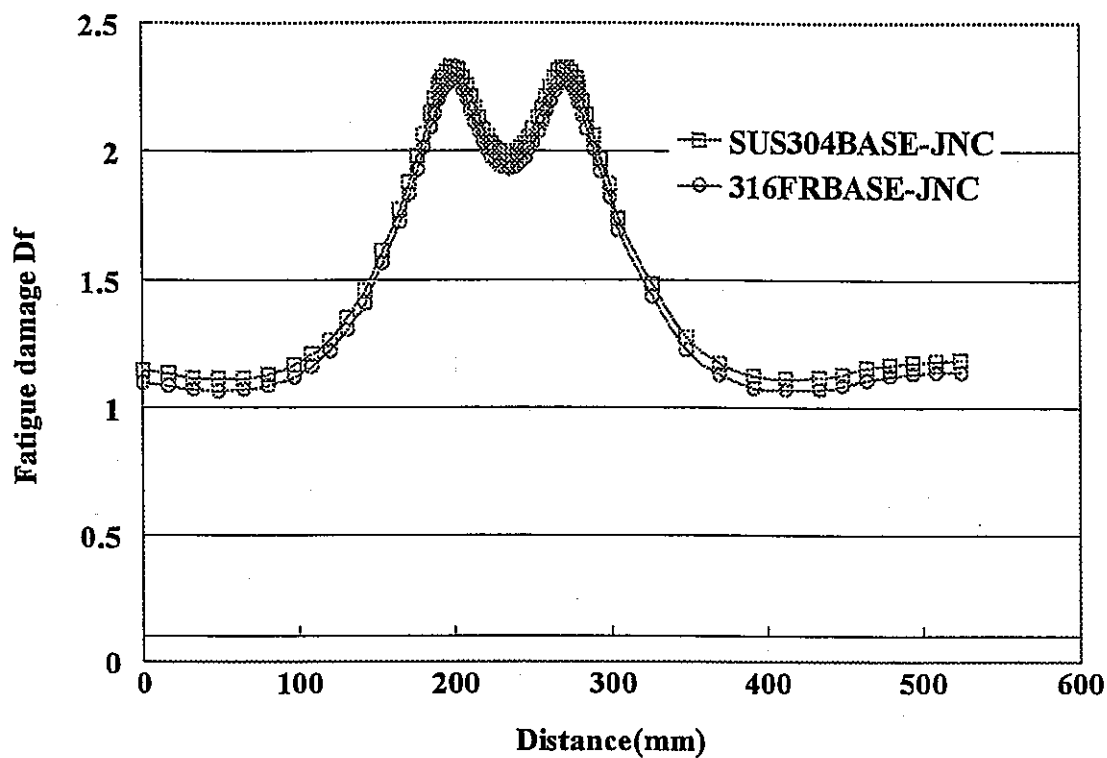


Fig.13 Distribution of fatigue damage factors on the surface of an inner vessel (Base metal)

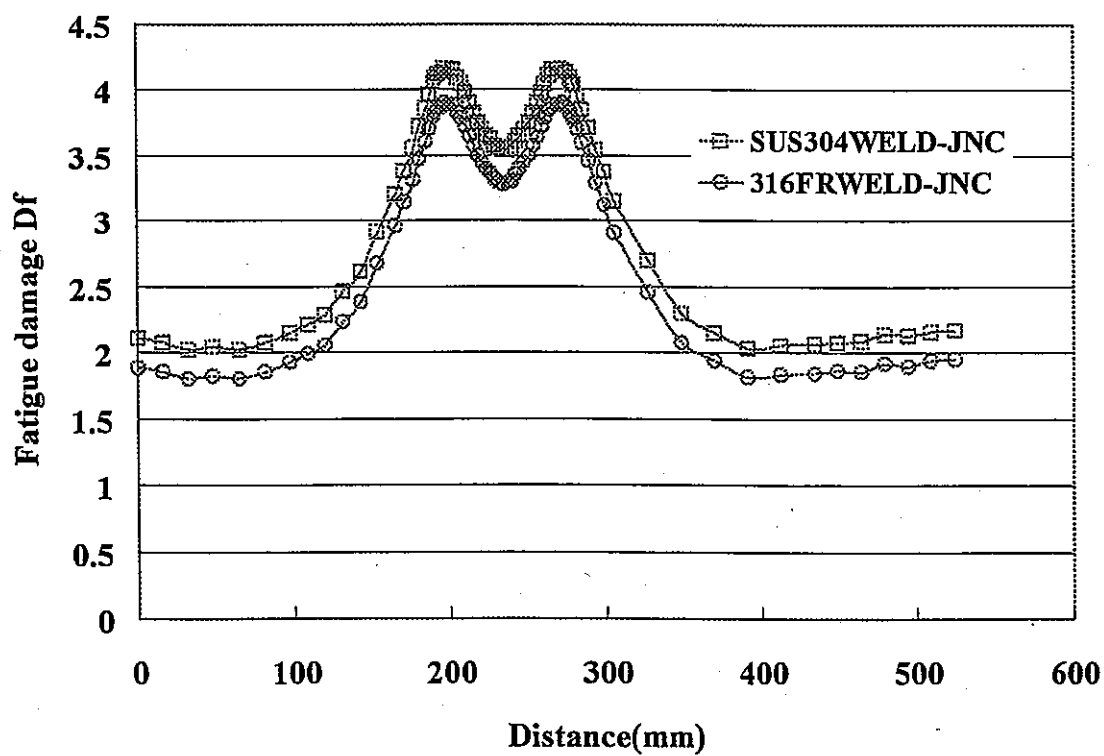


Fig.14 Distribution of fatigue damage factors on the surface of an inner vessel (Welded joint)

### 6.3. CREEP-FATIGUE EVALUATION RESULTS

From creep rupture curves of materials (304SS and 316FR) with estimated stress history, creep damage factors were calculated based on time fraction rule considering stress relaxation.

Predicted creep-fatigue damage on the inner vessel was as in Fig.15 and Fig.16. From these figures, creep-fatigue damage factors for all materials are beyond 1.0 at all of locations. It means that calculation results predicted possibility of crack initiation at all portions of the inner vessel after 1055 cycles of thermal transients.

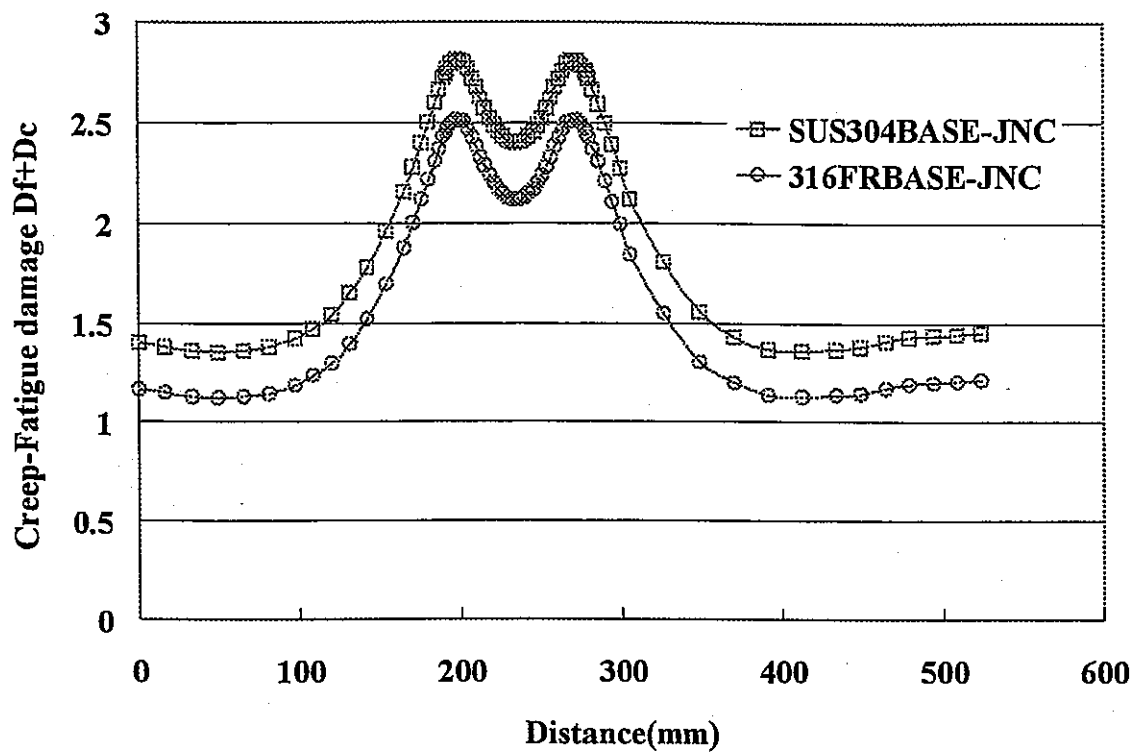


Fig.15 Distribution of creep-fatigue damage factors on the surface of an inner vessel (Base metal)

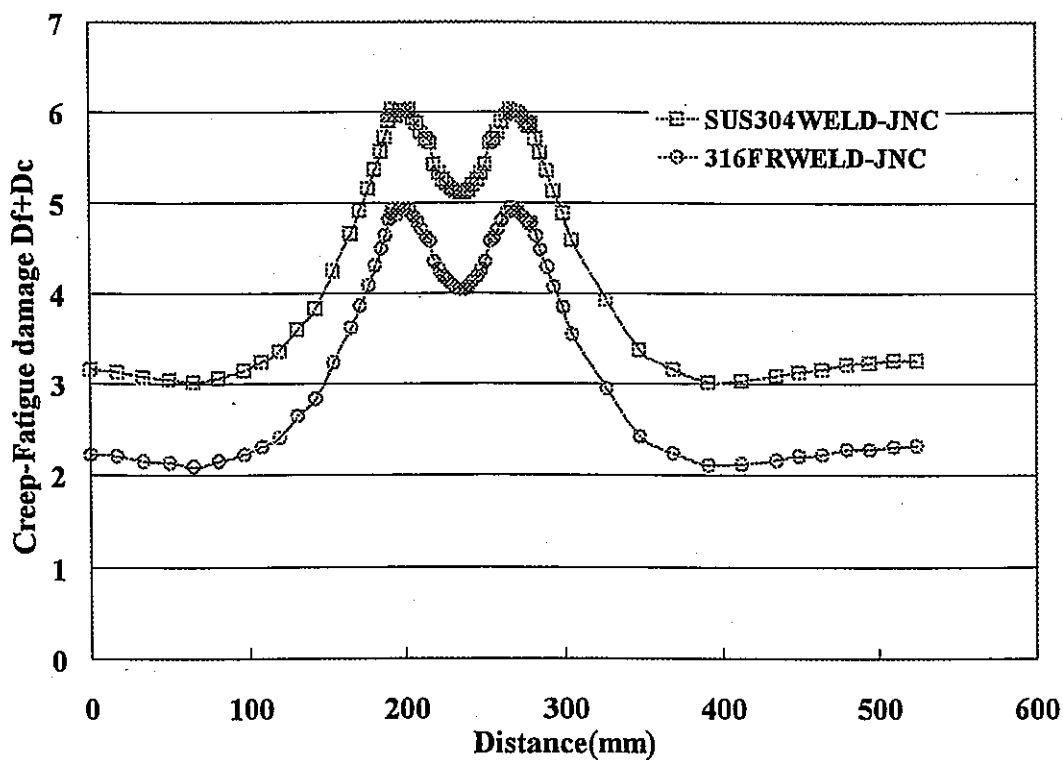


Fig.16 Distribution of creep-fatigue damage factors on the surface of an inner vessel (Welded joints)

## 7. DISCUSSIONS

Fig.17 is a sketch of initiated cracks on the surface of the inner vessel. Further sketches and photographs are attached in the Appendix 4. From comparison of calculated results with these experimental results, JNC procedure is considered to be conservative for 316FR. One of reasons is that yield strength difference between base and weld metals is less than  $\gamma_y=0.9$  in the most of strain range of this test. Another reason is that  $\alpha_R=10$  is adjusted factor to the weakest heat. These reasons are common with uni-axial material tests and SOUFFLE test, however, evaluation results of TTS test are considered to be more conservative than other tests. It is possible that  $q_w$  caused by thermal stress is less than one of mechanical stress, even if value of  $\gamma_y$  is the same.

## 8. FUTURE PLAN

In the next step of benchmark program, thermal transient strength evaluation of unfinished welded joints with penetration beads is planned.



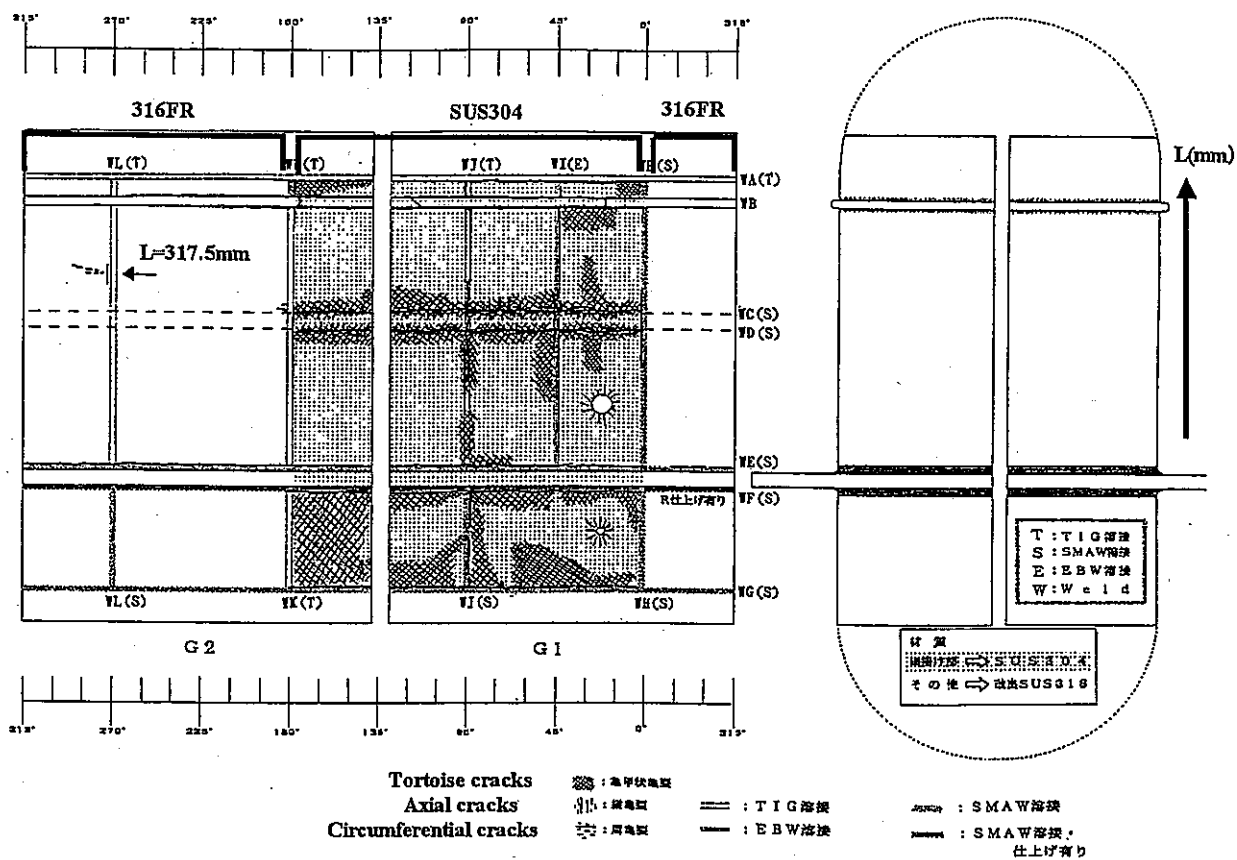


Fig.17 Distribution of initiated cracks on the surface of the inner vessel

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**ANNEXE 1 : Material properties of Japanese 304SS(SUS304)**

**ANNEXE 2 : Material properties of Japanese 316FR**

**ANNEXE 3 : Stress classification table obtained by thermal elastic analysis of the  
inner vessel model**

**ANNEXE 4 : Photographs and sketches of initiated clacks on both surface and  
section area of the inner vessel after 1055 cycles**

## **Appendix 1   Material properties of Japanese 304SS**

## SUS304, 316FR Fatigue Curve

$$\log_{10}(N_f)^{\frac{1}{2}} = A_0 + A_1 \cdot \log_{10} \Delta \varepsilon_t + A_2 \cdot (\log_{10} \Delta \varepsilon_t)^2 + A_3 \cdot (\log_{10} \Delta \varepsilon_t)^4$$

Unit

$T$  : *Temperature* ( °C )

$\varepsilon$  : *StrainRate*(mm/mm/sec)

$\Delta \varepsilon_t$  : *Total Strain Range* ( mm/mm )

$N_f$  : *Number of Cycles to Failure*

$A_0$	$1.621827 - 0.4567850 \times 10^{-7} \times T^2 \times R$
$A_1$	$1.131346 + 0.8665061 \times 10^{-8} \times T^2$
$A_2$	$0.3439663$
$A_3$	$-0.1374387 \times 10^{-1} + 0.4910723 \times 10^{-4} \times R$

Where  $R = \log_{10} \varepsilon$

## SUS304 Monotonic Stress-Strain Curve

$$(1) \sigma \leq \sigma_p$$

$$\varepsilon_e = \frac{\sigma}{E}$$

$$\varepsilon_p = 0$$

$$(2) \sigma > \sigma_p$$

$$\varepsilon_e = \frac{\sigma}{E}$$

$$\varepsilon_p = \left( \frac{\sigma - \sigma_p}{K} \right)^{\frac{1}{m}}$$

<Unit>

$\varepsilon_e$  (mm/mm),  $\varepsilon_p$  (mm/mm),  $\sigma$  (kg/mm<sup>2</sup>)

<Limit of total strain>

Maximum total strain  $(\varepsilon_e + \varepsilon_p)_{\max} \leq 0.03$  (mm/mm)

Temperature(°C) Parameter	315 ≤ T ≤ 650
E(kg/mm <sup>2</sup> )	315 ≤ T < 400    E = 2.040 × 10 <sup>4</sup> - 8.000T 400 ≤ T ≤ 650    E = 2.126 × 10 <sup>4</sup> - 10.125T
σ <sub>p</sub> (kg/mm <sup>2</sup> )	σ <sub>y</sub> - K(0.002) <sup>m</sup>
σ <sub>y</sub> (kg/mm <sup>2</sup> )	25.5655 - 5.58937 × 10 <sup>-2</sup> T + 1.04384 × 10 <sup>-4</sup> T <sup>2</sup> - 7.42535 × 10 <sup>-8</sup> T <sup>3</sup>
K(kg/mm <sup>2</sup> )	44.3068 - 1.78933 × 10 <sup>-2</sup> T
m	0.279395 + 7.749 × 10 <sup>-5</sup> T

## SUS304 Stress Range – Strain Range Relationship

$$\bullet \Delta\sigma/2 > \sigma_p$$

$$\log_{10}(\Delta\sigma - 2\sigma_p) = A_0 + A_1 \cdot \log_{10}(\Delta\varepsilon_t - \Delta\sigma/E)$$

$$\bullet \Delta\sigma/2 \leq \sigma_p$$

$$\Delta\sigma = E \cdot \Delta\varepsilon_t$$

Unit

$T$  : Temperature (°C)     $425 \leq T \leq 650$

$\Delta\sigma$  : StressRange (kg/mm<sup>2</sup>)

$\Delta\varepsilon_t$  : TotalStrainRange (mm/mm)

$E$  : ElasticModulus (kg/mm<sup>2</sup>)

$\sigma_p$  : ProportionalLimit(kg / mm<sup>2</sup>)

$A_0$	$0.9772687 + 0.6446708 \times 10^{-2} \times T - 0.4675557 \times 10^{-3} \times T^2 - 0.3724201 \times 10^{-8} \times T^3$
$A_1$	$3.690128 - 0.1847969 \times 10^{-1} \times T + 0.3544927 \times 10^{-4} \times T^2 - 0.2297822 \times 10^{-7} \times T^3$
$E$	$2.10236 \times 10^4 - 9.71895 \times T$
$\sigma_p$	$25.5655 - 5.58937 \times 10^{-2} \times T + 1.04384 \times 10^{-4} \times T^2 - 7.42535 \times 10^{-8} \times T^3$ $-(44.3068 - 1.78933 \times 10^{-2} \times T) \times (0.002)^{0.279395 + 7.749 \times 10^{-4} \times T}$

## SUS304 Creep Strain

$$\varepsilon_c = C_1(1 - e^{-r_1 t}) + C_2(1 - e^{-r_2 t}) + \varepsilon_m t$$

Unit

$T$  : Temperature ( $^{\circ}\text{C}$ )     $425 \leq T \leq 650$

$\sigma$  : Stress ( $\text{kg}/\text{mm}^2$ )     $0.1 \leq \sigma$

$t_R$  : CreepRuptureTime (hr)

$\varepsilon_m$  : Stationary Creep Strain Rate (mm/mm/hr)

$t$  : Time (hr)

$t_R$	$\log_{10}(\alpha_c t_R) = -17.54301 + \frac{26248.54}{T + 273.15} - \frac{6104.579}{T + 273.15} \log_{10} \sigma - \frac{425.0012}{T + 273.15} (\log_{10} \sigma)^2$		
$\varepsilon_m$	$62.416 \cdot \exp\left[-\frac{40812}{8.31 \cdot (T + 273.15)}\right] \cdot t_R^{-1.1335}$		
$C_1$	$1.2692 \cdot \varepsilon_m^{0.74491} / r_1$	$C_2$	$0.48449 \cdot \varepsilon_m^{0.81155} / r_2$
$r_1$	$103.37 \cdot t_R^{-0.72607}$	$r_2$	$17.255 \cdot t_R^{-0.86775}$

Where  $\alpha_c=1$



## SUS304 Creep Rupture Time

$$(T + 273.15) \{ \log_{10} (\alpha_R t_R) + C \}$$

$$= A_0 + A_1 \log_{10} \sigma + A_2 (\log_{10} \sigma)^2$$

Unit

$T$  : Temperature (°C)     $425 \leq T \leq 825$

$\sigma$  : Stress (kg/mm<sup>2</sup>)     $2 \leq \sigma$

$t_R$  : CreepRuptureTime (hr)

$C$	17.54301
$A_0$	26248.54
$A_1$	-6104.579
$A_2$	-425.0012

$\alpha_R$	Average : 1
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## **Appendix 2   Material properties of Japanese 316FR**

## 316FR Monotonic Stress-Strain Curve

$$(1) \sigma \leq \sigma_p$$

$$\varepsilon_e = \frac{\sigma}{E}$$

$$\varepsilon_p = 0$$

$$(2) \sigma > \sigma_p$$

$$\varepsilon_e = \frac{\sigma}{E}$$

$$\varepsilon_p = \left( \frac{\sigma - \sigma_p}{K} \right)^{\frac{1}{m}}$$

<Unit>

$\varepsilon_e$ (mm/mm),  $\varepsilon_p$ (mm/mm),  $\sigma$ (kg/mm<sup>2</sup>)

<Limit of total strain>

Maximum Total Strain  $(\varepsilon_e + \varepsilon_p)_{\max} \leq 0.03$ (mm/mm)

Temperature (°C)	
Parameter	315 ≤ T ≤ 650
E(kg/mm <sup>2</sup> )	315 ≤ T < 400    E = 2.040 × 10 <sup>4</sup> - 8.000T 400 ≤ T ≤ 650    E = 2.126 × 10 <sup>4</sup> - 10.125T
σ <sub>p</sub> (kg/mm <sup>2</sup> )	σ <sub>y</sub> - K(0.002) <sup>m</sup>
σ <sub>y</sub> (kg/mm <sup>2</sup> )	26.8073 - 5.04547 × 10 <sup>-2</sup> T + 8.03901 × 10 <sup>-5</sup> T <sup>2</sup> - 5.11282 × 10 <sup>-8</sup> T <sup>3</sup>
K(kg/mm <sup>2</sup> )	40.0909 - 9.69990 × 10 <sup>-3</sup> T
m	0.326245 + 6.13276 × 10 <sup>-5</sup> T

## 316FR Stress Range – Strain Range Relationship

$$\bullet \Delta\sigma/2 > \sigma_p$$

$$\log_{10}(\Delta\sigma - 2\sigma_p) = A_0 + A_1 \cdot \log_{10}(\Delta\varepsilon_t - \Delta\sigma/E)$$

$$\bullet \Delta\sigma/2 \leq \sigma_p$$

$$\Delta\sigma = E \cdot \Delta\varepsilon_t$$

Unit

$T$  : Temperature (°C)     $425 \leq T \leq 650$

$\Delta\sigma$  : StressRange (kg/mm<sup>2</sup>)

$\Delta\varepsilon_t$  : TotalStrainRange (mm/mm)

$E$  : ElasticModulus (kg/mm<sup>2</sup>)

$\sigma_p$  : ProportionalLimit(kg / mm<sup>2</sup>)

$A_0$	$4.139556 - 0.4434273 \times 10^{-2} \times T + 0.1354228 \times 10^{-5} \times T^2 + 0.1593061 \times 10^{-8} \times T^3$
$A_1$	$2.171727 - 0.7045263 \times 10^{-2} \times T + 0.7832692 \times 10^{-5} \times T^2 - 0.2083600 \times 10^{-8} \times T^3$
$E$	$2.10236 \times 10^4 - 9.71895 \times T$
$\sigma_p$	$26.8073 - 5.04547 \times 10^{-2} \times T + 8.03901 \times 10^{-5} \times T^2 - 5.11282 \times 10^{-8} \times T^3$ $-(40.0909 - 9.69990 \times 10^{-3} \times T) \times (0.002)^{0.326245 + 6.13276 \times 10^{-5} \times T}$

## 316FR Creep Strain

$$\varepsilon_c = C_1(1 - e^{-\eta t}) + C_2(1 - e^{-\eta t}) + \varepsilon_m t$$

Unit

$T$  : Temperature (°C)     $425 \leq T \leq 650$

$\sigma$  : Stress (kg/mm<sup>2</sup>)     $0.1 \leq \sigma$

$t_R$  : CreepRuptureTime (hr)

$\varepsilon_m$  : Stationary Creep Strain Rate (mm/mm/hr)

$t$  : Time (hr)

$t_R$	$\log_{10}(\alpha_c t_R) = -25.82042 + \frac{32232.27}{T + 273.15} - \frac{39.74271}{T + 273.15} \log_{10} \sigma - \frac{3481.803}{T + 273.15} (\log_{10} \sigma)^2$		
$\varepsilon_m$	$241.33 \cdot \exp\left[-\frac{51222}{8.31 \cdot (T + 273.15)}\right] \cdot t_R^{-1.1032}$		
$C_1$	$1.2692 \cdot \varepsilon_m^{0.74491} / r_1$	$C_2$	$0.48449 \cdot \varepsilon_m^{0.81155} / r_2$
$r_1$	$103.37 \cdot t_R^{-0.72607}$	$r_2$	$17.255 \cdot t_R^{-0.86775}$

Where  $\alpha_c=1$

## 316FR Creep Rupture Time

$$(T + 273.15) \{ \log_{10} (\alpha_R t_R) + C \}$$

$$= A_0 + A_1 \log_{10} \sigma + A_2 (\log_{10} \sigma)^2$$

Unit

$T$  : Temperature ( $^{\circ}\text{C}$ )     $425 \leq T \leq 825$

$\sigma$  : Stress ( $\text{kg/mm}^2$ )     $2 \leq \sigma$

$t_R$  : CreepRuptureTime (hr)

$C$	25.82042
$A_0$	32232.27
$A_1$	-39.74271
$A_2$	-3481.803

$\alpha_R$	Average : 1
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**Appendix 3   Stress classification table obtained by thermal elastic  
analysis of the inner vessel model**

[illegible]



**Appendix 4 Photographs and sketches of initiated cracks  
on both surface and section area of the inner vessel  
after 1055 cycles**

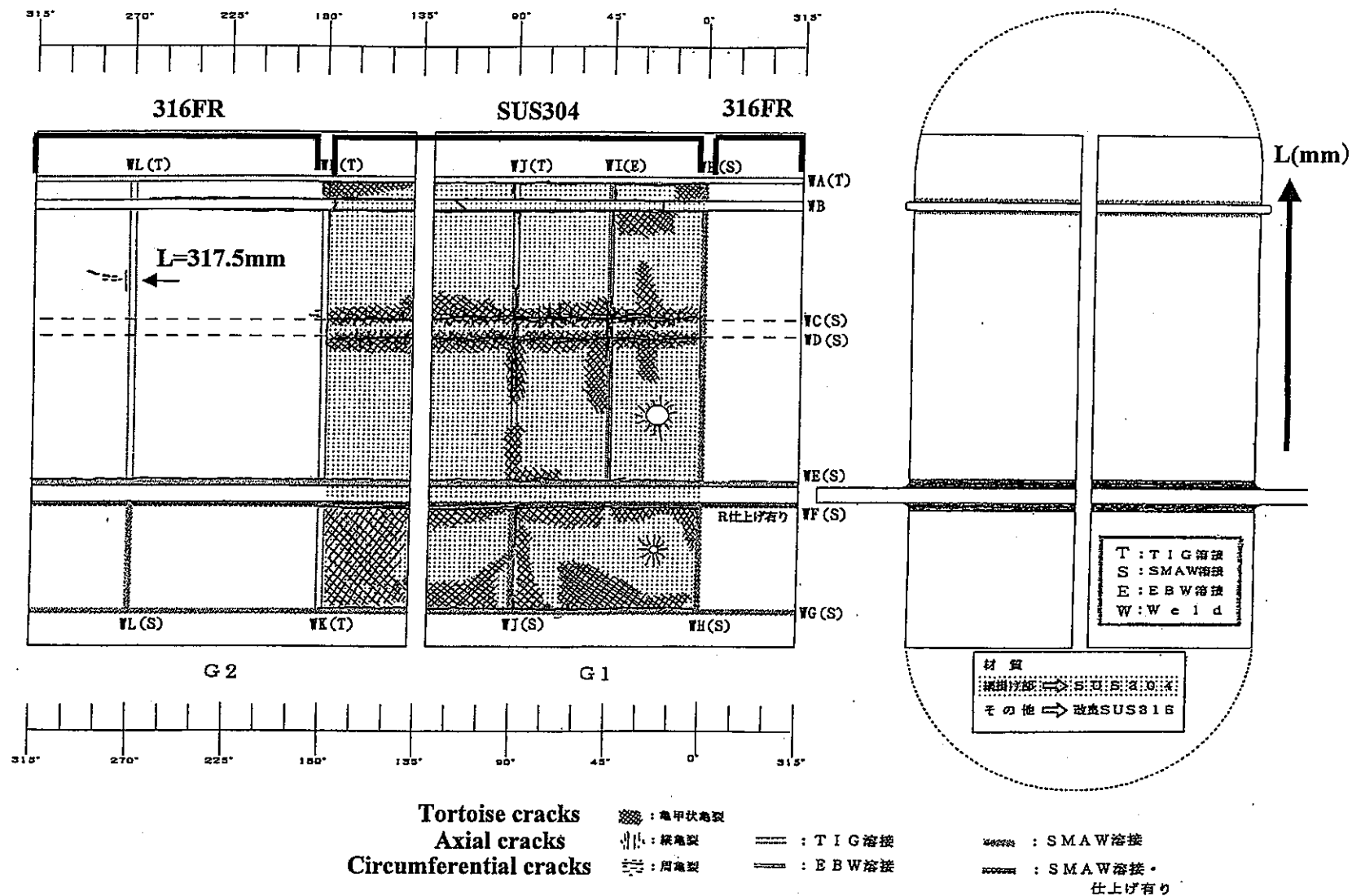


Fig.A4.1 Distribution of initiated cracks on the surface of the inner vessel

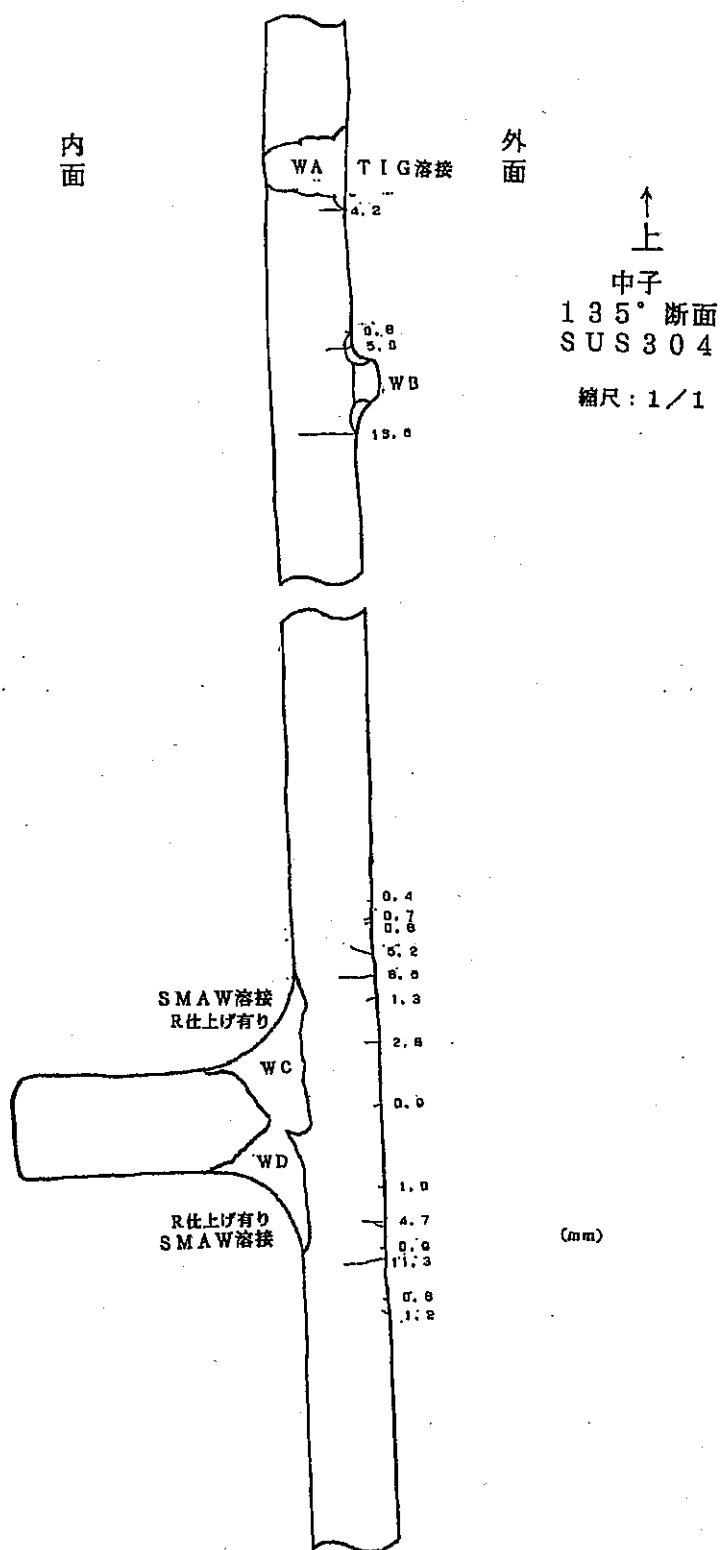


Fig.A4.2 Sketches of initiated cracks on a section area of the inner vessel after 1055 cycles  
(SUS304 135° ref: Fig.A4.1)

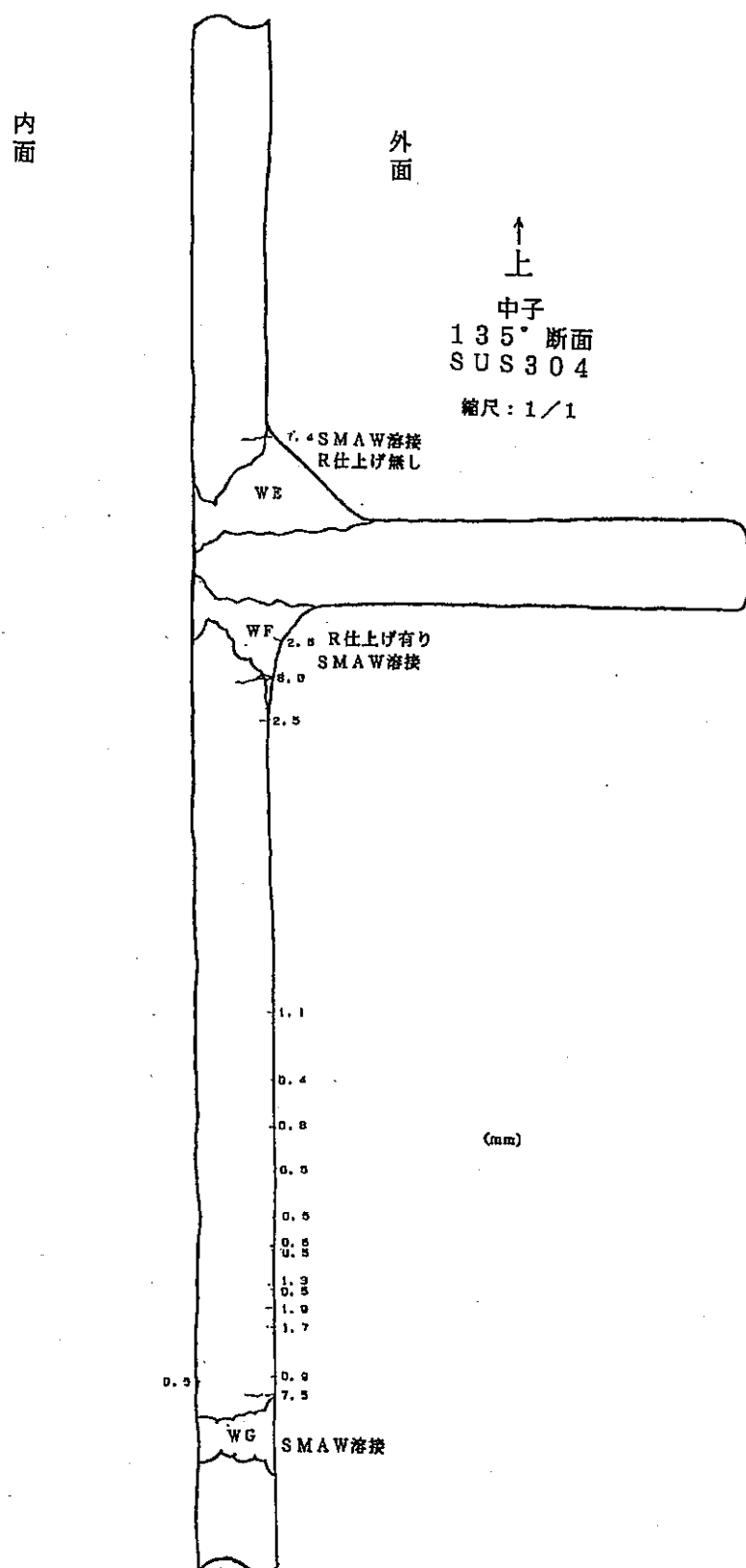


Fig.A4.3 Sketches of initiated cracks on a section area of the inner vessel after 1055 cycles  
(SUS304 135° ref: Fig.A4.1)

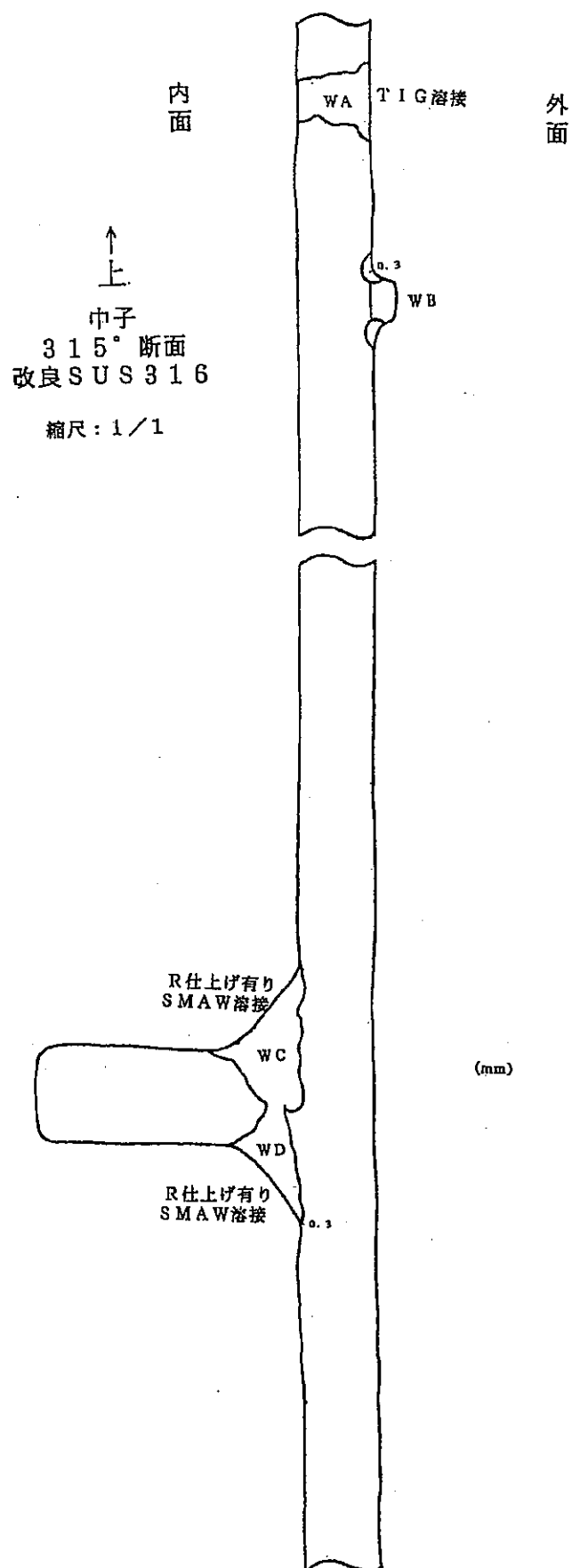


Fig.A4.4 Sketches of initiated cracks on a section area of the inner vessel after 1055 cycles  
(316FR 315° ref: Fig.A4.1)

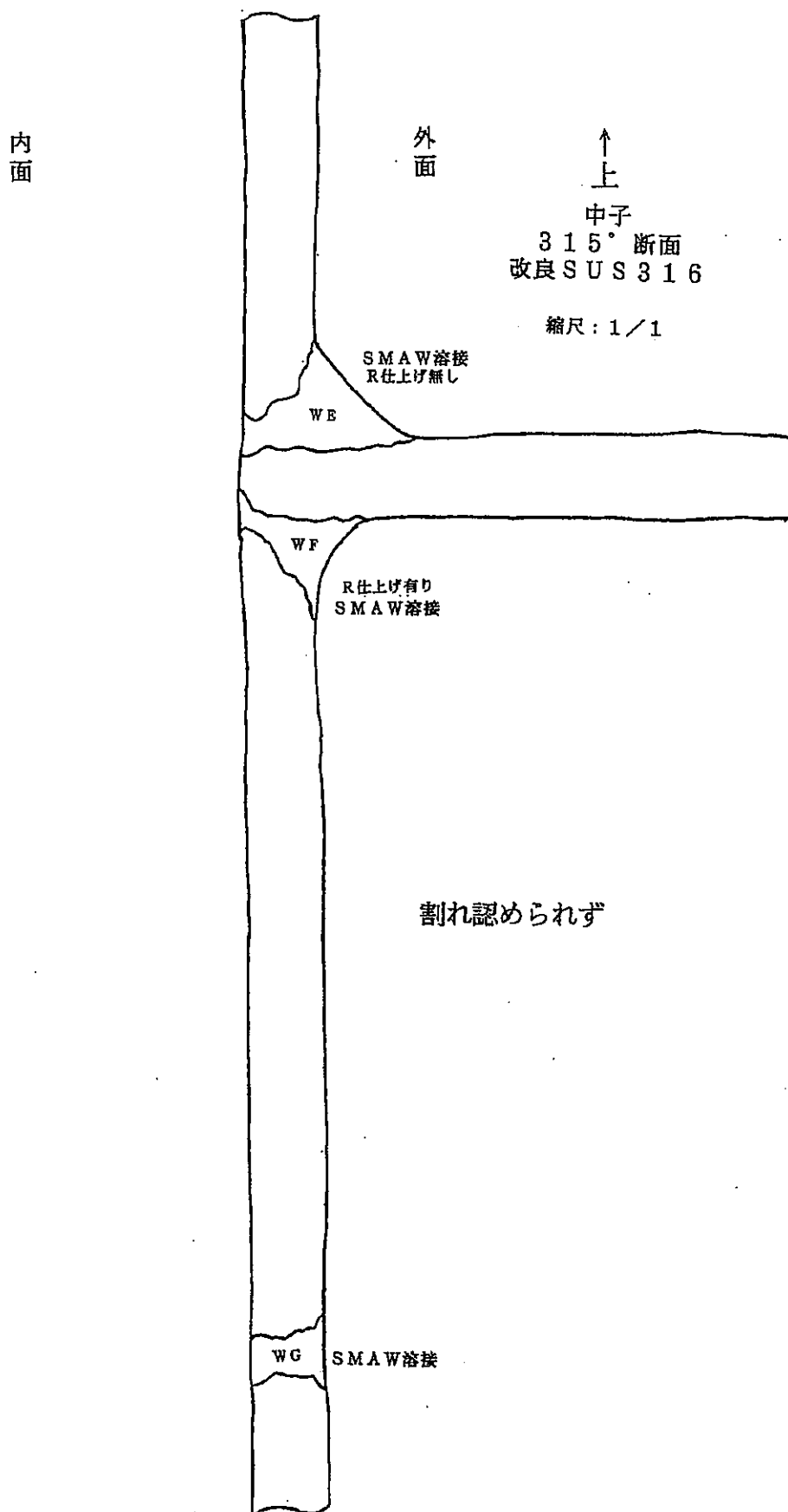


Fig.A4.5 Sketches of initiated cracks on a section area of the inner vessel after 1055 cycles  
(316FR 315° ref: Fig.A4.1)

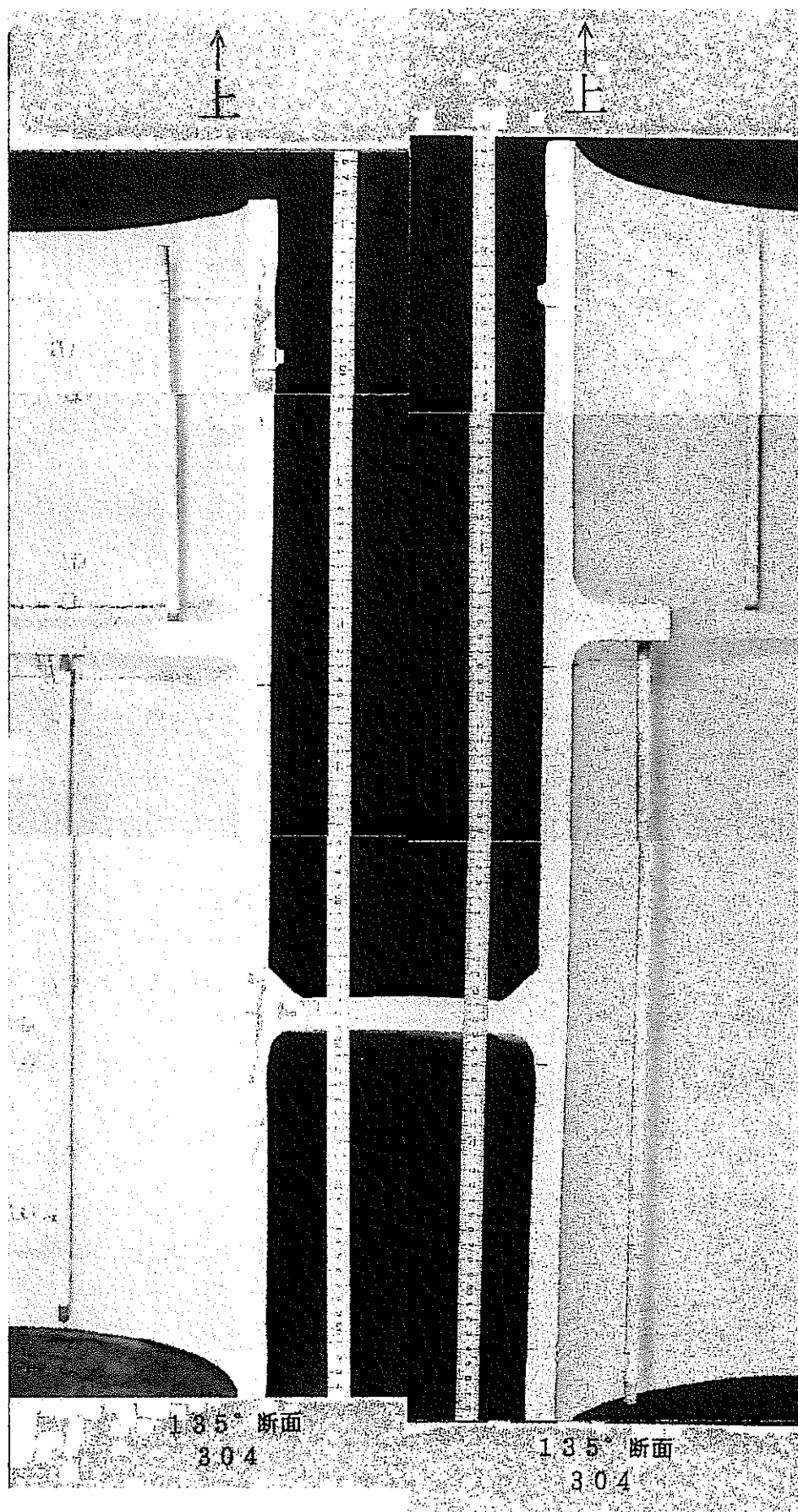


Fig.A4.6 Photograph of initiated cracks on a section area of the inner vessel after 1055 cycles  
(SUS304 135° ref: Fig.A4.1)

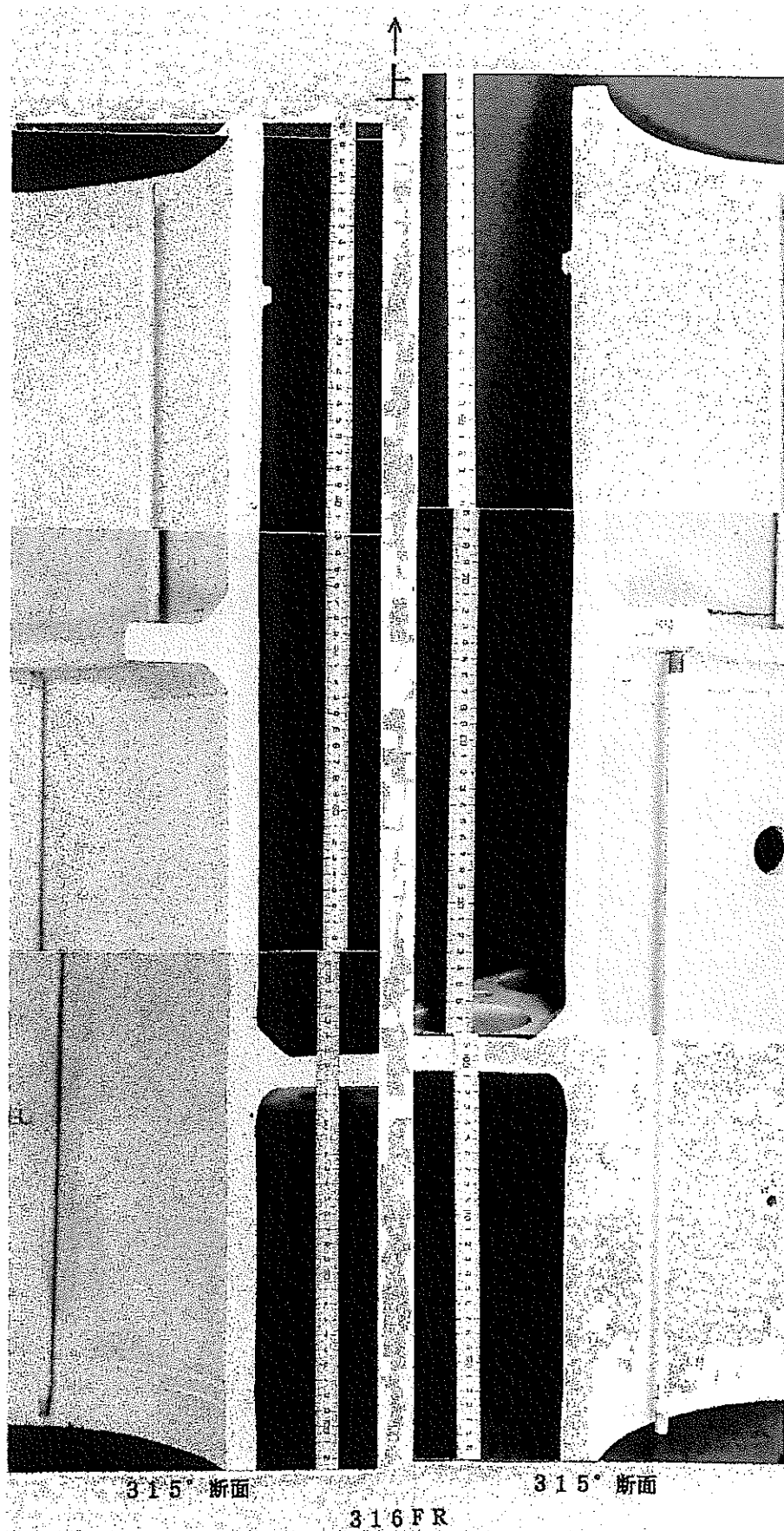


Fig.A4.7 Photograph of initiated cracks on a section area of the inner vessel after 1055 cycles  
(316FR 315° ref: Fig.A4.1)



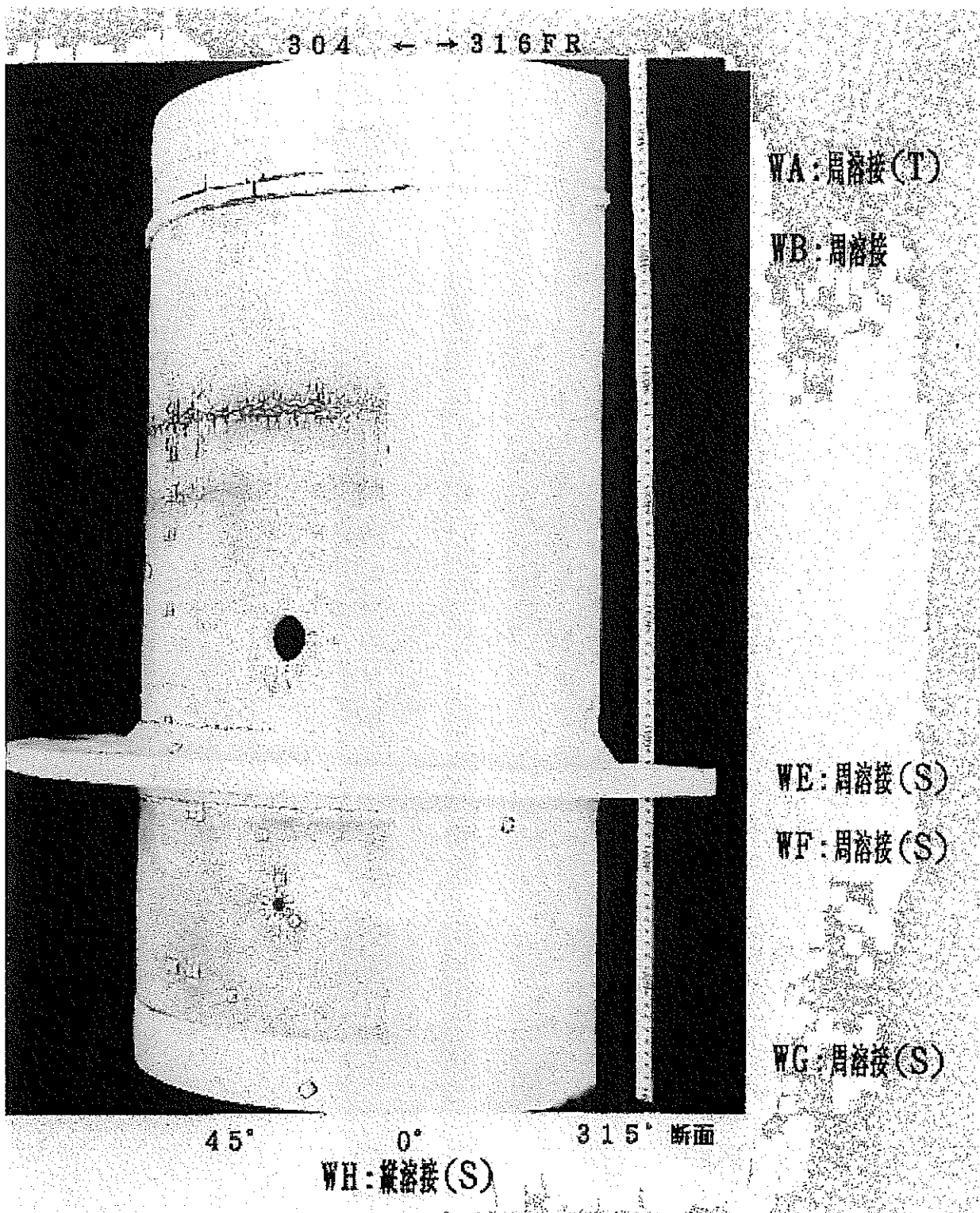


Fig.A4.8 Photograph of initiated cracks on the surface of the inner vessel after 1055 cycles (316FR-SUS304 315° - 45° ref: Fig.A4.1)

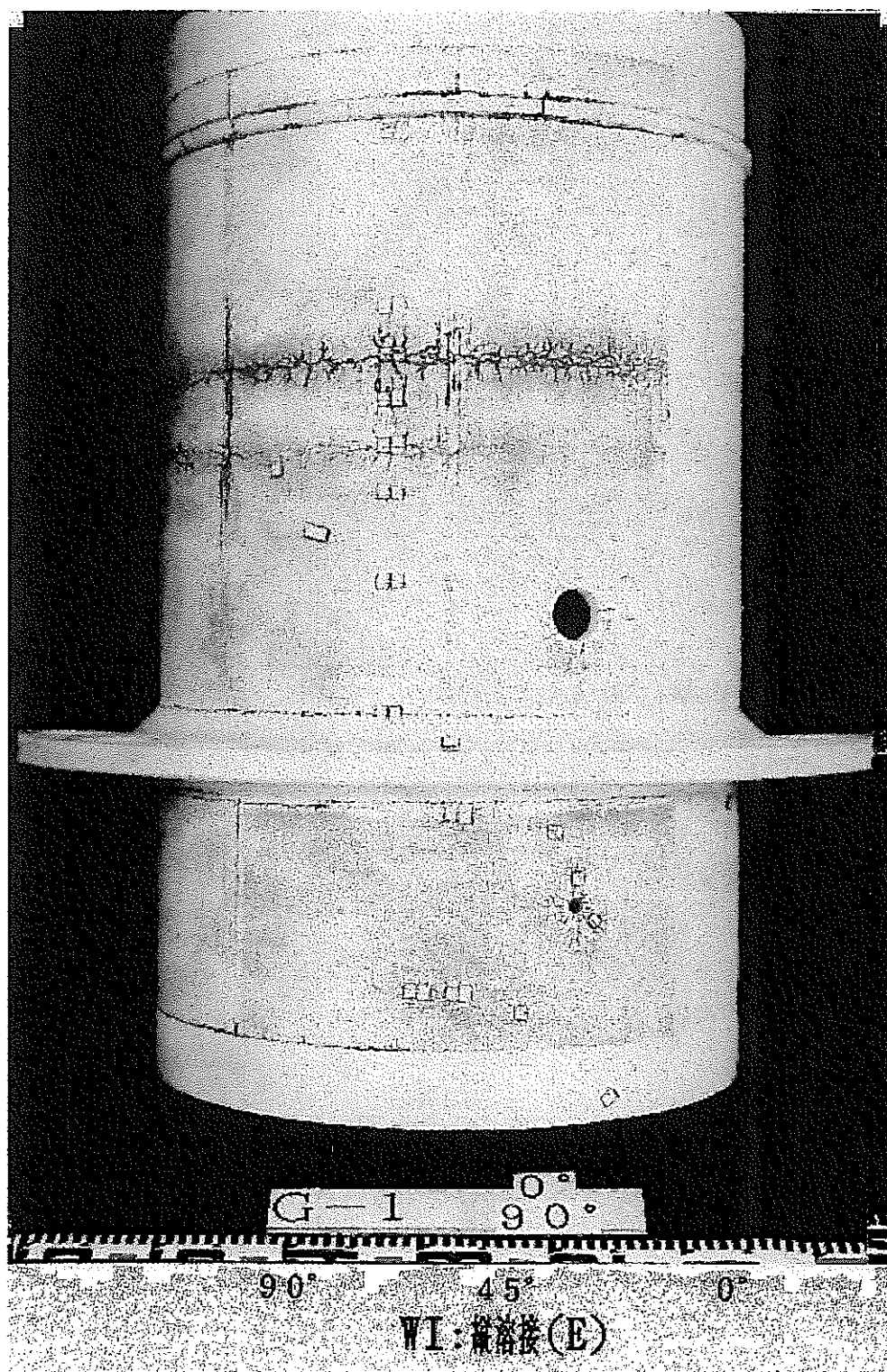


Fig.A4.9 Photograph of initiated cracks on the surface of the inner vessel after 1055 cycles  
(SUS304 0° - 90° ref: Fig.A4.1)

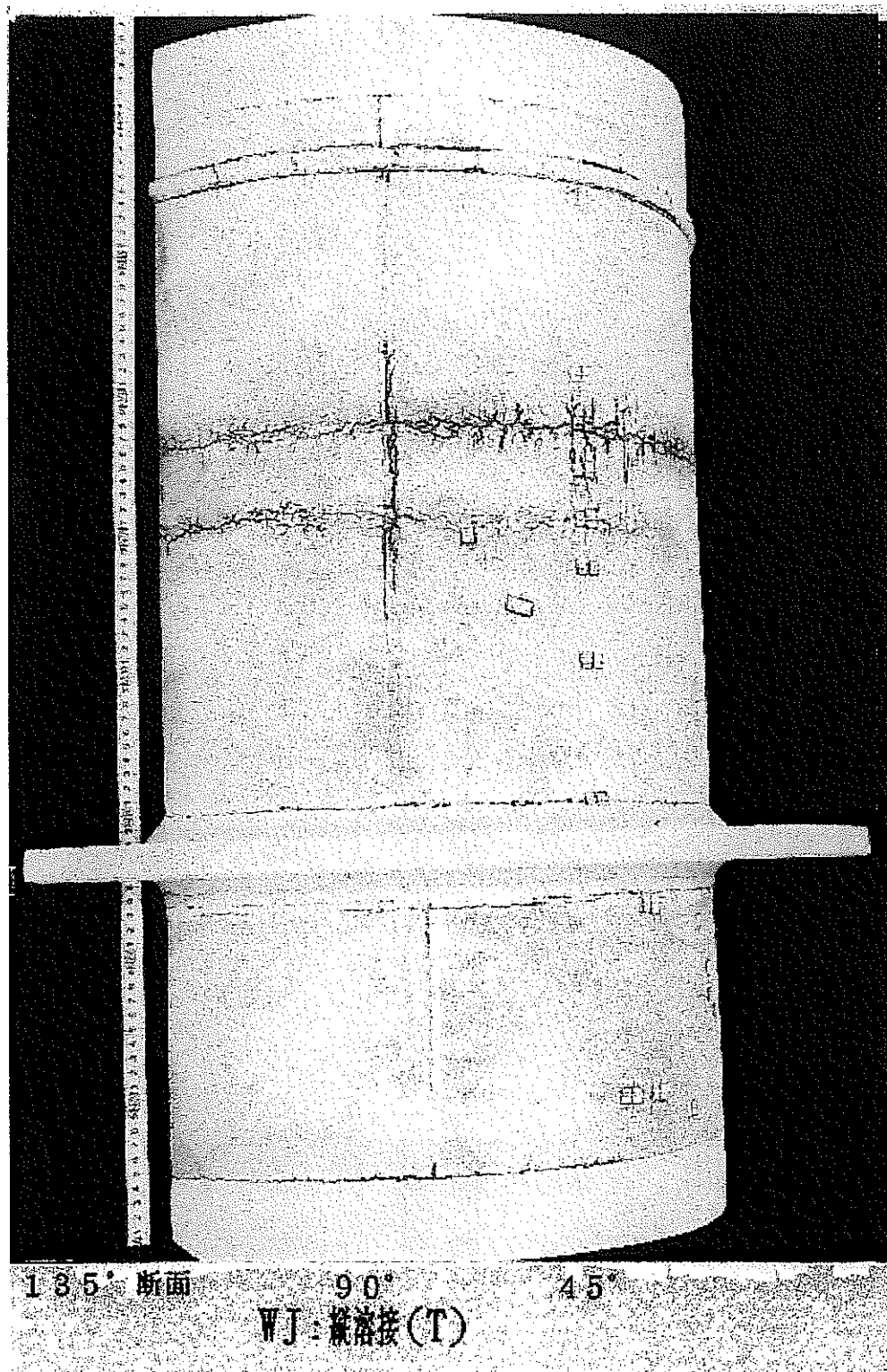


Fig.A4.10 Photograph of initiated cracks on the surface of the inner vessel after 1055 cycles  
(SUS304 45° - 135° ref: Fig.A4.1)

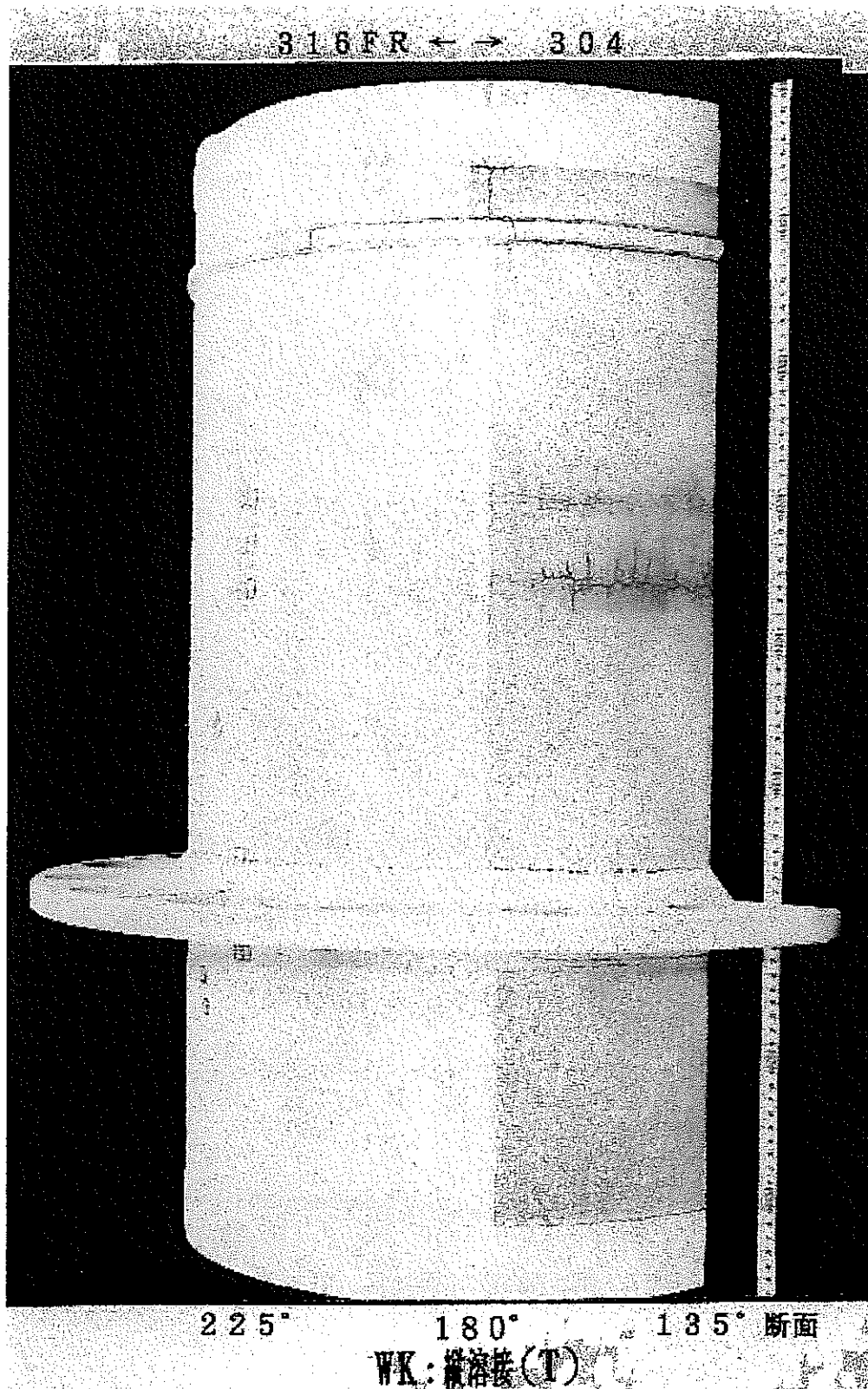


Fig.A4.11 Photograph of initiated cracks on the surface of the inner vessel after 1055 cycles  
(SUS304-316FR 135° - 225° ref: Fig.A4.1)

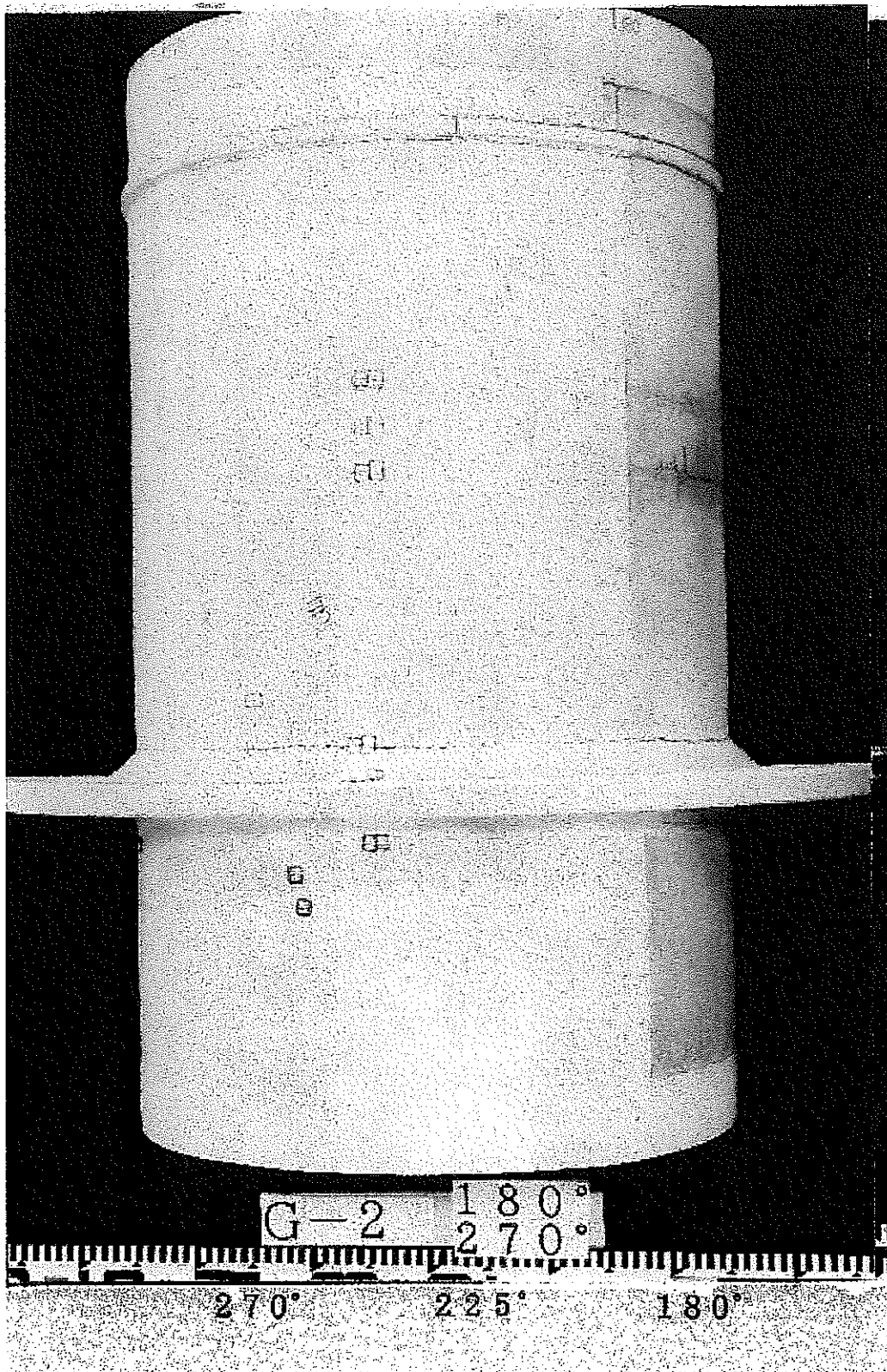


Fig.A4.12 Photograph of initiated cracks on the surface of the inner vessel after 1055 cycles  
(316FR 180° - 270° ref: Fig.A4.1)



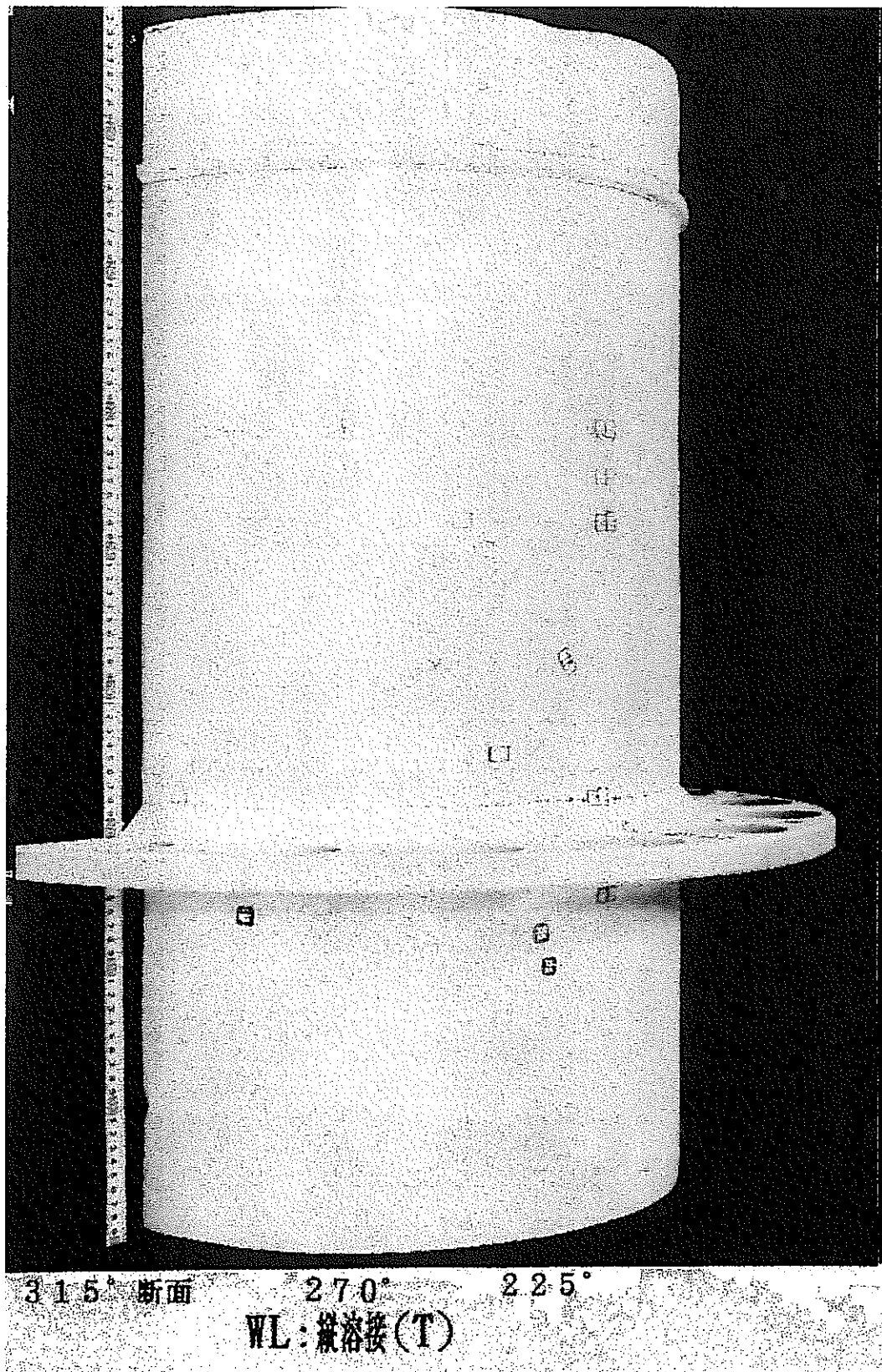


Fig.A4.13 Photograph of initiated cracks on the surface of the inner vessel after 1055 cycles  
(316FR 225° - 315° ref: Fig.A4.1)

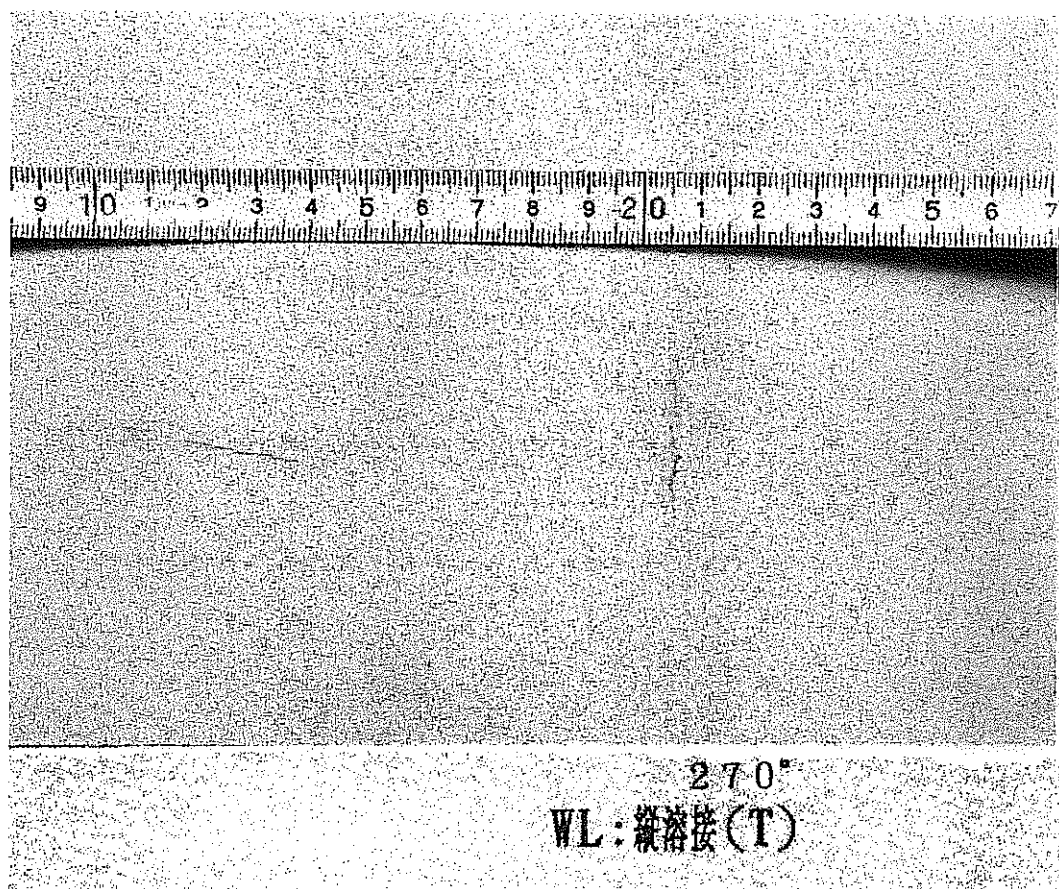


Fig.A4.14 Photograph of initiated cracks on the surface of the inner vessel after 1055 cycles  
(316FR 270° ref : Fig.A4.1)